





# DEVELOPMENT OF A CRASHWORTHY PEDESTRIAN RAIL

Submitted by

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16. Abstract The National Highway Traffic Safety Administration (NHTSA) estimated that approximately 4,300 pedestrian fatalities occurred in the United States in 2010. Risk of pedestrian injury is highest when crossing the street. In locations where pedestrians choose a more direct path and cross the street at non-designated crossing areas, driver expectations are violated and perception-reaction times are delayed, thus increasing risk to the pedestrian. Pedestrian rails may be placed adjacent to roadways to protect pedestrians from dangerous excursions into the roadway as well as from hazardous drop offs. Although numerous pedestrian rails have been designed, their performance has never been evaluated during vehicular impact events. Therefore, the Wisconsin Department of Transportation funded a study to develop a crashworthy pedestrian rail system which satisfies the Manual for Assessing Safety Hardware (MASH) TL-2 channelizer evaluation criteria. A total of twenty-five initial pedestrian rail concepts were designed, and four were advanced for final consideration and dynamic bogie testing. An aluminum rail with welded posts, rails, and spindles was selected for full-scale crash testing. The system consisted of 2-in. x 4-in. x ¼-in. x 43-in. tall (51-mm x 102-mm x 6-mm x 1,029-mm tall) posts with three 2-in. x 2-in. x ½-in. (51-mm x 51-mm x 3-mm) rail components at heights of 42 in. (1,067 mm), 24 <sup>15</sup> /16 in. (633 mm) and 7% in. (200 mm). Two full-scale crash tests were conducted according to MASH TL-2 test designation no. 2-90, but at impact angles of 25 and 0 degrees for test nos. APR-1 and APR-2, respectively. Both tests successfully satisfied the MASH channelizer evaluation criteria. However, the 0-degree impact showed that the pedestrian rail system was near the maximum ridedown acceleration limit. Thus, further modifications are recommended to improve the crashworthiness of the welded aluminum pedestrian rail design and to lower the occupant risk values.			
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This report was completed with funding from the Wisconsin Department of Transportation and the Federal Highway Administration, U.S. Department of Transportation. The contents of this report reflect the views and opinions of the authors who are responsible for the facts and the accuracy of the data presented herein. The contents do not necessarily reflect the official views or policies of the Wisconsin Department of Transportation nor the Federal Highway Administration, U.S. Department of Transportation. This report does not constitute a standard, specification, regulation, product endorsement, or an endorsement of manufacturers.

#### UNCERTAINTY OF MEASUREMENT STATEMENT

The Midwest Roadside Safety Facility (MwRSF) has determined the uncertainty of measurements for several parameters involved in standard full-scale crash testing and non-standard testing of roadside safety features. Information regarding the uncertainty of measurements for critical parameters is available upon request by the sponsor and the Federal Highway Administration. Bogie test nos. WIPR-1 through WIPR-4 were non-certified, dynamic component tests that were conducted for research and development purposes only.

### **INDEPENDENT APPROVING AUTHORITY**

The Independent Approving Authority (IAA) for the data contained herein was Mr. Scott Rosenbaugh, Research Associate Engineer.

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#### **1 INTRODUCTION**

## **1.1 Background**

The National Highway Traffic Safety Administration (NHTSA) estimated that approximately 4,300 pedestrian fatalities occurred in the United States in 2010 [1]. Leaf and Preusser estimated that only 5 percent of pedestrians would die when struck by a vehicle traveling at 20 miles per hour or less, while fatality rates of 40, 80, and nearly 100 percent would occur for vehicles striking pedestrians at 30, 40 and 50 mph or more, respectively [2]. Pedestrian fatalities may be related to transportation designs as well as human behaviors [3]. Many pedestrian-vehicle events are caused by motorists and pedestrians not understanding and/or obeying laws and safe behaviors pertaining to driving and walking [4].

Risk of pedestrian injury is highest when crossing the street. Many intersections have designated crosswalk areas for pedestrians to safely cross the street, and these marked areas inform drivers to be mindful of pedestrian traffic. However, pedestrians may choose a more direct path or be distracted and enter the roadway in non-marked areas. Pedestrian rails are often placed adjacent to roadways to protect pedestrians from hazardous drop offs or dangerous excursions into the roadway. Examples of such areas, as shown in Figure 1, include (1) sidewalks over culverts where a pedestrian rail may be necessary to separate pedestrians from hazardous drop offs or (2) busy streets where median fences may be used to deter pedestrians from crossing in non-designated crossing locations. In some cases where pedestrian rails are installed to prevent pedestrians form entering areas adjacent to right of way, as shown in Figure 2, the pedestrian rail may also prevent pedestrian maneuver options like escaping an errant vehicle. Thus, pedestrian rail design and placement should be carefully considered. Although numerous pedestrian rails have been designed, their performance has never been evaluated during vehicular impact events. The Wisconsin Department of Transportation provided some examples of vehicle impacts on pedestrian rails are shown in Figure 3. Pedestrian rails that have not been evaluated to vehicle impact safety performance standards may be hazardous to the passengers of errant vehicles due to disengaged components penetrating the windshield or occupant compartment, excessive vehicle decelerations, or vehicle instability and rollover.

#### **1.2 Objective**

The objective of this research project was to design a crashworthy pedestrian rail that will protect pedestrians from hazards while not posing an undue safety risk to motorists and pedestrians. The new pedestrian rail must meet the design standards associated with the Americans with Disabilities Act (ADA) [5] and the pedestrian rail standards contained in the American Association of State Highway and Transportation Officials (AASHTO) *Load and Resistance Factor Design (LRFD) Bridge Design Specifications* [6]. In addition, the pedestrian rail was evaluated according to the Test Level 2 (TL-2) safety performance criteria for longitudinal channelizers published in the AASHTO *Manual for Assessing Safety Hardware* (MASH) [7].

#### 1.3 Scope

The research objective was achieved through the completion of several tasks. First, a survey was conducted of the Midwest States Pooled Fund Program members to identify the most common locations and circumstances in which a crashworthy pedestrian rail would be warranted. Next, a review was conducted of existing pedestrian rail and fence designs from State Departments of Transportation (DOT) and product manufacturers. Potential fabrication

2

materials, such as aluminum, steel, wood, and polymers, were investigated. Design concepts were configured, and the preferred concepts were selected for further evaluation. Bogie tests were conducted on the selected design concepts to evaluate their performance behavior. Two full-scale vehicle crash tests were performed in accordance with the MASH TL-2 impact conditions for longitudinal channelizers. The test results were analyzed, evaluated, and documented. Finally, conclusions and recommendations were made that pertain to the safety performance of the new pedestrian rail system.



Figure 1. Examples of Pedestrian Rails



Figure 2. Pedestrian Rail Limiting Pedestrian Maneuver Options



Figure 3. Vehicle Impacts with Pedestrian Rails

#### **2 LITERATURE REVIEW**

## 2.1 Standards

The prototype design concepts considered within this research project must meet three standards and guidelines to satisfy the objectives stated earlier. The pedestrian rail must be ADA compliant and meet AASHTO *LRFD Bridge Design Specifications*, which ensures that the rail will be accessible for use by all people as well as safely function as a longitudinal channelizer. Additional pedestrian rail design criteria included the International Building Code (IBC) [8] and Occupational Safety and Health Administration (OSHA), Part 1910 [9]. The AASHTO *LRFD Bridge Design Specifications* were used for the rail loading requirements, as they varied between the standards. The final design concept would be evaluated according to the MASH Test Level 2 safety performance criteria for longitudinal channelizers [7].

#### 2.1.1 Americans with Disabilities Act Design Criteria

A pedestrian rail must be accessible to all people, including those with disabilities. The 2010 ADA Standards for Accessible Design sets forth handrail criteria [5]. The handrail needs to be continuous along the full length of the walkway and not be obstructed on the top or sides. The handrail top gripping surface should be a minimum of 34 in. (864 mm) and a maximum of 38 in. (965 mm) vertically above the walking surface. There should be a minimum of 1½ in. (38 mm) separation between the back surface of the handrail and any adjacent surface. The handrail gripping surface for a circular cross section shall have minimum and maximum outside diameters of 1¼ in. (32 mm) and 2 in. (51 mm), respectively. Non-circular cross sections shall have minimum and maximum perimeters of 4 in. (102 mm) and 6¼ in. (159 mm), respectively, with the diagonal cross section length no greater than 2¼ in. (57 mm). Maximum diagonal dimensions for a non-circular cross section are shown in Figure 4. If fittings are used, the handrail shall not rotate within them. When a vertical or horizontal force of 250 lb (1,112 N) is

applied on any point on the handrail, fasteners, mounting devices, or supporting structures, the allowable stresses shall not be exceeded.

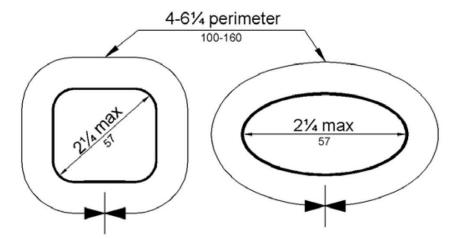


Figure 4. ADA Non-Circular Cross Section Dimensions

#### 2.1.2 AASHTO LRFD Bridge Design Specifications

The AASHTO *LRFD Bridge Design Specifications* also provided requirements for the design of a pedestrian rail [6]. Pedestrian rail height should be a minimum of 42 in. (1,067 mm) above the walkway. A clear spacing shall apply to the lower 27 in. (686 mm) of the railing where a 6-in. (152-mm) diameter sphere cannot pass thought the rail elements. The clear spacing in the upper section of the railing above 27 in. (686 mm) shall not allow an 8-in. (203-mm) diameter sphere to pass through the rail elements. Chain link or metal fabric fence should not have openings larger than 2 in. (51 mm).

Longitudinal railing elements must withstand a uniform live load of 50 lb/ft (730 N/m) simultaneously applied both transversely and vertically, along with a concentrated live load of 200 lb (890 N) applied at any point and in any direction on the longitudinal element, as shown in Figure 5. The posts are subjected to a concentrated live load,  $P_{LL}$ , defined in Equation 1. The concentrated live load  $P_{LL}$  shall be applied transversely at the center of gravity of the upper horizontal element. For a railing mounted taller than 5 ft (1.5 m),  $P_{LL}$  shall be applied at a point 5

ft (1.5 m) above the walkway. Chain link or metal fabric fence shall be designed for a distributed live wind load of 15  $lb/ft^2$  (718 N/m<sup>2</sup>) applied perpendicular to the entire mesh surface.

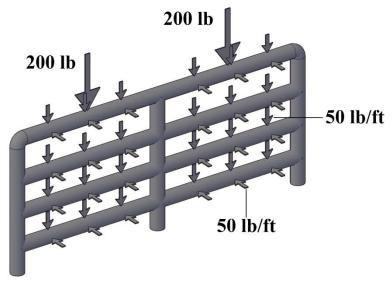


Figure 5. AASHTO Loading Criteria (Vertical 200-lb Point Load Shown)

$$P_{LL} = 200 + 50L \tag{1}$$

Where:  $P_{LL} = Post Point Live Load (lb)$ L = Post Spacing (ft)

## 2.1.3 International Building Code

The 2012 IBC [8] also contains handrail design criteria similar to the ADA code. The handrail shall be continuous along the full length of the walkway and not be obstructed on the top or side. The handrail top gripping surface should be a minimum of 34 in. (864 mm) and a maximum of 38 in. (965 mm) vertically above the walking surface. There should be a minimum separation of 1½ in. (38 mm) between the back surface of the handrail and any adjacent surface. Handrail gripping surfaces with circular cross sections shall have minimum and maximum outside diameter of 1¼ in. and 2 in. (32 mm and 51 mm), respectively. Non-circular cross sections shall have minimum and maximum surface perimeters of 4 in. and 6¼ in. (102 mm and 159 mm), respectively, with the cross section dimension of at least 1 in. (25 mm) but no greater

than 2¼ in. (57 mm). Edges shall have a minimum radius of 0.01 in. (0.25 mm). If fittings are used, the handrail shall not rotate within them. Handrails should be designed to resist a linear load of 50 lb/ft (730 N/m). Handrails should also be designed to resist a concentrated load of 200 lb (890 N) applied in any direction at any point along the top. Intermediate rails, balusters, and panel fillers should be designed to resist a concentrated load of 50 lb (222 N).

### 2.1.4 Occupational Safety & Health Administration (OSHA)

Handrail design criteria is also contained in *Part 1910 – Occupational Safety & Health Administration Regulations (Standards – 29 CFR)* [9]. A standard railing shall consist of a top rail, intermediate rail, and posts and shall have a vertical height of 42 in. (1,067 mm), as measured between the upper surface of top rail to the ground. The top rail shall be smooth throughout the length of the railing. Pipe railings, posts, and top and intermediate railings shall be at least  $1\frac{1}{2}$  in. (38 mm) nominal diameter with posts spaced not more than 8 ft (2.4 m) on center. The complete structure shall be capable of withstanding 200 lb (890 N) load applied in any direction at any point on the top rail.

#### 2.1.5 AASHTO MASH Longitudinal Channelizers

Longitudinal channelizers are intended to provide clear visual indication of the intended vehicle path through a construction zone. They are not intended to contain and redirect impacting vehicles. The vehicle is allowed to traverse through and behind the system. Thus, the impact performance criterion for longitudinal channelizers is different from those used for longitudinal barriers. For MASH TL-2 longitudinal channelizers, two full-scale crash tests are recommended, test designation no. 2-90 with an 1100C vehicle and test designation no. 2-91 with a 2270P vehicle [7]. The impact conditions for each test vehicle are a speed of 44 mph (70 km/h) and a critical impact angle between 0 and 25 degrees that maximize the risk of vehicle rollover and excessive vehicle decelerations.

#### 2.2 Existing Pedestrian Rail Designs

Four categories of pedestrian rails were considered: (1) concrete combination barriers, (2) plastic fences, (3) wood fences, and (4) metal rails. Concrete barriers are the most costly and are often used in combination with a metal rail or chain link fence to accommodate pedestrian safety in high-speed facilities. Metal rails are typically fabricated with aluminum or steel for strength and ease of construction. Wood fences are used for economic reasons. Current polymer fences are fabricated with polyvinyl chloride (PVC), high density polyethylene (HDPE), and fiber-reinforced polymers (FRP) for aesthetics and corrosion resistance. Most combination concrete barriers and pedestrian rail designs have been crash tested according to safety performance criteria. However, plastic, wood, and metal fences and rails historically have not been crash tested. The most prominent designs are categorized in the following sections. However, this is not an all-inclusive list of pedestrian rail designs.

## 2.2.1 Concrete Combination Traffic and Pedestrian Rail Designs

The Minnesota combination bridge rail is an example of a traffic and bicycle combination bridge rail that has been developed and successfully crash-tested [10]. This system successfully met all Test Level 3 (TL-3) safety performance criteria of National Cooperative Highway Research Program (NCHRP) Report No. 350, *Recommended Procedures for the Safety Performance Evaluation of Highway Features* [11]. This bridge rail utilized a 31<sup>7</sup>/<sub>8</sub>-in. (810-mm) high New Jersey safety shape barrier with steel panels formed from tubular steel and posts, and square vertical spindle bolted to the back-side vertical face of the concrete barrier. The steel rail extended 22<sup>1</sup>/<sub>2</sub> in. (572 mm) above the Jersey barrier, giving a total barrier height of 54<sup>3</sup>/<sub>8</sub> in. (1,381 mm). This bridge rail is a longitudinal barrier that contains and redirects impacting vehicles as well as prevents pedestrians from crossing at non-designated crossing locations, but

is more expensive and requires more installation time than a pedestrian-only rail channelizer. The Minnesota combination bridge rail is shown in Figure 6.



Figure 6. Minnesota Combination Traffic and Pedestrian Barrier [10]

#### **2.2.2 Plastic Fence Designs**

Plastic fence designs create separation between two areas and are typically fabricated using HDPE, FRP, or PVC. Many HDPE designs were observed for use in large animal containment. FRP designs were commonly used as safety handrails. PVC fences commonly serve as boundaries on personal properties.

HDPE fences are durable and virtually maintenance-free and stain resistant. HDPE is more resistant to shattering and splitting at low temperatures than common polymers. HDPE has very low material strength. The base of HDPE posts are commonly supported with a wood or metal insert. Examples of existing HDPE fences are shown in Figure 7 [12-14].

FRP is a composite material made of a polymer reinforced with fibers, usually glass. FRPs also have a low weight-to-strength ratio. FRP handrail systems are corrosion-resistant, giving them a long lifespan with little maintenance. UV inhibitors are added to the resin during fabrication, along with a synthetic surfacing veil, providing protections from UV weathering. For these reasons, FRP handrails are often used in extreme climate locations or facilities with highly corrosive chemicals. Most FRP rail systems are yellow for safety reasons but also can be fabricated in any color. Dynarail and SAFRAIL, as shown in Figure 8, are two of many FRP handrails [15-17].

PVC fencing is commonly found as decorative barriers to divide personal property. A PVC fence design offers virtually no maintenance with ultraviolet inhibitors in the vinyl to prevent it from changing color and material properties. PVC material may become brittle under low temperatures. PVC can come in a wide range of colors, but when heated, material strength properties decline. For this reason, PVC fences are usually white to reflect the sun. The PVC posts are commonly supported with a wood or metal insert. Examples of PVC fencing are shown in Figure 9 [18-20].







Figure 7. Examples of Existing HDPE Fences [12-14]







Figure 8. Examples of Existing FRP Handrail Systems [15-17]



Figure 9. Examples of Existing PVC Fences [18-20]





#### 2.2.3 Existing Wood Fence Designs

Wood fences are generally used to separate personal property by acting as boundary lines and to contain large animals. Non-treated wood can be highly susceptible to decay, rotting, and bug deterioration. For this reason, most wood fences require preservative treatment as well as continuous maintenance and repair. Wood material properties can vary significantly, so the strength of each fence system may vary. Wood fences historically have not been crash tested, and the post and rail components may be penetrate the windshield or occupant compartment when impacted by errant vehicles in some cases, as shown in Figure 3. Examples of wood fencing are shown in Figure 10 [21-24].

#### **2.2.4 Metal Barrier Designs**

The New Southern Wales Roads and Traffic Authority (RTA) developed two steel pedestrian rail concepts, the RTA Designed Pedestrian Fence and the Modified Welded Steel Mesh Fencing [25]. The RTA Designed Pedestrian Fence, as shown in Figure 11, was composed of customized steel posts and two rails connected with steel balusters. The balusters gave the barrier an anti-climb design. Although the fence was designed to collapse during impact to minimize damage on individual elements, evidence of crash testing was not provided. The staggered layout of the balusters permits visibility on both sides. This pedestrian fence design is preferred by the RTA.

The Modified Welded Steel Mesh, as shown in Figure 12, was designed to deform safely upon vehicle impact, although no evidence of crash testing was provided. It differed from the Pedestrian Fence in that it was more difficult to see through at acute angles. Near the bottom of the fence was a longitudinal 0.4-in. (10-mm) diameter galvanized steel cable that ran through each panel and post. The cable was tied and clamped at the end posts. To prevent the bottom from opening significantly when impacted, the bottom of the panels was secured with two heavy-gauge split links.

Based on a 1988 study in the United Kingdom, pedestrian rails placed near the roadway diminish the ability for pedestrians and vehicles to see one another [26]. This fact is most prominent when the pedestrian is a child who cannot see over the rail. Although the use of pedestrian rails has shown to effectively improve road safety, the lack of visibility has been shown to be detrimental to road safety, especially for children. For this reason, Pell & Baldwin LTD created a steel, pedestrian-only rail called the VISIFLEX pedestrian guardrail, which was a more visible rail for pedestrians and motorists [27]. The VISIFLEX pedestrian guardrail, as shown in Figure 13, was composed with only three components – standard panels with balusters, stub posts, and an end bar. All components were fabricated with galvanized steel. The balusters were placed at an angle and spaced appropriately for optimum visibility. The simple design allows for easy installation and repair of the VISIFLEX system.

The Iowa DOT designed a welded handrail as a pedestrian rail, as shown in Figure 14 [28]. The design consisted of two  $2\frac{1}{2}$ -in. (64-mm) diameter steel pipe rails. The top rail was 45 in. (1,143 mm) above the walkway, and the second rail was 24 in. (610 mm) above the walkway. The  $2\frac{1}{2}$ -in. (64-mm) diameter steel pipe posts were welded to the rail elements and an  $8\frac{1}{2}$ -in. x  $3\frac{1}{4}$ -in. x  $8\frac{1}{2}$ -in. (216-mm x 19-mm x 216-mm) steel plate at the base of the post. The steel plate was attached with four  $\frac{5}{8}$ -in. (16-mm) diameter steel stud concrete anchors, which fixed the pedestrian rail system to the ground.

The Washington State DOT designed a 42-in. (1,067-mm) tall aluminum handrail, as shown in Figures 15 and 16 [29]. Two horizontal 2<sup>1</sup>/<sub>2</sub>-in. (64-mm) diameter horizontal rails were spliced at the posts and were 4 in. (102 mm) and 42 in. (1,067 mm) above the walkway. Posts were 2<sup>1</sup>/<sub>2</sub> in. (64 mm) in diameter and spaced at 7 ft (2.1 m). The lower pipe surface was 4 in.

(102 mm) above the walkway surface. Eleven 1-in. (25-mm) diameter baluster pipes spanned vertically between the rails in each panel section.





Figure 10. Examples of Existing Wood Fences [21-24]





Figure 11. RTA Designed Pedestrian Barrier [25]

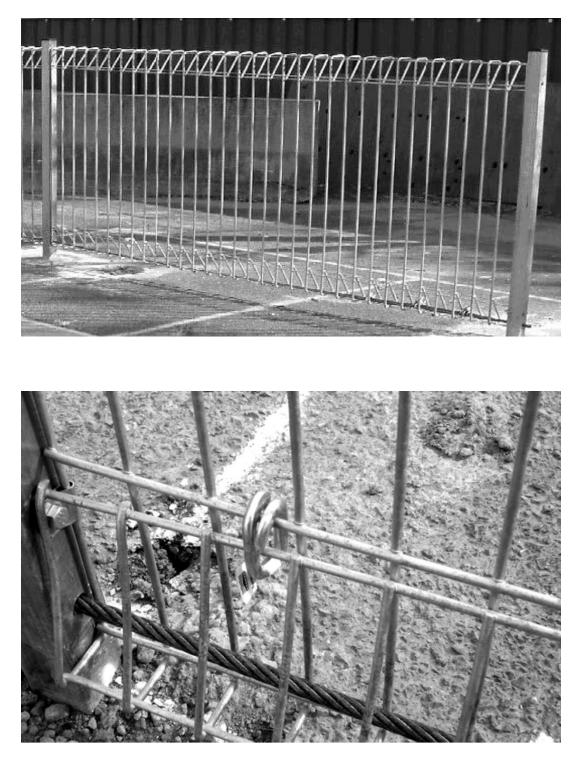


Figure 12. RTA Modified Steel Mesh [25]



Figure 13. VISIFLEX Pedestrian Guardrail [27]





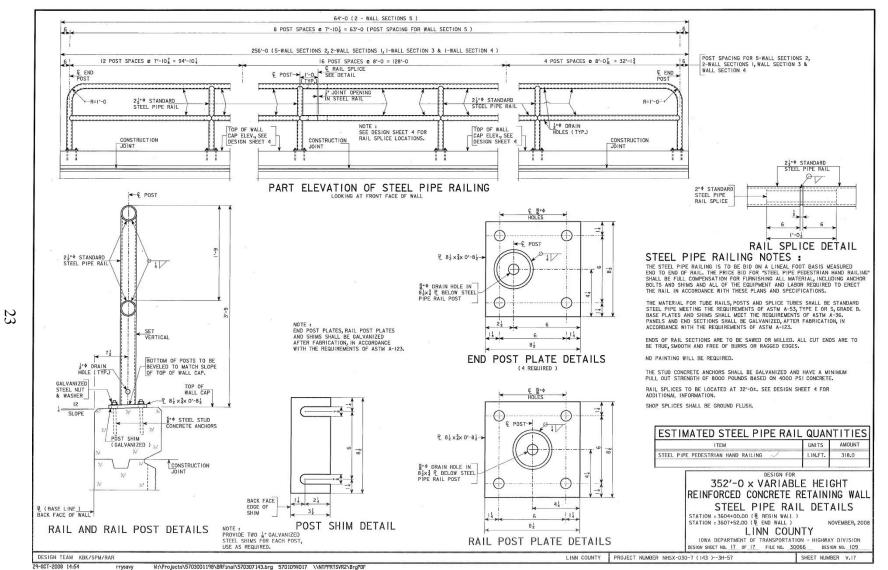


Figure 14. Iowa DOT Steel Pipe Rail [28]

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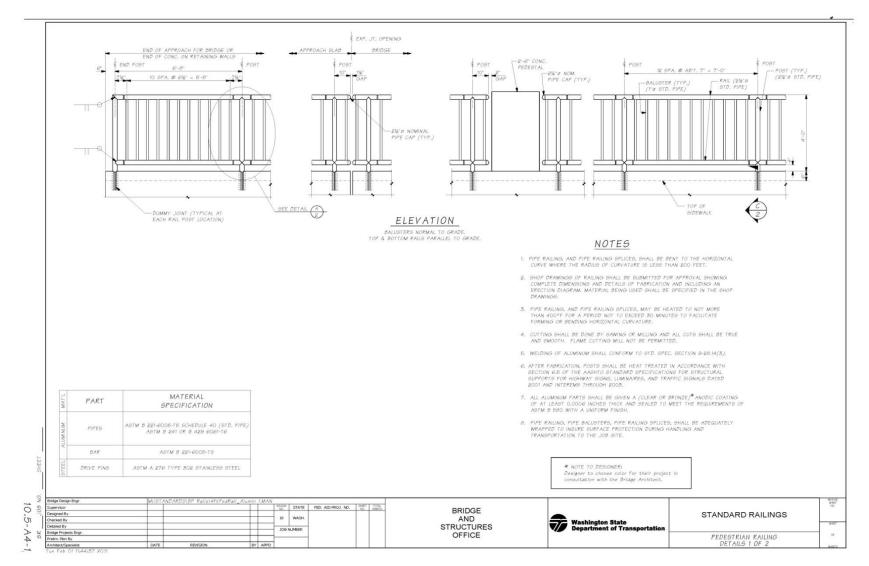


Figure 15. Washington DOT Standard Railing (Sheet 1 of 2) [29]

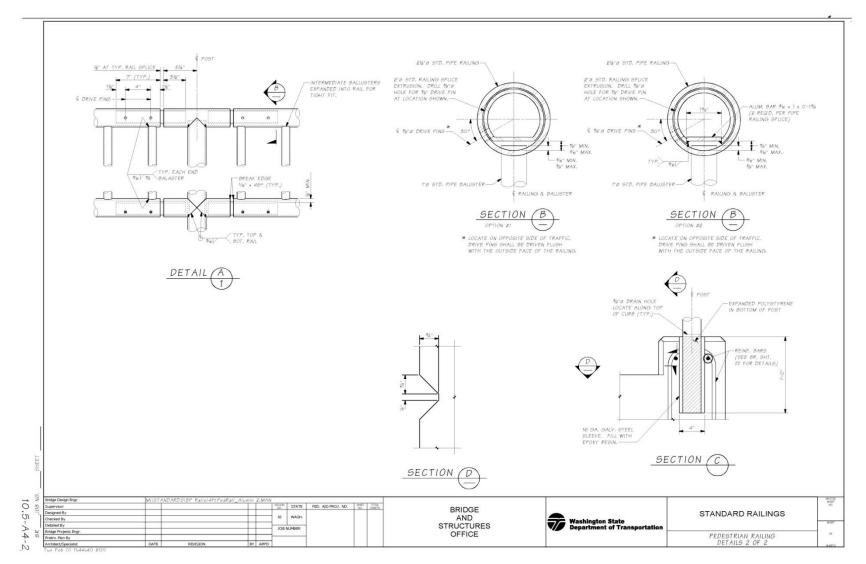


Figure 16. Washington DOT Standard Railing (Sheet 2 of 2) [29]

The Texas DOT pedestrian rail consisted of six horizontal rails, as shown in Figure 17 [30]. The top rail was a 3-in. (76 mm) standard steel pipe and the lower five rails were 2-in. (51 mm) standard steel pipe. Posts were 5 in. (127 mm) wide and spaced at a maximum of 10 ft (3.1 m) apart. The minimum rail height was 42 in. (1,067 mm) above the walkway.

The Ultra-tec Cable Railing Systems used varying cable diameters and frame constructions to accommodate for various uses, one of which was pedestrian rail [31]. Cables could be aligned horizontally or vertically across the frame, as shown in Figure 18 and Figure 19. The cables were spaced 3 in. (76 mm) from each other and had to each support a 400-lb (1,779-N) tension minimum. Support rail braces should be placed at a minimum spacing of 42 in. (1,067 mm). If the cables are not tensioned properly, the end posts may bend due to high cable tension.

An aluminum, pedestrian-only rail was designed by the Florida Department of Transportation (FDOT), to meet the AASHTO and ADA load and dimension requirements, as shown in Figure 20 through Figure 27 [32-33]. The rail consisted of structural tubes, pipes, and bars made of aluminum alloy 6061-T6. The end hoop sections of the rail were fabricated with alloy 6063-T5 for better formability. Two 2-in. x 2-in. x <sup>1</sup>/<sub>4</sub>-in. (51-mm x 51-mm x 6-mm) square tubes were used at each post location, separated by 5<sup>3</sup>/<sub>4</sub> in. (146 mm). Total post spacing was specified as 5 ft – 8 in. (1.7 m). The top horizontal member was a Schedule 10 2<sup>1</sup>/<sub>2</sub>-in. nominal pipe size (73-mm x 3-mm) round tube. The bottom and intermediate horizontal members were 2-in. x <sup>1</sup>/<sub>4</sub>-in. (51-mm x 51-mm x 51-mm x 6-mm) square tubes. Five infill panel options were specified including <sup>3</sup>/<sub>4</sub>-in. (19-mm) diameter round bar pickets. The pickets spanned between the intermediate and bottom longitudinal rails. This rail also specified an ADA-compliant handrail attachment.



Figure 17. Texas DOT Handrail [30]

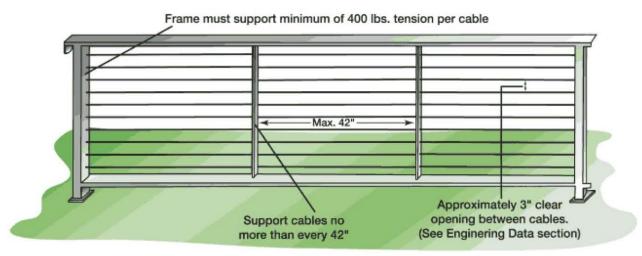


Figure 18. Horizontal Cable Frame [31]

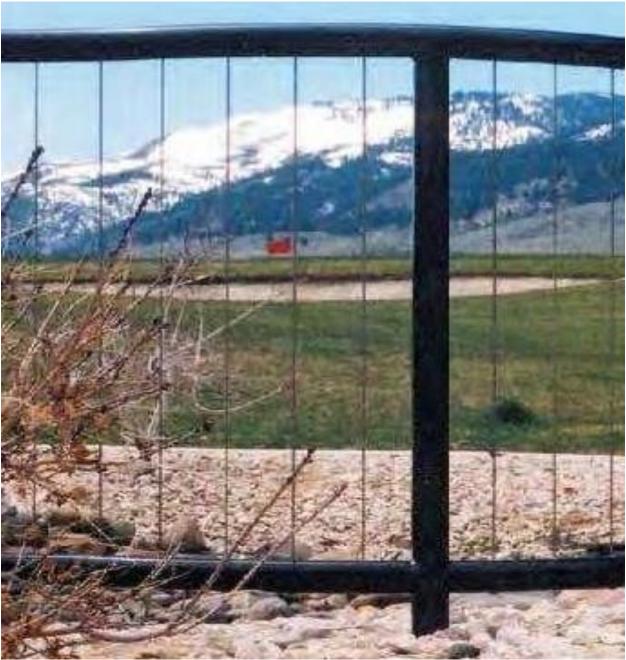
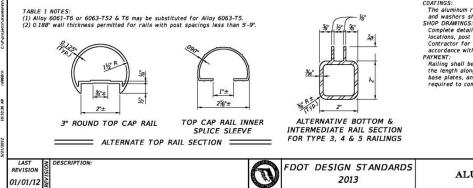


Figure 19. Vertical Cable Frame [31]

#### 3D VIEW OF RAILING WITH TYPE 1 - PICKET INFILL PANEL (42" Height shown, 54" Height Similar)

MEMBER	ALLOY <sup>(1)</sup>	DESIGNATION	OUTSIDE DIMENSION	WALL THICKNESS	
Posts	6061-T6	RT 2x2x.250	2.00" x 2.00"	0.250"	
Top Rail	6061-T6	2½" NPS (Sch. 10) 3" Round Top Cap Rail	2.875" 3.000"	0.120" 0.125" 0.125" 0.125" 0.125" 0.090"	
End Hoops	6063-T5	2½" NPS (Sch. 10) 3.00 0D x 0.125 Wall	2.875" 3.000"		
Top Rail Joint/Splice Sleeves	6063-T5	2.50 OD x 0.125 Wall Top Cap Rail Inner Sleeve	2.500" 2.800"		
Intermediate & Bottom Rail	6061-T6	RT 2x2x.250	2.00" x 2.00"	0.250" (2)	
Int. & Bottom Rail Post Connection Sleeve	6063-T5	1.50 OD x 0.125 Wall	1.500"	0.125"	
Handrail Joint/Splice Sleeves	6063-T5	1" NPS (Sch. 40)	1.315"	0.133"	
Handrails	6061-T6	11/2" NPS (Sch. 40)	1.900"	0.145"	
Handrail Support Bar	6061-T6	¾" Ø Round Bar	0.750"	N/A	
Pickets (Type 1 Infill Panel)	6061-T6	¾" Ø Round Bar	0.750"	N/A	
Infill Panel Members (Types 2 - 5)	6063-T5	Varies (See Details)	Varles	Varies	



#### DESIGN LOADS, GEOMETRY AND APPLICABILITY:

See the Instructions for Design Standards for the design loads, geometry and applicability requirements. GENERAL:

Adequate foundation support shall be provided for anchorage and stability against overturning. See Index No. 861 for special requirements and modifications for use on bridges. The railing shown on these drawings requires a handrail for ramps steeper than a 5% grade to conform with the requirements of the Americans with Disabilities Act (ADA)

NOTES

#### RAUS PANELS AND POSTS

Structural Extrusions, Tube, Pipe and Bar shall be in accordance with Table 1 and ASTM B221 or ASTM B429. Top, bottom and intermediate rail corner bends with maximum 4-0" post spacing, may be Alloy 6063-T6. Perforated panels (Type 5) shall be Alloy 3003-H14. Posts shall be fabricated and installed plumb,  $\pm$  1" tolerance when measured at 3'-6" above the foundation. Pickets and vertical panel elements shall be fabricated parallel to the posts, except that Type 2, 3 & 5 panel infills may be fabricated parallel to the longitudinal grade. Corners and changes tangential longitudinal alignment shall be made continuous with a 9<sup>th</sup> bend radius or terminate at adjoining sections with mitered end sections when handrails are not required. For changes in tangential longitudinal alignment greater than 45°, posts shall be positioned at a maximum distance of 2'-0" each side of the corner and shall not be located at the corner apex. For curved longitudinal alignments the top and bottom rails and handrails shall be shop bent to match the alignment radius.

BASE PLATES AND RAIL CAPS. Base Plates and Post Cap plates shall be in accordance with ASTM B209, Alloy 6061-T6.

SHIM PLATES:

Shim Plates shall be aluminum in accordance with ASTM B209, Alloy 6061 or 6063. Shim plates shall be used for foundation height adjustments greater than  $\frac{1}{2^n}$  and localized irregularities greater than  $\frac{1}{2^n}$ Field trim shim plates when necessary to match the contours of the foundation. Beveled shim plates may be used in lieu of trimmed flat shim plates shown. Stacked shim plates must be bonded together with adhesive bonding material and limited to a maximum total thickness of ½", unless longer anchor bolts are provided for the exposed thread length. ANCHOR BOLTS:

Anchor bolts shall be in accordance with ASTM F1554 Grade 36. Headless anchor bolts for Adhesive Anchors shall be threaded full length. Cutting of reinforcing steel is permitted for drilled hole installation. Expansion Anchors are not permitted. All anchor bolts shall have single self-locking hex nuts. Tack welding of the nut to the anchor bolt may be used in lieu of self-locking nuts. All nuts shall be in accordance with ASTM A563 or ASTM A194. Flat Washers shall be in accordance with ASTM F436 and Plate Washers (for long slotted holes only), shall be in accordance with ASTM A36 or ASTM A709 Grade 36. After the nuts have been snug tightened, the anchor bolt threads shall be distorted to prevent removal of the nuts. Distorted threads and tack welds shall be coated with a galvanizing compound in accordance with the Specifications

RESILIENT AND NEOPRENE PADS:

Resilient and Neoprene pads shall be in accordance with Specification Section 932 except that testing of the finished pads shall not be required. Neoprene pads shall be durometer hardness 60 to 80. IOINTS.

All welded joints are to be ground smooth. Expansion joints shall be spaced at a maximum 35-0". Field splices similar to the expansion joint detail may be approved by the Engineer to facilitate handling, but top rail must be continuous across a minimum of two posts. WELDING:

All welding shall be in accordance with the American Welding Society Structural Welding Code (Aluminum) ANSI/AWS D1.2 (current edition). Filler metal shall be either ER5183, ER5356 or ER5556. Nondestructive testing of welds is not required. Filler metal for plug welds and bend splices may be ER4043. COATINGS:

The aluminum railing shall be mill finish unless otherwise noted in the Contract Documents. All nuts, bolts and washers shall be hot-dip galvanized in accordance with Specification Section 962.

Complete details addressing project specific geometry (line & grade) showing post and expansion joint locations, post and panel type, anchor bolt installation "Case" or lengths, must be submitted by the Contractor for the Engineer's approval prior to fabrication of the railing. Shop drawings shall be in accordance with the Specifications

Railing shall be paid for per linear foot (Item No. 515-2-abb). Payment will be plan quantity measured as the length along the center line of the top rail, and includes rails, posts, pickets, panels, rail splice assembly, base plates, anchor bolts, nuts, washers, resilient or neoprene pads and all incidental materials and labor required to complete installation of the railing.

ALUMINUM PEDESTRIAN/BICYCLE RAILING

INDEX SHEET NO. NO. 862 1

Figure 20. FDOT Aluminum, Pedestrian-Only Rail (Sheet 1 of 8) [32-33]

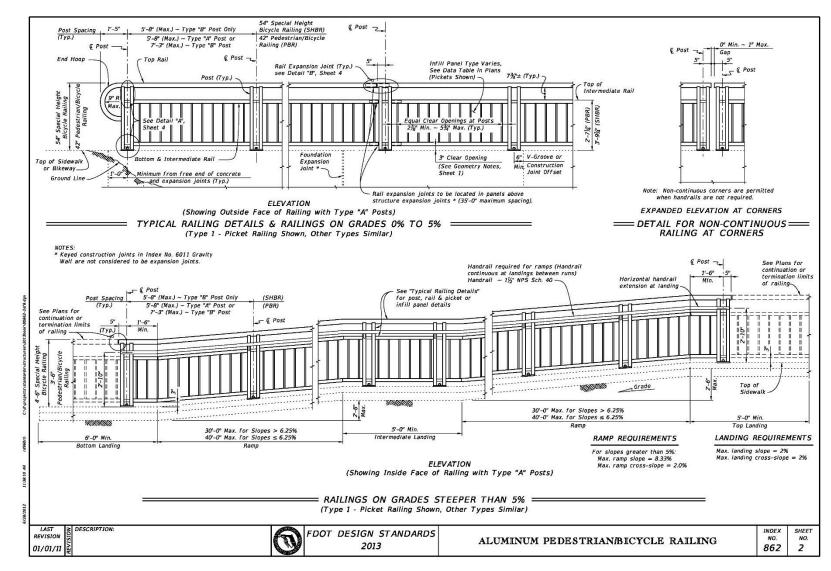


Figure 21. FDOT Aluminum, Pedestrian-Only Rail (Sheet 2 of 8) [32-33]

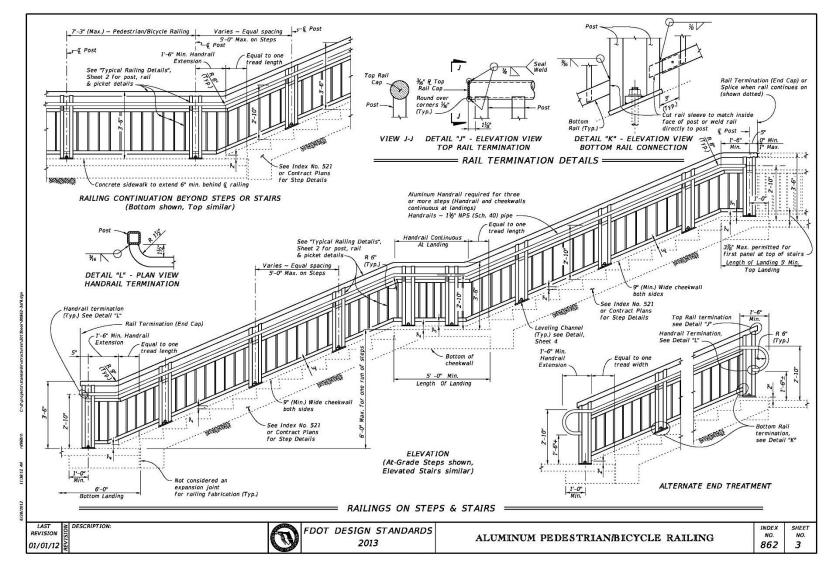


Figure 22. FDOT Aluminum, Pedestrian-Only Rail (Sheet 3 of 8) [32-33]

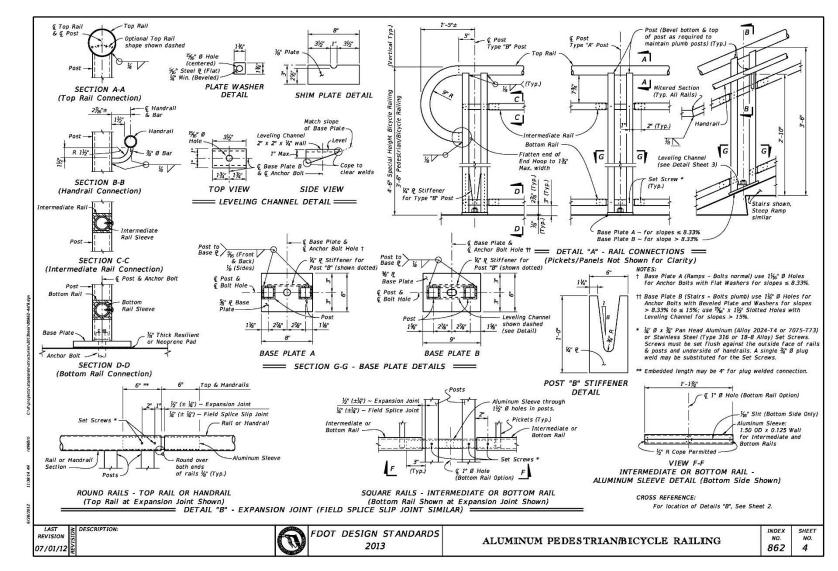


Figure 23. FDOT Aluminum, Pedestrian-Only Rail (Sheet 4 of 8) [32-33]

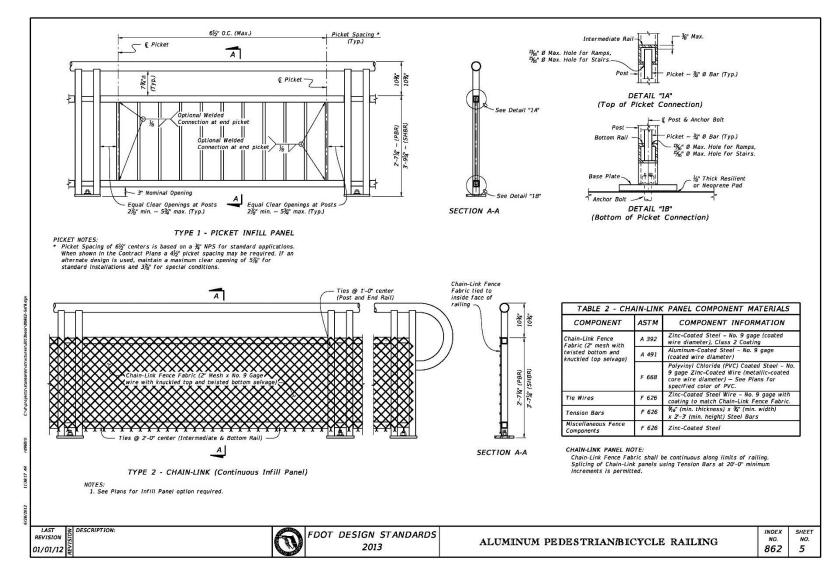


Figure 24. FDOT Aluminum, Pedestrian-Only Rail (Sheet 5 of 8) [32-33]

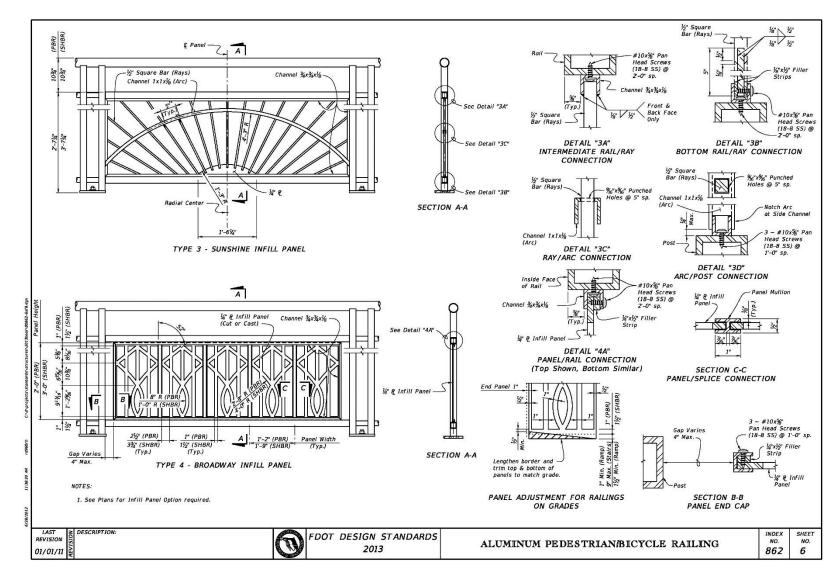


Figure 25. FDOT Aluminum, Pedestrian-Only Rail (Sheet 6 of 8) [32-33]

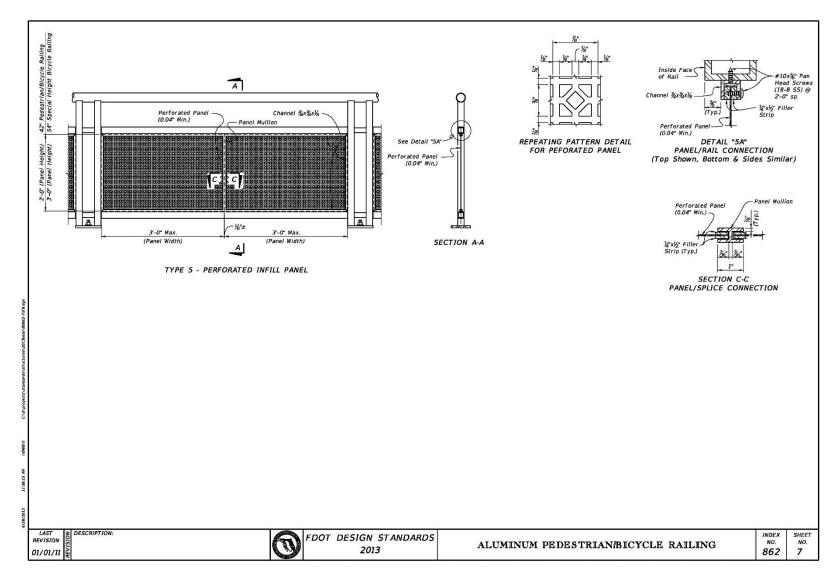


Figure 26. FDOT Aluminum, Pedestrian-Only Rail (Sheet 7 of 8) [32-33]

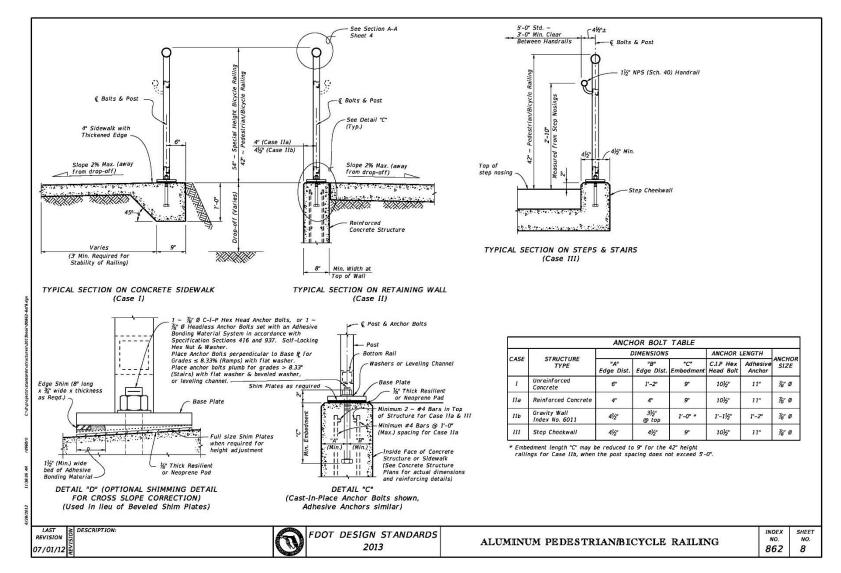


Figure 27. FDOT Aluminum, Pedestrian-Only Rail (Sheet 8 of 8) [32-33]

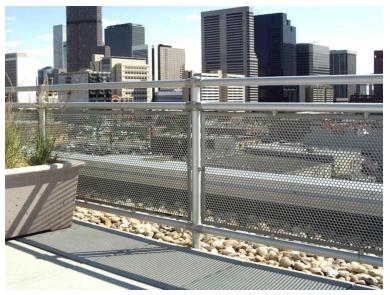
Prefabricated steel and aluminum rail fittings are available from many manufacturers. For example, Hollaender Manufacturing Company has many steel and aluminum systems, as shown in Figure 28 [34-35]. The Speed Rail system is an aluminum modular handrail system which is created from individual fittings and pipe sections [35]. This modular design allows for fast and simple fabrication. Repair of a modular system is less difficult, because the damaged section and fittings are the only components that need to be replaced. When impacted by a vehicle, the railing system may break into its individual elements, which may put the impacting vehicle's occupant, surrounding vehicles, and nearby pedestrians at risk of flying elements. Hollaender fabricated the railing systems to meet OSHA and IBC testing standards. A vast set of fitting sizes and shapes provide multiple design options.





Figure 28. Examples of Hollaender Rail Systems [34-35]





#### **3 EVALUATION OF PEDSESTRIAN RAIL NEEDS**

A survey was conducted to identify the most common locations and circumstances in which a crashworthy pedestrian rail would be warranted. This survey was important to find where and how these barriers would be installed. A copy of the survey that was sent to the Midwest States Pooled Fund members is shown in Appendix A. The survey results, as determined from nine member state DOTs, are shown in Table 1, while the Wisconsin DOT survey results are shown in Table 2. As stated previously, this project was funded by the Wisconsin DOT and their input was primarily used.

According to the 2011 Edition of the National Safety Council's (NSC) Injury Facts report, motor vehicle collisions with pedestrians are a significant concern and result in a fatality about one-third of the time [36]. According to NSC data from 2009, sixty percent of the pedestrian-to-motor-vehicle fatalities occur when the pedestrian tried to improperly cross a roadway or intersection. The desire to use the pedestrian rail to prevent pedestrians from crossing the street and non-designated crossing locations addresses the dangers associated with crossing the roadway at an unintended location, and may aid in reducing fatality and injury accidents between motor vehicles and pedestrians.

For the Pooled Fund, the highest-priority, crashworthy pedestrian rail need was identified for use on top of culverts. For the Wisconsin DOT, the most common, highest-priority, crashworthy pedestrian rail need was to prevent urban/suburban pedestrian crossings at nondesignated locations. Based on the two findings, the highest priority was to focus on preventing pedestrian crossings at non-designated locations, since the project was funded by the Wisconsin Department of Transportation.

# Table 1. Pooled Fund Member Responses to Highest Need–Pedestrian Rail Survey

Pedestrian Rail	Usefulness Summary:						
Locations/Circumstances	Not Useful Somew		Somewhat Use	ewhat Useful		Rank	
On top of culverts			3	5	1	1	
On top of retaining walls	1	1	4		3	2	
Prevent Jaywalking	2	3	3		1	3	
Rail around private/public property	2	3	3	1		4	
Other:							
Bike/pedestrian path separation from roadway					1		
Bike path hazard protection					1		
Sidewalk higher than surroundings			1				
On bridges				1			

40

# Table 2. Wisconsin DOT Response to Highest Need–Pedestrian Rail Survey

Pedestrian Rail	Usefulness Summary:					
Locations/Circumstances	ces Not Useful Somewhat Useful Very Us		Very Useful	Rank		
Prevent Jaywalking					X	1
On top of culverts			X			2
Rail around private/public property				X		3
On top of retaining walls		Х				4

#### **4 PRELIMINARY PEDESTRIAN RAIL DESIGN**

The pedestrian rail must: (1) meet AASHTO standards, (2) be ADA compliant, and (3) meet AASHTO MASH TL-2 criteria for longitudinal channelizers. Two additional design goals include a desire for the rail to be aesthetically pleasing and to allow pedestrians and motorists to be visible to one another. The pedestrian rail was also to be designed to eventually accommodate an ADA-compliant handrail.

#### **4.1 Design Load Calculations**

The calculations described herein were used to design an anchored, straight, pedestrian rail with uniform post spacing. The applied loads were defined by the requirements published in the AASHTO *LRFD Bridge Design Specifications* for a pedestrian rail [6]. These loads corresponded to the critical loading that was applied to the pedestrian rail structure, which generated the critical forces/stresses. The minimum available cross sections were determined to meet all load requirements. In addition to the loading requirements, a maximum allowable deflection was set to 4 in. (102 mm) for all longitudinal and vertical elements.

#### **4.1.1 Longitudinal Rail Element**

The longitudinal rail elements were designed to withstand two types of live loads: (a) a uniformly distributed load of 50 lb/ft (730 N/m) applied both transversely (z-axis) and vertically (y-axis) and (b) a concentrated load of 200 lb (890 N) applied at any point and in any direction. In general, stresses are maximized when the concentrated load can be superposed with the uniform loads in the transverse, vertical, or resultant direction. An example of the design loading conditions with a concentrated load acting vertically downward in the center of the top longitudinal beam is shown in Figure 29.

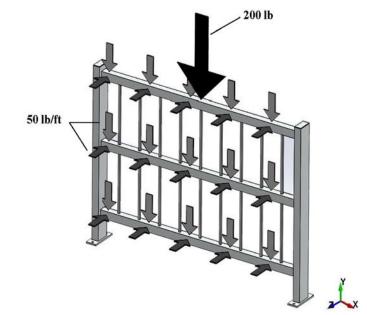


Figure 29. Example of Pedestrian Rail with Vertical Concentrated Load

The longitudinal rail element was assumed to act as a simply supported beam for the preliminary strength analysis. Total direct shear loads were calculated by summing uniform and concentrated loads together. For small-deflection, linear-elastic bending displacements, superposition can be used to estimate bending moments and stresses in beams subjected to diverse loading conditions. Thus, bending moments were calculated by superposing moments created by loads in the transverse ( $M_{uy}$ ) and vertical ( $M_{uz}$ ) directions, as shown in Equation 2. Note that because the uniform loads do not spatially vary in direction or magnitude, only the moment created by the concentrated load can vary.

In general, beam bending analysis must consider loads, stresses, and deflections in principal (i.e.,  $I_{yy}$  and  $I_{zz}$ ) and off-principal (i.e.,  $I_{yz}$ ) axis directions. However, the rail element was designed as a doubly-symmetric member, such that the product of inertia value ( $I_{yz}$ ) was equal to zero. Several cross sections were investigated and included a circular tube, a solid circular bar, square tube, and solid square bar. As a result, the longitudinal tensile or compressive bending stresses ( $f_b$ ) resulting from loads in vertical and transverse directions could be summed

together. Similarly, the maximum deflection was calculated by using the vector addition to superpose vertical and transverse deflections, as shown in Equation 3.

$$M_{u} = M_{uy} + M_{uz}$$
(2)  

$$M_{uz} = \frac{wL^{2}}{8}$$

$$M_{uy} = \frac{(w + w_{ow})L^{2}}{8} + \frac{PL}{4}$$

$$\delta = \sqrt{\delta_{y}^{2} + \delta_{z}^{2}}$$
(3)  

$$\delta_{y} = \frac{1}{EI} \int M_{z} dx = \left(\frac{5(w + w_{ow})/12 \times (L \times 12)^{4}}{384EI} + \frac{P(L \times 12)^{3}}{48EI}\right)$$

$$\delta_{z} = \frac{1}{EI} \int M_{y} dx = \left(\frac{5w/12 \times (L \times 12)^{4}}{384EI}\right)^{2} + \left(\frac{5w/12 \times (L \times 12)^{4}}{384EI}\right)^{2}$$

Where:

∴ δ

 $M_u$  = Applied Bending Moment (lb-in.)

 $M_{uy}$  = Applied Bending Moment Acting in the Y-Axis (lb-in.)

 $M_{uz}$  = Applied Bending Moment Acting in the Z-Axis (lb-in.)

- w = 50 lb/ft Distributed Load (lb/in.)
- L = Post Spacing (in.)
- $w_{ow}$  = Rail Weight (lb/in.)
- P = 200 lb Concentrated Load
- $\delta$  = Deflection (in.)
- $\delta_y$  = Deflection in Vertical (i.e., Y-Axis) due to w,  $w_{ow}$ , and P
- $\delta_z$  = Deflection in Transverse (i.e., Z-Axis) due to w
- I = Moment of Inertia (in.<sup>4</sup>)

E = Young's Modulus (psi)

#### **4.1.2 Vertical Post Element**

The posts were subjected to a concentrated live load,  $P_{LL}$ , as defined in Equation 1. The concentrated live load shall be applied transversely at the center of gravity of the upper horizontal element. The post was assumed to act as a single cantilever beam. The bending moment and deflection of the post were calculated, as shown in Equations 4 and 5, respectively.

Several cross sections were investigated for the post, including a circular tube, circular bar, square tube, square bar, rectangular tube, and rectangular bar.

$$M_p = P_{LL}(\mathbf{h}) \tag{4}$$

$$\delta = \frac{P_{LL}h^3}{3EI} \tag{5}$$

Where:

 $\begin{array}{ll} M_p = & \text{Bending Moment in Post (lb-in.)} \\ P_{LL} = & \text{Post Point Live Load (lb)} \\ h = & \text{Distance from Ground to Center of Gravity of Upper} \\ & \text{Horizontal Element (in.)} \\ \delta = & \text{Deflection (in.)} \\ E = & \text{Young's Modulus (psi)} \\ I = & \text{Moment of Inertia (in.}^4) \end{array}$ 

#### 4.1.3 Infill, Mesh, and Spindle Element

Mesh elements were designed to withstand the  $15\text{-lb/ft}^2$  (718-N/m<sup>2</sup>) load defined in the AASHTO *LRFD Bridge Design Specifications*. Where spindles or other infill spanned between rail elements,  $15 \text{ lb/ft}^2$  (718 N/m<sup>2</sup>) was also used as a design load for these elements. Because the applied load on the infill, mesh, or spindles was much, much less than for the posts and beams, the design of these elements would not control the shape or appearance of the pedestrian rail design and thus were not considered for the initial concepts. Furthermore, these components did not provide any structural support to the pedestrian rail.

#### **4.2 Material Selection**

#### 4.2.1 Material Consideration

The materials considered for the initial design of the pedestrian-only rail structure included: (1) steel, (2) aluminum, (3) FRP, (4) PVC, (5) HDPE, and (6) wood. All material types had benefits and disadvantages. General properties of each material were ranked Very Low, Low, Medium, High, Very High, or Not Applicable (NA) or listed as Yes or No, as shown in Table 3. Steel, aluminum, and FRP provided high material strength, and the polymer options had lower strengths and were assumed to act more brittle during impacts. A summary of all relevant material properties is shown in Table 4.

				• •			
Consideration/	Material						
Condition	Steel <sup>1</sup>	Aluminum <sup>2</sup>	PVC	FRP	HDPE	Wood	
Bending Strength $(f_b)$	Very High	High	Low	Medium	Very Low	Very Low	
Modulus of Elasticity (E)	Very High	High	Low	Medium	Very Low	Medium	
Brittleness	Low	Medium	High	High	Medium	High	
Formability	Very High	Low	NA	NA	NA	NA	
Cost	Medium	High	Medium	Very High	Medium	Low	
Component Weight <sup>3</sup>	Medium	Very Low	High	Low	High	Very High	
Prefabricated Connections	Yes	Yes	Yes	Yes	No	No	
Corrosion Resistance	Medium	Very High	Very High	Very High	Very High	Low	
Temperature Degradation	Very Low	Very Low	High	Low	Very High	Very Low	
UV Exposure Degradation	Very Low	Very Low	High <sup>4</sup>	High <sup>4</sup>	High <sup>4</sup>	Very Low	

Table 3. General Material Comparisons

1 – ASTM A992 Steel

2 – 6061-T6 Aluminum

3 – Weight of cross sections which meet load requirements

4 – Can be treated or painted to resist UV degradation

Table 4. Relevant Material Properties [37]

Material	Bending Strength ( $f_b$ )		Young's N	Aodulus (E)	Density	
	(psi)	(kPa)	(ksi)	(MPa)	$(lb/ft^3)$	$(kg/m^3)$
Steel <sup>1</sup>	50,000	345,000	29,000	199,950	503	8,060
Aluminum <sup>2</sup>	40,000	276,000	10,000	68,950	169	2,710
FRP	24,000	165,000	2,320	16,000	108	1,730
PVC	14,450	100,000	400	2,760	90	1,440
HDPE	4,800	33,000	200	1,380	59	950
Wood	1,550	11,000	1,700	11,720	31	500

1 – ASTM A992 Steel

2-6061-T6 Aluminum

Further evaluation of each material type was necessary to determine which material would provide the greatest benefits while keeping the initial designs to a manageable set. Although many variables should be considered when choosing the most efficient material, the primary selection criteria were aesthetics, strength, weight, cost, and workability.

#### 4.2.2 Aluminum

Aluminum had many properties which were desirable for the fabrication of a pedestrian rail. Aluminum has a very high strength-to-density ratio and is highly resistant to corrosion. Depending on the rail design, prefabricated aluminum fittings are also available. One disadvantage is that aluminum is difficult to weld, and when welded, aluminum loses much of its strength near the site of the weld. However, aluminum may be heat-treated at an additional cost to retain its original strength. Another disadvantage is that aluminum is a relatively expensive material and may be a target for theft.

#### 4.2.3 Steel

Steel has very high strength material properties and is about three times denser than aluminum. Steel is easily welded and formed to a desired shape with little to no loss in material strength. Prefabricated steel fittings are available. To reduce the effects of corrosion, the steel must be galvanized. The cost of steel is typically cheaper than aluminum.

#### 4.2.4 Polyvinyl Chloride (PVC)

PVC is a very common material used for plumbing and private property fencing. PVC has a low material strength when compared to aluminum and steel, and is about one-sixth the density of steel. PVC is corrosion-resistant, but the material properties and appearance degrade with Ultra Violet (UV) exposure. The PVC material must be treated or painted to reduce the effects of UV exposure. PVC material strength is also affected by temperature. The stiffness of the PVC material is reduced at high temperatures, potentially resulting in large deformations at

warm temperatures. To reduce the temperature effects, PVC should be painted with a light color, preferably white. PVC has prefabricated fittings used for pipes, which may allow the material to work as a handrail system. PVC is very brittle under impact loading, specifically at low temperatures. The cost of PVC is in the medium range when compared to other materials.

#### 4.2.5 Fiber-Reinforced Polymer (FRP)

The material strengths of FRP are much higher than other polymers due to the added strength from the internal reinforcing fibers of the material. FRP has about one-fifth the density of steel. It is corrosion-resistant and acts brittle under impact loading. The cost of FRP is much higher than all other materials considered.

## 4.2.6 High Density Polyethylene (HDPE)

HDPE is very similar to PVC, but the material strengths are lower. HDPE is corrosionresistant and has about one-ninth the density of steel. It must be protected from UV degradation with paint or an additive. HDPE material strength is also affected by temperature. At high temperatures, the stiffness decreases. This could potentially result in large deformations at warm temperatures. To reduce the temperature effects, HDPE should be painted with a light color, preferably white.

#### 4.2.7 Wood (Douglas Fir)

Douglas fir was considered for this project because of its high strength properties. Douglas fir has about one-sixteenth the density of steel. Wood has a low ultimate bending strength due to variability in the cross section from imperfections, such as cracks and knots. Wood is readily available and relatively inexpensive.

#### **5 INITIAL CONCEPT DEVELOPMENT**

#### **5.1 Preliminary Concept**

After a comprehensive literature review was completed on existing pedestrian rail systems and other commercially available railings, twenty-five pedestrian rail concepts were considered, as shown in Appendix B. The geometry of the pedestrian rail was the main focus, such that all concepts met the AASHTO *LRFD Bridge Design Specifications* loading criteria required for a pedestrian barrier. As stated previously, various materials were considered and included steel, aluminum, PVC, wood, HDPE, and FRP. Material types were considered based on aesthetics, strength, weight, cost, and workability. The handrail, infill, and connections were not designed during the initial development phase. Only one rail segment is shown for each concept. However, all preliminary concepts could later be designed as either a long, continuous railing system or as individual segments.

#### **5.2 Refined Concepts**

Following a review of the preliminary concepts, several concepts and materials were eliminated. Further investigation showed that the cost of aluminum was comparable to steel. Thus, since aluminum would fracture upon impact more easily than steel is lighter weight, aluminum options were added. Due to the significant cost of FRP, it was eliminated. HDPE was eliminated due to its limited application and having a low material strength, especially at high temperatures. Many designs were not pursued based on aesthetics and feasibility of fabrication.

Seven preliminary concepts were further developed and included: two modular aluminum concepts (designated AM-1 and AM-2), one welded aluminum concept (designated AW2), two PVC concepts (designated PVC1 and PVC2), and two wood concepts (designated WOOD1 and WOOD2). The system details are described in the following sections, and components were obtained to fabricate prototype segments of each concept. Connections were specified, but

further development, such as weld details for applicable systems, were not designed during this phase.

#### 5.2.1 Design Concept AM-1

Concept AM-1 consisted of a modular aluminum system with vertical spindles welded to the horizontal rail. The modular assembly simplified installation. The aluminum material was lightweight for transportation and fabrication. The spindles may be solid or hollow aluminum cross sections. It was recommended that the spindles be clipped in or welded to both the center and bottom rails in order to reduce flying debris when impacted. Details of design concept AM-1 are shown in Figures 30 through 34. Photographs of the fabricated design are shown in Figures 35 and 36.

#### 5.2.2 Design Concept AM-2

Concept AM-2 was very similar to concept AM-1. The only change for this design was to use a 2-in. x 2-in. (51-mm x 51-mm) steel mesh in place of the vertical aluminum spindles. The mesh would require panel clips or welds at the connections to the center and bottom rail components in order to secure it in place. This option was presented to provide variability in aesthetics of this structural design. Details of design concept AM-2 are shown in Figures 37 through 41. This concept was not fabricated due to the similarity between concepts AM-1 and AM-2.

#### 5.2.3 Design Concept AW2

Concept AW2 utilized aluminum posts and rails with rectangular cross sections. Post-torail connections were welded at the connection surface. The connections were tack welded for illustrative purposes only. The aluminum material was lightweight for transportation and fabrication. The spindles may be solid or hollow aluminum cross sections. Spindles will need to be welded at both connections to the center and bottom rails. Details of design concept AW2 are shown in Figures 42 through 46. Photographs of the fabricated design are shown in Figures 47 and 48.

#### 5.2.4 Design Concept PVC1

Concept PVC1 consisted of a modular PVC system. The modular assembly simplified installation. Initial fabrication at the Midwest Roadside Safety Facility (MwRSF) utilized available plastic base connections, but it was determined that this base connection would not be as stable as desired. Thus, the base connection would need to be redesigned. The rail elements were overdesigned, as T-shaped PVC fittings for the connection between posts and rails were not available with two different diameters between the vertical and horizontal connection slots. It was noted during fabrication that the girth of concept PVC1 may reduce needed visibility near the side of the road. Details of design concept PVC1 are shown in Figures 49 through 53. Photographs of the fabricated design are shown in Figures 54 and 55.

## 5.2.5 Design Concept PVC2

Concept PVC2 utilized PVC posts and rails, with circular cut-out sections in the post at each post-to-rail connection. Horizontal rail elements were attached with a vertical steel reinforcing bar through the ends inside the PVC post to ensure that the rail elements did not shift individually within the system. A base connection for design concept PVC2 was not designed or fabricated. Fabrication of the PVC2 system was simplistic. Three variations utilized the same post-to-rail connection method with different post and rail sizes and segment geometry.

## 5.2.5.1 Design Concept PVC2-a

Concept PVC2-a was the original design that was fabricated with 4<sup>1</sup>/<sub>2</sub>-in. (114-mm) diameter rails. Details of design concept PVC2-a are shown in Figures 56 through 59. Photographs of the fabricated design are shown in Figures 60 and 61.

#### 5.2.5.2 Design Concept PVC2-b

Concept PVC2-b decreased the rail diameter to 2<sup>7</sup>/<sub>8</sub> in. (73 mm). The reduced cross section of the system resulted in an extra rail element added to meet the AASHTO requirement of 6-in. (152-mm) minimum spacing between elements. This alteration allowed for the post spacing to be increased from 54 in. (1,372 mm) to 60 in. (1,524 mm). Details of design concept PVC2-b are shown in Figures 62 through 66. Concept PVC2-b was not fabricated due to its similarity to PVC2-a, and to the 2<sup>1</sup>/<sub>2</sub>-in. (64-mm) diameter PVC pipe not being readily available at the time of fabrication.

#### 5.2.5.3 Design Concept PVC2-c

The design of concept PVC2-c altered that of concept PVC2-b to utilize local, readilyavailable material, since the  $2\frac{7}{8}$ -in. (73-mm) diameter PVC was not readily available. The post diameter was reduced from  $6\frac{5}{8}$  in. (168 mm) to  $4\frac{1}{2}$  in. (114 mm), and the rail diameter was reduced from  $2\frac{7}{8}$  in. (73 mm) to  $2\frac{3}{8}$  in. (60 mm). The cross section changes resulted in a post spacing reduction from 60 in. (1,524 mm) to 48 in. (1,219 mm) to maintain strength requirements. Details of design concept PCV2-c are shown in Figures 67 through 71. Photographs of the fabricated design are shown in Figures 72 and 73.

#### **5.2.6 Design Concept WOOD1**

Concept WOOD1 consisted of Douglas Fir wood post and rail elements. The design details specified that a steel fitting be used for the post-to-rail connection, but this connection was not readily available and was altered during fabrication. Instead of a steel bracket, 1½-in. (38-mm) diameter steel conduit was used as a post-to-rail connection. The post and rail were auger-drilled, and then the conduit was set approximately 1½ in. (38 mm) deep within these holes to secure the connection. The solid steel spindles were replaced with a ½-inch (13-mm) steel conduit during fabrication to reduce weight and cost of the section. Details of design

concept WOOD1 are shown in Figures 74 through 78. Photographs of the fabricated design are shown in Figures 79 and 80.

#### 5.2.7 Design Concept WOOD2

Concept WOOD2 utilized Douglas Fir for post and rail elements. Although a square cutout was initially considered for inserting the rails into the posts, fabrication would be more difficult than circular cutouts. Thus, 3<sup>1</sup>/<sub>2</sub>-in. (89-mm) round holes were drilled into the post, and the square rail ends were cut down to a 3<sup>1</sup>/<sub>2</sub>-in. (89-mm) diameter head for easy insertion into the post cutout. Details of design concept WOOD2 are shown in Figures 81 through 85. Photographs of the fabricated design are shown in Figures 86 and 87.

#### **5.3 Discussion**

The initial pedestrian rail concepts were submitted to the project sponsor for review and comment as well as to select preferred concepts based on aesthetics, cost, installation, maintenance, and sight lines. Some of the sponsor's concerns included the possibility for the rail to obstruct a driver's visual line of sight at critical locations (such as near intersections), the need to treat a wood railing system on a regular basis to prevent degradation, the labor of heat-treating welded aluminum, and the possibility of system components fracturing away from the frame and becoming projectile hazards to pedestrians or drivers. The comments were considered and applied to eliminate numerous concepts. The concepts made from PVC material were eliminated mainly due to lack of aesthetic appeal, difficulty in the design and fabrication of post and rail connections, and instability of each PVC segment. The Douglas Fir wood concepts were eliminated due to the concern of long-term durability, warping of the wood sections, and splinter hazards to pedestrians, vehicle occupants, and bystanders. After eliminating the concepts configured with PVC and Douglas Fir materials, both modular and welded aluminum railing systems were pursued further.

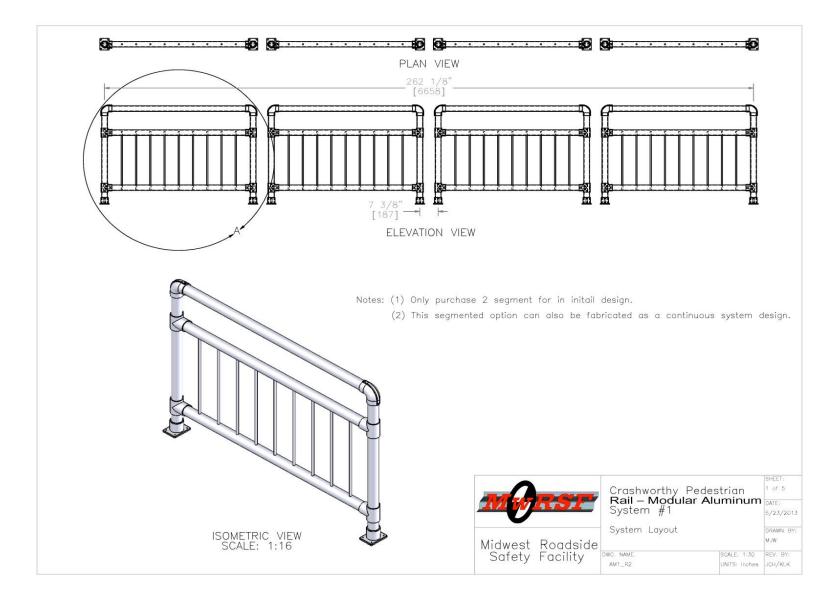


Figure 30. Aluminum Modular Rail with Spindles, Design Concept AM-1 (Sheet 1 of 5)

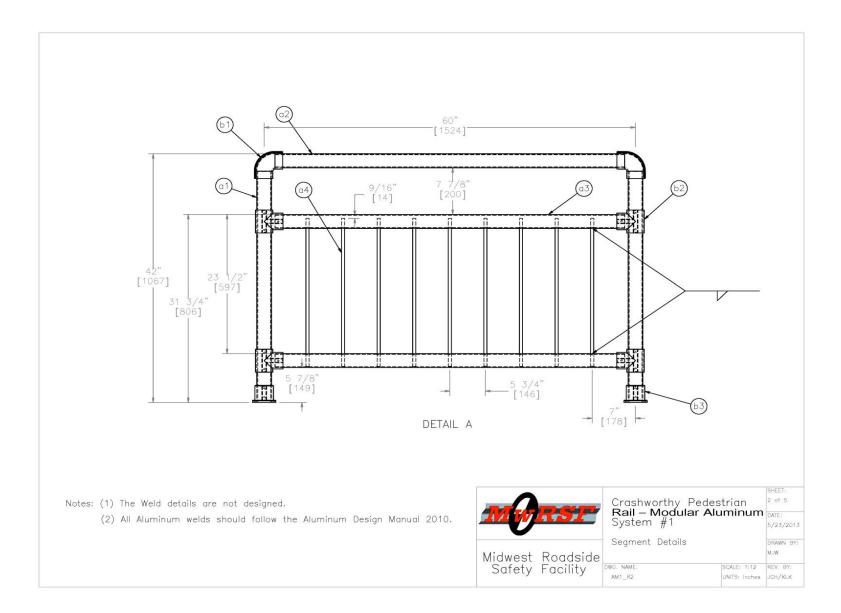


Figure 31. Aluminum Modular Rail with Spindles, Design Concept AM-1 (Sheet 2 of 5)

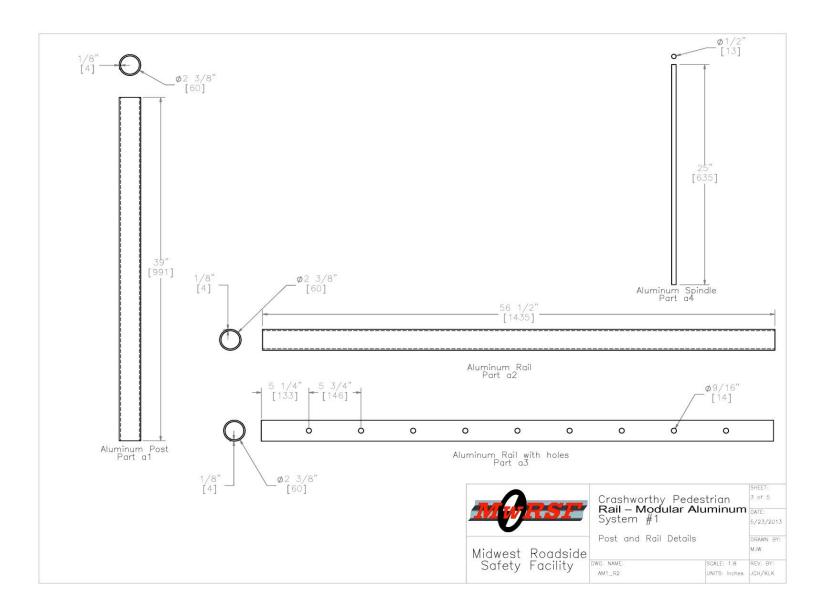


Figure 32. Aluminum Modular Rail with Spindles, Design Concept AM-1 (Sheet 3 of 5)

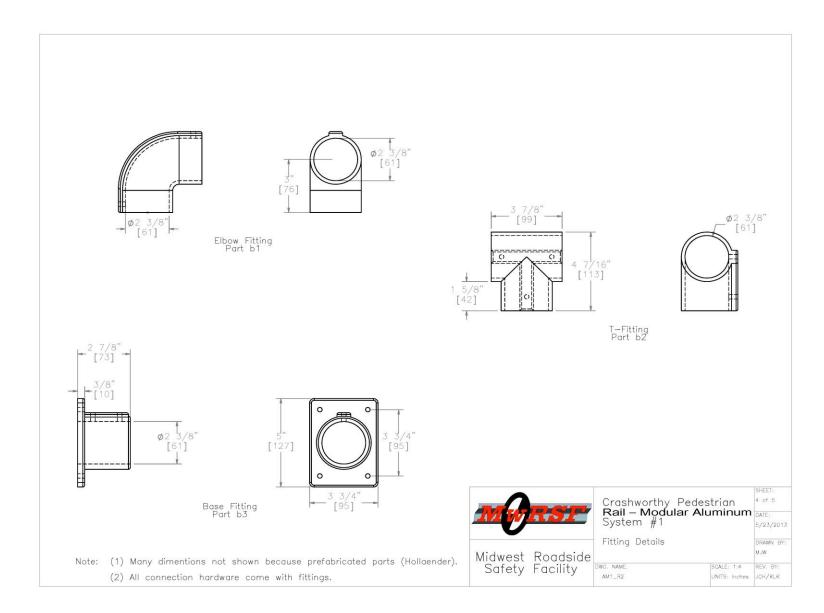


Figure 33. Aluminum Modular Rail with Spindles, Design Concept AM-1 (Sheet 4 of 5)

ii				and a start of strength to	
ltem No.	QTY.	Description	Material Spec	Hollander Part	Hardware Guide
a2	4	2" Dia. Schedule 40 Pipe Rail, 56 1/2" long	6061–T6 Aluminum	<del></del> :	
a1	8	2" Dia. Schedule 40 Pipe Post, 39" long	6061-T6 Aluminum		-
a3	8	2" Dia. Schedule 40 Pipe Rail with Holes, 56 1/2" long	6061-T6 Aluminum	_	
a4	36	1/2" Dia. Spindle, 25" long	6061-T6 Aluminum		3
b1	8	2" Elbow-Fitting	6061-T6 Aluminum	No. 3 Elbow	
b2	16	2" T-Fitting	6061–T6 Aluminum	No. 5 Tee	
b3	8	2" Base-Fitting	6061-T6 Aluminum	No. 47 Base	

Notes: (1) All aluminum pipe properties, dementions, and prices came from Metals Depot (www.metalsdepot.com/).

- (2) All aluminum fittings are prefabricated components from Hollaender Speed-Rail (www.hollaender.com/?page=speedrail).
- (3) There are alternate (heavy duty) fittings for stability if needed.
- (4) Hollaender may prefabricate the rail with spindles.

		Crashworthy Peo Rail – Modular	destrian	SHEET: 5 of 5
	177.4	System #1	Aluminum	DATE: 5/23/2013
Midwest	Roadside	Description of View		DRAWN BY: MJW
Safety		DWG. NAME, AM1_R2	SCALE: 1:12 UNITS: Inches	REV. BY: JCH/KLK

Figure 34. Aluminum Modular Rail with Spindles, Design Concept AM-1 (Sheet 5 of 5)



Figure 35. Aluminum Modular Rail with Spindles, Design Concept AM-1



Figure 36. Aluminum Modular Rail with Spindles, Design Concept AM-1

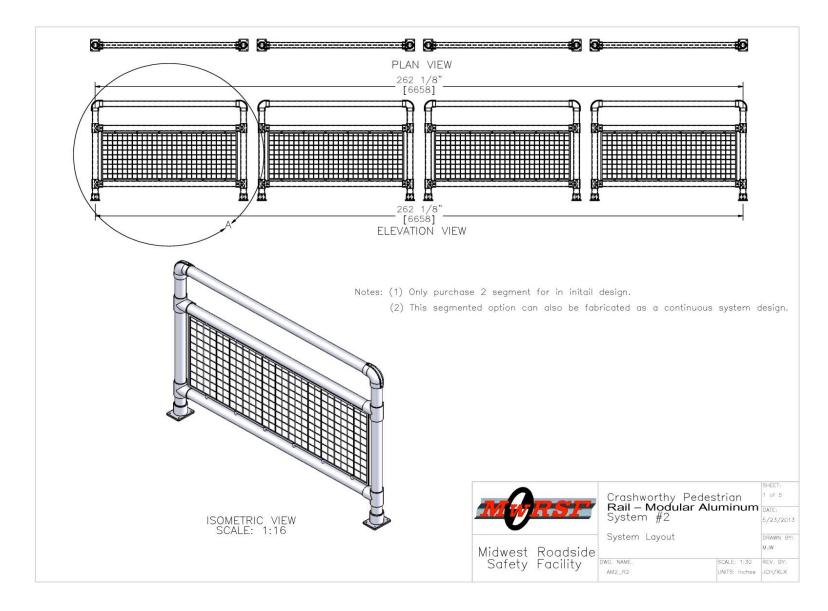


Figure 37. Aluminum Modular Rail with Wire Mesh, Design Concept AM-2 (Sheet 1 of 5)

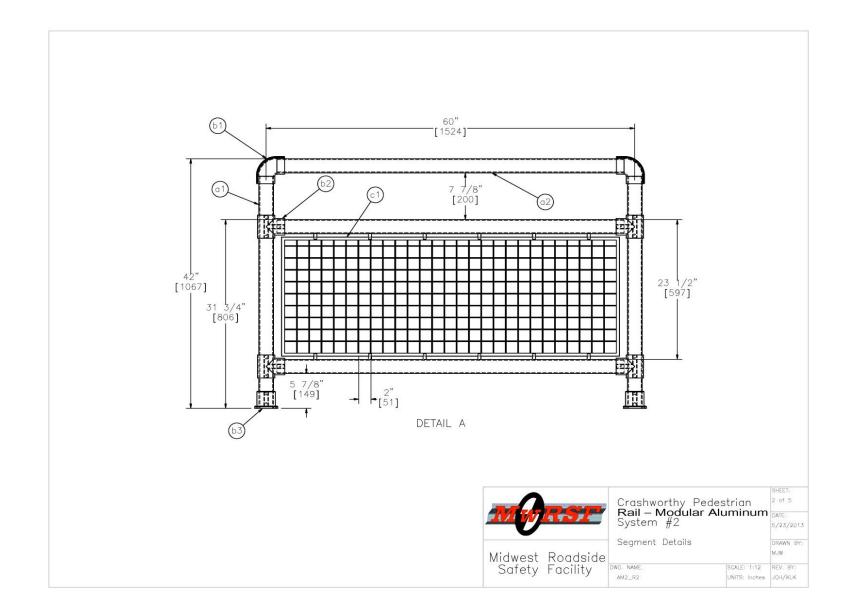


Figure 38. Aluminum Modular Rail with Wire Mesh, Design Concept AM-2 (Sheet 2 of 5)

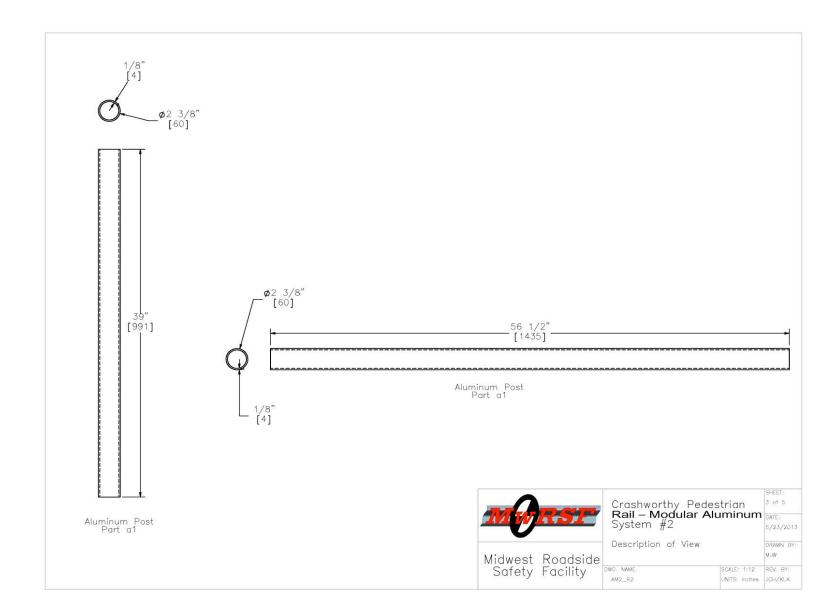


Figure 39. Aluminum Modular Rail with Wire Mesh, Design Concept AM-2 (Sheet 3 of 5)

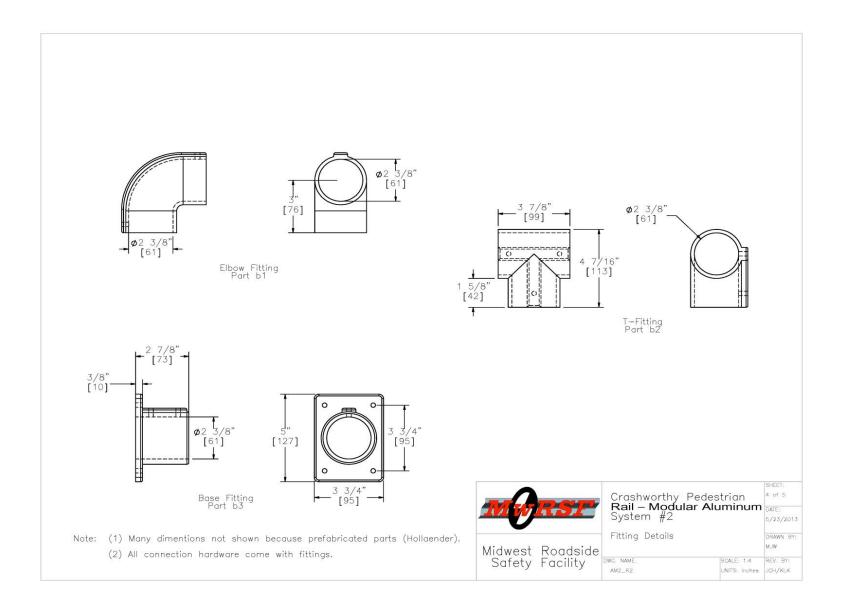


Figure 40. Aluminum Modular Rail with Wire Mesh, Design Concept AM-2 (Sheet 4 of 5)

		oradiniarity readduita	n Rail — Segmented Aluminum wit		
Item No.	QTY.	Description	Material Spec	Hollander Part	Hardware Guide
a1	8	2" Dia. Schedule 40 Pipe Post, 39" long	6061-T6 Aluminum		
a2	12	2" Dia. Schedule 40 Pipe Rail, 56 1/2" long	6061-T6 Aluminum		
b1	8	2" Elbow-Fitting	6061-T6 Aluminum	No. 3 Elbow	
b2	16	2" T–Fitting	6061-T6 Aluminum	No. 5 Tee	
b3	8	2" Base-Fitting	6061-T6 Aluminum	No. 47 Base	
c1	4	55x20" Steel Wire Mesh, 2x2" Gaps	A36 Steel		

Notes: (1) All aluminum pipe properties, dementions, and prices came from Metals Depot (www.metalsdepot.com/).

- (2) All aluminum fittings are prefabricated components from Hollaender Speed-Rail (www.hollaender.com/?page=speedrail).
- (3) There are alternate (heavy duty) fittings for more stability if needed.



Figure 41. Aluminum Modular Rail with Wire Mesh, Design Concept AM-2 (Sheet 5 of 5)

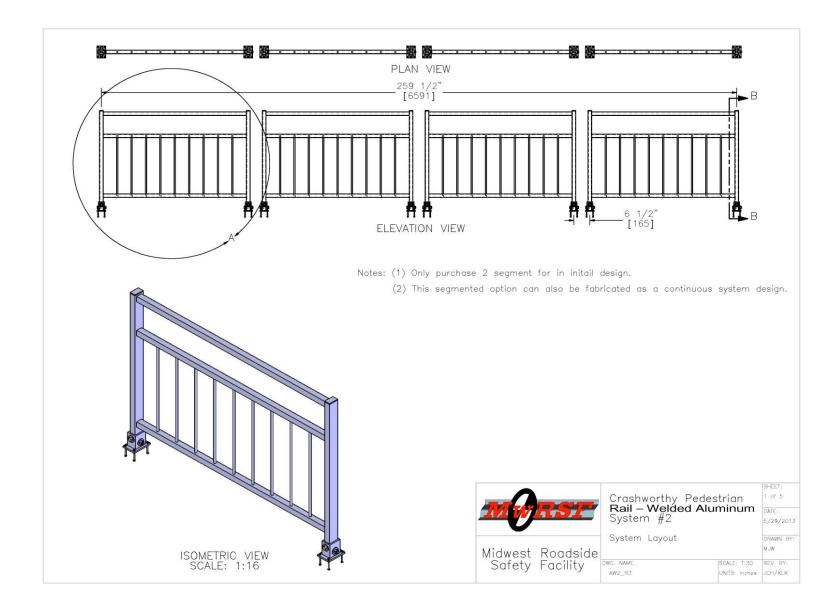


Figure 42. Welded Aluminum Rail, Design Concept AW2 (Sheet 1 of 5)

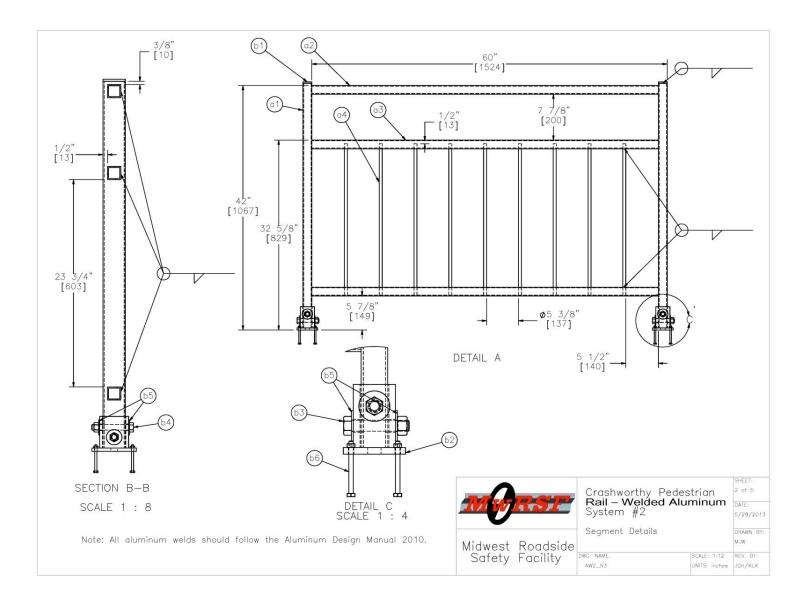


Figure 43. Welded Aluminum Rail, Design Concept AW2 (Sheet 2 of 5)

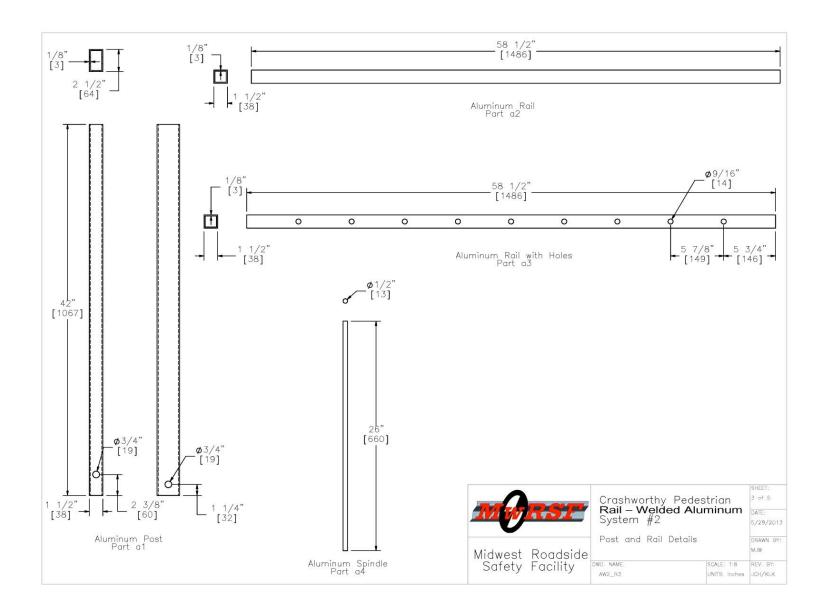


Figure 44. Welded Aluminum Rail, Design Concept AW2 (Sheet 3 of 5)

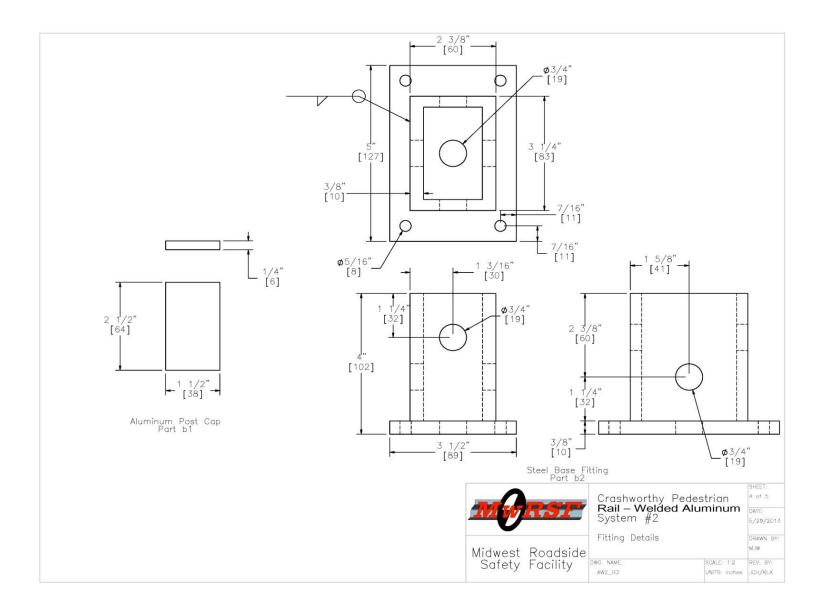


Figure 45. Welded Aluminum Rail, Design Concept AW2 (Sheet 4 of 5)

ltem No.	QTY.	Description	Material Spec	Hardware Guide
a1	8	2 1/2 x 1 1/2 x 1/8" Aluminum Post - 42" long	6061-T6	
a2	4	1 1/2 x 1 1/2 x 1/8" Aluminum Rail — 58 1/2" long	6061-T6	
aЗ	8	1 1/2 x 1 1/2 x 1/8" Aluminum Rail with holes - 58 1/2" long	6061-T6	
a4	36	1/2" Dia. Aluminum Spindle — 26" long	6061-T6	
b1	8	Aluminum Post Cap	6061-T6	
b2	8	Aluminum Post Base Fitting	6061-T6	
b3	8	5/8" Dia. x 3 1/4" Long Hex Head Bolt and Nut	A307	FBX16a
b4	8	5/8" Dia. x 4 1/4" Long Hex Head Bolt and Nut	A307	FBX16a
b5	32	5/8" Dia. Flat Washer	ASTM F844 or Grade 2 Steel	FWC16a
b6	32	1/4" Dia. 3" long Hex Bolt and Nut	A307	FBX06a

		Crashworthy	/ Pedestrian <b>Ied Aluminum</b>	SHEET: 5 of 5
		System #2	led Aluminum	DATE: 5/29/2013
V		Bill of Materia	19	DRAWN BY:
Midwest	Roadside	Diff of Materia		MJW
Safety	Facility	DWG: NAME. AW2_R3	SCALE: None UNITS: Inches	REV. BY: JCH/KLK

Figure 46. Welded Aluminum Rail, Design Concept AW2 (Sheet 5 of 5)



Figure 47. Welded Aluminum Rail, Design Concept AW2







January 18, 2016 MwRSF Report No.TRP-03-321-15

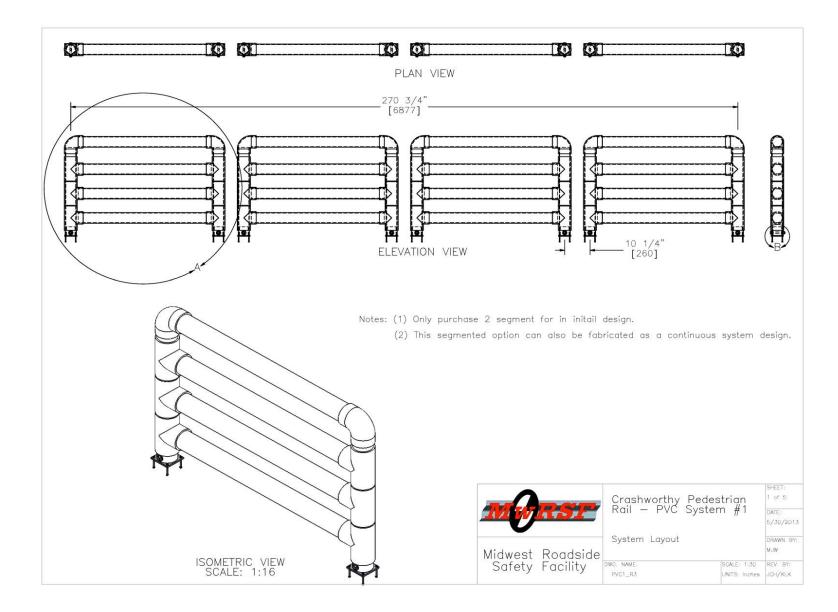


Figure 49. Modular PVC Rail, Design Concept PVC1 (Sheet 1 of 5)

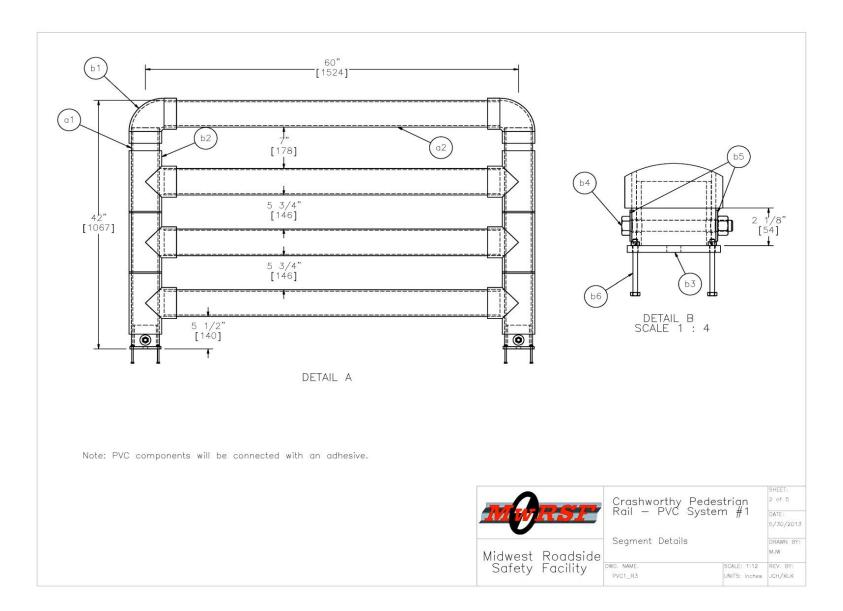


Figure 50. Modular PVC Rail, Design Concept PVC1 (Sheet 2 of 5)

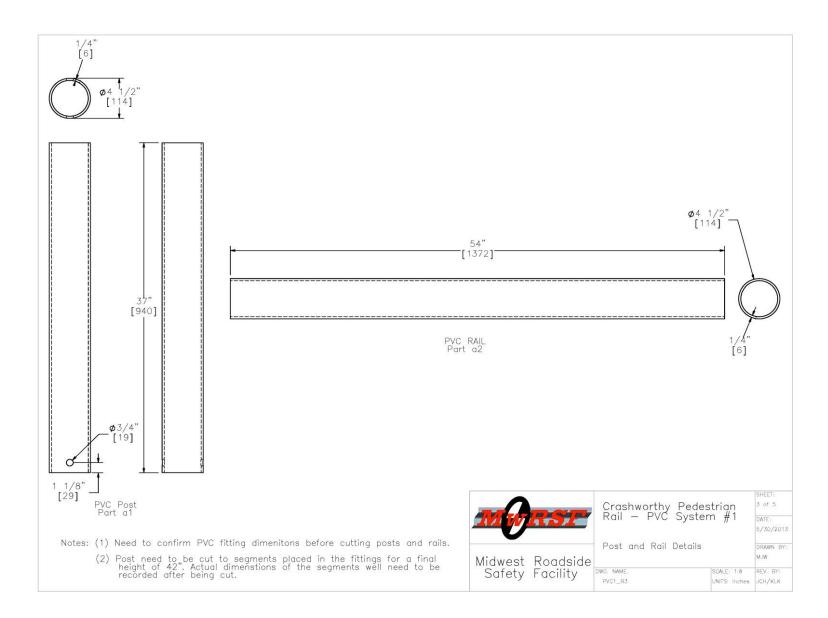


Figure 51. Modular PVC Rail, Design Concept PVC1 (Sheet 3 of 5)

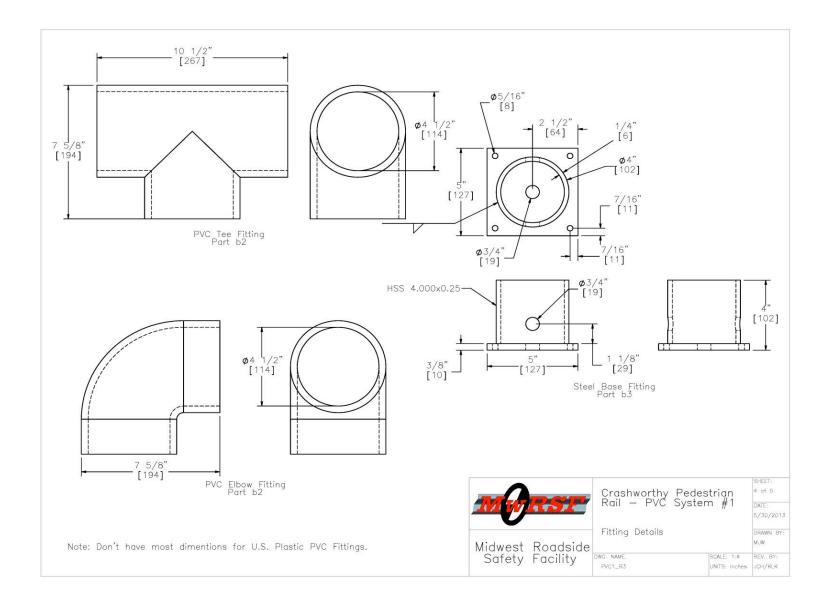


Figure 52. Modular PVC Rail, Design Concept PVC1 (Sheet 4 of 5)

Item No.	QTY.	Description	Material Spec	U.S. Plastic Parts	Hardware Guide
a1	8	4" Dia. Schedule 40 Post Pipe, 37" long	PVC		
a2	16	4" Dia. Schedule 40 Rail Pipe, 54" long	PVC		
b1	8	4" Dia. Elbow—Fitting	PVC	#28410	
b2	24	4" Dia. Tee Fitting	PVC	#28434	
b3	8	Post Base Fitting	A36		
b4	8	5/8" Dia. x 5 1/2" Long Hex Head Bolt and Nut	A307		FBX16a
b5	16	5/8" Dia. Flat Washer	ASTM F844 or Grade 2 Steel		FWC16a
b6	32	1/4" Dia. 3" long Hex Bolt and Nut	A307		FBX06a

Notes: (1) All PVC pipe and fitting properties, dementions, and prices came from U.S. Plastic (www.usplastic.com).

(2) All pipe and fitting were desined with schedule 40. Schedule 80 is also avalible if needed.

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Figure 53. Modular PVC Rail, Design Concept PVC1 (Sheet 5 of 5)



Figure 54. Modular PVC Rail, Design Concept PVC1

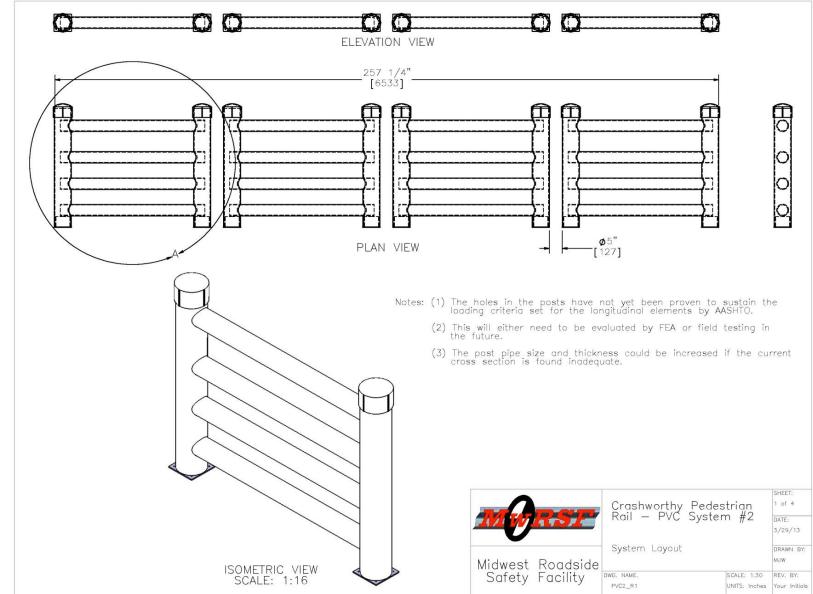






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Figure 55. Modular PVC Rail, Design Concept PVC1



MJW SCALE: 1:30 REV. BY: UNITS: Inches Your Initials

Figure 56. Modular PVC Rail, Design Concept PVC2-a (Sheet 1 of 4)

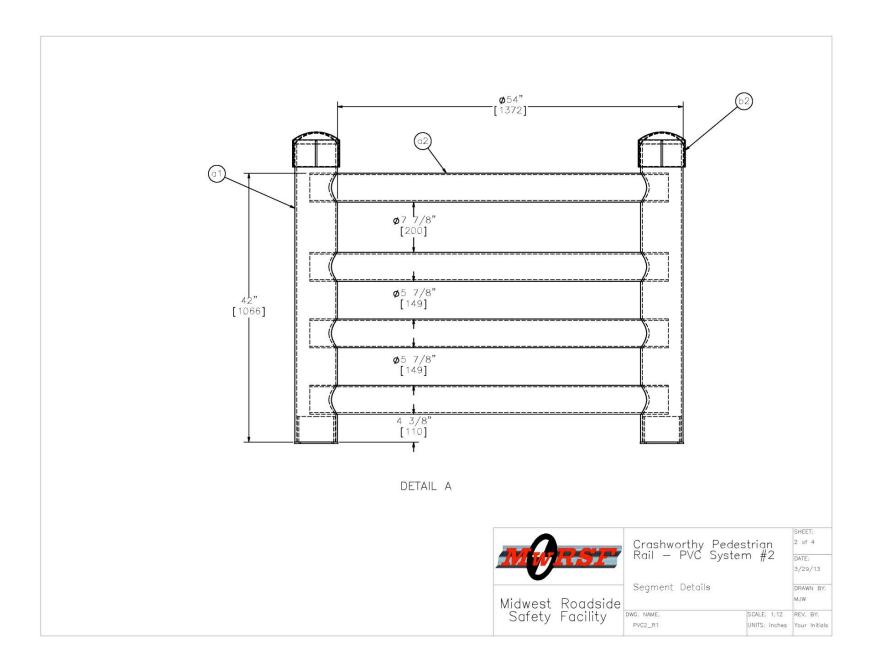
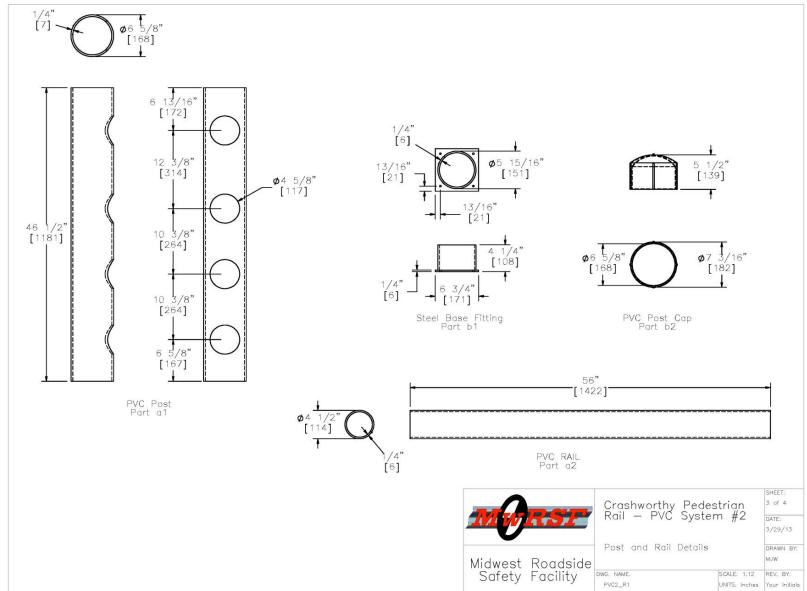


Figure 57. Modular PVC Rail, Design Concept PVC2-a (Sheet 2 of 4)



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			Crashworthy Pedestrian	Rail – Segmented PVC	
ltem No.	QTY.	Description	Material Spec	U.S. Plastic Parts	File Name
a1	8	6" Dia. End Post Pipe, 46.5" long	PVC	#26311	PVC2_5x.258x44.5_post
۵2	16	4" Dia. Rail Pipe, 56" long	PVC	#26310	PVC2_4x.237x56_rail
b1	8	Post Base Fitting	A36	-	PVC2_base_fitting_A36
b2	8	6"Dia. Post-Cap-Fitting	PVC	#28447	PVC2_6_post-cap

Notes: (1) All PVC pipe and fitting properties, dementions, and prices came from U.S. Plastic (www.usplastic.com).

(2) All pipe and fitting were desined with schedule 40. Schedule 80 is also availble if needed.

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		Crashworthy Pedes Rail — PVC Syster	trian	SHEET: 4 of 4
		Rail — PVC Syster	n #2	DATE: 3/29/13
Midwest	Roadside	Bill of Materials		DRAWN BY: MJW
	Facility	DWG. NAME. PVC2_R1	SCALE: None UNITS: Inches	REV. BY: Your Initial

January 18, 2016 MwRSF Report No.TRP-03-321-15

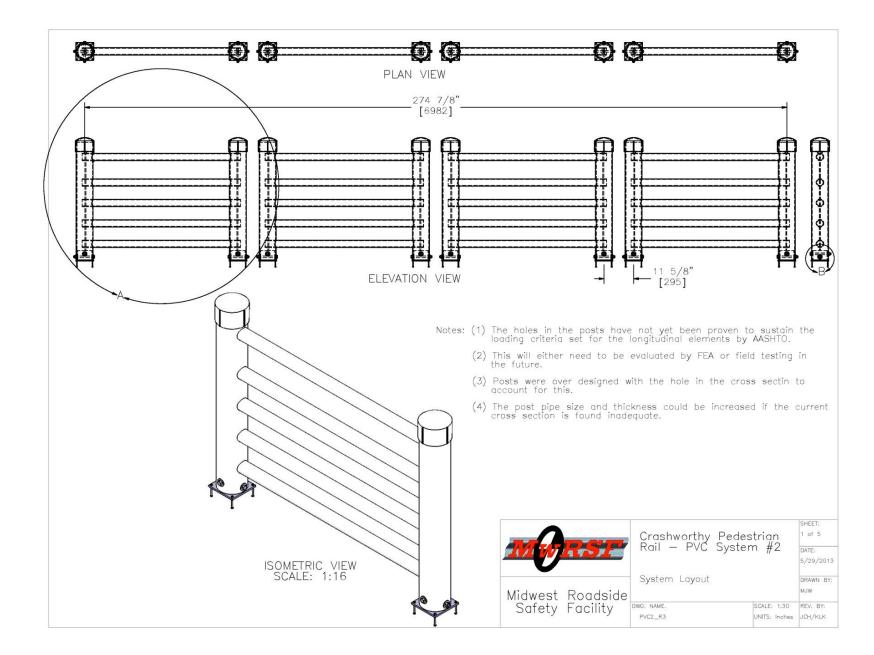


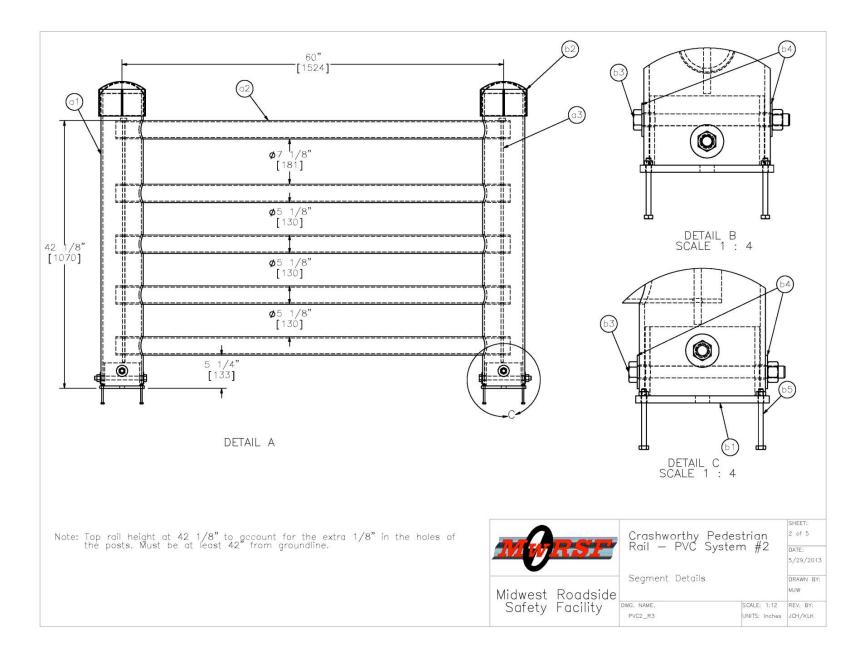
Figure 60. Modular PVC Rail, Design Concept PVC2-a





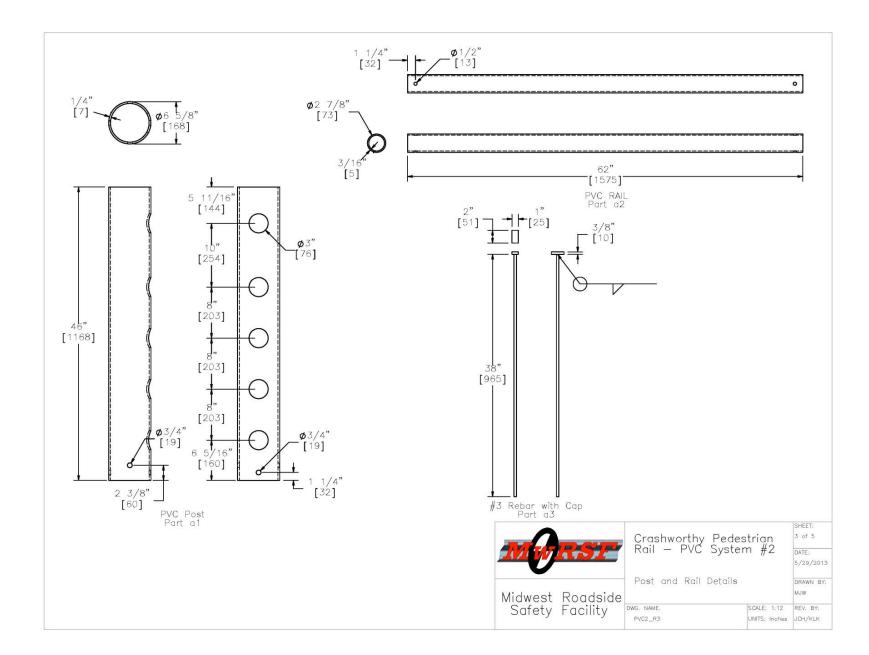
Figure 61. Modular PVC Rail, Design Concept PVC2-a





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Figure 63. Modular PVC Rail, Design Concept PVC2-b (Sheet 2 of 5)



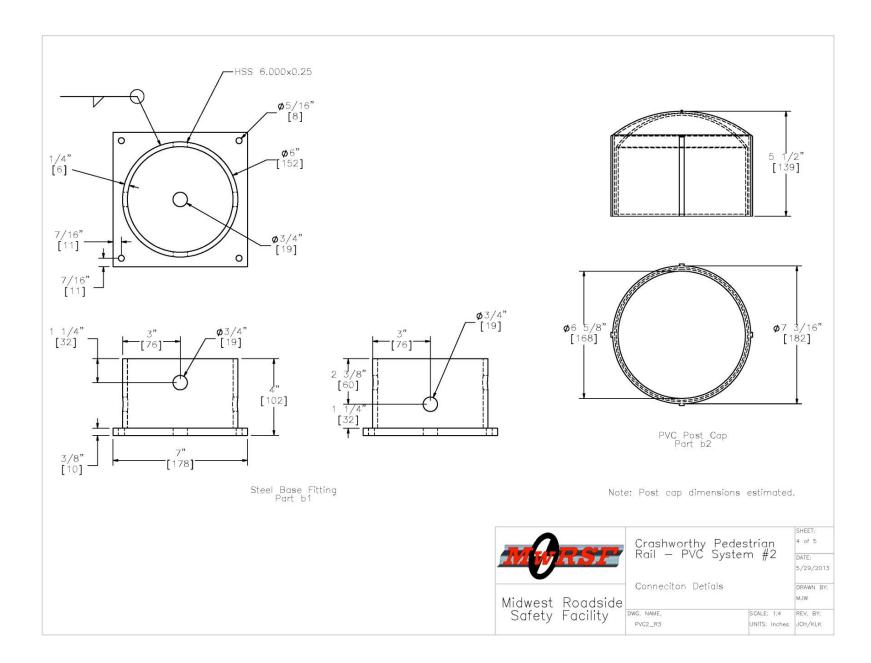


Figure 65. Modular PVC Rail, Design Concept PVC2-b (Sheet 4 of 5)

		Crashworthy Pede	estrian Rail — Segmented PVC		
ltem No.	QTY.	Description	Material Spec	U.S. Plastic Parts	Hardware Guide
a1	8	6" Dia. Schule 40 End Post Pipe, 46" long	PVC	#26508	
a2	20	2 1/2" Dia. Schedule 40 Rail Pipe, 62" long	PVC	#26511	
a3	8	#3 Sraight Rebar with cap, 38" long	Grade 60		
b1	8	Post Base Fitting	A36	<u>121</u>	
b2	8	6" Dia. Post-Cap-Fitting	PVC	#28447	
b3	16	5/8" Dia. x 7 5/8" Long Hex Head Bolt and Nut	A307		FBX16a
b4	32	5/8" Dia. Flat Washer	ASTM F844 or Grade 2 Steel		FWC16a
b5	32	1/4" Dia. 3" long Hex Bolt and Nut	A307		FBX06a

Notes: (1) All PVC pipe and fitting properties, dementions, and prices came from U.S. Plastic (www.usplastic.com).

(2) All pipe and fitting were desined with schedule 40. Schedule 80 is also availble if needed.

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		Crashworthy Pedestrian Rail — PVC System #2	SHEET: 5 of 5
	AND A	Rail — PVC System #2	DATE: 5/29/201
Midwest	Roadside	Bill of Materials	DRAWN BY MJW

Figure 66. Modular PVC Rail, Design Concept PVC2-b (Sheet 5 of 5)

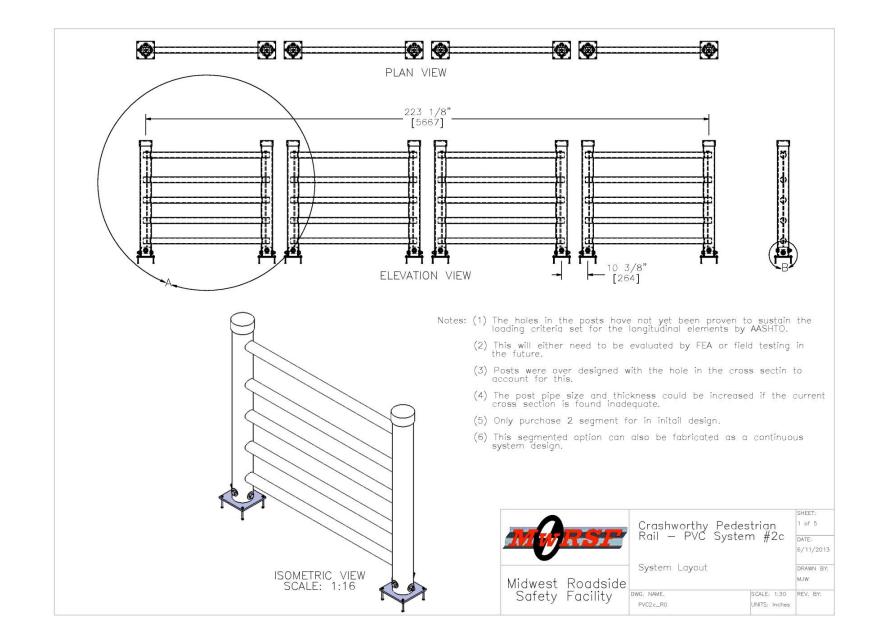


Figure 67. Modular PVC Rail, Design Concept PVC2-c (Sheet 1 of 5)

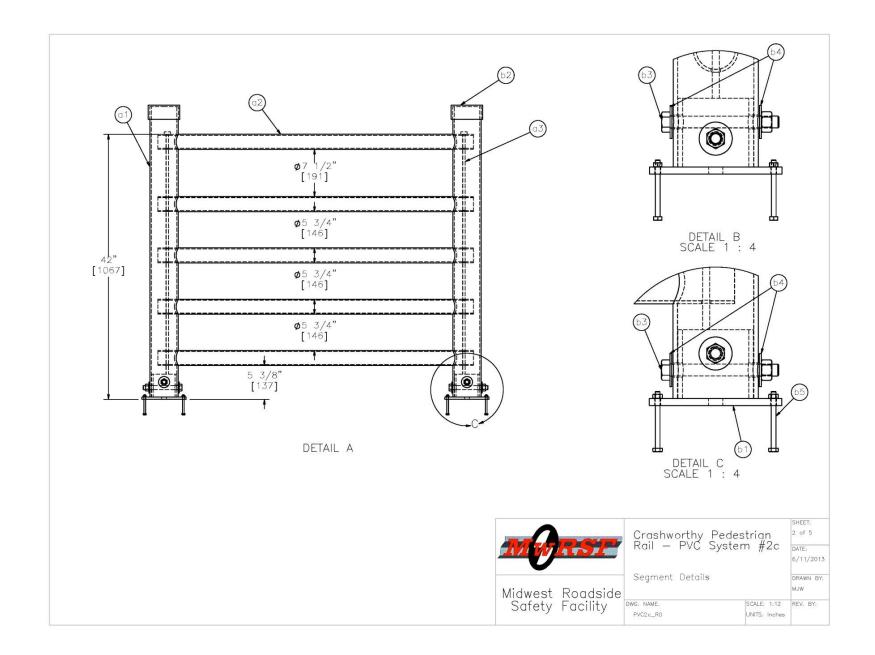


Figure 68. Modular PVC Rail, Design Concept PVC2-c (Sheet 2 of 5)

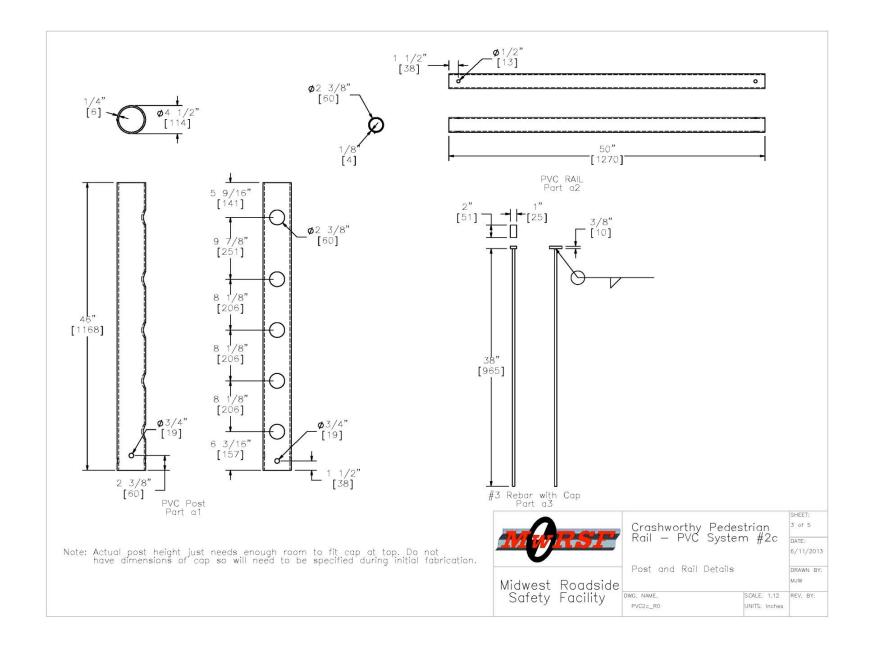
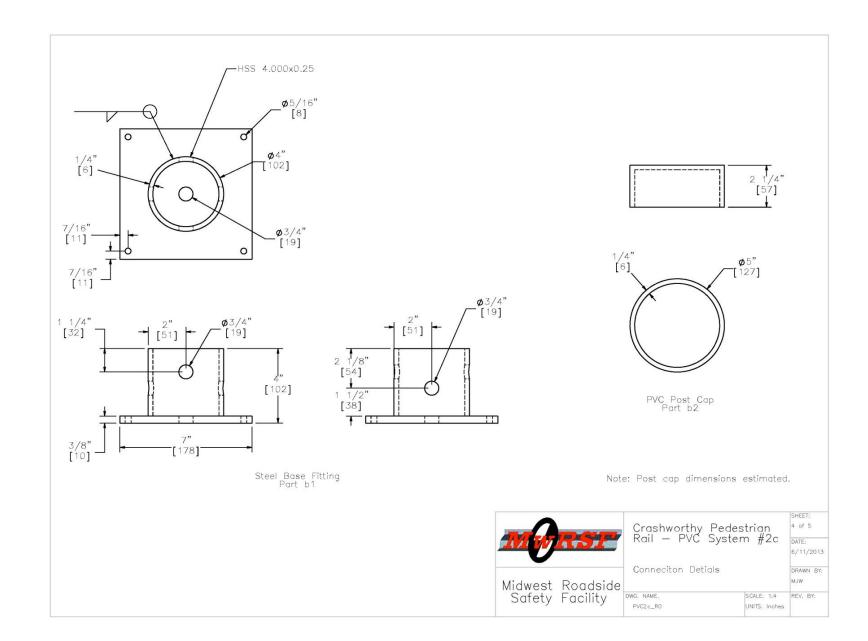


Figure 69. Modular PVC Rail, Design Concept PVC2-c (Sheet 3 of 5)

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Crashworthy Pedestrian Rail — Segmented PVC					
ltem No.	QTY.	Description	Material Spec	Hardware Guide	
a1	8	4" Dia. Schule 40 End Post Pipe, 46" long	PVC		
a2	20	2 1/2" Dia. Schedule 40 Rail Pipe, 50" long	PVC		
aЗ	8	#3 Sraight Rebar with cap, 38" long	Grade 60		
b1	8	Post Base Fitting	A36		
b2	8	6" Dia. Post-Cap-Fitting	PVC		
b3	16	5/8" Dia. x 6 5/8" Long Hex Head Bolt and Nut	A307	FBX16a	
b4	32	5/8" Dia. Flat Washer	ASTM F844 or Grade 2 Steel	FWC16a	
b5	32	1/4" Dia. 3" long Hex Bolt and Nut	A307	FBX06a	

Notes: (1) All PVC pipe and fitting properties, dementions, and prices came from U.S. Plastic (www.usplastic.com).

(2) All pipe and fitting were desined with schedule 40. Schedule 80 is also availble if needed.

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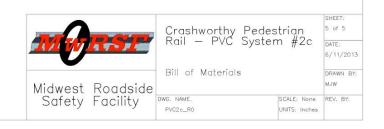




Figure 72. Modular PVC Rail, Design Concept PVC2-c



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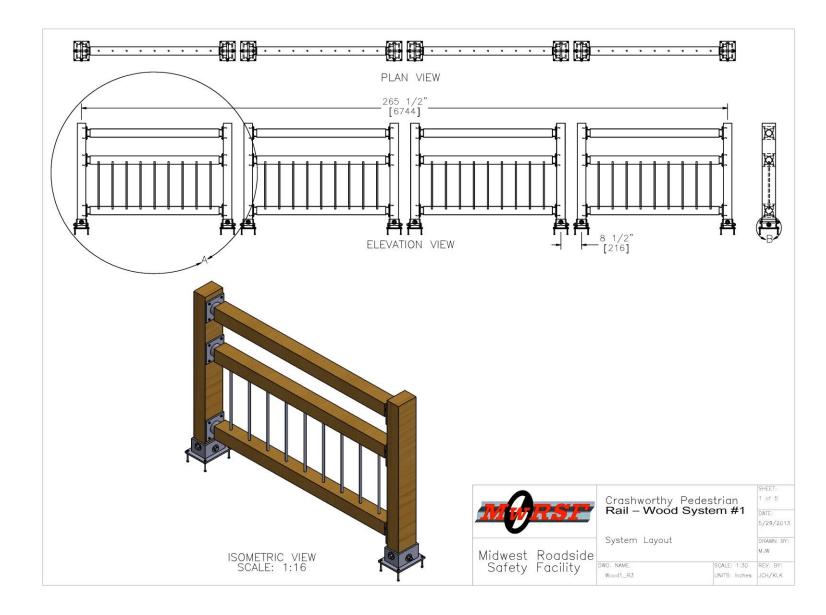


Figure 74. Modular Wood Rail, Design Concept WOOD1 (Sheet 1 of 5)

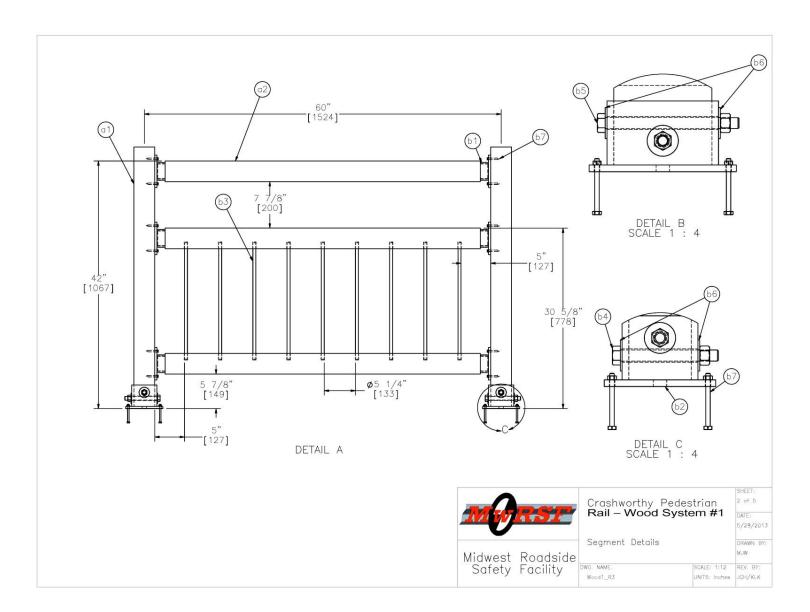


Figure 75. Modular Wood Rail, Design Concept WOOD1 (Sheet 2 of 5)

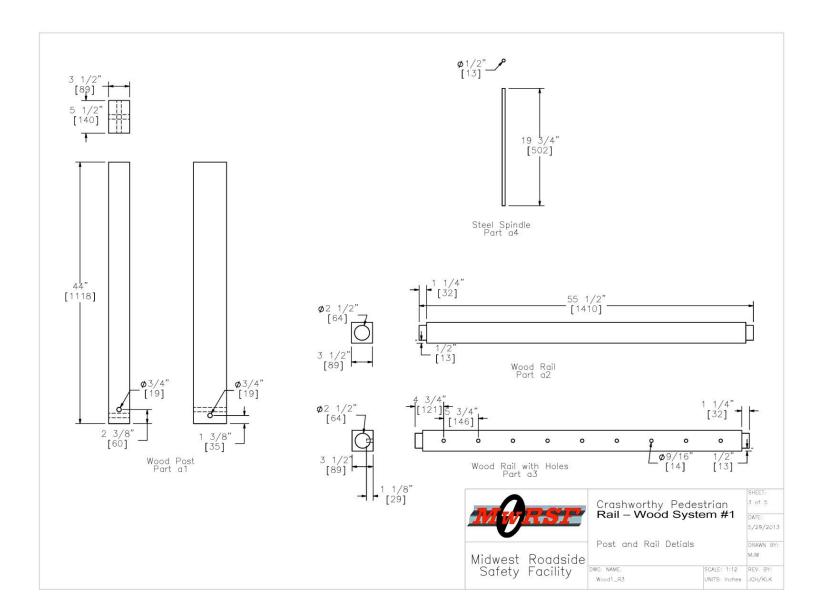


Figure 76. Modular Wood Rail, Design Concept WOOD1 (Sheet 3 of 5)

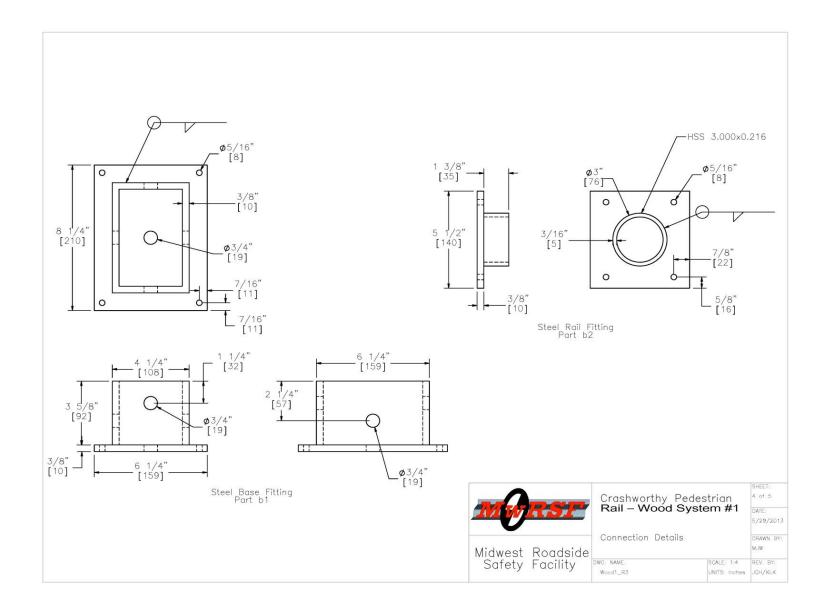


Figure 77. Modular Wood Rail, Design Concept WOOD1 (Sheet 4 of 5)

Item No.	QTY.	Description	Material Spec	Hardware Guide
a1	8	4x6" Wood Post - 44" long	Grade Dense No. 1 Douglas Fir-Larch	
a2	4	4x4" Wood Rail - 55 1/2" long	Grade Dense No. 1 Douglas Fir-Larch	
a3	8	4x4" Wood Rail with Holes - 55 1/2" long	Grade Dense No. 1 Douglas Fir-Larch	
b1	24	Steel Rail Fitting	A36	
b2	8	Steel Base Fitting	A36	
b3	36	1/2" Dia. Steel Spindle - 19 3/4" long	A36	
b4	8	5/8" Dia. x 5 3/8" Long Hex Head Bolt and Nut	A307	FBX16a
b5	8	5/8" Dia. x 7 3/8" Long Hex Head Bolt and Nut	A307	FBX16a
b6	32	5/8" Dia. Flat Washer	ASTM F844 or Grade 2 Steel	FWC16a
Ь7	32	1/4" Dia. 3" long Hex Bolt and Nut	A307	FBX06a
	96	1/4" Dia. x 1 3/4" Long Hex Head Lag Screw	A307	

		Crashworthy Pedestrian Rail – Wood System #1	SHEET: 5 of 5
	ATT I	Rail – Wood System #1	DATE: 5/29/2013
V	t Roadside	Bill of Materials	DRAWN BY:
Midwest	Roadside		WJW

Figure 78. Modular Wood Rail, Design Concept WOOD1 (Sheet 5 of 5)



Figure 79. Fabricated Refined Design Concept WOOD1

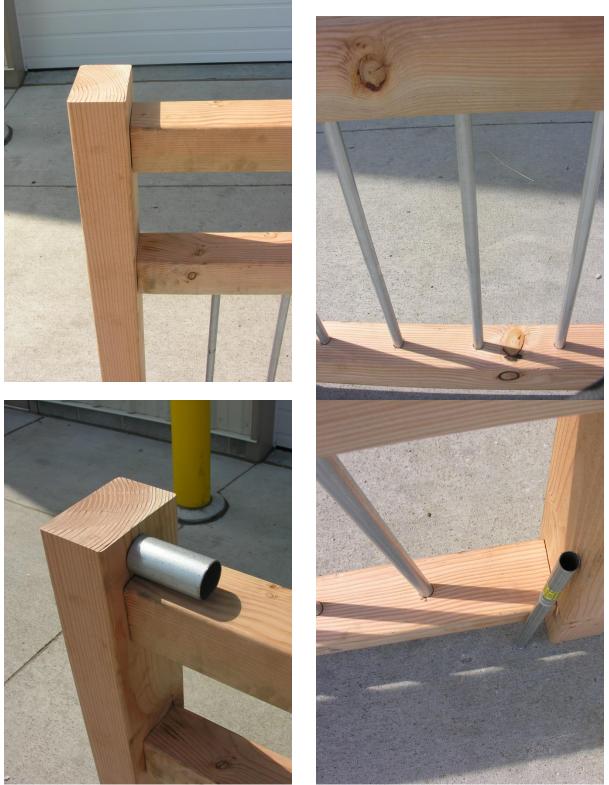


Figure 80. Fabricated Refined Design Concept WOOD1

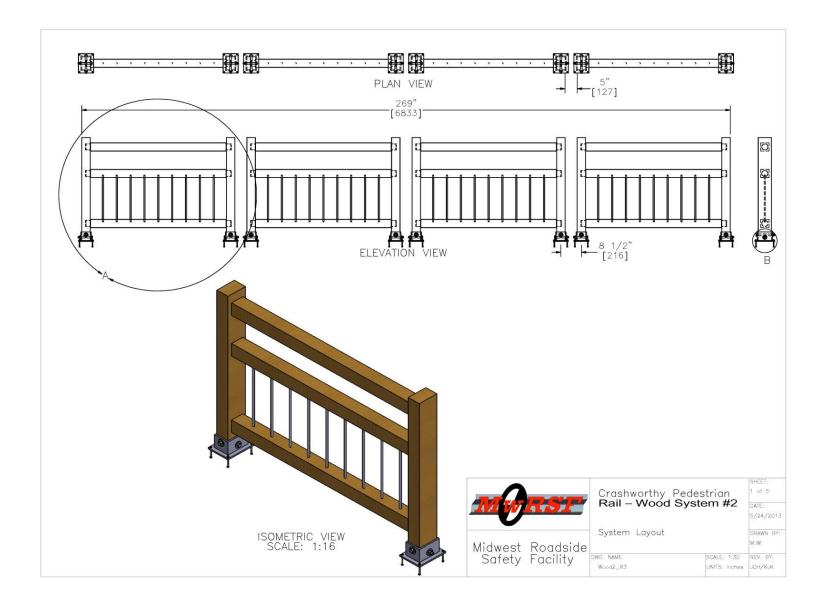


Figure 81. Cutout Wood Rail, Design Concept WOOD2 (Sheet 1 of 5)

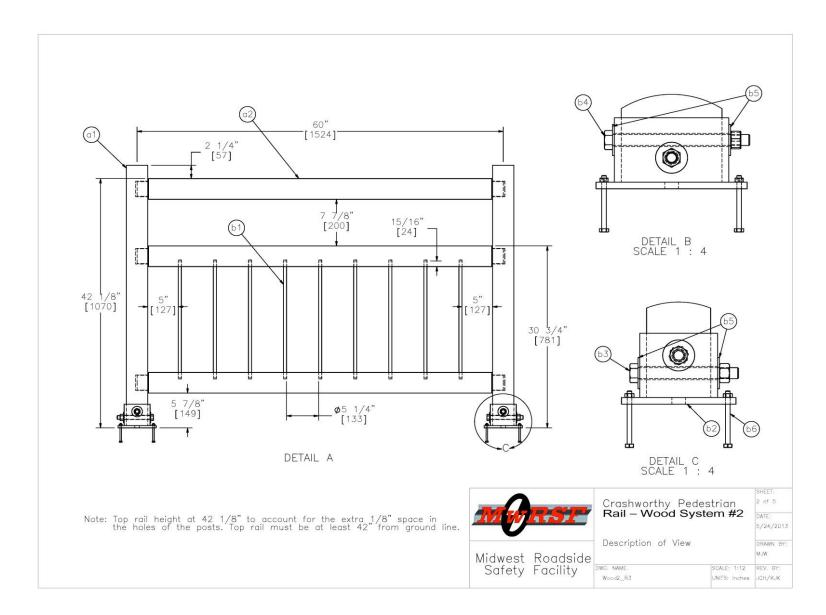


Figure 82. Cutout Wood Rail, Design Concept WOOD2 (Sheet 2 of 5)

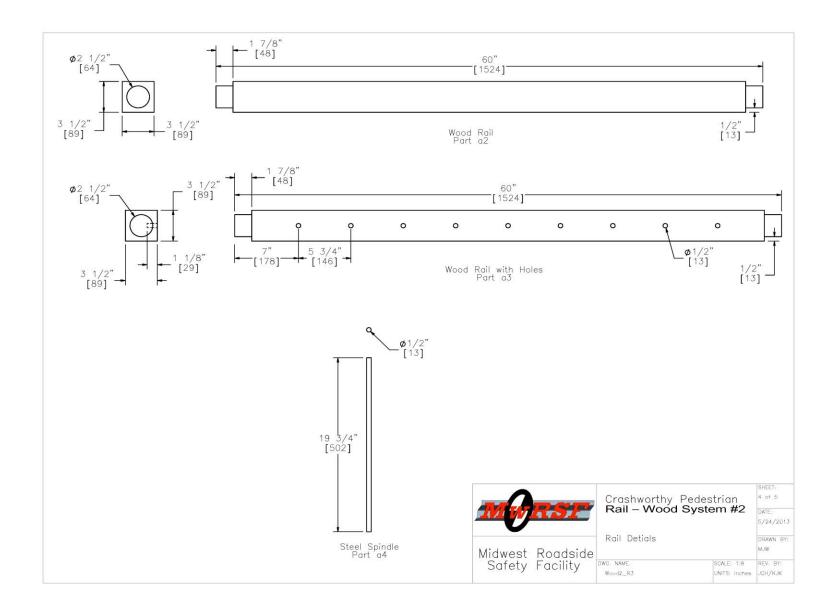


Figure 83. Cutout Wood Rail, Design Concept WOOD2 (Sheet 3 of 5)

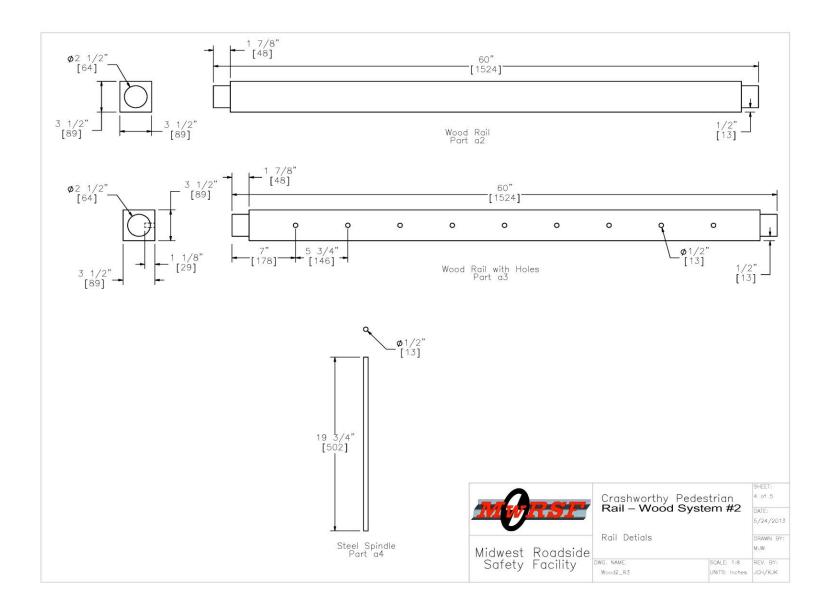


Figure 84. Cutout Wood Rail, Design Concept WOOD2 (Sheet 4 of 5)

		Crashworthy Pedestrian Rail -	ooginentea mooa	
ltem No.	QTY.	Description	Material Spec	Hardware Guide
a1	8	4x6" Wood End Post - 44" long	Grade Dense No. 1 Douglas Fir	
a2	4	4x4" Wood Rail — 60" long	Grade Dense No. 1 Douglas Fir	
a3	8	4x4" Wood Rail with Holes- 60" long	Grade Dense No. 1 Douglas Fir	
b1	36	1/2" Dia. Steel Spindle - 20" long	A36	
b2	8	Steel Base Fitting	A36	
b3	8	5/8" Dia. x 5 1/2" Long Hex Head Bolt and Nut	A307	FBX16a
b4	8	5/8" Dia. x 7 3/8" Long Hex Head Bolt and Nut	A307	FBX16a
Ь5	32	5/8" Dia. Flat Washer	ASTM F844 or Grade 2 Steel	FWC16a
b6	32	1/4" Dia. 2" long Hex Bolt and Nut	A307	FBX06a

		Crashworthy Pedestrian Rail – Wood System #2		SHEET: 5 of 5
	AND L	Rail – Wood	d System #2	DATE: 5/24/2013
Midwest	Roadside	Bill of Material	S	DRAWN BY: MJW
	Facility	DWG. NAME, Wood2_R3	SCALE: None UNITS: Inches	REV. BY: JCH/KJK

Figure 85. Cutout Wood Rail, Design Concept WOOD2 (Sheet 5 of 5)



Figure 86. Fabricated Refined Design Concept WOOD2



Figure 87. Fabricated Refined Design Concept WOOD2



January 18, 2016 MwRSF Report No.TRP-03-321-15

### **6 PEDESTRIAN RAIL DESIGNS**

For the initial design, simplified load cases were assumed. The AASHTO *LRFD Bridge Design Specifications* denote design live loads on the longitudinal rail, vertical post, and any infill components of a pedestrian rail [6]. Additional load scenarios and assumptions were considered to determine detailed designs for: (1) rail member, (2) post member, (3) infill member, (4) post-to-rail connection, (5) post-to-base connection, (6) infill-to-rail, and (7) anchor ages.

## 6.1 Rail Component

The AASHTO *LRFD Bridge Design Specifications* specifies that the design live load of each longitudinal element shall include the application of two uniform loads of 50 lb/ft (730 N/m) or 4.17 lb/in. (730 N/m) and a concentrated load of 200 lb (890 N), acting simultaneously. Superposition of forces should be used to replicate loads in two principal directions based on the use of a doubly symmetric beam. The uniform loads shall be applied both vertically and transversely. The concentrated load may be applied in any direction to maximize the forces in the member. The system was designed with the concentrated load applied vertically on the rail, as shown in Figure 29. Simply supported and fixed-end configurations were assumed, and the maximum shears and moments were determined for design purposes. The length used for the rail design was 60 in. (1,524 mm).

# 6.1.1 Concentrated Load

The concentrated load applied at the support of the longitudinal element produced the maximum shear in the rail. The shear in the rail is shown in Figure 88. The maximum shear stress in a simply supported beam with a concentrated load was calculated using Equation 6. A 200-lb (890 N) concentrated load applied at either support of the longitudinal element produced a

maximum shear of approximately 200 lb (890 N) at either support with no shear elsewhere along the rail.

$$R_1 = V_{max}$$
 (when a < b)  $= \frac{Pb}{L} = \frac{(200 \ lb)(60 \ in.)}{(60 \ in.)} = 200 \ lb$  (6)

Where:

R<sub>1</sub>= Support Reaction of Simply Supported Beam under a Point Load [lb]

- V<sub>max</sub>= Maximum Shear Force in Rail due to Point Load, Virtually at One Support [lb] 200 lb
- a= Distance from Concentrated Load to End of Rail [in.] 0 in.
- b= Location of Concentrated Load Relative to End of Rail Component [in.] – 60 in.
- P= Concentrated Live Load for Rails [lb]
- L= Rail Length [in.] 60 in.

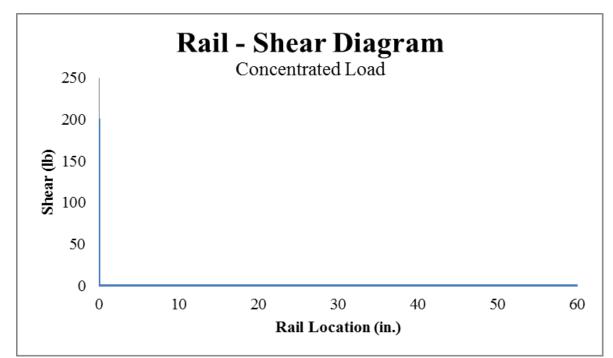


Figure 88. Rail Shear Diagram - Concentrated Load Virtually at Support

The concentrated load placed at the midspan of the beam maximizes the moment at the midspan in the rail when the ends are simply supported, as shown in Figure 89. The maximum moment at the midspan was calculated using Equation 7. The maximum moment resulting from

a 200-lb (890-N) concentrated load applied in any direction at the midspan of a 60-in. (1.5-m) rail span was calculated to be 3,000 lb-in. (339 N-m).

$$M_{max} = \frac{PL}{4} = \frac{(200 \ lb)(60 \ in.)}{4} = 3,000 \ lb - in. \text{ or } 250 \ lb - ft \tag{7}$$

Where:

M<sub>max</sub>= Maximum Bending Moment in Rail due to Midspan Point Load [lb-in.]

P= Concentrated Midspan Live Load for Rails [lb] – 200 lb

L= Rail Length [in.] - 60 in.

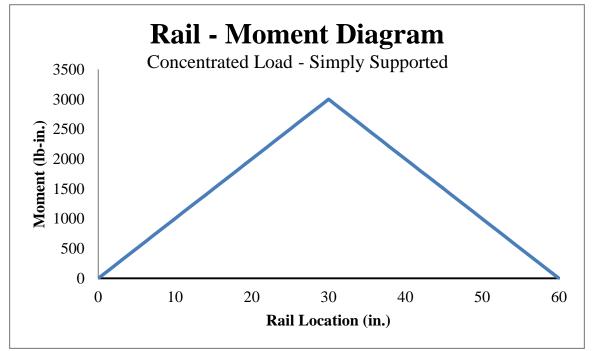


Figure 89. Rail Moment Diagram - Concentrated Load, Simply Supported Ends

# 6.1.2 Uniform Load

The shear in a simply supported the rail due to a uniformly distributed load is shown in Figure 90, with the maximum shear force occurring at the ends. The maximum shear force is equal to the support reaction, which can be calculated using Equation 8. The maximum shear force in a rail element with a 4.17-lb/in. (730-N/m) uniform load over a span of 60 in. (1.5 m) was calculated to be 125 lb (556 N).

$$R = V_{max} = \frac{wL}{2} = \frac{(4.17 \frac{lb}{in.})(60 in.)}{2} = 125 \ lb \tag{8}$$

Where: R= Support Reaction of Simply Supported Beam due to Uniform Load [lb]

V<sub>max</sub>= Maximum Shear Force in Rail due to Uniform Load [lb]

w= Distributed Live Load [lb/in.] – 4.17 lb/in.

L= Rail Length [in.] - 60 in.

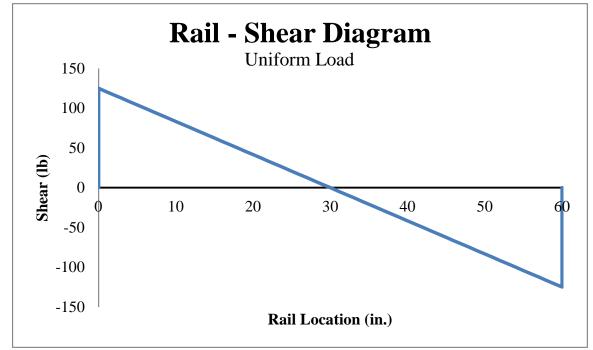


Figure 90. Rail Shear Diagram – Uniformly Distributed Load, Simply Supported Ends

When the ends are assumed to be simply supported, the maximum moment from a uniformly distributed load occurs at the midspan, as shown in Figure 91. The maximum moment, calculated using Equation 9 with a 4.17-lb/in. (730-N/m) uniform load over a 60-in. (1.5-m) span, was 1,876.5 lb-in. (213.4 N-m), which was located at the midpoint of the longitudinal member.

$$M_{max} = \frac{wL^2}{8} = \frac{(4.17 \frac{lb}{in.})(60 in.)^2}{8} = 1,876.5 \ lb - in. \ or \ 156.25 \ lb - ft \tag{9}$$

Where: M<sub>max</sub>= Maximum Bending Moment in Rail due to Uniform Load [lb-in.]

- w= Uniform Design Live Load [lb/in.] 4.17 lb/in.
- L= Rail Length [in.] 60 in.

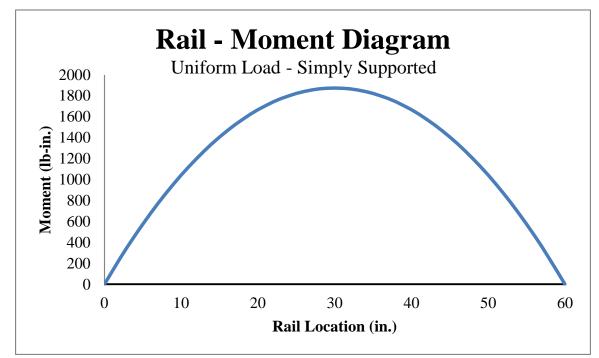


Figure 91. Rail Moment Diagram - Uniformly Distributed Load, Simply Supported Ends

# 6.1.3 Combined Concentrated and Uniform Loads

The total design loads for the longitudinal rail element must consider the combination of loading in two directions. AASHTO criteria specify that the two uniform loads must be applied vertically and transversely, but the concentrated load may be applied at any point and in any direction on the rail element [6]. The maximum shear and bending effect of the combined loading from the two uniform loads (ie., transverse and vertical) and the concentrated load acting in either the vertical (z-axis) or transverse (y-axis) directions. For the purposes of this design, the

concentrated load was assumed to act in the vertical direction (z-axis). However, since it could be applied transversely, a doubly symmetric section would be most efficient.

The maximum shear force for both the concentrated and uniform loads occurs at the end of the rail. Using results from Figures 88 and 90, these loads can be combined into a resultant shear force using Equation 10 and a maximum shear force of 348.2 lb (1,549 N).

$$V = \sqrt{V_z^2 + V_y^2} = \sqrt{325^2 + 125^2} = 348.2 \text{ lb}$$
(10)

Where:

 $V_z$  = Maximum Vertical Shear at End of Rail [lb] = 200 lb + 125 = 325 lb  $V_y$  = Maximum Transverse Shear Force at End of Rail [lb] = 125 lb

The combined bending moment resulting from the three separate loads acting on the longitudinal member can be calculated using the combined bending formula shown in Equation 11. The rail element was designed as a doubly symmetric member, meaning  $I_{zz} = I_{yy} = I$ , y = z = C, and the product of inertia value,  $I_{yz}$ , is equal to zero. Elimination of the  $I_{yz}$  terms and simple algebra were used to obtain the form shown in Equation 12. To simplify this equation and acquire the maximum tensile or compressive stress in Equation 13, either y and z or  $M_y$  and  $M_z$  need to have opposite signs. Using the relation of section properties given in Equation 14, the formula can be further simplified to Equation 15. This relationship implies that moments acting about two orthogonal axes over a doubly symmetric cross section can be combined to determine a maximum bending stress in the cross section. In this case, the maximum applied moment would be determined as the sum of the maximum bending moments from the loads applied both vertically and transversely and used to size the symmetric beam section. Assuming the point load is acting in the same plane as one of the distributed loads to maximize reactions, then the maximum bending moment in the rail element would be the combination of the maximum

bending moment for two distributed loads, plus the bending moment from a concentrated load applied at the center of the rail span, or 6,750 lb-in. (762.8 N-m) using Equation 16 and shown in Figure 92.

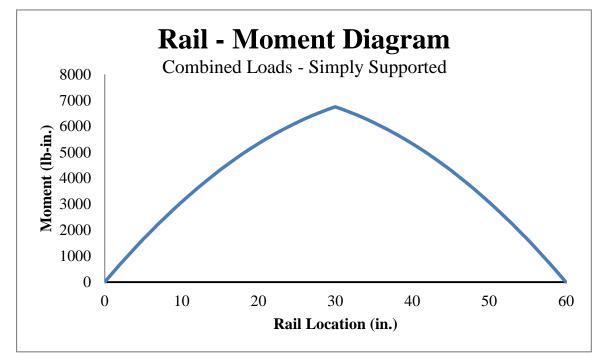


Figure 92. Rail Moment Diagram – Combined Loads, Simply Supported Ends

$$\sigma_{xx} = \frac{(M_y I_{zz} + M_z I_{yz})z - (M_z I_{yy} + M_y I_{yz})y}{(I_{yy} I_{zz} - I_{yz}^2)}$$
(11)

$$\sigma_{xx} = \frac{M_y z}{I_{yy}} - \frac{M_z y}{I_{zz}}$$
(12)

$$\sigma_{xx,max} = \frac{M_y C}{I} - \frac{M_z C}{I} \tag{13}$$

$$S = \frac{1}{c} \tag{14}$$

$$\sigma_{xx,max} = \frac{|M_y|}{s} + \frac{|M_z|}{s} \tag{15}$$

$$M = \left| M_{\mathcal{Y}} \right| + \left| M_{z} \right| \tag{16}$$

Where:

 $\sigma_{xx} = \text{Tensile or Compressive Stress Acting on Surface Perpendicular to$ the X-Direction [psi] $<math display="block">\sigma_{xx,max} = \text{Maximum Tensile or Compressive Stress Acting on Surface$ Perpendicular to the X-Direction [psi]M = Combined Moment [lb-ft]M<sub>y</sub> = Moment in the Y-Direction [lb-ft]M<sub>z</sub> = Moment in the Z-Direction [lb-ft]I<sub>zz</sub> = Moment of Inertia with Respect to the Z-Axis [in.<sup>4</sup>]I<sub>yz</sub> = Products of Inertia with Respect to the X- and Y-Axes [in.<sup>4</sup>]I<sub>yy</sub> = Moment of Inertia with Respect to the Y-Axis [in.<sup>4</sup>]Z = Distance from the Neutral Axis in the Z-Direction [in.]Y = Distance from the Neutral Axis in the Y-Direction [in.]S = Section Modulus [in<sup>3</sup>]I = Moment of Inertia I<sub>yy</sub> = I<sub>zz</sub> [in.<sup>4</sup>]C = Distance from Neutral Axis |y| = |z| [in.]

# 6.2 Post Component

The vertical member of a pedestrian rail must be designed for a concentrated live load,  $P_{LL}$ , applied transversely on the post at the center of gravity of the uppermost longitudinal element.  $P_{LL}$  is determined from Equation 13.8.2-1 in the AASHTO *LRFD Bridge Design Specifications* [6] and is shown in Equation 17. The magnitude of  $P_{LL}$  with a 60-in. (1.5-m) post spacing is 450 lb (2,000 N).

$$P_{LL} = 200 + 50L = 200 \ lb + 4.17 \ \frac{lb}{in.} (60 \ in.) = 450 \ lb \tag{17}$$

Where 
$$P_{LL}$$
= Concentrated Live Load for Posts [lb]  
L= Post Spacing [in.] – 60 in.

The post members were analyzed as a cantilever beam, with the fixed end represented by a rigid anchorage at the base of the post. The shear and moment diagrams correspond to a concentrated load,  $P_{LL}$ , applied transversely to the post element at the mid-height of the top rail [41 in. (1,041 mm) above ground], as shown in Figure 93 and Figure 94, respectively. Both the maximum shear load and bending moment in the post component is located at the base of the post, nearest the connection to the baseplate. The maximum shear in the post is equal to  $P_{LL}$  = 450 lb (2,000 N). The maximum bending moment in the post behaving as a fixed-end cantilever element was determined with Equation 18 and is 18,450 lb-in. (2,085 N-m).

$$M_{max} = P_{LL}h = (450 \ lb)(41 \ in.) = 18,450 \ lb - in. \text{ or } 1,537.5 \ lb - ft.$$
(18)

Where  $M_{max}$ = Maximum Bending Moment in Post [lb-in.]  $P_{LL}$ = Concentrated Live Load for Posts [lb] – 450 lb h= Height at which Transverse Point Load is Applied [in.] – 41 in.

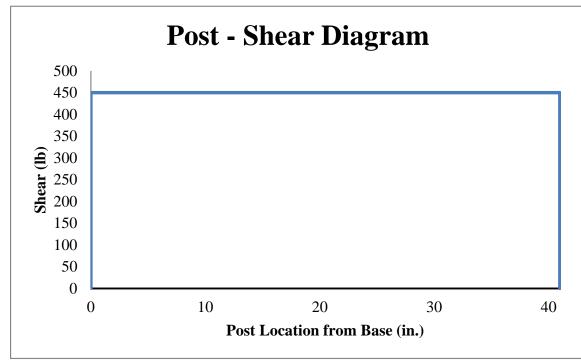


Figure 93. Post Shear Diagram

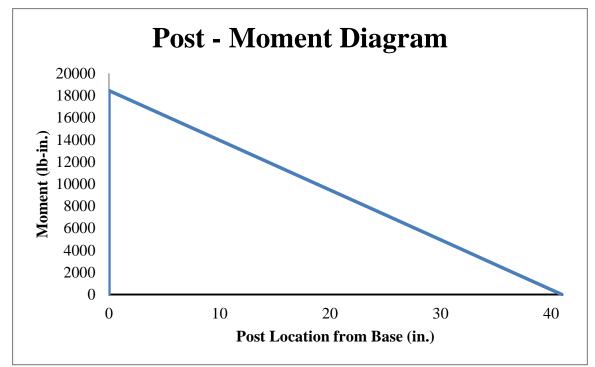


Figure 94. Post Moment Diagram

# 6.3 Infill

The infill region of a pedestrian rail system is the area between two vertical posts and longitudinal rails where mesh or spindle designs may be implemented to meet maximum opening requirements and add aesthetic characteristics to the rail system. The AASHTO *LRFD Bridge Design Specifications* specify that the members or panel within this area must support a 15-lb/ft<sup>2</sup> (718-N/m<sup>2</sup>) load over the entire infill area [6]. With a rail span of 60 in. (1,524 mm), nine ½-in. (13-mm) spindles would be required with a 5¾-in. (146-mm) maximum gap width [6]. The maximum spindle length between rail components was assumed to be 24¼ in. (616 mm), based on the preliminary designs. The average tributary area for each of the nine spindles was 151.56 in.<sup>2</sup> (0.098 m<sup>2</sup>). The 15-lb/ft<sup>2</sup> (718-N/m<sup>2</sup>) load distributed over the tributary area of the spindle equates to a uniform load, *w*, of 0.651 lb/in. (114 N/m) over the 24¼-in. (616-mm) length of the spindle member. The shear diagram is shown in Figure 95. The maximum shear force in a

spindle was calculated with Equation 19 based on an assumption of simply supported ends. The maximum midspan moment in the spindles was calculated with Equation 20. The moment diagram is shown in Figure 96.

When evaluating a mesh infill panel, the capacity needs to exceed 15  $lb/ft^2$  (718 N/m<sup>2</sup>). The maximum shear and moment is dependent on the types of mesh panel selected.

$$V_{max} = \frac{wL}{\frac{2}{m}} = \frac{(0.651\frac{lb}{in})(24.25in.)}{\frac{2}{m}} = 7.9 \ lb \tag{19}$$

$$M_{max} = \frac{wL^2}{8} = \frac{(0.651\frac{lb}{in})(24.25in.)^2}{8} = 47.85\ lb - in. = 4.0\ lb - ft$$
(20)

Where:

 $V_{max}$  = Maximum Shear Force in Spindle [lb]  $M_{max}$  = Maximum Bending Moment in Spindle [lb-in.] L = Length of the Spindle Member [in.] - 24.25 in. w = Uniform Load [lb/in.] - 0.651 lb/in.

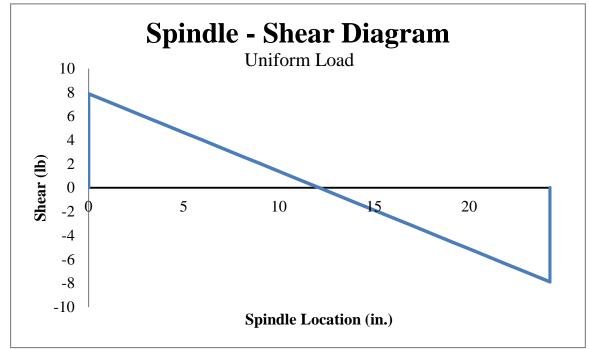


Figure 95. Spindle Shear Diagram – Uniform Load, Simply Supported Ends

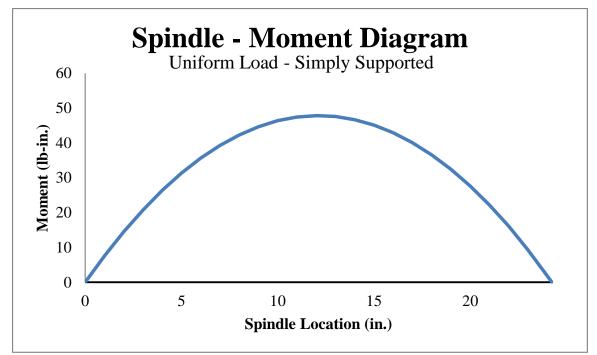


Figure 96. Spindle Moment Diagram - Uniform Load, Simply Supported Ends

# **6.4 Connections**

The connections between the rail, post, and base components are essential for transferring loads between elements and to the anchoring system. It was assumed that the reactions at each joint would be fully transferred through the connection. Therefore, the shear and moment capacity of each connection must be greater than the calculated reactions at the member ends. The connections that were evaluated included post-to-rail, post-to-base, infill-to-rail, and concrete anchors.

## 6.4.1 Post-to-Rail Connection

While the rail member designs utilized an assumption of simply supported ends to maximize midspan moments, the ends were assumed to be fixed for connection design to maximize applied moment at the ends. This assumption was also more realistic, as a welded or fitted connection would likely be used. When the ends are fixed, the maximum moment at the ends of the rail due to a concentrated load is shown in Figure 97. When the ends are fixed, the maximum moment at the ends of the rail from the uniform load is shown in Figure 98.

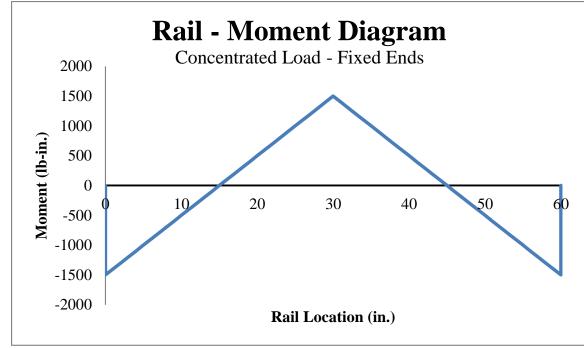


Figure 97. Rail Moment Diagram - Concentrated Load, Fixed-Fixed Ends

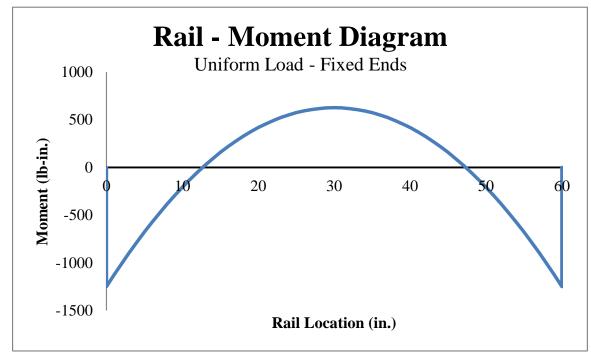


Figure 98. Rail Moment Diagram - Uniformly Distributed Load, Fixed-Fixed Ends

## 6.4.1.1 Maximum Shear Force

The required strength of the post-to-rail connection was calculated using shear and bending moments at the supported ends of the longitudinal rail element. The shear at the end of a fixed-fixed beam was calculated using Equation 10 and found to be 348.2 lb (1,549 N) at the post-to-rail connection.

#### 6.4.1.2 Maximum Bending Moment

The bending moment at the support of a fixed-fixed beam from the distributed load along the entire beam is given in Equation 21. Using w = 4.17 lb/in. (730 N/m) and L = 60 in. (1,524 mm), Equation 21 yielded a maximum moment of 1,251 lb-in. (141.3 N-m) at each end of the longitudinal rail member, the location of the post-to-rail connection.

$$M_{end} = \frac{wL^2}{12} = \frac{(4.17\frac{lb}{in})(60 \text{ in.})^2}{12} = 1,251 \ lb - in. \text{ or } 104.2 \ lb-ft.$$
(21)

Where:

 $M_{end}$  = End Moment Reaction due to Distributed Load [lb-in.] w= Distributed Design Live Load [lb/in.] – 4.17 lb/in. L= Rail length [in.] – 60 in.

The shear reaction at the support of a fixed-fixed beam due to a 200-lb (890-N) concentrated load was calculated using Equation 22. The design bending moment was determined by Equation 23. The shear and bending moment depends on the longitudinal distance, a, away from the support to the concentrated load, which can be applied at any point on the longitudinal member. To maximize the moment due to point load, the differential of the bending moment in Equation 23 was set to zero to determine the longitudinal distance between the fixed end support to the concentrated point load. This calculation yielded a longitudinal distance of a=(2/3)L and b=(1/3)L. Applying the corresponding values to the equation, the maximum bending moment formed at the fixed end was 1,778 lb-in. (201 N-m).

$$V(\max \text{ when } a > b) = \frac{Pa^2}{L^3}(a+3b)$$
 (22)

$$M(\max \text{ when } a > b) = \frac{Pa^2b}{L^2}$$
(23)

Where:

v = Shear Moment due to Concentrated Load [lb]
a = Longitudinal Distance between Concentrated and Load Considered Support [in.]
b = L - a [in.]
L = Post Spacing [in.]
M = End Moment due to Concentrated Load [lb-in.]

The combined shear loading in the post-to-rail connection was the same as the maximum shear load in the rail, which was 348.2 lb (1,549 N). The same concept of combining vertical and transverse bending moments for the rail applies at the connection as well. The combination of a reaction in the post-to-rail connection from the uniform and concentrated loads in the vertical plane plus a transverse uniform load was 4,280 lb-in. (483.6 N-m), as shown in Figure 99.

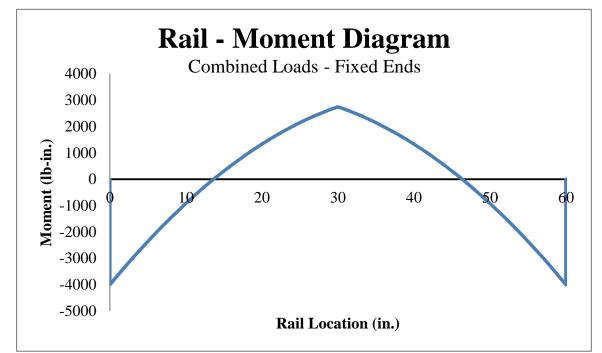


Figure 99. Rail Moment Diagram - Combined Loads, Fixed-Fixed Ends

### 6.4.2 Post-to-Base Assembly Connection

The required strength of the post-to-base connection was calculated using the design shear and bending moment at the base of the post element from the design loading conditions from Section 6.2. The shear and moment reactions at the post-to-base connection were 450 lb (2,000 N) and 18,450 lb-in. (2,085 N-m), respectively.

In addition to the applied shear and moment, each baseplate would be subjected to an axial force based on the sum of the rail loads that each post experiences. Each concept had three rail members, and based on the pedestrian rail loads defined by the AASHTO LRFD Bridge Specifications [6], one would have a concentrated load and all three would have a uniform load in the lateral and vertical directions. The resultant shear forces from these applied loads on the rails produced the maximum axial force on the baseplate, calculated to be 200 lb + [125 lb \* 3] = 575 lb.

The maximum vertical force imparted to the baseplate can be determined from the maximum moment experienced at the base, 18,450 lb-in. (2,085 N-m), divided by the depth of the post. The maximum vertical force is calculated using Equation 24.

$$P_{max} = \frac{M_{max}}{d} = \frac{18,450 \text{ lb-in.}}{d}$$
(24)

Where:  $P_{max} = Maximum$  Vertical Force on Baseplate [in.]  $M_{max} = Maximum$  Moment at Base of Post [lb-in.] d = Depth of Post [in.]

Also, from the American Institute of Steel Construction (AISC) Steel Design Guide Series 1, the required bending moment of the baseplate with a large eccentricity is based upon a combined loading of the axial force, 575 lb (2,557 N), and the moment 18,450 lb-in. (2,085 Nm), on the baseplate [41-42]. The free-body diagram of this system is shown in Figure 100. The required bending moment per width,  $M_{pl}$ , for the baseplate design is shown in Equation 25. The supporting concrete under the baseplate is assumed to have dimensions at least twice each dimension of the baseplate. While this equation was derived for steel baseplates, the variables were modified for an aluminum baseplate and should produce similar results.

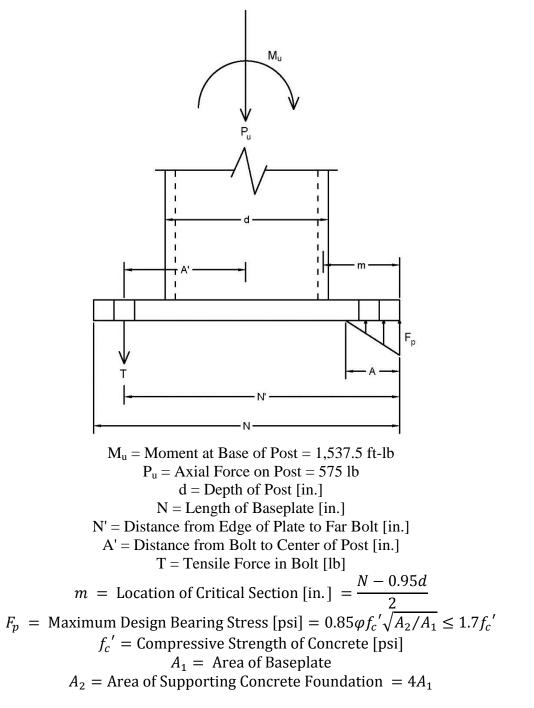


Figure 100. Baseplate Loads with Combined Axial Force and Bending Moment with Large Eccentricity

$$M_{pl} = \sum M_m \tag{25}$$

Where:  $M_{pl}$  = Required Bending Moment on Baseplate [in.-lb/in.]  $M_m$  = Maximum Moment at Location m, which is the Critical Section [in.-lb]

## 6.4.3 Infill-to-Rail Connection

The infill between the rail components varied by design concept and included aluminum spindles between the rail components, or a mesh infill between the post and rail members. To maximize the moment at the midspan of the spindle, a simply supported end connection was assumed in Section 6.3 for the spindle member design. However, for the connection between the spindle and rail, the ends were assumed to be fixed-fixed to maximize the moment at the connections, which would also be more representative of a welded or fitted connection. With a 24<sup>1</sup>/<sub>4</sub>-in. (616-mm) long spindle and a uniform load of 0.651 lb/in. (114 N/m), the moment diagram is shown in Figure 101. Maximum shear at the spindle connection was the same as for the spindle member, 7.89 lb (35.1 N). The maximum bending moment located at the end of the spindle-to-rail connection was 31.9 lb-in. (3.6 N-m), as calculated by Equation 26.

$$M_{end} = \frac{wL^2}{12} = \frac{(0.651\frac{lb}{in})(24.25in)^2}{12} = 31.9 \ lb - in = 2.66 \ lb - ft \tag{26}$$

Where:  $M_{end}$  = Maximum Bending Moment at End of Spindle [lb-in.] L = Spindle Location [in.] – 24.25 in. w = Uniform Load [lb/in.] – 0.651 lb/in.

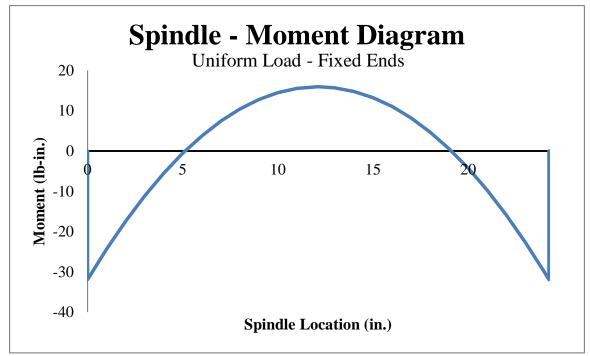


Figure 101. Spindle Moment Diagram – Uniformly Distributed Load, Fixed-Fixed Ends

## 6.4.4 Concrete Anchorage

The base of the post was previously assumed to have a fixed-end condition. In order for this assumption to be true, the base of the post was rigidly fixed to a concrete foundation with anchors. Wedge anchors would not likely allow a damaged system to be removed and reinstalled multiple times. Therefore, a threaded rod secured through the base fitting with an epoxy adhesive anchoring system was selected. The shear and bending moment induced at the base of the post were calculated in Sections 6.2 and 6.3.2 to be 450 lb (2,000 N) and 18,450 lb-in. (2,085 N-m), respectively.

If two bolts were utilized, the required shear load on each bolt was 225 lb (1,000 N). The bending moment at the base of the post transfers into a tensile force prying up on the bolt. The size of the baseplate and length of the moment arm between the anchor bolts influence the

magnitude of the tensile stress on the bolts' cross sections. The general equation for the magnitude of tensile force is shown in Equation 27.

$$P_{max} = \frac{M_{max}}{x} = \frac{(18,450 \text{ lb-in.})}{x \text{ in.}}$$
(27)

Where:

 $P_{max}$  = Tensile Force Acting Upward on Anchor Bolts [lb]  $M_{max}$  = Moment at the Base of the Post [lb-in.] x = Distance between Anchor Bolts [in.]

## **7 DESIGN OF PROTOTYPE PEDESTRIAN RAILS**

## 7.1 Introduction

The mechanical properties of aluminum can vary depending on the alloy, shape, thickness, and existence of weld-affected zones. The process of welding aluminum at a connection location significantly reduces the strength of the material surrounding the weld location. While heat treatment can be applied to regain most of the material strength in weld-affected zones, the heat treatment was not desired. Thus, the pedestrian rail was designed using the lower weld-affected material strengths. A common aluminum alloy, 6061-T6, was selected for all of the pedestrian rail designs. The mechanical properties of non-welded 6061-T6 aluminum were provided in Table A.3.4 in the Aluminum Design Manual (ADM) [38] and are shown in Table 5 for extrusions, sheets, and plates. The mechanical properties of weld-affected 6061-T6 aluminum were provided in Table A.3.5 in the ADM and are shown in Table 5 for all shapes and plate/sheet thicknesses, t, less than and greater than <sup>3</sup>/<sub>8</sub> in. (10 mm).

Non-welded Strength Extrusions, All Thicknesses ksi (MPa)		Non-Welded Strength Sheet & Plate, $0.010 \le t \le 4.000$ in. ksi (MPa)		Weld-Affected Strength All Shapes, t ≤ 0.375 in. ksi (MPa)		Weld-Affected Strength All Shapes, t > 0.375 in. ksi (MPa)	
F <sub>tu</sub>	38 (260)	F <sub>tu</sub>	42 (290)	F <sub>tuw</sub>	24 (165)	$F_{tuw}$	24 (165)
F <sub>ty</sub>	35 (240)	$F_{ty}$	35 (240)	$F_{tyw}$	15 (105)	$F_{tyw}$	11 (80)
F <sub>cy</sub>	35 (240)	$F_{cy}$	35(240)	$F_{cyw}$	15 (105)	F <sub>cyw</sub>	11 (80)
F <sub>su</sub>	24 (165)	$F_{su}$	27 (185)	$F_{suw}$	15 (105)	F <sub>suw</sub>	15 (105)

Table 5. Aluminum Alloy 6061-T6 Material Strengths [38]

Where:	$F_{tu}$ = Tensile Ultimate Strength
	$F_{ty}$ = Tensile Yield Strength
	$F_{cy}$ = Compressive Yield Strength
	$F_{su}$ = Shear Ultimate Strength
	F <sub>tuw</sub> = Tensile Ultimate Strength of Weld-Affected Zones
	$F_{tyw}$ = Tensile Yield Strength of Weld-Affected Zones
	$F_{cyw}$ = Compressive Yield Strength of Weld-Affected Zones
	$F_{suw}$ = Shear Ultimate Strength of Weld-Affected Zones

Each component was designed using Load and Resistance Factor Design (LRFD) equations from the ADM in which the required strength,  $R_u$ , needs to be less than or equal to the design strength,  $\phi R_n$ , from equation B.3-1 in the ADM, as given in Equation 28 [38].

$$R_u \le \varphi R_n \tag{28}$$

Where	R <sub>u</sub> = Required Strength
	R <sub>n</sub> = Nominal Strength
	$\varphi$ = Resistance Factor
	$\phi R_n = Design Strength$

# **7.2 Section Capacities**

The shear and flexural capacities of each rail, post, and spindle cross section were determined for limit states using equations in the ADM.

# 7.2.1 Shear

# 7.2.1.1 Rectangular tubes

The shear capacity of a non-welded section of a flat web support on both edges (e.g. rectangular tube) from Section G.2 of the ADM is given in Equation 29.

$$\varphi V_n = \varphi F_s A_w \tag{29}$$

Where

$$\label{eq:phi} \begin{split} \phi V_n &= \text{Nominal Shear Capacity [kip]} \\ F_s &= \text{Shear Stress Corresponding to Shear Strength from} \\ & \text{Table 6 [ksi]} \\ A_w &= \text{Area of Web} = d^*t \text{ [in.]} \\ \phi &= 0.90 \end{split}$$

## Table 6. Rectangular Tubes Shear Strength [38]

Limit State	$F_s$	b/t	Slenderness Limits		
yielding	$F_{sy}$	$b/t \le S_1$	$S_1 = \frac{B_s - F_{sy}}{1.25D_s}$		
inelastic buckling	$B_s - 1.25 D_s b/t$	$S_1 < b/t < S_2$			
elastic buckling	$S_2 = \frac{C_s}{1.25}$				
buckling $(1.25b/t)^2$ $5_2 = 5_2^2 = 1.25$ $F_{sy} =$ Shear Yield Strength [ksi] $D_s =$ Buckling Constant Slope [ksi] $B_s =$ Buckling Constant Intercept [ksi] $C_s =$ Buckling Constant Intersection $b =$ Clear Height of the Web for Unstiffened Webs [in.] $t =$ Web Thickness [in.] $d =$ Full Depth of Section [in.] $F_{sy} = F_{sy}$ if Non-Welded and $F_{syw}$ if Welded					

For non-welded 6061-T6 aluminum extrusions, the buckling constants can be calculated using equations provided in Table 1-1 in the ADM, as follows:

$$B_s = 27.2 \text{ ksi}$$
  
 $D_s = 0.141 \text{ ksi}$   
 $C_s = 79$ 

The slenderness limits  $S_1$  and  $S_2$  were then calculated for non-welded sections using the relationship  $F_{sy} = 0.6F_{ty} = 0.6 * 35$  ksi = 21 ksi from Table A.3.1 in ADM and Equations 30 and 31.

$$S_1 = \frac{B_s - F_{sy}}{1.25D_s} = \frac{27.2 - 21}{1.25 * 0.141} = 35$$
(30)

$$S_2 = \frac{C_s}{1.25} = \frac{79}{1.25} = 63.2 \tag{31}$$

The slenderness limits  $S_1$  and  $S_2$  were then calculated for welded sections using the relationship  $F_{syw} = 0.6F_{tyw} = 0.6 * 15$  ksi = 9 ksi from Table A.3.1 in ADM and Equations 32 and 33.

$$S_1 = \frac{B_s - F_{syw}}{1.25D_s} = \frac{12-9}{1.25*0.051} = 47$$
(32)

$$S_2 = \frac{C_s}{1.25} = \frac{158}{1.25} = 126 \tag{33}$$

For weld-affected zones of 6061-T6 aluminum extrusions, the buckling constants can be calculated using equations provided in Table 1-2 in the ADM, as follows:

If thickness is less than or equal to 0.375 in. (10 mm):  $B_s = 12 \text{ ksi}$   $D_s = 0.051 \text{ ksi}$  $C_s = 158$ 

If thickness is greater than 0.375 in. (10 mm):  $B_s = 8.6 \text{ ksi}$  $D_s = 0.031 \text{ ksj}$ 

$$D_{\rm s} = 0.031$$
$$C_{\rm s} = 187$$

## 7.2.1.2 Round and Oval Tubes

The shear capacity of round or oval tubes from Section G.3 of the ADM is given in Equation 34.

$$\varphi V_n = \varphi F_s A_g / 2 \tag{34}$$

Where:  $\phi V_n$ = Nominal Shear Capacity [kip]  $F_s$  = Shear Stress Corresponding to Shear Strength from Table 7 [ksi]  $A_g$  = Gross Area [in.<sup>2</sup>]  $\phi$  = 0.90

Since the buckling constants are dependent on the material type and slenderness limits,

these values are the same as rectangular tubes in Section 7.2.1.1.

## Table 7. Round or Oval Tubes Shear Strength [38]

Limit State	$F_s$	$\lambda_t$	Slenderness Limits			
yielding	$F_{sy}$	$\lambda_t \leq S_1$	$S_1 = \frac{1.3B_s - F_{sy}}{1.63D_s}$			
inelastic buckling	$1.3B_s - 1.63D_s \lambda_t$	$S_1 < \lambda_t < S_2$				
elastic buckling	$\frac{1.3\pi^2 E}{(1.25 \ \lambda_t)^2}$	$\lambda_t \ge S_2$	$S_2 = \frac{C_s}{1.25}$			
$\lambda_t = 2.9 \left(\frac{R_b}{t}\right)^{5/8} \left(\frac{L_v}{R_b}\right)^{1/4}$						
$R_b =$ Mid-Thickness Radius of a Round Tube or Maximum Mid-Thickness						
Radius of an Oval Tube [in.] t = Thickness of Tube [in.]						
$L_v =$ Length of Tube from Maximum to Zero Shear Force [in.]						
$F_{sy} = F_{sy}$ if Non-Welded and $F_{syw}$ if Welded						

## 7.2.1.3 Solid Sections

For solid sections, the nominal shear capacity was not provided in the ADM but calculated using Equation 35.

$$\varphi V_n = \varphi F_{sy} A_g \tag{35}$$

Where:

$$\varphi V_n$$
= Nominal Shear Capacity [kip]  
 $F_{sy}$  = Shear Yield Strength [ksi] =  $F_{sy}$  if Non-Welded and  $F_{syw}$  if Welded  
 $F_{sy}$  = 0.6 $F_{ty}$  = 0.6 \* 35 ksi = 21 ksi  
 $F_{syw}$  = 0.6 $F_{tyw}$  = 0.6 \* 15 ksi = 9 ksi  
 $A_g$  = Gross Area [in.<sup>2</sup>]  
 $\varphi$  = 0.90

# 7.2.2 Flexure

## 7.2.2.1 Rectangular Tubes

The general equation for the nominal flexural capacity of a closed-shape aluminum section, excluding pipes and round tubes, for the limit states of tensile yielding and tensile rupture is defined in Section F.8 in the ADM and shown in Equation 36. The flexural strength is

a function of the section modulus on the tension side of the neutral axis,  $S_t$ . In pure bending, half of a rectangular cross section is subjected to tension, while the other half is subjected to compression. This relation leads to the assumption that  $S_t$  of an aluminum tube section is half of the corresponding section modulus for the full section.

$$\varphi M_n = \varphi F_b S_t \tag{36}$$

Where:  $\phi M_n$ = Nominal Flexural Capacity [kip-ft]  $F_b$ = Flexural Strength [ksi]  $S_t$ = Section Modulus on the Tension Side of the Neutral Axis [in.<sup>3</sup>]  $\phi$ = 0.90 for Yielding, 0.75 for Rupture

The flexural strength,  $F_b$ , for non-welded members in the yielding and rupture limit states is given by Equations 37 and 38. The flexural strength of a member within a weld-affected zone is defined differently and is explained in the next section. The tension coefficient,  $k_t$ , of the 6061 alloy with T6 temper is specified in the ADM as 1.0.

$$F_{b-yielding} = 1.30F_{ty} \tag{37}$$

$$F_{b-rupture} = 1.42F_{tu}/k_t \tag{38}$$

Where:

$$\begin{array}{lll} \mbox{ re: } & F_{b\mbox{-yielding}} = & \mbox{Flexural Strength in Yielding Limit State [ksi]} \\ & F_{b\mbox{-rupture}} = & \mbox{Flexural Strength in Rupture Limit State [ksi]} \\ & F_{ty} = & \mbox{Tensile Yield Strength [ksi]} \\ & k_t = & \mbox{Tension Coefficient} \end{array}$$

A section which has been welded uses a flexural strength,  $F_b$ , in yielding and rupture limit states determined by Equations 39 and 40. These equations use a combination of the tensile yield or ultimate strengths and the weld-affected yield or ultimate strength, with each contribution based on the proportion of the cross section in tension affected by the weld ( $A_{wzt}$ ) to the gross cross-sectional area of the member in tension ( $A_{gt}$ ). The ADM defines the weldaffected zone as any part within 1 in. (25.4 mm) of the centerline of the weld [38]. If the entire cross section is in the weld-affected zone, then  $A_{wzt}=A_{gt}$  and the equations simplify to include only the tensile yield or ultimate strength of welded aluminum.

$$F_{b-yielding} = 1.30[F_{ty}\left(1 - \frac{A_{wzt}}{A_{gt}}\right) + F_{tyw}\left(\frac{A_{wzt}}{A_{gt}}\right)]$$
(39)

$$F_{b-rupture} = 1.42 \left[ F_{tu} \frac{\left(1 - \frac{A_{wzt}}{A_{gt}}\right)}{k_t} + F_{tuw} \left(\frac{A_{wzt}}{A_{gt}}\right) \right]$$
(40)

Where:	F <sub>b-yielding</sub> =	Flexural Strength in Yielding Limit State [ksi]
	F <sub>b-rupture</sub> =	Flexural Strength in Rupture Limit State [ksi]
	F <sub>ty</sub> =	Tensile Yield Strength [ksi]
	$F_{tyw} =$	Tensile Yield Strength of Weld-Affected Zone [ksi]
	$F_{tu} =$	Tensile Ultimate Strength [ksi]
	$F_{tuw} =$	Tensile Ultimate Strength of Weld-Affected Zone [ksi]
	$A_{wzt} =$	Cross-Sectional Area of the Weld-Affected Zone in Tension
		[in. <sup>2</sup> ]
	$A_{gt} =$	Gross Cross-Sectional Area of Element in Tension [in. <sup>2</sup> ]
	$A_{gt}=k_t=$	Tension Coefficient

## 7.2.2.2 Pipe and Round Tubes

The nominal flexural capacity of pipes and round tubes should be calculated for the limit states of compressive yielding, tensile yielding, tensile rupture, and local buckling, as defined in Section F.6 in the ADM. For the compressive yielding limit state, nominal flexural capacity is given in Equation 41. For the tensile yielding limit state, nominal flexural capacity is given in Equation 42. For the tensile rupture limit state, nominal flexural capacity is given in Equation 43. For the local buckling limit state, the nominal flexural capacity is given in Equation 44.

$$\varphi M_n = \varphi 1.17 F_{cy} S \tag{41}$$

Where:

 $\phi M_n$ = Nominal Capacity in Flexural Compressive Yielding [kip-ft]  $F_{cy}$ = Compressive Yield Strength [ksi] S= Section Modulus [in.<sup>3</sup>]  $\phi$ = 0.90

$$\varphi M_n = \varphi 1.17 F_{ty} S \tag{42}$$

Where:  $\phi M_n$ = Nominal Capacity in Flexural Tensile Yielding [kip-ft]  $F_{ty}$ = Tensile Yield Strength [ksi] S= Section Modulus [in.<sup>3</sup>]  $\phi$ = 0.90

$$\varphi M_n = \varphi 1.24 \frac{F_{tu}S}{k_t} \tag{43}$$

Where:  $\phi M_n$ = Nominal Capacity in Flexural Tensile Rupture [kip-ft]  $F_{tu}$ = Tensile Yield Strength [ksi] S= Section Modulus [in.<sup>3</sup>]  $k_t$  = Tension Coefficient  $\phi$ = 0.75

$$\varphi M_n = \varphi F_b S \tag{44}$$

Where:

:  $\phi M_n$ = Nominal Capacity in Flexural Local Bucking [kip-ft]  $F_b$ = Flexural Strength as Determined by Table 8 [ksi]  $S_t$ = Section Modulus on the Tension Side of the Neutral Axis [in.<sup>3</sup>]  $\phi$ = 0.90

Table 8. Pipe Flexural Local Buckling Strength [38]

Limit State	$F_b$	$\frac{R_b}{t}$	Slenderness Limits			
upper inelastic buckling	$B_{tb} - D_{tb} \sqrt{\frac{R_b}{t}}$	$\frac{R_b}{t} \le S_1$	$S_1 = \left(\frac{B_{tb} - B_t}{D_{tb} - D_t}\right)^2$			
lower inelastic buckling	$B_t - D_t \sqrt{\frac{R_b}{t}}$	$S_1 < \frac{R_b}{t} < S_2$				
elastic buckling	$\frac{\pi^2 E}{16\left(\frac{R_b}{t}\right)\left(1+\frac{\sqrt{R_b/t}}{35}\right)}$	$\frac{R_b}{t} \ge S_2$	$S_2 = C_t$			
$D_t = Buckling Constant Slope [ksi]$						
B <sub>t</sub> = Buckling Constant Intercept [ksi]						
$C_t$ = Buckling Constant Intersection						
D <sub>tb</sub> = Buckling Constant [ksi]						
$B_{tb} = Buckling Constant Intercept [ksi]$						
$R_b = Mid$ -Thickness Radius of a Round Tube or Maximum Mid-Thickness						
Radiu	is of an Oval Tube	[in.]				

t = Thickness of Tube [in.]

For non-welded 6061-T6 aluminum pipe, the buckling constants can be calculated using equations provided in Tables B.4.2 and Table 1-1 in the ADM, as follows:

 $\begin{array}{l} B_t = 43.2 \; ksi \\ D_t = 1.558 \; ksi \\ C_t = 141 \\ B_{tb} = 64.8 \; ksi \\ D_{tb} = 4.458 \; ksi \end{array}$ 

46.

The corresponding slenderness limits for non-welded pipe are shown in Equations 45 and

$$S_1 = \left(\frac{B_{tb} - B_t}{D_{tb} - D_t}\right)^2 = \left(\frac{64.8 - 43.2}{4.458 - 1.558}\right)^2 = 55.48$$
(45)

$$S_2 = C_t = 141$$
 (46)

For weld-affected zones of 6061-T6 aluminum pipe, the buckling constants can be calculated using equations provided in Tables B.4.2 and Table 1-2 in the ADM, as follows:

If thickness is less than or equal to 0.375 in. (10 mm):

 $\begin{array}{l} B_t = 19.5 \; ksi \\ D_t = 0.654 \; ksi \\ C_t = 390 \\ B_{tb} = 29.2 \; ksi \\ D_{tb} = 1.539 \; ksi \end{array}$ 

If thickness is greater than 0.375 in. (10 mm):

 $\begin{array}{l} B_t = 14.1 \; ksi \\ D_t = 0.425 \; ksi \\ C_t = 524 \\ B_{tb} = 21.1 \; ksi \\ D_{tb} = 0.999 \; ksi \end{array}$ 

The corresponding slenderness limits for welded pipe with thicknesses less than 0.375 in.

(10 mm) are shown in Equations 47 and 48.

$$S_1 = \left(\frac{B_{tb} - B_t}{D_{tb} - D_t}\right)^2 = \left(\frac{29.2 - 19.5}{1.539 - 0.654}\right)^2 = 120.1$$
(47)

$$S_2 = C_t = 390$$
 (48)

The corresponding slenderness limits for welded pipe with thicknesses greater than 0.375 in. (10 mm) are shown in Equations 49 and 50.

$$S_1 = \left(\frac{B_{tb} - B_t}{D_{tb} - D_t}\right)^2 = \left(\frac{21.1 - 14.1}{0.999 - 0.425}\right)^2 = 148.7$$
(49)

$$S_2 = C_t = 524$$
 (50)

## 7.3 Connection Capacity

#### 7.3.1 Welds

The filler material used in welding two aluminum elements together is dependent on the alloy specification of the two elements being welded. Table M.9.1 in the ADM specifies the desired filler alloy to be used for a welded connection, which is 5356 aluminum alloy for welds between two elements of the 6061 alloy. The corresponding tensile ultimate strength,  $F_{tuw}$ , and shear ultimate strength,  $F_{suw}$ , of the 5356 filler alloy from Table J.2.1 in the ADM are 35 ksi (240 MPa) and 17 ksi (115 MPa), respectively. The ADM considers the stress on an aluminum weld to be a shear stress, so weld capacities are calculated as a nominal shear strength from Section J.2.2.2 in the ADM, as shown in Equation 51.

$$\varphi R_n = \varphi F_{sw} L_{we} \tag{51}$$

Where

 $\phi R_n =$  Nominal Weld Shear Strength [lb]

 $F_{sw}$  = Shear Strength of Weld [psi], which is the Least of:

- a) The Product of the Weld Filler's Shear Ultimate Strength and the Effective Throat =  $F_{suw(filler)} * e =$ 17,000 *psi* \* *e*
- b) For Base Metal in Shear at the Weld-Base Metal Joint, the Product of the Base Metal's Welded Shear Ultimate Strength and the Fillet Size  $S_w$  at the Joint =  $F_{suw(base metal)} * S_w = 15,000 \ psi * S_w$
- c) For Base Metal in Tension at the Weld-Base Metal Joint, the Product of the Base Metal's Welded Tensile Ultimate Strength and the Fillet Size  $S_w$  at the Joint

$$= F_{tuw(base metal)} * S_w = 24,000 \ pst * S_w$$
  
L<sub>we</sub> = Weld Effective Length [in.]

$$\Phi = 0.75$$

The ADM did not provide a specific method for calculating the flexural capacity of a weld, so a calculation was derived based on the nominal shear strength of the weld and moment of inertia of the weld group, as shown in Equation 52. Detailed calculations of the nominal shear strength and moment capacity are provided in Appendix C.

$$\varphi M_n = \frac{\varphi F_{suw}I}{c}$$
(52)  
Where:  $\varphi M_n =$ Moment Capacity of Weld [ft-lb]  
 $F_{suw} =$  Shear Ultimate Strength of the Weld Filler,  $F_{suw(filler)}$   
 $c =$ Distance to Neutral Axis [in.]  
 $\varphi = 0.75$ 

## 7.3.2 Baseplate

Where:

The aluminum design manual does not specify a design procedure for baseplates. Therefore, two steel baseplate design equations were utilized as specified in the AISC *Steel Construction Manual* and *Steel Design Guide Series 1* [41-42]. Method no. 1 was from Page 14-6 in the *Steel Construction Manual*, describing how the minimum baseplate thickness can be determined using the maximum tensile force acting on the baseplate with Equation 53.

Using Equation 53, the nominal capacity of the baseplate is calculated with Equation 54. An example of the baseplate design for Concept AW2-A is shown in Appendix C.

$$t_{min} = l \sqrt{\frac{2P_u}{\varphi F_y BN}}$$
(53)  

$$t_{min} = \text{Minimum Baseplate Thickness [in.]}$$

$$l = \text{The Greater of } m \text{ and } n \text{ [in.]}:$$

$$m = \frac{N - 0.95d}{2}$$

$$n = \frac{B - 0.80b_f}{2}$$
B = Baseplate Width [in.]  
N = Baseplate Depth [in.]  
b\_f = Post Flange Width [in.]  
d = Post Depth [in.]  
F\_y = Yield Stress [psi]  
P\_u = Maximum Vertical Force from Equation (24 [lb])  
 $\varphi = 0.90$ 

$$\varphi P_n = \varphi \frac{F_y BN}{2} \left(\frac{t}{l}\right)^2$$
(54)  
Where:  $\varphi P_n = \text{Nominal Baseplate Strength [lb]}$   
 $t = \text{Baseplate Thickness [in.]}$   
 $l = \text{The Greater of } m \text{ and } n \text{ [in.]}$ :  
 $m = \frac{N - 0.95d}{2}$   
 $n = \frac{B - 0.80b_f}{2}$   
B = Baseplate Width [in.]  
N = Baseplate Depth [in.]  
 $b_f = \text{Post Flange Width [in.]}$   
 $d = \text{Post Flange Width [in.]}$   
 $f_y = \text{Yield Stress [psi]}$   
 $\varphi = 0.90$ 

An alternative equation from the AISC *Steel Design Guide Series 1* [41-42] combines the axial force and moment on the baseplate, and the minimum can be determined using the required bending moment on the baseplate with Equation 55. Using method no. 2, the nominal capacity of the baseplate is calculated with Equation 56. An example of the baseplate design for Concept AW2-A is shown in Appendix C.

$$t_{min} = \sqrt{\frac{4M_{pl}}{\varphi F_{y}}}.$$
(55)  
Where:  $t_{min} = \text{Minimum Baseplate Thickness [in.]}$   
 $F_y = \text{Yield Stress [psi]}$   
 $M_{pl} = \text{Required Bending Moment per Width from Equation (25
[lb-in./in.]]
 $\varphi = 0.90$   
 $\varphi M_n = \frac{\varphi F_y t^2}{4}$ 
(56)  
Where:  $M_n = \text{Nominal Bending Moment per Width [lb-in./in.]}$   
 $t = \text{Baseplate Thickness [in.]}$   
 $F_y = \text{Yield Stress [psi]}$$ 

For the welded aluminum concepts, the base of the post or a sleeve was welded to the baseplate; therefore,  $F_y$  was set equal to  $F_{tyw} = 15,000$  psi. For the modular concept, the baseplate was a cast aluminum part made from aluminum alloy 535. According to Table A.3.6 in the ADM, the tensile yield strength for alloy 535 is 13,500 psi (93 MPa).

 $\phi = 0.90$ 

## 7.3.3 Anchors

Steel bolts were preferred over aluminum due to availability. Certain types of steel react with aluminum; therefore, the grade of anchor bolts was selected to be compatible with 6061-T6 aluminum. Threaded anchor rods that were embedded into a concrete foundation using an epoxy adhesive were selected as they are the easiest to install and would allow the rail system to be repaired without replacing anchors. The threaded rod was configured with ASTM A193 Grade B7 steel.

The shear and flexural stresses have two different effects on the anchor. The shear stress that is transferred to the anchor is resisted completely by the shear capacity of the anchor bolts, while the flexural stress is assumed to concentrate about a moment arm, resulting in an upward tension on the anchorage rods. The procedure in the *Building Code Requirements for Structural Concrete (ACI 318-11)* [40] was used to determine the appropriate size and strength required of the anchor bolts while using Powers Fasteners AC100+ Gold epoxy. The minimum bond strength of the Powers Fastener epoxy is 1,450 psi (10.0 MPa) for threaded rods up to 7/8 in. (22 mm) in diameter. The equations to find the compared required strength of the steel, concrete, and bond are shown in Table D.4.1.1 in ACI 318-11 [40] and in Table 9. The concrete foundation has a minimum compressive strength of 2,500 psi (17.2 MPa), a minimum thickness of 7 in. (178 mm), and outer dimensions at least 10 in. (254 mm) away from the nearest anchor.

		Ancho	or group*		
Failure mode	Single anchor	Individual anchor in a group	Anchors as a group		
Steel strength in tension (D.5.1)	φN <sub>sa</sub> ≥ N <sub>ua</sub>	¢N <sub>sa</sub> ≥ N <sub>ua,i</sub>			
Concrete breakout strength in tension (D.5.2)	<i>φ</i> N <sub>cb</sub> ≥ N <sub>ua</sub>		$\phi N_{cbg} \ge N_{ua,g}$		
Pullout strength in tension (D.5.3)	∳N <sub>pn</sub> ≥ N <sub>ua</sub>	∲N <sub>pn</sub> ≥ N <sub>ua,i</sub>			
Concrete side-face blowout strength in tension (D.5.4)	∲N <sub>sb</sub> ≥ N <sub>ua</sub>		φN <sub>sbg</sub> ≥N <sub>ua,g</sub>		
Bond strength of adhesive anchor in tension (D.5.5)	∲N <sub>a</sub> ≥ N <sub>ua</sub>		∳N <sub>ag</sub> ≥ N <sub>ua,g</sub>		
Steel strength in shear (D.6.1)	¢V <sub>sa</sub> ≥ V <sub>ua</sub>	∳V <sub>sa</sub> ≥ V <sub>ua,i</sub>			
Concrete breakout strength in shear (D.6.2)	¢V <sub>cb</sub> ≥ V <sub>ua</sub>		¢V <sub>cbg</sub> ≥ V <sub>ua,g</sub>		
Concrete pryout strength in shear (D.6.3)	¢V <sub>cp</sub> ≥ V <sub>ua</sub>		¢V <sub>cpg</sub> ≥ V <sub>ua,g</sub>		
Required strengths for steel and pullout failure modes shall be calculated for the most highly stressed anchor in the group.					

## Table 9. Strength of Anchors [40]

## 7.3.3.1 Tension

The designs that were considered in Section 5.2 utilized rails spanning between two posts. One anchor plate was attached to each post, as shown in Figure 29. Calculations were performed to determine whether or not a two-bolt anchor plate design was sufficient to withstand the design loads. Assuming a nearly rigid system and worst-case loading conditions means that nearly all bending load is applied to one anchor, meaning that one anchor rod (e.g., front anchor rod) would support all of the tension load and one anchor rod (e.g., rear anchor rod) would not be loaded, due to bending loads on the frame. Thus, each anchor was treated independently in the calculations.

ACI318-11 compares five different failure criteria for the anchorage system under tensile loading to determine the final capacity of the anchor: steel strength ( $N_{sa}$ ), concrete breakout

strength (N<sub>cb</sub>), pullout strength (N<sub>pn</sub>), concrete side-face blowout strength (N<sub>sb</sub>), and bond strength of adhesive anchor  $(N_a)$  [40].

The equation used to determine the steel strength of an anchor rod in tension is calculated with Equation D-2 in ACI 318-11 [40] and is shown in Equation 57.

$$\varphi N_{sa} = \varphi A_{se,N} f_{uta} \tag{57}$$

 $\phi N_{sa}$  = Nominal Strength of an Anchor in Tension [in.<sup>2</sup>] Where:  $A_{se,N}$  = Effective Cross-Sectional Area of an Anchor in Tension  $[in.^2]$ f<sub>uta</sub> = Steel Strength, Minimum [1.9f<sub>ya</sub>, 125,000 psi] f<sub>va</sub> = Yield Strength of Anchor [psi]  $\omega = 0.75$ 

Determination of the concrete breakout strength is given in equation D-3 in ACI 318-11 [40] and by Equation 58. The project concrete failure area,  $A_{nc}$ , is estimated as the base of the rectilinear geometrical figure that results from projecting the failure surface outward 1.5h<sub>ef</sub> from the centerlines of the anchor.

$$\varphi N_{cb} = \varphi \frac{A_{Nc}}{A_{Nco}} \Psi_{ed,N} \Psi_{c,N} \Psi_{cp,N} N_b$$
(58)

Where:

 $\phi N_{cb}$  = Nominal concrete breakout strength in Tension of a Single Anchor [lb]

- $A_{nc}$  = Projected Concrete Failure Area of a Single Anchor or Group of Anchors [in.<sup>2</sup>]
- $A_{Nco}$  = Projected Concrete Failure Area of a Single Anchor [in.<sup>2</sup>]  $=9h_{ef}^{2}$
- Anchor Embedment Depth [in.]  $h_{ef} =$

 $\Psi_{ed,N}$  = Modification Factor for Edge Effects for Single Anchor or Anchor Groups Tension in Tension ( $\leq 1.0$ ) If  $c_{a,min} \ge 1.5 h_{ef}$ ,  $\Psi_{ed,N} = 1.0$ H

$$f c_{a,min} < 1.5 h_{ef} \Psi_{ed,N} = 0.7 + 0.3 \frac{c_{a,min}}{1.5 h_{ef}}$$

- Distance from the Center of an Anchor to the Edge of the  $c_{a1} =$ Concrete in one Direction [in.]
- Distance from the Center of an Anchor to the Edge of the  $c_{a2} =$ Concrete in Direction Perpendicular to  $c_{a1}$  [in.]
- $c_{a,min}$  = Minimum Distance from the Center of an Anchor to the Edge of the Concrete [in.]

- $\Psi_{c,N}$  = Modification Factor Based on Presence or Absence of Cracks in Concrete
  - = 1.25 for Cast-In Anchors
  - = 1.4 for Post-Installed Anchors

$$\begin{split} \Psi_{cp,N} &= \text{Modification Factor for Post-Installed Anchors for} \\ & \text{Uncracked Concrete without Supplementary Reinforcement} \\ & \text{If } c_{a,\min} \geq c_{ac}, \Psi_{cp,N} = 1.0 \\ & \text{If } c_{a,\min} < c_{ac}, \Psi_{cp,N} = \frac{c_{a,\min}}{c_{ac}} \\ & c_{ac} = \text{Critical Edge Distance [in.]} = 2h_{ef} \text{ (for Adhesive Anchors)} \\ & \text{N}_{b} = \text{Basic Concrete Breakout Strength of a Single Anchor in} \\ & \text{Tension} = k_{c}\lambda_{a}\sqrt{f_{c}'}h_{ef}^{1.5} \\ & \text{k}_{c} = 24 \text{ for Cast-In Anchors} \\ &= 17 \text{ for Post-Installed Anchors} \\ & f_{c}' = \text{Compressive Strength of Concrete [psi]} = 2,500 \text{ psi} \end{split}$$

The pullout strength ( $N_{pn}$ ) is not applicable for adhesive anchors, according to Section D.5.3 in ACI 318-11. The concrete side-face blowout strength ( $N_{sb}$ ) is not applicable unless deep anchors exist (where  $h_{ef} > 2.5c_{a1}$ ), which is not the case for these designs. The bond strength of an adhesive anchor in tension is given by equation D-18 in ACI 318-11 [40] and is shown in Equation 59.

 $\phi = 0.65$ 

$$\varphi N_a = \varphi \frac{A_{Na}}{A_{Nao}} \Psi_{ed,Na} \Psi_{cp,Na} N_{ba}$$
<sup>(59)</sup>

Where:

$$\begin{split} \varphi N_{a} &= \text{Nominal Bond Strength in Tension of a Single Anchor [lb]} \\ A_{Na} &= \text{Projected Influence Area of a Single Anchor or Group of Anchors [in.<sup>2</sup>]} \\ A_{Nao} &= \text{Projected Influence Area of a Single Adhesive Anchor with an Edge Distance Equal to or Greater than <math>c_{Na}$$
 [in.<sup>2</sup>],  $A_{Nao} = (2c_{Na})^{2}$  $c_{Na} = \text{Critical Distance } = 10d_{a}\sqrt{\frac{\tau_{uncr}}{1100}}$  $d_{a} = \text{Diameter of Anchor [in.]}$  $\tau_{uncr} = \text{Uncracked Shear Stress [psi]}$  $\Psi_{ed,Na} = \text{Modification Factor for Edge Effects for Single Anchors or Anchor Groups Loaded in Tension If <math>c_{a,\min} \ge c_{Na}, \Psi_{ed,Na} = 1.0$  $\text{If } c_{a,\min} < c_{Na}, \Psi_{ed,Na} = 0.7 + 0.3 \frac{c_{a,\min}}{c_{Na}}$  $c_{a1} = \text{Distance from the Center of an Anchor to the Edge of the Edge of the Edge of the Stress from the Center of an Anchor to the Edge of the Stress of the Edge of the Stress from the Center of the Edge of the Edge of the Stress from the Center of the Edge of the Edge of the Stress from the Center of the Edge of the Edge of the Stress from the Center of the Edge of the Stress from Stress from the Center of the Edge of the Edge of the Stress from Stress from Stress from the Center of the Edge of the Stress from the Center of the Edge of the Stress from Stress$ 

Concrete in one Direction [in.]

- $c_{a2}$  = Distance from the Center of an Anchor to the Edge of the Concrete in Direction Perpendicular to  $c_{a1}$  [in.]
- $c_{a,min}$  = Minimum Distance from the Center of an Anchor to the Edge of the Concrete [in.]

 $\Psi_{cp,Na}$ = Modification Factor for Adhesive Anchors in Uncracked Concrete without Supplementary Reinforcement

If  $c_{a,min} \ge c_{ac}$ ,  $\Psi_{cp,Na} = 1.0$ If  $c_{a,min} < c_{ac}$ ,  $\Psi_{cp,Na} = \frac{c_{a,min}}{c_{ac}}$   $c_{ac}$ = Critical Edge Distance [in.]  $N_{ba} = \lambda_a \tau_{cr} \pi d_a h_{ef}$   $h_{ef}$ = Anchor Embedment Depth [in.]  $\tau_{cr}$ = Characteristic Bond Stress [psi]  $\lambda_a = 1.0$  for Normal-Weight Concrete  $\varphi = 0.65$ 

## 7.3.3.2 Shear

ACI318-11 compares three different failure criteria of the anchorage system under shear loading to determine the final capacity of the anchor: steel strength ( $V_{sa}$ ), concrete breakout strength ( $V_{cb}$ ), and concrete pryout strength ( $V_{cp}$ ) [40]. Two anchors with a spacing, s, can be loaded in shear at the same time, so the two anchors should be considered a group.

The equation used to determine the steel strength of an anchor rod in tension is calculated with Equation D-29 in ACI318-11 [40] and is shown in Equation 60.

$$\varphi V_{sa} = \varphi 0.6A_{se,V} f_{uta} \tag{60}$$

Where:  $\phi V_{sa} =$  Nominal Strength of an Anchor in Shear [lb]  $A_{se,V} =$  Effective Cross-Sectional Area of an Anchor in Shear [in.<sup>2</sup>]  $f_{uta} =$  Steel Strength, Minimum [1.9 $f_{ya}$ , 125,000 psi]  $f_{ya} =$  Yield Strength of Anchor [psi]  $\phi = 0.75$ 

Determination of the concrete breakout strength is given in equation D-31 in ACI318-11 [40] and by Equation 61. The project concrete failure area,  $A_{vc}$ , is estimated as the base of the rectilinear geometrical figure that results from projecting the failure surface outward  $1.5c_{a1}$  from the centerline of the anchors.

$$\varphi V_{cbg} = \varphi \frac{A_{Vc}}{A_{Vco}} \Psi_{ec,V} \Psi_{ed,V} \Psi_{c,V} \Psi_{h,V} V_b$$
(61)

Where:

 $\phi V_{cbg}$  = Nominal Concrete Breakout Strength in Shear of a Group of

Anchors [lb]

A<sub>Vc</sub> = Projected Concrete Failure Area of a Single Anchor or Group of Anchors [in.<sup>2</sup>]

 $A_{Vco}$  = Projected Concrete Failure Area of a Single Anchor [in.<sup>2</sup>] =  $4.5(c_{a1})^2$ 

 $h_a$  = Depth of Concrete Foundation [in.]

 $\Psi_{ec,V}$  = Modification Factor for Anchors Based on Eccentricity of Applied Loads =  $\frac{1}{\sqrt{1-2a'}}$ 

$$\frac{1+\frac{2e_V}{3c_{a1}}}{\left(1+\frac{2e_V}{3c_{a1}}\right)}$$

 $e'_V$  = Eccentricity of Applied Shear Force [in.] = 0

 $\Psi_{ed,V}$  = Modification Factor for Edge Effects for Single Anchor or Anchor Groups Loaded in Shear

If 
$$c_{a2} \ge 1.5 c_{a1}, \Psi_{ed,V} = 1.0$$

If 
$$c_{a2} < 1.5 c_{a1}, \Psi_{ed,V} = 0.7 + 0.3 \frac{C_{a2}}{1.5 c_{a1}}$$

- $c_{a1}$  = Distance from the Center of an Anchor to the Edge of the Concrete in one Direction [in.]
- $c_{a2}$  = Distance from the Center of an Anchor to the Edge of the Concrete in Direction Perpendicular to  $c_{a1}$  [in.]
- $c_{a,min}$  = Minimum Distance from the Center of an Anchor to the Edge of the Concrete [in.]
- $\Psi_{c,V}$  = Modification Factor Based on Presence or Absence of Cracks in Concrete
  - = 1.4 for No Cracking at Service Loads
  - = 1.0 for Anchors in Cracked Concrete without Supplementary Reinforcement
  - = 1.2 for Anchors in Cracked Concrete with Supplementary Reinforcement
  - = 1.4 for Anchors in Cracked Concrete with Supplementary Reinforcement Enclosed within Stirrups
- $\Psi_{h,V}$ = Modification Factor for Anchors Located in Concrete

If 
$$h_a < 1.5c_{a1}, \Psi_{h,V} = \sqrt{\frac{1.5c_{a1}}{h_a}} \ge 1.0$$

Otherwise  $\Psi_{h,V} = 1.0$ 

 $V_b$  = Basic Concrete Breakout Strength of a Single Anchor in Shear is Equal to the Smaller of:

$$V_b = \left(7\left(\frac{l_e}{d_a}\right)^{0.2}\sqrt{d_a}\right)\lambda_a\sqrt{f_c'}(c_{a1})^{1.5}$$
$$V_b = 9\lambda_a\sqrt{f_c'}(c_{a1})^{1.5}$$

 $l_e = h_{ef}$  for Anchors with a Constant Stiffness over the Length of Embedded Section

d<sub>a</sub> = Diameter of Anchor [in.]  $f_c$ ' = Compressive Strength of Concrete [psi] = 2,500 psi  $\phi = 0.75$ 

The concrete pryout of an adhesive anchor in shear is given by equation D-41 in ACI318-

11 [40] and is shown in Equation 62.

$$\varphi V_{cpg} = \varphi k_{cp} N_{cpg} \tag{62}$$

Where:

 $\varphi V_{cpg}$  = Nominal Concrete Pryout Strength in Shear of Anchor Group [lb]

 $k_{cp}$  = Coefficient for Pryout Strength

$$= 1.0 \text{ for } h_{ef} < 2.5 \text{ in}$$

- = 2.0 for  $h_{ef} \ge 2.5$  in.
- h<sub>ef</sub> = Anchor Embedment Depth [in.]

$$N_{cpg} = Concrete Pryout Strength for Adhesive Anchors is Lesser of 
 $N_{ag} = \frac{A_{Na}}{A_{Nao}} \Psi_{ec,Na} \Psi_{ed,Na} \Psi_{cp,Na} N_{ba}$$$

$$N_{cbg} = \frac{A_{Nc}}{A_{Nco}} \Psi_{ec,N} \Psi_{ed,N} \Psi_{c,N} \Psi_{cp,N} N_{b}$$

- $N_{ag} =$ Nominal Bond Strength in Tension of an Anchor Group
- A<sub>Na</sub>= Projected Influence Area of a Single Anchor or Group of Anchors [in.<sup>2</sup>]
- A<sub>Nao</sub>= Projected Influence Area of a Single Anchor with an Edge Distance Equal to or Greater than  $c_{Na}$  [in.<sup>2</sup>].  $A_{Nao} =$  $(2c_{Na})^2$

$$c_{Na} = \text{Critical Distance} = 10 d_a \sqrt{\frac{\tau_{uncr}}{1100}}$$

- $d_a =$ Diameter of Anchor [in.]
- $\tau_{uncr}$  = Uncracked Shear Stress [psi]

$$\Psi_{ec,Na}$$
 = Modification Factor for Anchors Based on Eccentricity of  
Applied Loads =  $\frac{1}{\left(1+\frac{2e'_N}{3c_{a1}}\right)}$ 

 $e'_N$  = Eccentricity of Applied Tension Force [in.] = 0

 $\Psi_{ed,Na}$ =Modification Factor for Edge Effects for Single Anchors or Anchor Groups Loaded in Tension

c

If 
$$c_{a,min} \ge c_{Na}$$
,  $\Psi_{ed,Na} = 1.0$ 

If 
$$c_{a,min} < c_{Na}, \Psi_{ed,Na} = 0.7 + 0.3 \frac{c_{a,min}}{c_{Na}}$$

- $c_{a1}$  = Distance from the Center of an Anchor to the Edge of the Concrete in One Direction [in.]
- $c_{a2}$  = Distance from the Center of an Anchor to the Edge of the Concrete in Direction Perpendicular to c<sub>a1</sub> [in.]
- $c_{a,min}$  = Minimum Distance from the Center of an Anchor to the Edge of the Concrete [in.]

 $\Psi_{cp,Na}$ = Modification Factor for Adhesive Anchors in uncracked Concrete without Supplementary Reinforcement

If 
$$c_{a,min} \ge c_{ac}$$
,  $\Psi_{cp,Na} = 1.0$   
If  $c_{a,min} < c_{ac}$ ,  $\Psi_{cp,Na} = \frac{c_{a,min}}{c_{ac}}$ 

c<sub>ac</sub>= Critical Edge Distance [in.]

$$N_{ba} = \lambda_a \tau_{cr} \pi d_a h_{ef}$$

- h<sub>ef</sub> = Anchor Embedment Depth [in.]
- $\tau_{cr}$  = Characteristic Bond Stress [psi]
- $\lambda_a = 1.0$  for Normal-Weight Concrete
- $N_{cbg}$  = Nominal Concrete Breakout Strength in Tension of Anchor Group [lb]
- A<sub>nc</sub> = Projected Concrete Failure Area of a Single Anchor or Group of Anchors [in.<sup>2</sup>]
- $A_{\text{Nco}}$  = Projected Concrete Failure Area of a Single Anchor [in.<sup>2</sup>] =  $9h_{ef}^2$
- h<sub>ef</sub> = Anchor Embedment Depth [in.]
- $\Psi_{ec,N}$  = Modification Factor for Anchors Based on Eccentricity of Applied Loads =  $\frac{1}{\left(1+\frac{2e'_N}{3c_{a1}}\right)}$
- $\Psi_{ed,N}$  = Modification Factor for Edge Effects for Single Anchors or Anchor Groups Loaded in Tension
  - in Tension ( $\leq 1.0$ )
  - If  $c_{a,\min} \ge 1.5 h_{ef}$ ,  $\Psi_{ed,N} = 1.0$

If 
$$c_{a,min} < 1.5 h_{ef}, \Psi_{ed,N} = 0.7 + 0.3 \frac{C_{a,min}}{1.5 h_{ef}}$$

- $\Psi_{c,N}$  = Modification Factor Based on Presence or Absence of Cracks in Concrete
  - = 1.25 for Cast-In Anchors
  - = 1.4 for Post-Installed Anchors
- $\Psi_{cp,N}$ = Modification Factor for Post-Installed Anchors for Uncracked Concrete without Supplementary Reinforcement If  $c_{a,min} \ge c_{ac}$ ,  $\Psi_{cp,N} = 1.0$

f c<sub>a,min</sub> < c<sub>ac</sub>, 
$$\Psi_{cp,N} = \frac{c_{a,min}}{c_{ac}}$$

- $c_{ac}$  = Critical Edge Distance =  $2h_{ef}$  (for Adhesive Anchors)
- $N_b$  = Basic Concrete Breakout Strength of a Single Anchor in Tension =  $k_c \lambda_a \sqrt{f_c'} h_{ef}^{1.5}$
- $k_c = 24$  for Cast-In Anchors = 17 for Post-Installed Anchors  $f_c$ ' = Compressive Strength of Concrete [psi] = 2,500 psi  $\varphi = 0.75$

The calculations for determining the nominal capacities of the anchors for each concept are shown in Appendix C, while details are provided in Section 7.4.

#### 7.3.4 Modular Cast Aluminum

For the modular system, cast aluminum connections including tees and elbows were provided with the system. Since these connections are specialized parts based on the selected rail and post sizes, the capacities of these parts were not evaluated.

## 7.4 Final Designs

## 7.4.1 Introduction

Two concepts from Section 5.2 were further refined into four concepts and had final design calculations completed. Those concepts included the Modular Aluminum AM-1 design (Section 5.2.1) and three variations of the Welded Aluminum AW2 design (Section 5.2.3). The capacity of each component of the final four rail designs to be tested (AW2-A, AW2-C, AM-1, and AW2-D), as well as the required design loads, are shown in Appendix C. The system drawings for each of the concepts are shown in Figures 102 through 121.

The smallest section that met the required design loads was determined for each component. The sections were then evaluated to determine if they were commonly available by aluminum suppliers. If not, the next smallest section that was commonly available was selected for the final design. In some cases, the thickness of the section was optimized to match the minimum base metal thickness for a given weld size.

For the three AW2 concepts, a <sup>3</sup>/<sub>8</sub>-in. (10-mm) thick baseplate was selected, even though Concept AW2-C did not meet the required design loads based on the method no. 1 equations for the nominal capacity. None of the concepts met the baseplate design with method no. 2 equations, and since these equations were derived with columns with a large axial force, they may not be applicable to this situation. However, the smaller baseplate dimensions were selected for two reasons. First, the baseplate capacity equations were derived for steel connections and are believed to be inherently conservative. Second, it was not desired that the anchor bolts incur damage when impacted dynamically. If a larger and/or thicker baseplate were selected, the greater plate capacity may cause a greater load imparted to the anchor bolts, which could cause permanent deformation. The researchers believed that a <sup>3</sup>/<sub>8</sub>-in. (10-mm) thick baseplate was sufficient to sustain the AASHTO *LRFD Bridge Design Specifications* pedestrian rail live loads and had the ability to verify the loads with a static component test, if necessary.

#### 7.4.2 AW2-A Welded Aluminum

Concept AW2-A designated 2-in. x 4-in. x 43-in. tall (51-mm x 102-mm x 6-mm x 1,029-mm tall) posts with three 2-in. x 2-in. x <sup>1</sup>/<sub>8</sub>-in. (51-mm x 51-mm x 3-mm) rail components at heights of 42 in. (1,067 mm),  $34\frac{1}{8}$  in. (867 mm), and  $7\frac{1}{8}$  in. (200 mm). The rails were secured to the posts with  $\frac{1}{2}$ -in. (3-mm) fillet welds at each connection. Nine  $\frac{1}{2}$ -in. x 24<sup>1</sup>/<sub>4</sub>-in. (13-mm x 13-mm x 616-mm) square spindles were used as the infill design between the mid and bottom rail components, connected with <sup>1</sup>/<sub>8</sub>-in. (3-mm) fillet welds. Each post member was welded to a 3-in. x 7<sup>1</sup>/<sub>2</sub>-in. x <sup>3</sup>/<sub>8</sub>-in. (76-mm x 191-mm x 9.5-mm) baseplate with a <sup>1</sup>/<sub>4</sub>-in. (6mm) fillet weld at the connection. The baseplate had two <sup>1</sup>/<sub>2</sub>-in. (13-mm) holes spaced at 6 in. (152 mm) to accommodate two <sup>3</sup>/<sub>8</sub>-in. (9.5-mm) threaded anchor rods, each embedded 5 in. (127 mm) into 1,450-psi (10.0-MPa) minimum bond strength epoxy adhesive secured through the baseplate with a <sup>3</sup>/<sub>8</sub>-in. (9.5-mm) dia. ASTM A194 Grade 8M nut. The concrete foundation has a minimum compressive strength of 2,500 psi (17.2 MPa), a minimum thickness of 7 in. (178 mm), and outer dimensions at least 10 in. (254 mm) away from the nearest anchor. The static post deflection was estimated to be 0.19 in. (4.8 mm), and the static rail deflection was estimated to be 0.32 in. (8 mm). Detailed schematic drawings of design AW2-A are shown in Figures 102 through 105, 109, and 114. A photograph of design AW2-A is shown in Figure 122.

#### 7.4.3 AW2-C Welded Aluminum

Concept AW2-C designated 2-in. x 3-in. x <sup>1</sup>/<sub>8</sub>-in.x 43-in. tall (51-mm x 76-mm x 3-mm x 1,029-mm tall) posts with three 2-in. x 2-in. x ½-in. (51-mm x 51-mm x 3-mm) rail components at heights of 42 in. (1,067 mm),  $34\frac{1}{8}$  in. (867 mm), and  $7\frac{1}{8}$  in. (200 mm). The rails were secured to the posts with <sup>1</sup>/<sub>8</sub>-in. (3-mm) fillet welds at each connection. Nine <sup>1</sup>/<sub>2</sub>-in. x <sup>1</sup>/<sub>2</sub>-in. x 24<sup>1</sup>/<sub>4</sub>-in. (13mm x 13-mm x 616-mm) square spindles were used as the infill design between the mid and bottom rail components, connected with 1/8-in. (3-mm) fillet welds. Each post member was connected to a 3<sup>1</sup>/<sub>2</sub>-in. x 7<sup>1</sup>/<sub>2</sub>-in. x <sup>3</sup>/<sub>8</sub>-in. (89-mm x 191-mm x 9.5-mm) baseplate with a retention sleeve. The 3<sup>1</sup>/<sub>2</sub>-in. (89-mm) tall retention sleeve was constructed using <sup>1</sup>/<sub>4</sub>-in. (6-mm) aluminum plates to form a sleeve for the post. The outer dimensions of the sleeve were  $2\frac{5}{8}$  in. x  $3\frac{5}{8}$  in. (67) mm x 92 mm), and the inner dimensions were  $2\frac{1}{8}$  in. x  $3\frac{1}{8}$  in. (54 mm x 79 mm). The sleeve was completely welded to the baseplate with a  $\frac{3}{16}$ -in. (4.8-mm) fillet weld. A  $\frac{3}{8}$ -in. (9.5-mm) hole was drilled longitudinally through both the sleeve and post components at a height of 2 in. (51 mm) from the surface of the baseplate, so a <sup>1</sup>/<sub>4</sub>-in. dia. x 3-in. long (6-mm dia. x 76-mm long) ASTM A193 Grade B8M bolt could be fastened through the post and sleeve together, secured with a <sup>1</sup>/<sub>4</sub>-in. (6-mm) dia. ASTM A194 Grade 8M nut. The baseplate had two <sup>1</sup>/<sub>2</sub>-in. (13-mm) holes spaced at 6 in. (152 mm) to accommodate two <sup>3</sup>/<sub>8</sub>-in. (9.5-mm) threaded anchor rods, each embedded 5 in. (127 mm) into 1,450-psi (10.0-MPa) minimum bond strength epoxy adhesive and secured through the baseplate with a <sup>3</sup>/<sub>8</sub>-in. (9.5-mm) dia. ASTM A194 Grade 8M nut. The concrete foundation has a minimum compressive strength of 2,500 psi (17.2 MPa), a minimum thickness of 7 in. (178 mm), and outer dimensions at least 10 in. (254 mm) away from the nearest anchor. The static post deflection was estimated to be 0.70 in. (18 mm), and the static rail deflection was estimated to be 0.32 in. (8 mm). Detailed drawings of design AW2-C are shown in Figures 102, 103, 106 through 109, and 114. A photograph of design AW2-C is shown in Figure 123.

### 7.4.4 AM-1 Modular Aluminum

Concept AM-1 was a standard modular system, the Speed Rail® that is available through Hollaender. The system was designed by Hollaender according to the AASHTO pedestrian rail loads [6] and the rail, post, spindle, and baseplate were verified by MwRSF in Appendix C. Hollaender's Speed-Rail® system is composed of 6061-T6 aluminum circular tube rail and post members, with tee, elbow, and cross fittings utilized as connections. Various standard bases are available depending on the combination of the required strength and anchoring options [35].

The Hollaender modular system uses 2-in. dia. x 39-in. long (51-mm dia. x 991-mm long) schedule 80 posts, with 2-in. dia. x 56½-in. long (51-mm dia. x 1,435-mm long) schedule 40 rails, and standard bases for anchoring, depending on the style and required capacity. Hollaender's two-hole No. 48 Heavy-Duty Base Flange was selected as the base connection bracket for anchoring the system to the concrete. Two ¾-in. (9.5-mm) threaded anchor rods were embedded 5 in. (127 mm) into 1,450-psi (10.0-MPa) minimum bond strength epoxy adhesive and secured through the baseplate with a ¾-in. (9.5-mm) dia. ASTM A194 Grade 8M nut. The concrete foundation has a minimum compressive strength of 2,500 psi (17.2 MPa), a minimum thickness of 7 in. (178 mm), and outer dimensions at least 10 in. (254 mm) away from the nearest anchor. The static post deflection was estimated to be 1.18 in. (30 mm), and the static rail deflection was estimated to be 0.26 in. (7 mm). Detailed drawings of design AM-1 are shown in Figures 102, 103, 110 through 113, and 115. A photograph of design AM-1 is shown in Figure 124.

#### 7.4.5 AW2-D Welded Aluminum

Concept AW2-D was similar to Concept AW2-A, with differences in the rail locations and spindle length. Concept AW2-D designated 2-in. x 4-in. x 4/2-in. x 4/2-in. tall (51-mm x 102mm x 6-mm x 1,029-mm tall) posts with three 2-in. x 2-in. x  $\frac{1}{8}$ -in. (51-mm x 51-mm x 3-mm) rail components at heights of 42 in. (1,067 mm),  $24^{15}/16$  in. (633 mm), and 7% in. (200 mm). The rails were inserted into cutouts in the posts at each rail location and secured to the face of the posts with <sup>1</sup>/<sub>8</sub>-in. (3-mm) fillet welds at each connection. The post-to-rail connection was more rigid than Concept AW2-A due to the interaction of the end of the rail with both post faces. Nine <sup>1</sup>/<sub>2</sub>-in. x <sup>1</sup>/<sub>2</sub>-in. x 32<sup>1</sup>/<sub>8</sub>-in. (13-mm x 13-mm x 816-mm) square spindles were spanned between the top and bottom rail and were inserted through the middle rail. The spindles were welded with <sup>1</sup>/<sub>8</sub>in. (3-mm) fillet welds at each rail location. Each post member was welded to a 3-in. x 7<sup>3</sup>/<sub>4</sub>-in. x <sup>3</sup>/<sub>8</sub>-in. (76-mm x 191-mm x 9.5-mm) baseplate with a <sup>1</sup>/<sub>4</sub>-in. (6-mm) fillet weld at the connection. The baseplate had two <sup>5</sup>/<sub>8</sub>-in. (16-mm) holes spaced at 6<sup>1</sup>/<sub>4</sub> in. (159 mm) to accommodate two <sup>1</sup>/<sub>2</sub>in. (13-mm) diameter threaded anchor rods, each embedded 5 in. (127 mm) into 1,450-psi (10.0-MPa) minimum bond strength epoxy adhesive and secured through the baseplate with a <sup>1</sup>/<sub>2</sub>-in. (13-mm) diameter ASTM A194 Grade 8M nut. The concrete foundation has a minimum compressive strength of 2,500 psi (17.2 MPa), a minimum thickness of 7 in. (178 mm), and outer dimensions at least 10 in. (254 mm) away from the nearest anchor. The static post deflection was estimated to be 0.19 in. (4.8 mm), and the static rail deflection was estimated to be 0.32 in. (8 mm). Detailed drawings of design AW2-D are shown in Figures 116 through 121. A photograph of design AW2-D is shown in Figure 125.

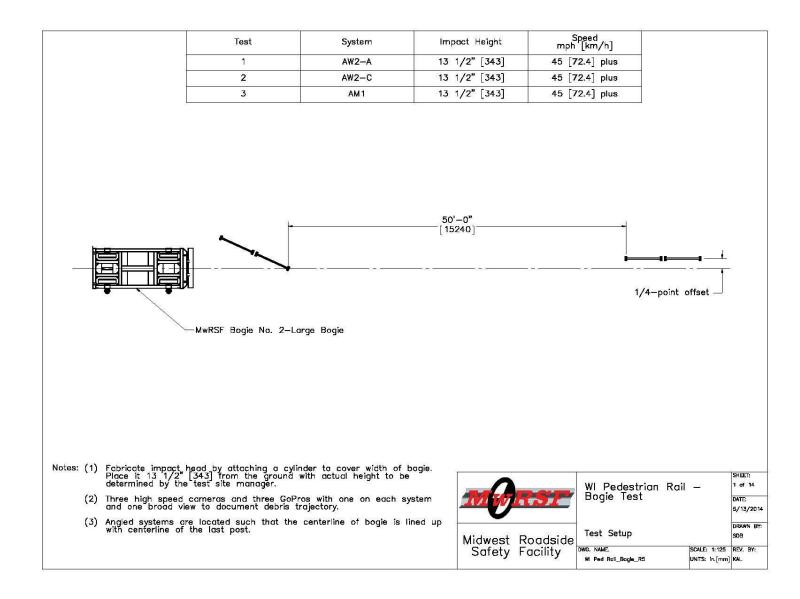


Figure 102. Pedestrian Rail Test Setup, Designs AW2-A, AW2-C and AM-1

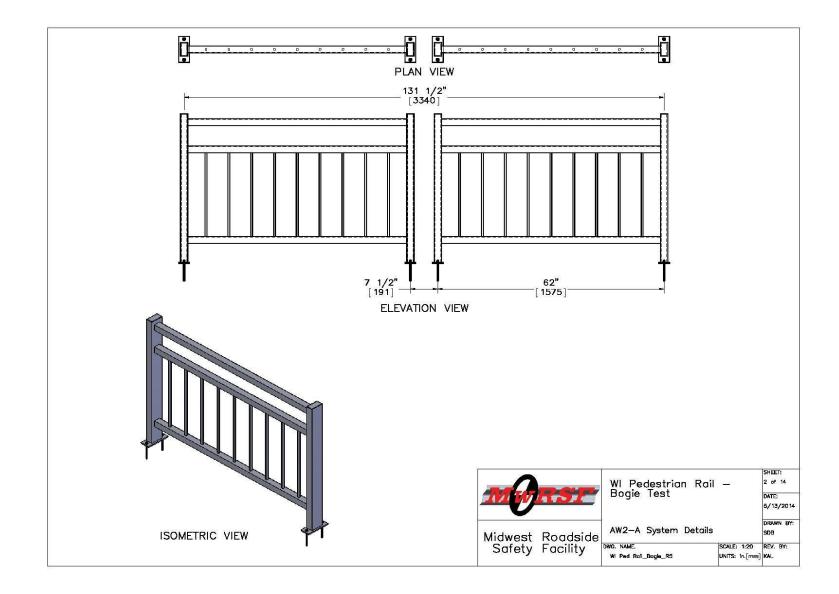


Figure 103. Pedestrian Rail Test Setup, Designs AW2-A, AW2-C and AM-1

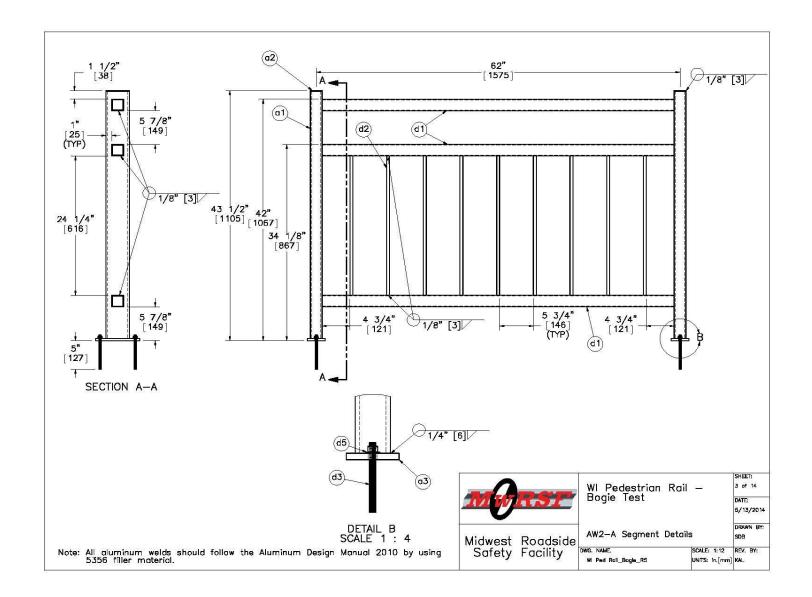


Figure 104. Pedestrian Rail Design AW2-A

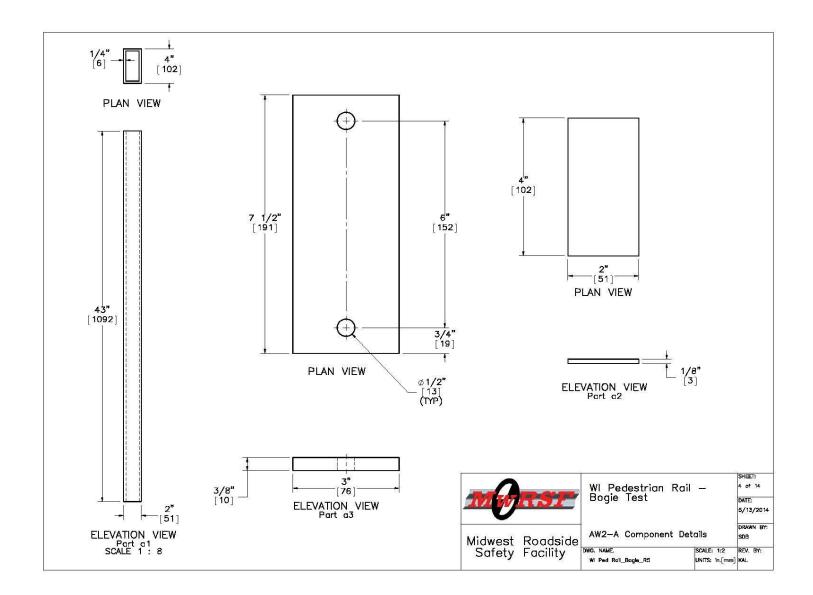


Figure 105. Pedestrian Rail Design AW2-A

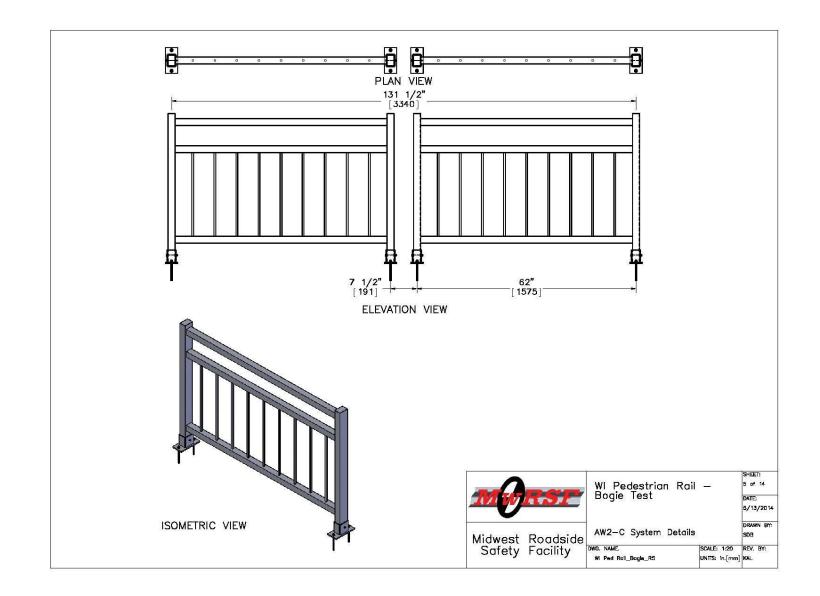


Figure 106. Pedestrian Rail Design AW2-C

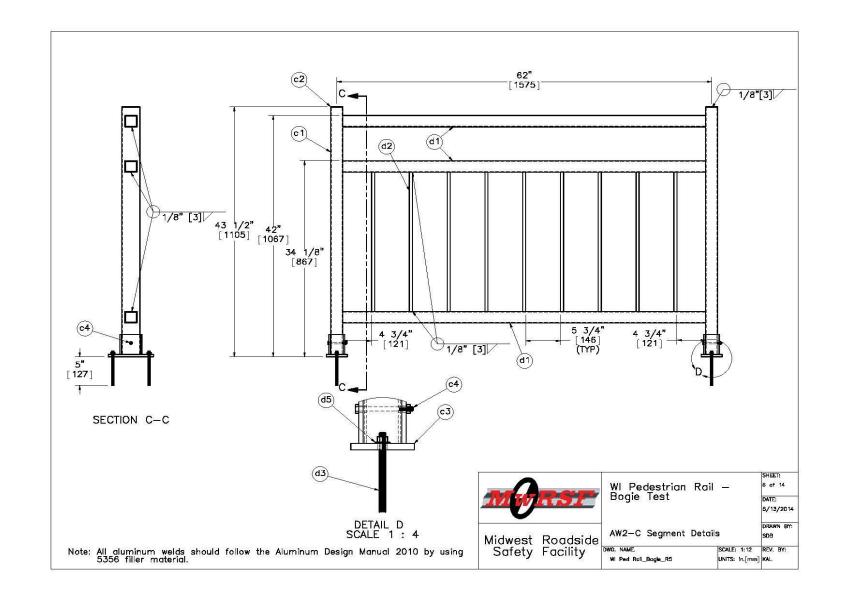
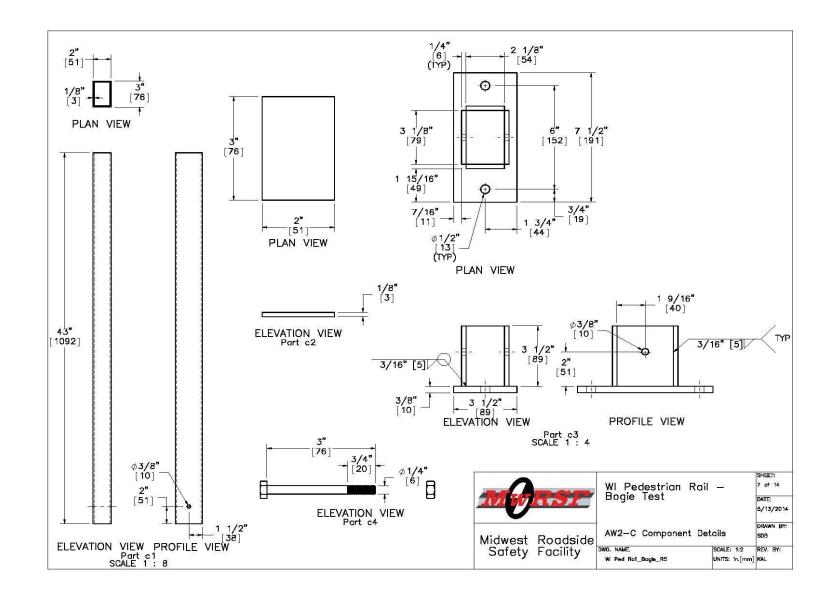


Figure 107. Pedestrian Rail Design AW2-C



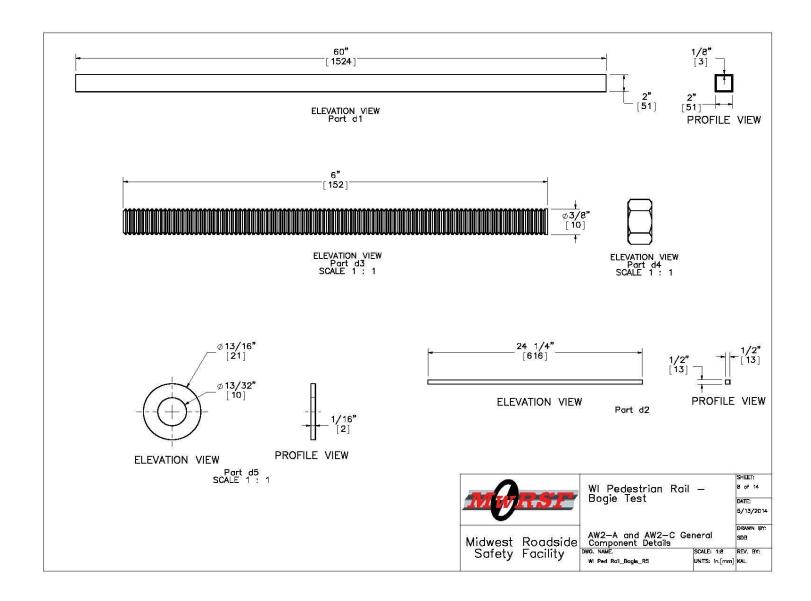


Figure 109. General Components, Pedestrian Rail Designs AW2-A and AW2-C

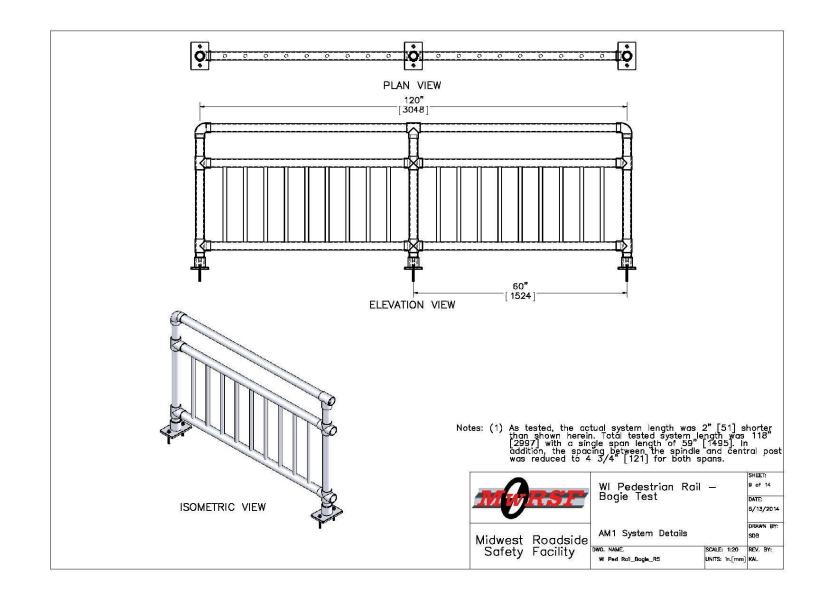


Figure 110. Pedestrian Rail Design AM-1

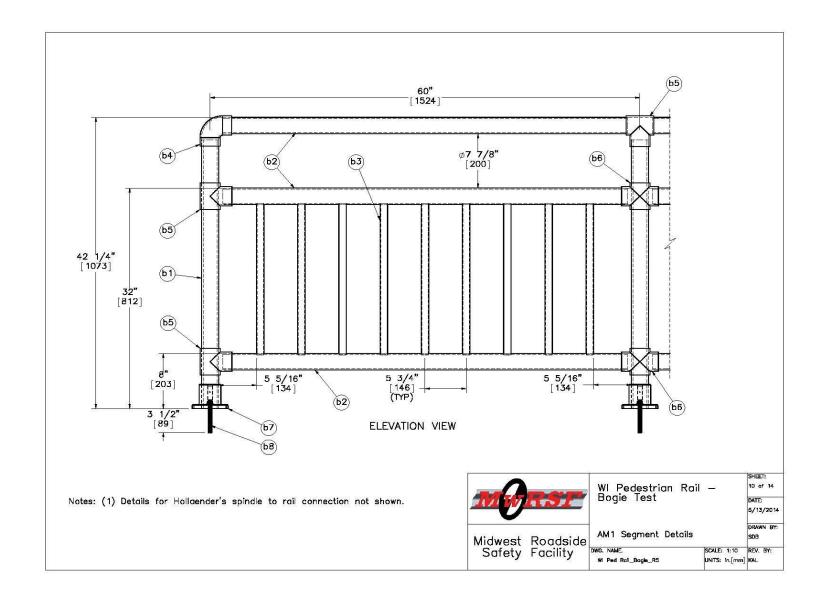


Figure 111. Pedestrian Rail Design AM-1

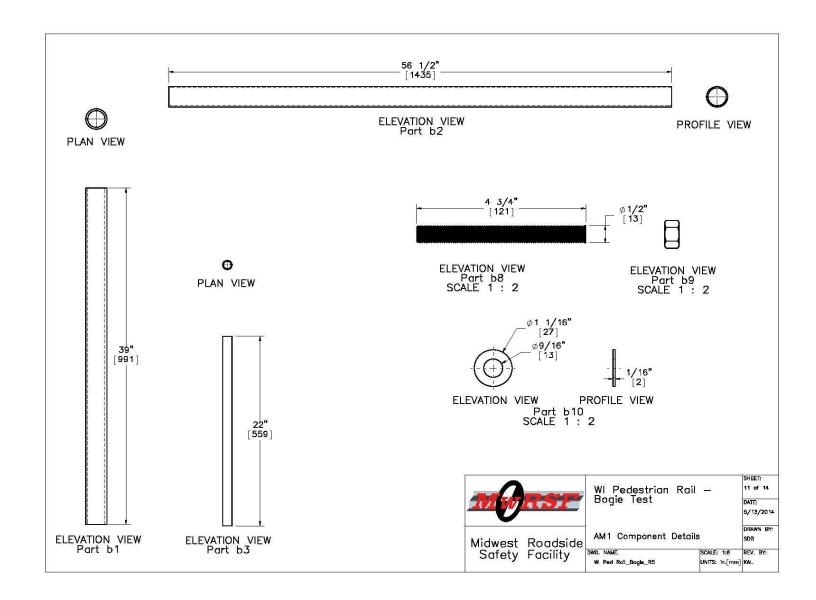
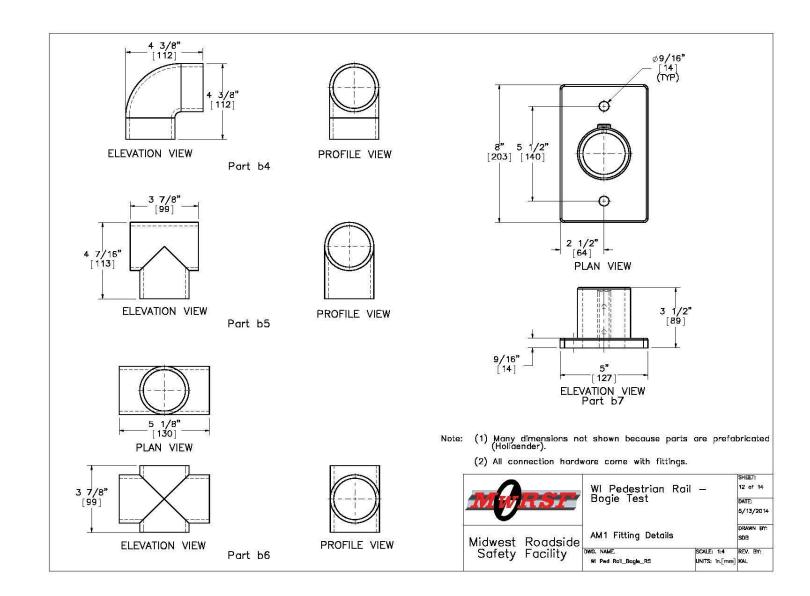


Figure 112. Pedestrian Rail Design AM-1



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Figure 113. Pedestrian Rail Design AM-1

Item No.	QTY.	Description	Material Spec
a1	8	2"x4"x1/4" [51x102x6] Aluminum Post, 43" [1092] long	6061-T6
۵2	8	Aluminum Post Cap — 1/8" [3] Plate	6061 <b>-</b> ⊺6
α3	8	Aluminum Post Base	6061-T6
d1	12	2"x2"x1/8" [51x51x3] Aluminum Roil - 60" [1524] long	6061-T6
d2	36	1/2"x1/2" [13x13] Square Aluminum Spindle - 24 1/4" [616] long	6061-T6
d3	16	3/8" [10] Dia. Threaded Rod	ASTM A193 Grade B7 Galv.
d4	16	3/8" [10] Dia. Nut	ASTM A194 Grade 8M Galv.
d5	16	3/8" [10] Dia. SAE Flat Washer	ASTM F436 Type 1 Galv.
d6	3 <del></del> -1	Ероху	Powers Fasteners AC100+ Gold Minimum bond strength = 1,450 psi [10.0 MPd

Item No.	QTY.	Description	Material Spec
c1	8	2"x3"x1/8" [51x76x3] Aluminum Post, 43" [1092] long	6061–T6
c2	8	Aluminum Post Cap — 1/8" [3] Plate	6061-T6
c3	8	Aluminum Post Base	6061–T6
c4	8	1/4" [6] Dia., 3" [76] Long Bolt and Nut	Bolt ASTM A193 Grade B8M Class 2, Nut ASTM A194 Grade 8M
d1	12	2"x2"x1/8" [51x51x3] Aluminum Rail - 60" [1524] long	6061–T6
d2	36	1/2"x1/2" [13x13] Square Aluminum Spindle - 24 1/4" [616] long	6061-T6
d3	16	3/8" [10] Dia. Threaded Rod	ASTM A193 Grade B7 Galv.
d4	16	3/8" [10] Dia. Nut	ASTM A194 Grade 8M Galv.
d5	16	3/8" [10] Dia. SAE Flat Washer	ASTM F436 Type 1 Galv.
d6	3 <b>77</b> 6	Ероху	Powers Fasteners AC100+ Gold Minimum bond strength = 1,450 psi [10.0 MPa]

		WI Pedestrian Rail Bogie Test		SHEET: 13 of 14
		bogie lest		DATE: 5/13/2014
Midwest	Roadside	AW2—A and AW2—C Bill Materials	of	drawn by: SDB
	Facility	And the second	SCALE: None UNITS: in.[mm]	REV. BY: KAL

Figure 114. Bill of Materials, Pedes	trian Rail Designs AW2-A and AW2-

ltern No.	QTY.	Description	Material Specification	Hollaender Part No.
b1	6	2" [51] Dia. Schedule 80 post, 39" [991] long	6061—T6 Aluminum	99231
b2	12	2" [51] Dia. Schedule 40 rail, 56 1/2" [1435] long	6061-T6 Aluminum	98221
b3	36	3/4" [19] Dia. Schedule 10 picket, 22" [559] long	6063–T6 Aluminum	5 <b></b> 3
b4	4	No. 3 Elbow (2" [51])	6061–T6 Aluminum	09020
b5	10	No. 5 Tee (2" [51])	6061-T6 Aluminum	09040
b6	4	No. 7 Cross (2" [51])	6061-T6 Aluminum	09090
b7	6	No. 48 Heavy-Duty Base Flange (2" [51], 2-hole)	6061-T6 Aluminum	28200
b8	12	1/2" [13] Dia. Threaded Rod	ASTM A193 Grade B7 Galv.	
b9	12	1/2" [13] Dia. Nut	ASTM A194 Grade 8M Galv.	1
ь10	12	1/2" [13] Dia. SAE Flat Washer	ASTM F436 Type 1 Galv.	=
Ь11	<u>0.00</u>	Ероху	Powers Fasteners AC100+ Gold Minimum bond strength = 1,450 psi [10.0 MPa]	8 <u>—</u> 3

Notes: (1) All aluminum fittings are prefabricated components from Hollaender Speed-Rail (www.hollaender.com/?page=speedrail).

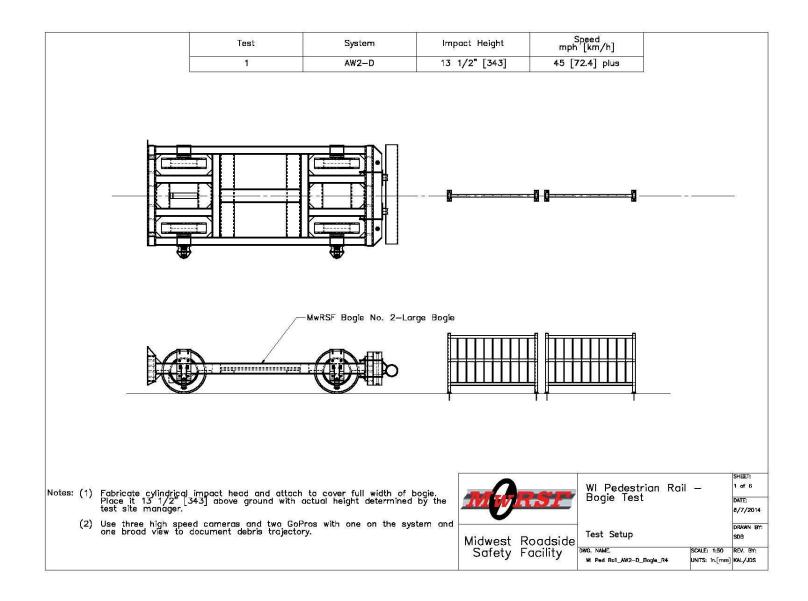
(2) All aluminum pipe properties and dimensions from Hollaender.

(3) Order as Hollaender picket rail system.

(4) Pickets (Part b3) may be substituted for comparable aluminum tube.

	RSF	WI Pedestrian Rail Bogie Test		SHEET: 14 of 14 DATE: 5/13/2014
Midwest	Roadside	AM1 Bill of Materials		drawn by: SDB
	Facility	DWG. NAME. Wi Ped Roll_Bogle_R5	SCALE: NONE UNITS: in.[mm]	REV. BY: KAL

Figure 115. Bill of Materials, Continued, Pedestrian Rail Design AM-1



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Figure 116. Pedestrian Rail Test Setup, Design AW2-D

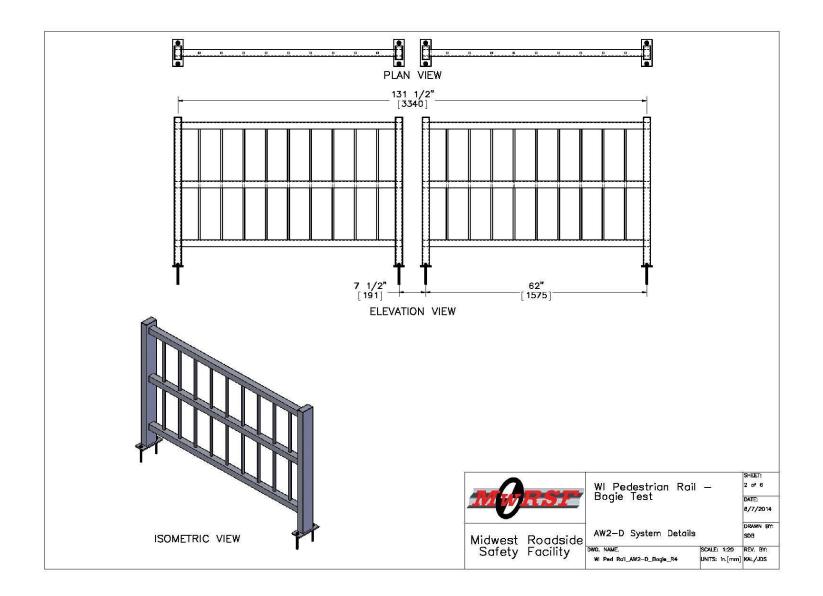


Figure 117. Pedestrian Rail Design AW2-D

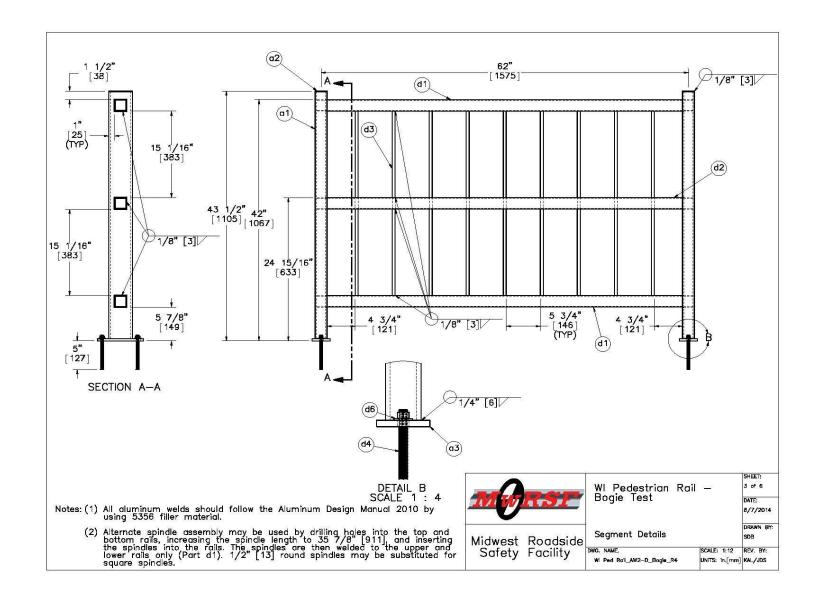
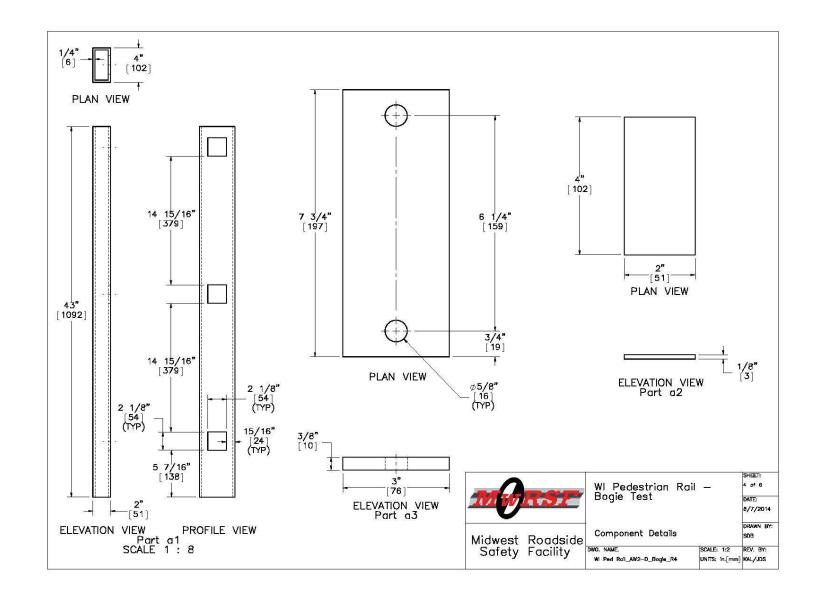
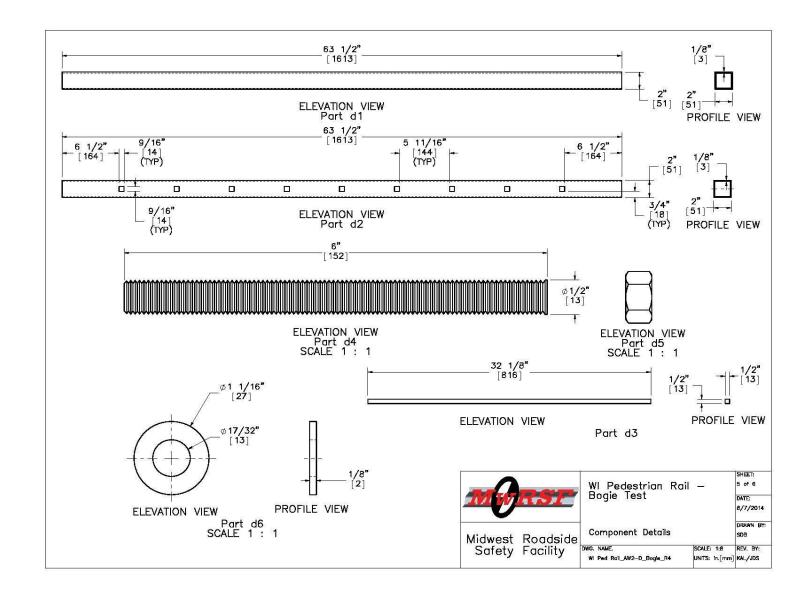


Figure 118. Pedestrian Rail Design AW2-D





Item No.	QTY.	Description	Material Spec
a1	4	2"x4"x1/4" [51x102x6] Aluminum Post, 43" [1092] long	6061-T6
۵2	4	Aluminum Post Cap — 1/8" [3] Plate	6061–⊺6
α3	4	Aluminum Post Base	6061-T6
d1	4	2"x2"x1/8" [51x51x3] Aluminum Rail — 63 1/2" [1613] long	6061-T6
d2	2	2"x2"x1/8" [51x51x3] Aluminum Rail - 63 1/2" [1613] long with holes	6061-T6
d3	18	1/2"x1/2" [13x13] Square Aluminum Spindle - 32 1/8" [816] long	6061 <b>-</b> T6
d4	8	1/2" [13] Dia. Steel Threaded Rod	ASTM A193 Grade B7 Galv.
d5	8	1/2" [13] Dia. Steel Nut	ASTM A194 Grade 8M Galv.
d6	8	1/2" [13] Dia. Steel SAE Flat Washer	ASTM F436 Type 1 Galv.
d7	-	Ероху	Powers Fasteners AC100+ Gold Minimum bond strength = 1,450 psi [10.0 MPa]

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	RSF	WI Pedestrian Ro Bogie Test	il —	SHEET: 6 of 6 DATE: 8/7/2014
Midwest	Roadside	Bill of Materials		drawn by SDB
Safety		DWG. NAME. WI Ped Roll_AW2-D_Bogle_R4	SCALE: None UNITS: In.[mm]	REV. BY: KAL/JDS

Figure 121. Bill of Materials, Pedestrian Rail Design AW2-D



Figure 122. Pedestrian Rail AW2-A



Figure 123. Pedestrian Rail AW2-C



Figure 124. Pedestrian Rail AM-1

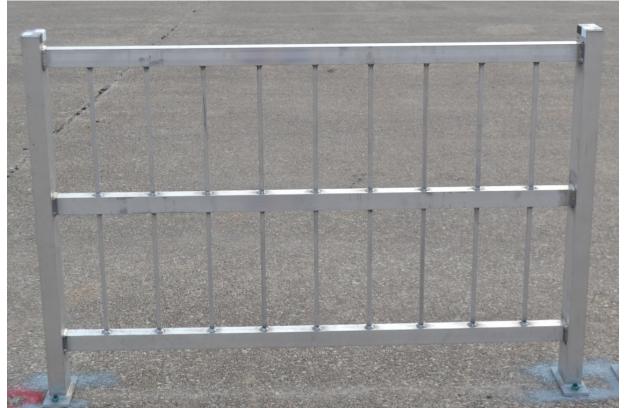


Figure 125. Pedestrian Rail AW2-D

### **8 COMPONENT TESTING CONDITIONS**

## 8.1 Purpose

According to TL-2 of MASH, longitudinal channelizing systems must be subjected to two full-scale vehicle crash tests which include impacts with both a passenger car and a pickup truck at a nominal speed of 44 mph (70 km/h) and a critical angle between 0 and 25 degrees. In order to evaluate the four pedestrian rail concepts, bogie test were undertaken and impacting similar to the MASH TL-2 test conditions in lieu of full-scale crash testing. The bogie was configured with a bumper similar in height and shape to the 1100C small car to evaluate how the pedestrian rails fracture upon impact with a low impact height. Ideal impact performance characteristics for the pedestrian rails included: clean and consistent component fracture, no anchor damage, component trajectory away from the windshield and undercarriage, and the potential for no vehicle instability and low occupant risk. Although the bogie was not configured with a windshield, floorpan, or body panels, the trajectory of components was evaluated to determine if the potential for occupant compartment or windshield deformation or penetration existed.

#### 8.2 Scope

Four test runs, consisting of seven bogie tests were conducted on four pedestrian rail concepts, as described in Section 7.4. Each concept was mounted to the existing concrete tarmac and was configured as a two-panel system. The target impact conditions included a speed of 45 mph (72 km/h) and two different impact orientations: 25 degrees and within the spindle region of the first panel, or 0 degrees for an end-on impact. The fourth concept (AW2-D) was only evaluated in the end-on orientation, due to its similarity to the first concept (AW2-A). The systems were impacted 13<sup>5</sup>/<sub>8</sub> in. (346 mm) above the groundline. The seven crash tests are

summarized in Table 10. The test setups are shown in Figure 102 through Figure 121. Photographs from a typical test set up are shown in Figure 126. Material specifications, mill certifications, and certificates of conformity for the pedestrian rail concepts are shown in Appendix D.









Figure 126. Typical Bogie Testing Setup

Run No.	Test No.	Design Concept	Target Impact Velocity mph (km/h)	Impact Angle (deg)
WIPR-1	WIPR-1-1	AW2-A	45 (72)	25
WIFK-1	WIPR-1-2	AW2-A	45 (72)	0 (end-on)
WIPR-2	WIPR-2-1	AW2-C	45 (72)	25
WIFK-2	WIPR-2-2	AW2-C	45 (72)	0 (end-on)
WIPR-3	WIPR-3-1	AM-1	45 (72)	25
WIFK-3	WIPR-3-2	AM-1	45 (72)	0 (end-on)
WIPR-4	WIPR-4	AW2-D	45 (72)	0 (end-on)

Table 10. Bogie Testing Matrix

#### 8.3 Test Facility

Physical testing of the pedestrian rail concepts was conducted at the MwRSF Proving Grounds, which is located at the Lincoln Air Park on the northwest side of the Lincoln Municipal Airport in Lincoln, Nebraska. The facility is approximately 5 miles (8 km) northwest from the University of Nebraska-Lincoln's city campus.

## **8.4 Equipment and Instrumentation**

Several pieces of equipment and instrumentation were utilized to collect and record data during the dynamic bogie tests, including a bogie vehicle, accelerometers, a retroreflective speed trap, high-speed and standard-speed digital video, and still cameras.

#### 8.4.1 Bogie Vehicle

A rigid-frame bogie vehicle was used to impact the rail prototypes. A variable-height, detachable impact head was used in the testing program. The bogie head was constructed of 6-in. (152-mm) diameter, <sup>1</sup>/<sub>4</sub>-in. (13-mm) thick standard steel pipe. The impact head was bolted to the bogie vehicle, creating a rigid frame with an impact height of 13<sup>5</sup>/<sub>8</sub> in. (346 mm). The bogie with the impact head is shown in Figure 127. The bogie weight, including the mountable impact head and accelerometers, was 5,166 lb (2,343 kg).



Figure 127. Rigid-Frame Bogie Vehicle on Guidance Track

A pickup truck with a reverse cable tow system was used to propel the bogie to a target impact speed of 45 mph (72 km/h). When the bogie approached the end of the guidance system, it was released from the tow cable, allowing it to be free-rolling when it impacted the system. A remote-control braking system was installed on the bogie, allowing it to be brought safely to rest after the test.

# **8.4.2** Accelerometers

No accelerometer readings were recorded for run nos. WIPR-1 or WIPR-2. For run nos. WIPR-3 and WIPR-4, an accelerometer system was mounted on the bogie vehicle near its center of gravity to measure the acceleration in the longitudinal, lateral, and vertical directions. However, only the longitudinal acceleration data was processed and reported.

The SLICE-2 unit was a modular data acquisition system manufactured by Diversified Technical Systems, Inc. (DTS) of Seal Beach, California. The acceleration sensors were mounted inside the body of a custom built SLICE 6DX event data recorder and recorded data at 10,000 Hz to the onboard microprocessor. The SLICE 6DX was configured with 7 GB of non-volatile flash

memory, a range of  $\pm 500$  g's, a sample rate of 10,000 Hz, and a 1,650 Hz (CFC 1000) antialiasing filter. The "SLICEWare" computer software program and a customized Microsoft Excel worksheet were used to analyze and plot the accelerometer data.

## 8.4.3 Retroreflective Optic Speed Trap

The retroreflective optic speed trap was used to determine the speed of the bogie vehicle before impact. Five retroreflective targets, spaced at approximately 18-in. (457-mm) intervals, were applied to the side of the vehicle. When the emitted beam of light was reflected by the targets and returned to the Emitter/Receiver, a signal was sent to the data acquisition computer, recording at 10,000 Hz, as well as the external LED box activating the LED flashes. The speed was then calculated using the spacing between the retroreflective targets and the time between the signals. LED lights and high-speed digital video analysis are only used as a backup in the event that vehicle speeds cannot be determined from the electronic data.

### 8.4.4 Digital Photography

No photographic documentation was collected for run no. WIPR-1. For run nos. WIPR-2 and WIPR-3, three AOS high-speed digital video cameras, four GoPro digital video cameras, and one JVC digital camera were used to document each test. For run no. WIPR-4, two AOS high-speed digital video cameras, three GoPro digital video cameras, and one JVC digital camera were used to document the test. The AOS high-speed cameras had a frame rate of 500 frames per second, the GoPro video cameras had a frame rate of 120 frames per second, and the JVC digital video cameras had a frame rate of 29.97 frames per second. The cameras were placed laterally away from the prototype pedestrian rails, with a view perpendicular to the bogie's direction of travel. A Nikon D50 digital still camera was used to document pre- and post-test conditions for all tests.

# **8.5 Data Processing**

The electronic accelerometer data obtained in dynamic testing was filtered using the SAE Class 60 Butterworth filter conforming to the SAE J211/1 specifications [43]. The pertinent acceleration signal was extracted from the bulk of the data signals. The processed acceleration data was then multiplied by the mass of the bogie to get the impact force using Newton's Second Law. Next, the acceleration trace was integrated to find the change in velocity versus time. The initial velocity of the bogie, as calculated from the retroreflective optical speed trap data, was then used to determine the bogie velocity as a function of time, using the change in velocity data. The calculated velocity trace was integrated to find the bogie's displacement. This displacement was also used as the system displacement. Combining the previous results, a force versus deflection curve was plotted for each test. Finally, integration of the force versus deflection curve provided the energy versus deflection curve for each test.

## 9 DYNAMIC BOGIE TESTING RESULTS AND DISCUSSION

### 9.1 Run No. WIPR-1 (Test Nos. WIPR-1-1 and WIPR-1-2)

Run no. WIPR-1 was conducted during a practice run when the brakes malfunctioned. This run consisted of test nos. WIPR-1-1 and WIPR-1-2 occurring successively. For test no. WIPR-1-1, the bogie impacted the pedestrian rail concept AW2-A oriented at an angle to the vehicle, at an unknown speed and an angle of 25 degrees. For test no. WIPR-1-2, the bogie then impacted the pedestrian rail concept AW2-A oriented end-on to the vehicle, at an unknown speed and an angle of 0 degrees. These two tests were conducted without collecting videos, speed trap data, or accelerometer data. The systems prior to impact are shown in Figures 128 and 129.

Damage to the systems impacted during test nos. WIPR-1-1 and WIPR-1-2 is shown in Figures 130 through 133. The systems impacted during test nos. WIPR-1-1 and WIPR-1-2 encountered damage to both the first and second panels.

The damage to the first panel in test no. WIPR-1-1 consisted of:

- fractured welds between (1) downstream post and its baseplate (2) bottom and middle horizontal rails and the downstream post, and (3) top rail and both posts;
- downstream post baseplate bent due to prying;
- upstream post twisted and bent downstream;
- spindles detached from the horizontal rails; and
- all threaded anchors deformed slightly.

The damage to the second panel in test no. WIPR-1-1 consisted of:

- fractured welds between both posts and their baseplates,
- partially fractured welds between (1) top and middle horizontal rails and both posts and (2) bottom horizontal rail and downstream post,
- both post baseplates bent due to prying, and

• all threaded anchors deformed slightly.

The damage to the first panel in test no. WIPR-1-2 consisted of:

- fractured welds between (1) both posts and their baseplates and (2) all three horizontal rails and both posts,
- both post baseplates bent due to prying,
- upstream post bent above bottom horizontal rail (or at bumper height),
- some spindles detached from middle horizontal rail, and
- all threaded anchors deformed slightly.

The damage to the second panel in test no. WIPR-1-2 consisted of:

- fractured welds between (1) both posts and their baseplates, (2) the middle and bottom horizontal rails and both posts, and (3) the top rail and the downstream post;
- both post baseplates bent due to prying;
- some spindles detached from middle and bottom horizontal rails; and
- all threaded anchors deformed slightly.



Figure 128. System Panels and Anchor, Run No. WIPR-1



Figure 129. System Installation, Run No. WIPR-1

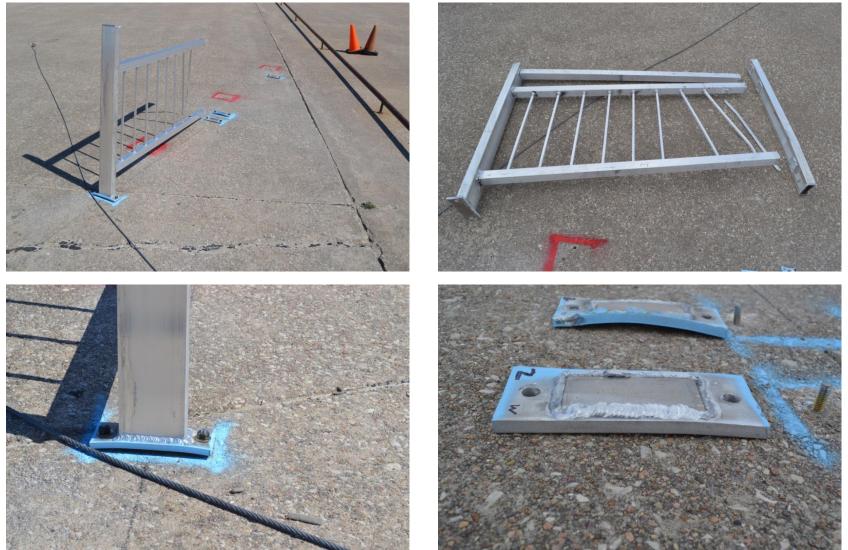


Figure 130. First Panel Damage, Run No. WIPR-1, Test No. WIPR-1-1

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Figure 131. Second Panel Damage, Run No. WIPR-1, Test No. WIPR-1-1



Figure 132. First Panel Damage, Run No. WIPR-1, Test No. WIPR-1-2



Figure 133. Second Panel Damage, Run No. WIPR-1, Test No. WIPR-1-2

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### 9.2 Run No. WIPR-2 (Test Nos. WIPR-2-1 and WIPR-2-2)

Run no. WIPR-2 consisted of two tests conducted successively, with test no. WIPR-2-1 followed by test no. WIPR-2-2. During test no. WIPR-2-1, the bogie, traveling at a speed of 50.7 mph (81.6 km/h), impacted the pedestrian rail concept AW2-C oriented at a 25-degree angle to the vehicle. For test no. WIPR-2-2, the bogie then impacted the pedestrian rail concept AW2-C at a speed of 48.5 mph (78.1 km/h), with the rail oriented end-on to the vehicle (i.e., 0 degrees). The systems prior to impact are shown in Figures 134 and 135.

Time-sequential photographs are shown in Figures 136 and 137. Damage to system nos. WIPR-2-1 and WIPR-2-2 is shown in Figures 138 through 141. The systems impacted during test nos. WIPR-2-1 and WIPR-2-2 encountered damage to both the first and second panels.

The damage to the first panel in test no. WIPR-2-1 consisted of:

- fractured welds between downstream post socket and its baseplate,
- downstream post sheared below middle horizontal rail,
- upstream post bent downstream at its base,
- all three horizontal rails bent at upstream post,
- some spindles were deformed and some detached from horizontal rails, and
- all threaded anchors deformed slightly.

The damage to the second panel in test no. WIPR-2-1 consisted of:

- fractured welds between both post sockets and their baseplates,
- upstream post bent above bottom horizontal rail,
- downstream post bent at middle and bottom horizontal rails,
- bottom horizontal rail bent, and
- some spindles deformed.

The damage to the first panel in test no. WIPR-2-2 consisted of:

- fractured welds between (1) both post sockets and their baseplates and (2) bottom horizontal rail and both posts,
- upstream post bent and fractured below middle horizontal rail,
- downstream post fractured between middle and bottom horizontal rails, and
- some spindles detached from horizontal rails and encountered deformations.

The damage to the second panel in test no. WIPR-2-2 consisted of:

- fractured welds between (1) downstream post socket and its baseplate, (2) top horizontal rail and upstream post, (3) middle horizontal rail and downstream post, and (4) the bottom horizontal rail and both posts;
- upstream post socket fractured and tore;
- upstream post bent above bottom horizontal rail (at bumper height) and tore below bottom horizontal rail; and
- downstream post bent below bottom horizontal rail.



Figure 134. System Panels and Anchors, Run No. WIPR-2



Figure 135. System Installation, Run No. WIPR-2







0.010 sec



0.020 sec



0.030 sec







0.050 sec



0.060 sec



0.070 sec



0.080 sec



0.090 sec



0.100 sec



0.110 sec

Figure 136. Time-Sequential Photographs, Run No. WIPR-2, Test No. WIPR-2-1







0.010 sec



0.020 sec



0.030 sec







0.050 sec



0.060 sec



0.070 sec



0.080 sec



0.090 sec



0.100 sec



0.110 sec

Figure 137. Time-Sequential Photographs, Run No. WIPR-2, Test No. WIPR-2-2



Figure 138. First Panel Damage, Run No. WIPR-2, Test No. WIPR-2-1



Figure 139. Second Panel Damage, Run No. WIPR-2, Test No. WIPR-2-1



Figure 140. First Panel Damage, Run No. WIPR-2, Test No. WIPR-2-2



Figure 141. Second Panel Damage, Run No. WIPR-2, Test No. WIPR-2-2

#### 9.3 Run No. WIPR-3 (Test Nos. WIPR-3-1 and WIPR-3-2)

Run no. WIPR-3 consisted of two tests conducted successively, with test no. WIPR-3-1 followed by test no. WIPR-3-2. During test no. WIPR-3-1, the bogie, traveling at a speed of 46.6 mph (75.0 km/h), impacted the pedestrian rail concept AM-1 oriented at a 25-degree angle to the vehicle. For test no. WIPR-3-2, the bogie then impacted the pedestrian rail concept AM-1 at a speed of 43.3 mph (68.1 km/h), with the rail oriented end-on to the vehicle (i.e., 0 degrees). The systems prior to impact are shown in Figures 142 and 143.

Time-sequential photographs are shown in Figures 144 and 145. Damage to the pedestrian rail concepts impacted during test nos. WIPR-3-1 and WIPR-3-2 is shown in Figures 146 through 149. The posts, rails, and spindles from test nos. WIPR-3-1 and WIPR-3-2 disengaged, thus generating a fair amount of debris and concerns of flying projectiles. Damage to the systems consisted of fractured post socket couplers and post-to-rail connection joints, as well as deformed posts.

Force versus displacement and energy versus displacement curves created from the DTS-SLICE accelerometer data are shown in Figures 150 and 151 for test nos. WIPR-3-1 and WIPR-3-2, respectively. A total of 387.8 kip-in. (42.8 kJ) of energy was absorbed by the system in test no. WIPR-3-1 through 80.7 in. (2,050 mm) of displacement, while a total of 452.5 kip-in. (51.1 kJ) of energy was absorbed by the system in test no. WIPR-3-2 through 78.2 in. (1,986 mm).



Figure 142. System Panels and Anchor, Run No. WIPR-3



Figure 143. System Installation, Run No. WIPR-3



0.110 sec

Figure 144. Time-Sequential Photographs, Run No. WIPR-3, Test No. WIPR-3-1



Figure 145. Time-Sequential Photographs, Run No. WIPR-3, Test No. WIPR-3-2



Figure 146. System Damage, Run No. WIPR-3, Test No. WIPR-3-1



Figure 147. System Damage, Run No. WIPR-3, Test No. WIPR-3-1



Figure 148. System Damage, Run No. WIPR-3, Test No. WIPR-3-2



Figure 149. System Damage, Run No. WIPR-3, Test No. WIPR-3-2

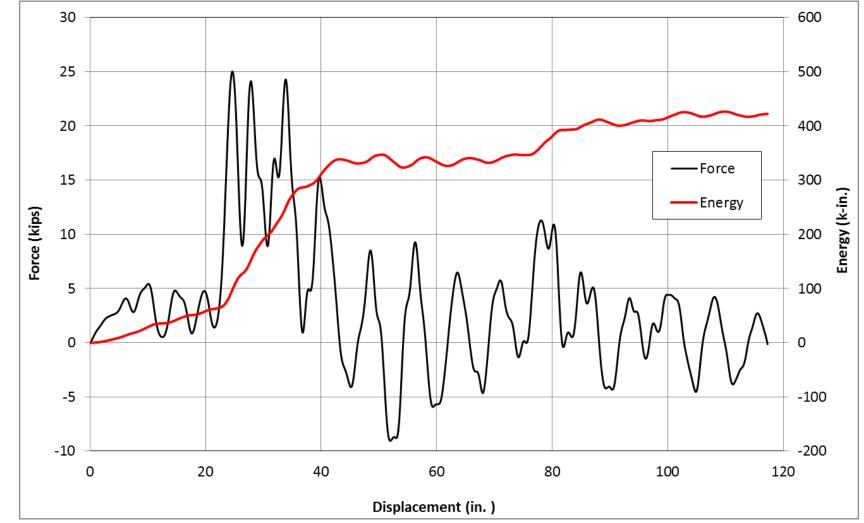


Figure 150. Force vs. Displacement and Energy vs. Displacement, Run No. WIPR-3, Test No. WIPR-3-1

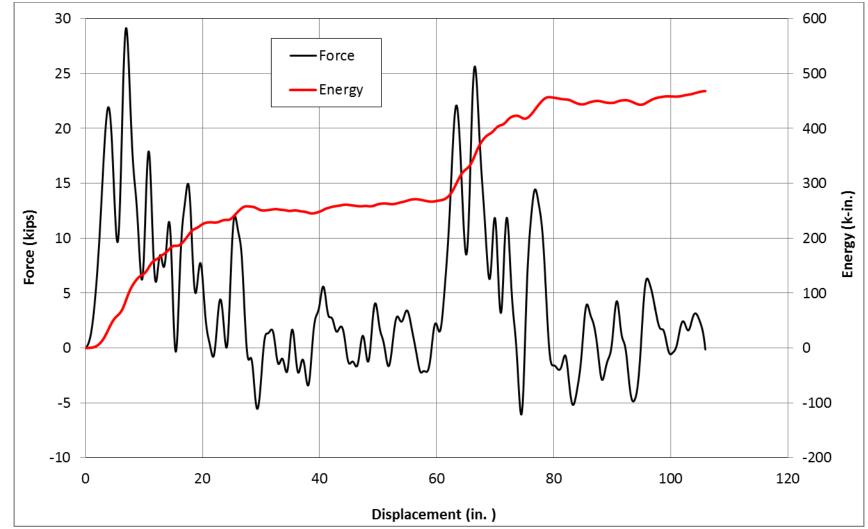


Figure 151. Force vs. Displacement and Energy vs. Displacement, Run No. WIPR-3, Test No. WIPR-3-2

## 9.4 Run No. WIPR-4 (Test No. WIPR-4)

Run no. 4 consisted of a single test, test no. WIPR-4, during which the bogie impacted the pedestrian rail concept AW2-D at a speed of 45.8 mph (73.8 km/h), with the rail oriented end-on to the vehicle (i.e., 0 degrees). The system prior to impact are shown in Figures 152 and 153.

Time-sequential photographs are shown in Figure 154. Damage incurred during test no. WIPR-4 is shown in Figures 155 and 156. The rail encountered damage to both the first and second panels. The damage to the first panel consisted of:

- fractured welds between both posts and their baseplates,
- all baseplates deformed,
- upstream post bent above bottom horizontal rail,
- downstream post bent at the middle horizontal rail,
- some spindles deformed, and
- upstream end of bottom horizontal rail crushed and bent.

The damage to the second panel consisted of:

- both posts bent at middle horizontal rail,
- some spindles deformed, and
- downstream end of bottom horizontal rail bent.

Force versus displacement and energy versus displacement curves created from the DTS-SLICE accelerometer data are shown in Figure 157. A total of 310.4 kip-in. (35.1 kJ) of energy was absorbed by the system in test no. WIPR-4 through 78.1 in. (1,984 mm) of displacement. However, a total of 306.8 kip-in. (34.7 kJ) of energy was absorbed by the system through 18.0 in. (457 mm) of displacement when all posts had fractured from their baseplates and the system was moving out in front of the bogie vehicle.



Figure 152. System Panels and Anchor, Run No. WIPR-4

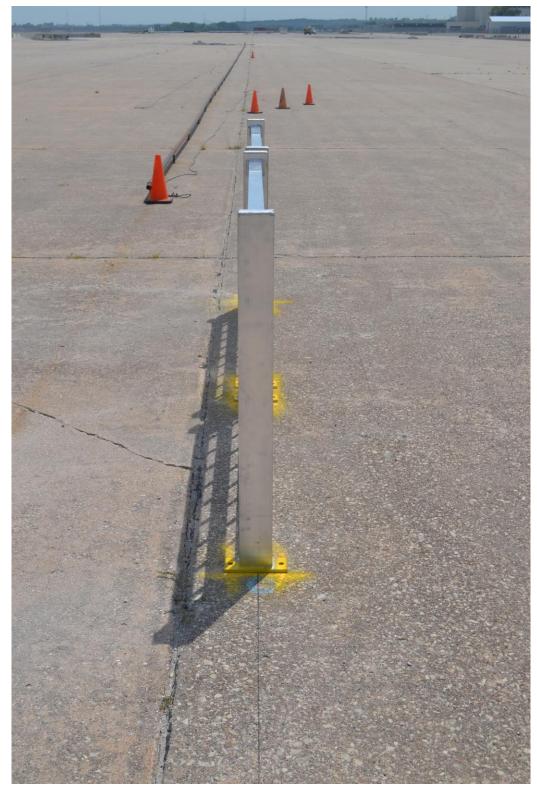


Figure 153. System Installation, Run No. WIPR-4





0.110 sec

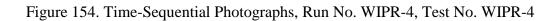




Figure 155. First Panel Damage, Run No. WIPR-4, Test No. WIPR-4



Figure 156. Second Panel Damage, Run No. WIPR-4, Test No. WIPR-4

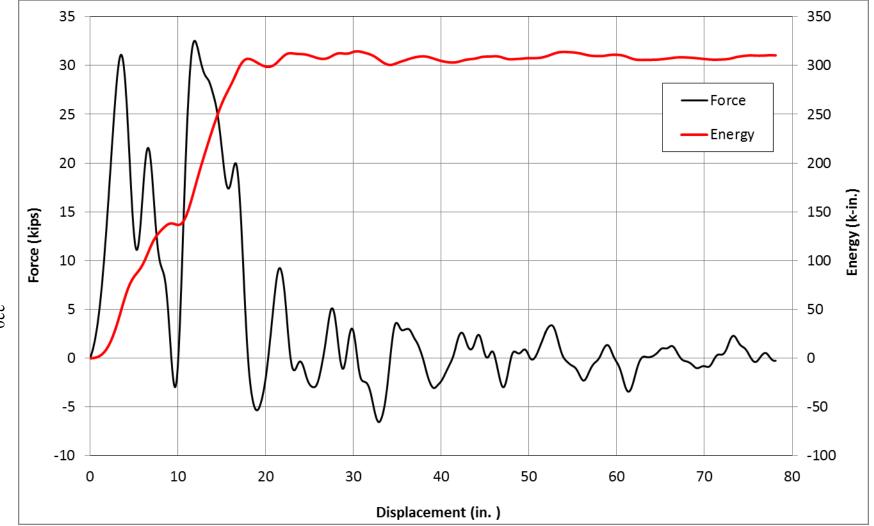


Figure 157. Force vs. Displacement and Energy vs. Displacement, Run No. WIPR-4, Test No. WIPR-4

## 9.5 Discussion

Four runs of seven bogie tests were conducted on four pedestrian rail concepts. Each concept was configured as a two-panel system. They were impacted at approximately 45 mph (72.4 km/h) and evaluated in two different impact orientations, except for the fourth concept (run no. WIPR-4). First, each concept was impacted at a 25-degree angle and within the spindle region of the first panel. Second, each concept was impacted using an end-on orientation. For run no. WIPR-4, only the end-on orientation was evaluated.

The performance of the post with welded baseplate (run no. WIPR-1 and WIPR-4) appeared to provide a cleaner fracture compared to the concept involving a post inserted into a socket that was welded to a baseplate (run no. WIPR-2). Minor deformation was found on all the baseplates. Permanent deformations of the baseplates could be eliminated by increasing the baseplate thickness. The <sup>3</sup>/<sub>8</sub>-in. (9.5-mm) diameter anchors exhibited slight permanent deformations during the tests of the two welded concepts (run nos. WIPR-1 and WIPR-2). Increasing anchor diameter to <sup>1</sup>/<sub>2</sub>-in. (12.7-mm) eliminated this permanent deformation, as shown in run nos. WIPR-3 and WIPR-4.

The upper and middle horizontal rails fractured or disengaged from the posts, rode over the top of the bogie, and posed the potential for windshield penetration and deformation during the first three system configurations when impacted end-on. In addition, based on the results of the first six bogie tests, the critical orientation was believed to occur under end-on impacts. All of the systems fractured cleanly when impacted at a 25-degree angle broke away and did not exhibit much potential for vehicle intrusion. Therefore, the last bogie test was only conducted using an end-on orientation.

The middle rail was lowered to more closely align with the bumper heights of the pickup truck and small car in order to improve dynamic impact behavior, as shown in test no. WIPR-4.

The change helped the system behave more rigidly instead of as individual posts and included: (1) lowering the middle horizontal rail, (2) extending the spindles from the top to bottom rail and passing the spindles through the middle rail, (3) increasing the anchor size to ½ in. (12.7 mm), and (4) inserting the rails into the posts. Therefore, design concept AW2-D (test no. WIPR-4) was recommended to be evaluated through full-scale vehicle crash testing according to the AASHTO MASH TL-2 safety performance criteria for longitudinal channelizers.

# **10 FULL-SCALE TEST REQUIREMENTS AND EVALUATION CRITERIA**

## **10.1 Test Requirements**

Longitudinal channelizers, such as pedestrian rail, must satisfy impact safety standards in order to be declared eligible for federal reimbursement by the Federal Highway Administration (FHWA) for use on the National Highway System (NHS). For new hardware, these safety standards consist of the guidelines and procedures published in MASH [7]. According to TL-2 of MASH, longitudinal channelizing systems must be subjected to two full-scale vehicle crash tests. The two required full-scale crash tests are noted below:

- 1. Test Designation No. 2-90 consists of a 2,425-lb (1,100-kg) passenger car impacting the system at a nominal speed of 44 mph (70 km/h) and a critical angle between 0 and 25 degrees.
- 2. Test Designation No. 2-91 consists of a 5,000-lb (2,268-kg) pickup truck impacting the system at a nominal speed of 44 mph (70 km/h) and a critical angle between 0 and 25 degrees.

The test conditions of TL-2 longitudinal barriers are summarized in Table 11. According to MASH, the critical impact angle for channelizers should be selected to maximize the risk of vehicle rollover and/or excessive vehicle decelerations. During discussions with FHWA personnel, the 0-degree impact angle would likely provide the greatest risk of excessive vehicle decelerations, and could also cause vehicle instability and windshield and occupant compartment deformation and/or penetration. The 25-degree impact angle could cause vehicle instability and windshield and occupant compartment deformation and/or penetration. The 25-degree impact angle could cause vehicle instability and windshield and occupant compartment deformation and/or penetration. Therefore, impact angles of 0- and 25-degrees were both deemed critical as vehicle instability and occupant risk could occur with either impact angle. Other impact angles between those values would likely be less critical.

Test Article	Test	Test Vehicle	Imp	act Condit			
	Designation No.		Sp	eed	Angle	Evaluation Criteria <sup>1</sup>	
			mph	km/h	(deg)		
Longitudinal Channelizer	2-90	1100C	44	70	0-25	C,D,F,H,I,N	
	2-91	2270P	44	70	0-25	C,D,F,H,I,N	

Table 11. MASH TL-2 Crash Test Conditions

<sup>1</sup> Evaluation criteria explained in Table 12.

Test no. 2-90 was deemed most critical as the small car has a greater potential for excessive vehicle decelerations, due to its smaller mass, and vehicle instability, due to overriding components. The small car also has lower hood, windshield, and floorpan heights, which would make it more susceptible to occupant compartment and windshield penetration and deformation with the channelizer. Therefore, after discussion with FHWA personnel, two tests with the small car were deemed critical initially: test no. 2-90 with an 1100C small car impacting at 0 degrees and test no. 2-90 with an 1100C small car impacting at 25 degrees. If the results of either test no. 2-90 test indicated that channelizer had the potential to cause excessive vehicle deceleration, vehicle instability, or occupant compartment or windshield penetration or deformation with the 2270P pickup truck, then additional test no. 2-91 tests would be conducted.

MASH is unclear in regards to the use of a centerline impact versus a quarter-point impact when testing channelizers. Therefore, the choice was made to use a centerline impact scenario for all full-scale tests, as the vehicle would likely interact with a greater number of panels with a greater risk of excessive vehicle deceleration. Further, a quarter-point impact typically is used to evaluate the potential for vehicle rollover and this is not as large of a concern for the channelizer system.

## **10.2 Evaluation Criteria**

Evaluation criteria for full-scale vehicle crash testing are based on three appraisal areas: (1) structural adequacy, (2) occupant risk, and (3) vehicle trajectory after collision. Criteria for structural adequacy are intended to evaluate the ability of the longitudinal channelizer to perform acceptably through either redirection, controlled penetration, or controlled vehicle stopping. Occupant risk evaluates the degree of hazard to occupants in the impacting vehicle. Post-impact vehicle trajectory is a measure of the potential of the vehicle to result in a secondary collision with other vehicles and/or fixed objects, thereby increasing the risk of injury to the occupants of the impacting vehicle and/or other vehicles. For longitudinal channelizers, penetration of the vehicle behind the test article is acceptable. These evaluation criteria are summarized in Table 12 and defined in greater detail in MASH. All full-scale vehicle crash tests were conducted and reported in accordance with the procedures provided in MASH.

In addition to the standard occupant risk measures, the Post-Impact Head Deceleration (PHD), the Theoretical Head Impact Velocity (THIV), and the Acceleration Severity Index (ASI) were determined and reported on the test summary sheet. Additional discussion on PHD, THIV and ASI is provided in MASH.

Structural Adequacy	C.	Acceptable test article performance may be by redirection, controlled penetration, or controlled stopping of the vehicle.							
Occupant	D.	Detached elements, fragments or other debris from the test article should not penetrate or show potential for penetrating the occupant compartment, or present an undue hazard to other traffic, pedestrians, or personnel in a work zone. Deformations of, or intrusions into, the occupant compartment should not exceed limits set forth in Section 5.3 and Appendix E of MASH.							
	F.	The vehicle should remain upright during and after collision. The maximum roll and pitch angles are not to exceed 75 degrees.							
	H.	Occupant Impact Velocity (OIV) (see Appendix A, Section A5.3 of MASH for calculation procedure) should satisfy the followin limits:							
Risk		Occupant Impact Velocity Limits							
		Component	Preferred	Maximum					
		Longitudinal and Lateral	30 ft/s (9.1 m/s)	40 ft/s (12.2 m/s)					
	I.	The Occupant Ridedown Acceleration (ORA) (see Append Section A5.3 of MASH for calculation procedure) should satisfollowing limits:							
		Occupant Ridedown Acceleration Limits							
		Component	Preferred	Maximum					
		Longitudinal and Lateral	15.0 g's	20.49 g's					
Vehicle Trajectory	N.	Vehicle trajectory behind the test article is acceptable.							

 Table 12. MASH Evaluation Criteria for Longitudinal Channelizers

## **11 TEST CONDITIONS**

## **11.1 Test Facility**

The testing facility is located at the Lincoln Air Park on the northwest side of the Lincoln Municipal Airport and is approximately 5 miles (8.0 km) northwest of the University of Nebraska-Lincoln city campus.

#### **11.2 Vehicle Tow and Guidance System**

A reverse cable tow system with a 1:2 mechanical advantage was used to propel the test vehicle. The distance traveled and the speed of the tow vehicle were one-half those of the test vehicle. The test vehicle was released from the tow cable before impact with the longitudinal channelizer system. A digital speedometer on the tow vehicle increased the accuracy of the test vehicle impact speed.

A vehicle guidance system developed by Hinch [44] was used to steer the test vehicle. A guide flag, attached to the left-front wheel and the guide cable, was sheared off before impact with the system. The <sup>3</sup>/<sub>8</sub>-in. (9.5-mm) diameter guide cable was tensioned to approximately 3,500 lb (15.6 kN) and supported both laterally and vertically every 100 ft (30.5 m) by hinged stanchions. The hinged stanchions stood upright while holding up the guide cable, but as the vehicle was towed down the line, the guide flag struck and knocked each stanchion to the ground.

### **11.3 Test Vehicles**

For test no. APR-1, a 2006 Kia Rio was used as the test vehicle. The curb, test inertial, and gross static vehicle weights were 2,421 lb (1,098 kg), 2,428 lb (1,101 kg), and 2,599 lb (1,179 kg), respectively. The test vehicle and vehicle dimensions are shown in Figure 158 and Figure 159, respectively.



Figure 158. Test Vehicle, Test No. APR-1

Date:	10/23/2	2014		ſ	Fest Num	ber:		APR-1		Model:	Rio	<u></u>
Make:	Kia	ı		Vehicle I.D.#: knade1235761					576194884			
Tire Size:	P186/65	5/R14			Y	ear:	20	06	(	Odometer: _	138090	)
*(All Measurem	de)		32psi									
							Vehicle Geometry in. (mm)					
							6		a_65	(1651)	b 57 1/2	(1461)
a m —		-  -0-			+	Ve	4 ehicle	-nt	c <u>167</u>	(4242)	d <u>35 1/2</u>	(902)
		i		Z					e <u>98 3/4</u> g 16 1/2	(2508) (419)	f <u>32 3/4</u> h 36 4/9	(832)
<u> </u>						<u>)</u>		<u> </u>	i 15	(381)	j 21	(533)
									k 12	(305)	1 24	(610)
				))				t	m56 1/2	(1435)	n 57 3/4	(1467)
									0_35	(889)	p_2	(51)
						<u>d</u>	1	<b>T T</b>	q_23 1/2	(597)	r151/4	(387)
° j i						~U	k	i 9	s <u>11 3/4</u>	(298)	t 63 1/4	(1607)
	f	h +	е		d		t			0	ront 10 3/4	(273)
		W <sub>front</sub>	С		₩re	ar				_	Rear 11 1/4	(286)
Mass Distribution lb (kg)										e (F) <u>3</u>	(76)	
		(3(3)	DE	876	(375)						$e(R) = \frac{21/2}{4}$	(64)
Gross Static	LF <u>800</u> LR 484	(363)	RF_ RR		(375)						t (F) <u>6</u> t (R) 15 3/4	(152) (400)
	LK 404	(220)	KK_	407	(222)						Гуре Gaso	
Weights lb (kg)	Curb		Test	Inerti	al	Gros	ss Stat	ic			Size 1.0	
W-front	1580	(717)		1532				(738)		Transmitio		
W-rear	841	(381)	1 <del></del>	896	(406)	3		(441)		(	Automatic	Manual
W-total	2421	(1098)	_	2428	(1101)	_	2599	(1179)		Q	FWD RWD	4WD
GVWR Ratings Dummy Data												
Front		1918			Type: Hybrid II							
					Mass: <u>171 lb</u>							
Total     3638     Seat Position: Passenger												
Note any damage prior to test: <u>Minor small dents</u>												

Figure 159. Vehicle Dimensions, Test No. APR-1

For test no. APR-2, a 2006 Kia Rio was used as the test vehicle. The curb, test inertial, and gross static vehicle weights were 2,424 lb (1,100 kg), 2,437 lb (1,105 kg), and 2,599 lb (1,179 kg), respectively. The test vehicle and vehicle dimensions are shown in Figure 160 and Figure 161, respectively.

The longitudinal component of the center of gravity (c.g.) was determined using the measured axle weights. The vertical component of the c.g. for the 1100C vehicle was determined utilizing a procedure published by SAE [45]. The location of the final c.g. for each test vehicle is shown in Figures 159 and 161. Data used to calculate the location of the c.g. and ballast information are shown in Appendix E.

Square, black- and white-checkered targets were placed on the vehicle for reference to be viewed from the high-speed digital video cameras and aid in the video analysis. Round, checkered targets were placed on the center of gravity on the left-side door, the right-side door, and the roof of the vehicle. Target locations are shown for each vehicle in Figures 162 and 163.

The front wheels of the test vehicle were aligned to vehicle standards, except the toe-in value was adjusted to zero so that the vehicles would track properly along the guide cable. A 5B flash bulb was mounted on each vehicle's dash and was fired by a pressure tape switch mounted at the impact corner of the bumper. The flash bulb was fired upon initial impact with the test article to create a visual indicator of the precise time of impact on the high-speed videos. A remote controlled brake system was installed in the test vehicle, so the vehicle could be brought safely to a stop after the test.



Figure 160. Test Vehicle, Test No. APR-2

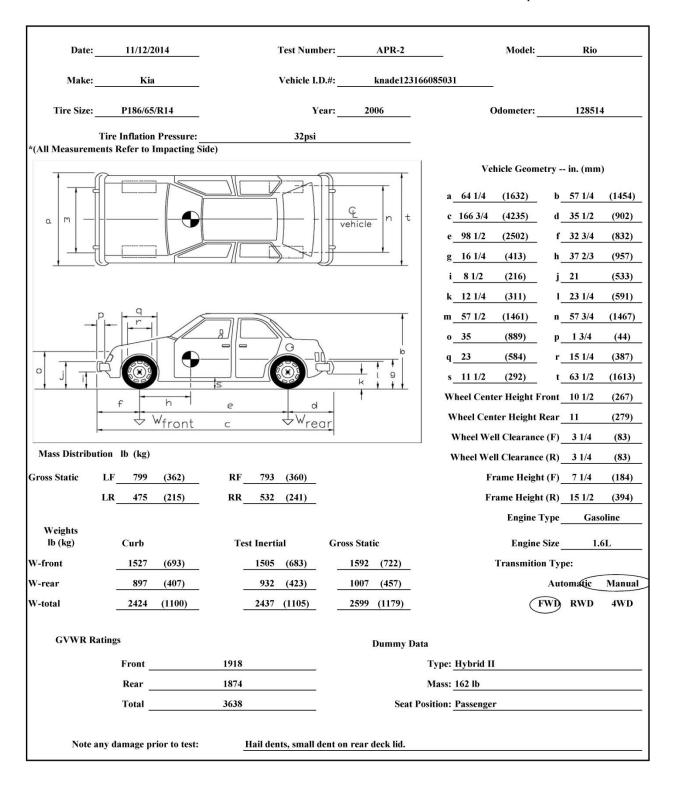


Figure 161. Vehicle Dimensions, Test No. APR-2

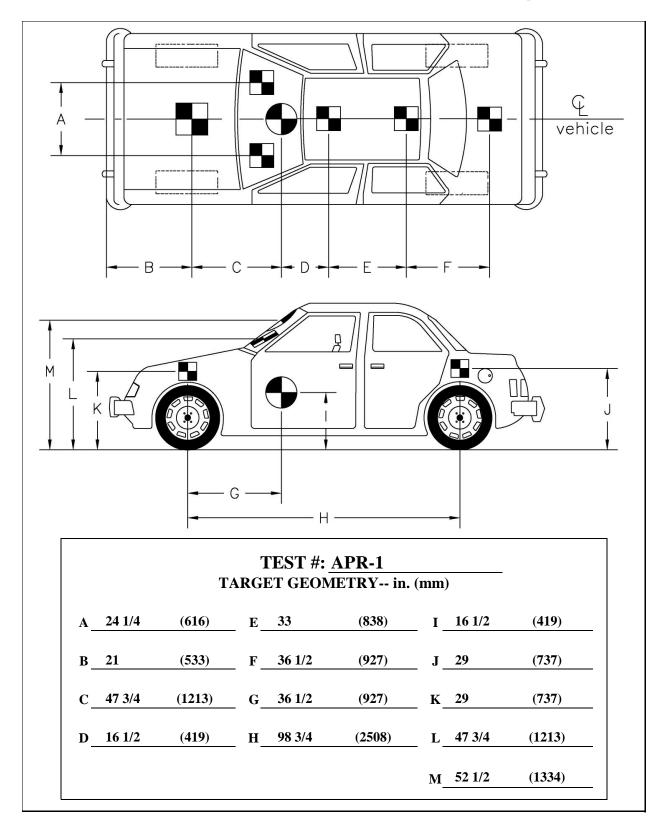


Figure 162. Target Geometry, Test No. APR-1

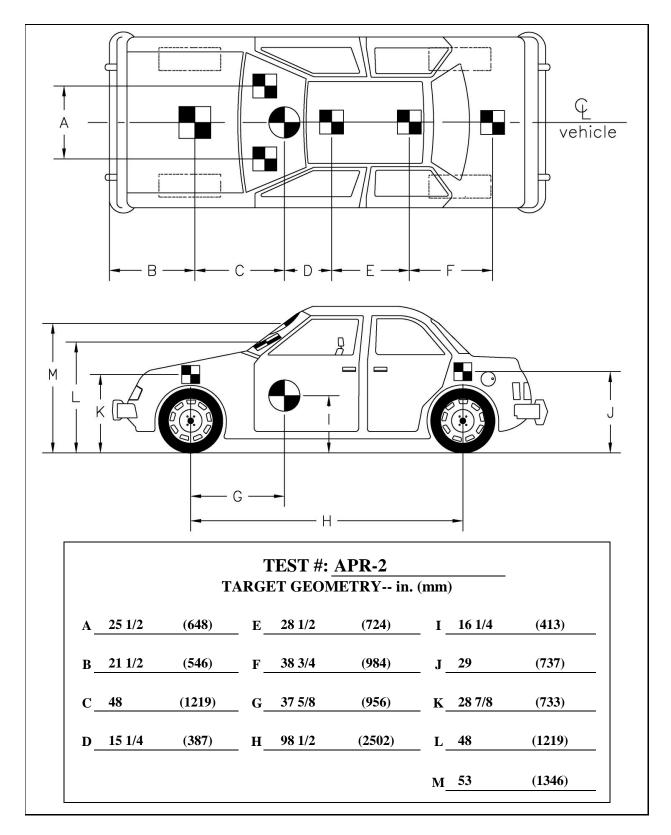


Figure 163. Target Geometry, Test No. APR-2

#### **11.4 Simulated Occupant**

For test nos. APR-1 and APR-2, a Hybrid II 50<sup>th</sup>-Percentile, Adult Male Dummy, equipped with clothing and footwear, was placed in the right-front seat of the test vehicle with the seat belt fastened. The dummy, which had an approximate weight of 170 lb (77 kg), was represented by model no. 572, serial no. 451, and was manufactured by Android Systems of Carson, California. As recommended by MASH, the dummy was not included in calculating the c.g location.

#### **11.5 Data Acquisition Systems**

### **11.5.1 Accelerometers**

Two environmental shock and vibration sensor/recorder systems were used to measure the accelerations in the longitudinal, lateral, and vertical directions. All of the accelerometers were mounted near the center of gravity of each test vehicle. The electronic accelerometer data obtained in dynamic testing was filtered using the SAE Class 60 and the SAE Class 180 Butterworth filters conforming to SAE J211/1 specifications [43].

The first accelerometer system, DTS, was a two-arm piezoresistive accelerometer system manufactured by Endevco of San Juan Capistrano, California. Three accelerometers were used to measure each of the longitudinal, lateral, and vertical accelerations independently at a sample rate of 10,000 Hz. The accelerometers were configured and controlled using a system developed and manufactured by Diversified Technical Systems, Inc. (DTS) of Seal Beach, California. More specifically, data was collected using a DTS Sensor Input Module (SIM), Model TDAS3-SIM-16M. The SIM was configured with 16 MB SRAM and eight sensor input channels with 250 kB SRAM/channel. The SIM was mounted on a TDAS3-R4 module rack. The module rack was configured with isolated power/event/communications, 10BaseT Ethernet and RS232 communication, and an internal backup battery. Both the SIM and module rack were

crashworthy. The "DTS TDAS Control" computer software program and a customized Microsoft Excel worksheet were used to analyze and plot the accelerometer data.

The second system, SLICE-2, was a modular data acquisition system manufactured by DTS. The acceleration sensors were mounted inside the body of the custom-built SLICE 6DX event data recorder and recorded data at 10,000 Hz to the onboard microprocessor. The SLICE 6DX was configured with 7 GB of non-volatile flash memory, a range of  $\pm$ 500 g's, a sample rate of 10,000 Hz, and a 1,650 Hz (CFC 1000) anti-aliasing filter. The "SLICEWare" computer software program and a customized Microsoft Excel worksheet were used to analyze and plot the accelerometer data.

#### **11.5.2 Rate Transducers**

An angle rate sensor, the ARS-1500, with a range of 1,500 degrees/sec in each of the three directions (roll, pitch, and yaw) was used to measure the rates of rotation of the test vehicles. The angular rate sensor was mounted on an aluminum block inside the test vehicle near the center of gravity and recorded data at 10,000 Hz to the DTS SIM. The raw data measurements were then downloaded, converted to the proper Euler angles for analysis, and plotted. The "DTS TDAS Control" computer software program and a customized Microsoft Excel worksheet were used to analyze and plot the angular rate sensor data.

A second angle rate sensor system, mounted inside the body of the SLICE-2 event data recorder was used to measure the rates of rotation of the test vehicle. The SLICE MICRO Triax ARS had a range of 1,500 degrees/sec in each of the three directions (roll, pitch, and yaw) and recorded data at 10,000 Hz to the onboard microprocessor. The raw data measurements were then downloaded, converted to the proper Euler angles for analysis, and plotted. The "SLICEWare" computer software program and a customized Microsoft Excel worksheet were used to analyze and plot the angular rate sensor data.

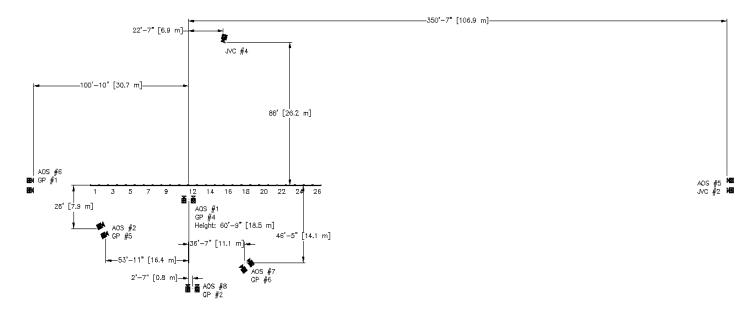
## **11.5.3 Retroreflective Optic Speed Trap**

The retroreflective optic speed trap was used to determine the speed of the bogie vehicle before impact. Five retroreflective targets, spaced at approximately 18-in. (457-mm) intervals, were applied to the side of the vehicle. When the emitted beam of light was reflected by the targets and returned to the Emitter/Receiver, a signal was sent to the data acquisition computer, recording at 10,000 Hz, and activated the External LED box. The speed was then calculated using the spacing between the retroreflective targets and the time between the signals. LED lights and high-speed digital video analysis are only used as backups in the event that vehicle speeds cannot be determined from the electronic data.

# **11.5.4 Digital Photography**

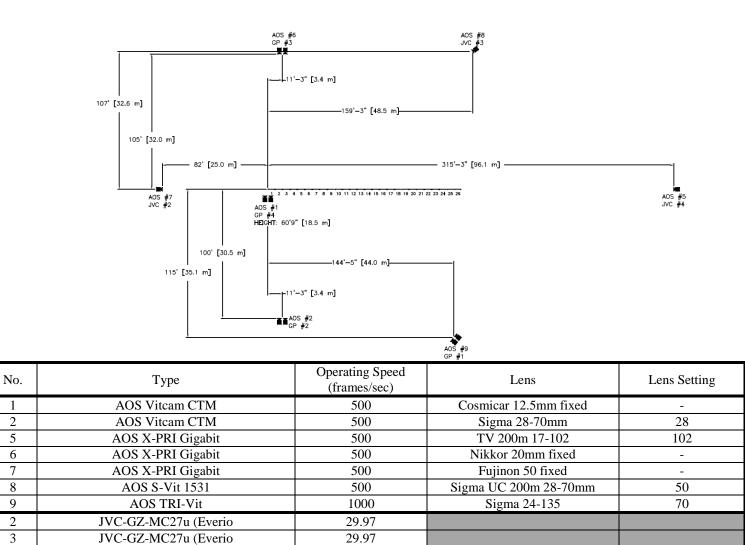
Six AOS high-speed digital video cameras, five GoPro digital video cameras, and two JVC digital video cameras were utilized to film test no. APR-1. Seven AOS high-speed digital video cameras, four GoPro digital video cameras, and three JVC digital video cameras were utilized to film test no. APR-2. Camera details, camera operating speeds, lens information, and a schematic of the camera locations relative to the system for each crash test are shown in Figures 164 and 165 for test nos. APR-1 and APR-2, respectively.

The high-speed videos were analyzed using ImageExpress MotionPlus and RedLake MotionScope software programs. Actual camera speed and camera divergence factors were considered in the analysis of the high-speed videos. A Nikon D50 digital still camera was used to document pre- and post-test conditions for all tests.



No.	Туре	Operating Speed (frames/sec)	Lens	Lens Setting
1	AOS Vitcam CTM	500	Cosmicar 12.5mm fixed	-
2	AOS Vitcam CTM	500	Sigma 28-70mm	28
5	AOS X-PRI Gigabit	500	Canon 200m	102
6	AOS X-PRI Gigabit	500	Fujinon-50mm fixed	-
7	AOS X-PRI Gigabit	500	Nikkor 28mm fixed	-
8	AOS S-Vit 1531	500	Sigma UC 200m 28-70mm	35
2	JVC – GZ-MC27u (Everio)	29.97		
4	JVC – GZ-MG27u (Everio)	29.97		
1	GoPro Hero 3	120		
2	GoPro Hero 3	120		
4	GoPro Hero 3+	120		
5	GoPro Hero 3+	120		
6	GoPro Hero 3+	120		

Figure 164. Camera Locations, Speeds, and Lens Settings, Test No. APR-1



29.97

120

120

120

120

GoPro Hero 3+ Figure 165. Camera Locations, Speeds, and Lens Settings, Test No. APR-2

JVC-GZ-MC27u (Everio

GoPro Hero 3

GoPro Hero 3

GoPro Hero 3+

239

4

1

2

3

#### **12 DESIGN DETAILS**

The 150-ft (45.7-m) long channelizer system was comprised of twenty-six aluminum pedestrian rail panels. Design details for test nos. APR-1 and APR-2 are shown in Figures 166 through 171. Photographs of the as-tested system are shown in Figures 172 through 174. Material specifications, mill certifications, and certificates of conformity for the system materials are shown in Appendix D.

Each panel utilized 2-in. x 4-in. x <sup>1</sup>/<sub>4</sub>-in. x 43-in. tall (51-mm x 102-mm x 6-mm x 1,029mm tall) posts with three 2-in. x 2-in. x <sup>1</sup>/<sub>8</sub>-in. (51-mm x 51-mm x 3-mm) rail components at heights of 42 in. (1,067 mm),  $24^{15}/_{16}$  in. (633 mm), and 7% in. (200 mm). The rails were inserted into cutouts in the posts at each rail location and secured to the face of the posts with <sup>1</sup>/<sub>8</sub>-in. (3mm) fillet welds at each connection. Nine <sup>1</sup>/<sub>2</sub>-in. x <sup>1</sup>/<sub>2</sub>-in. x 32<sup>1</sup>/<sub>8</sub>-in. (13-mm x 13-mm x 816-mm) square spindles spanned between the top and bottom rails and were inserted through the middle rail. The spindles were welded with <sup>1</sup>/<sub>8</sub>-in. (3-mm) fillet welds at each rail location. Each post member was welded to a 3-in. x 7<sup>3</sup>/<sub>4</sub>-in. x <sup>3</sup>/<sub>8</sub>-in. (76-mm x 191-mm x 9.5-mm) baseplate with a <sup>1</sup>/<sub>4</sub>-in. (6-mm) fillet weld at the connection. The baseplate had two <sup>5</sup>/<sub>8</sub>-in. (16-mm) holes spaced at 6<sup>1</sup>/<sub>4</sub> in. (159 mm) to accommodate two <sup>1</sup>/<sub>2</sub>-in. (13-mm) diameter threaded anchor rods, each embedded 5 in. (127 mm) into 1,450-psi (10.0-MPa) minimum bond strength epoxy adhesive and secured through the baseplate with a <sup>1</sup>/<sub>2</sub>-in. (13-mm) diameter ASTM A194 Grade 8M nut. The concrete foundation had a minimum compressive strength of 2,500 psi (17.2 MPa), a minimum thickness of 7 in. (178 mm), and outer dimensions at least 10 in. (254 mm) away from the nearest anchor. The panels were spaced  $5\frac{1}{2}$  in. (140 mm) away from each other.

During baseplate fabrication, jigs were built and slots were cut into the baseplates to aid in welding. Note, these slots do not appear in the system drawings. Examples of these slots are shown in Figure 174. Drawings from the fabricator are shown in Appendix F.

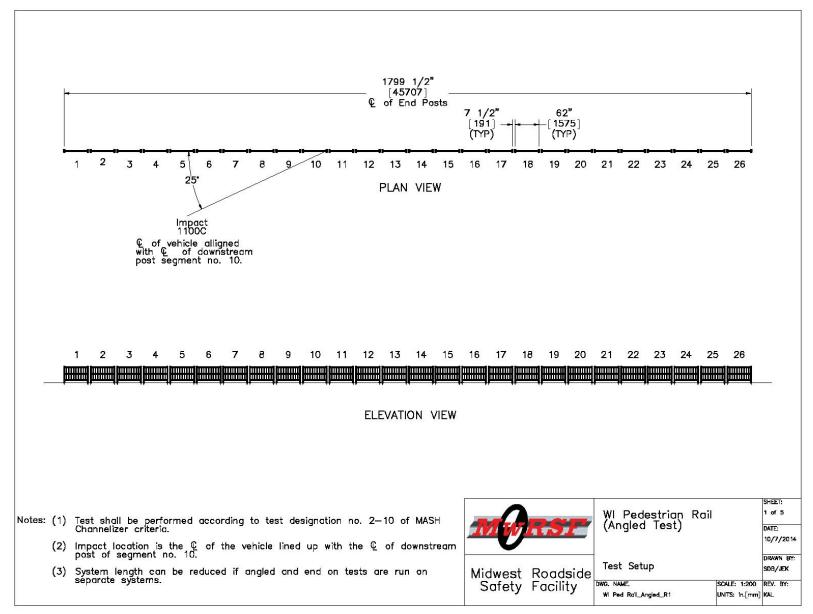


Figure 166. Test Installation Layout, Test No. APR-1

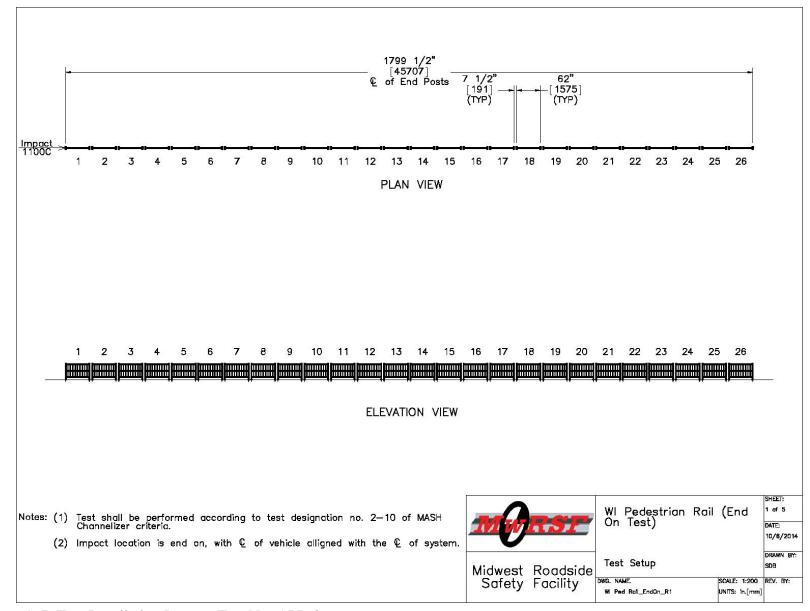


Figure 167. Test Installation Layout, Test No. APR-2

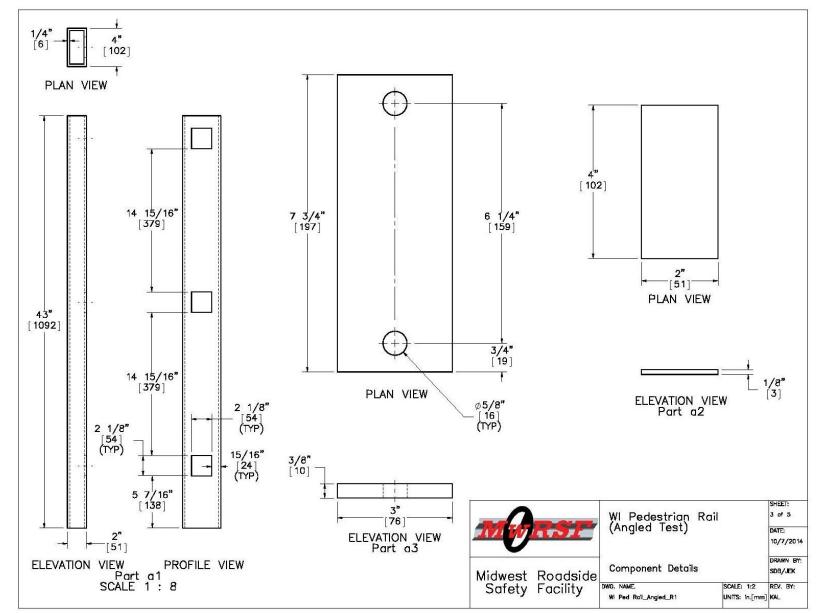


Figure 168. Component Details, Test Nos. APR-1 and APR-2

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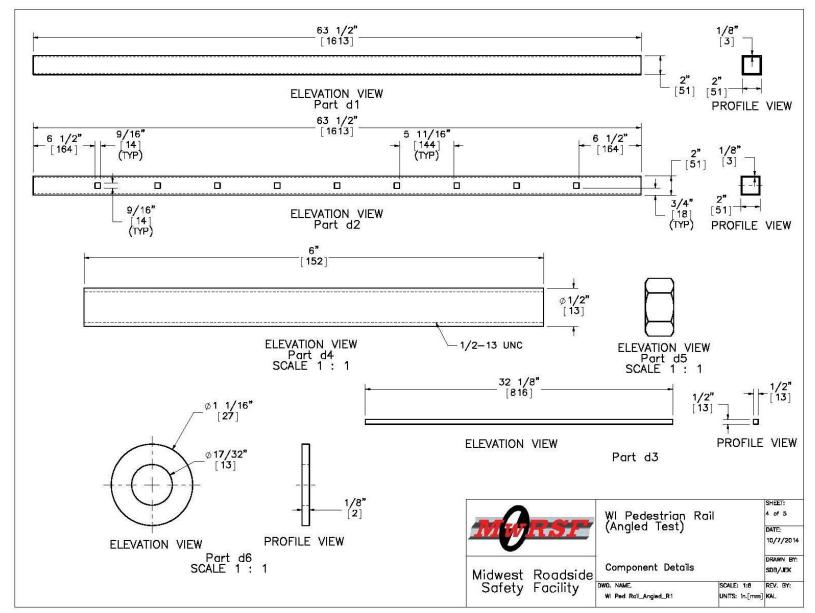


Figure 169. Component Details, Test Nos. APR-1 and APR-2

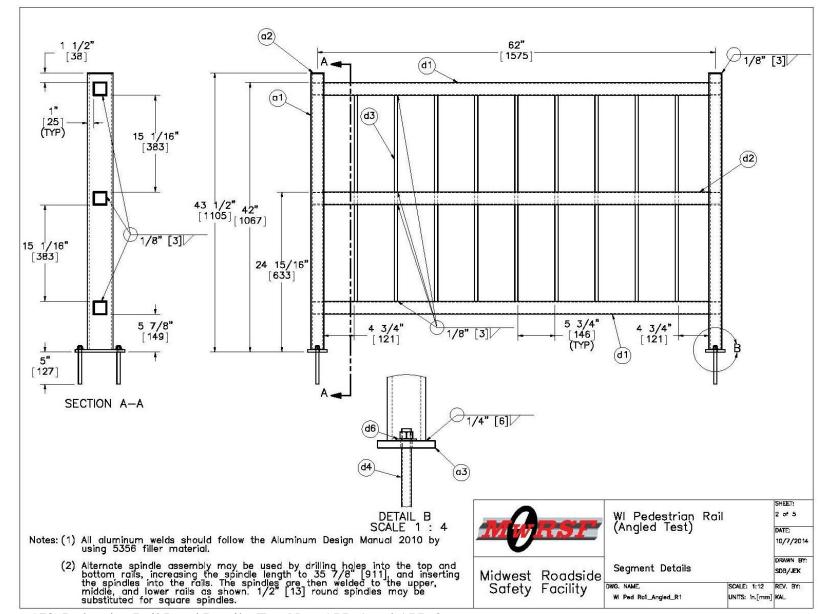


Figure 170. Pedestrian Rail Panel Details, Test Nos. APR-1 and APR-2

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Item No.	QTY.	Description	Material Spec		
o1	52	2"x4"x1/4" [51x102x6] Aluminum Post, 43" [1092] long	6061–T6		
a2	52	Aluminum Post Cap — 1/8" [3] Plate	6061-T6		
a3	52	Aluminum Post Base	6061–T6		
d1	52	2"x2"x1/8" [51x51x3] Aluminum Rail - 63 1/2" [1613] long	6061–T6		
d2	26	2"x2"x1/8" [51x51x3] Aluminum Rail - 63 1/2" [1613] long with holes	6061–T6		
d3	234	1/2"x1/2" [13x13] Square Aluminum Spindle - 32 1/8" [816] long	6061–T6		
d4	104	1/2" [13] Dia. UNC, 6" [152] Long Threaded Rod	ASTM A193 Grade B7		
d5	104	1/2" [13] Dia. Steel Nut	ASTM A194 Grade 8M Galv.		
d6	104	1/2" [13] Dia. Steel SAE Flat Washer	ASTM F436 Type 1 Galv.		
d7		Ероху	Powers Fasteners AC100+ Gold Minimum bond strength = 1,450 psi [10.0 MPa]		

-0		WI Pedestrian	Rail		SHEET: 5 of 5
MAN	RSE	WI Pedestrian (Angled Test)			DATE: 10/7/2014
V		Bill of Materials			DRAWN BY:
Midwest	Roadside				SDB/JEK
Safety	Facility	DWG. NAME. Wi Ped Roll_Angled_R1	i	SCALE: None UNITS: In.[mm]	REV. BY:

Figure 171. Bill of Materials, Test Nos. APR-1 and APR-2





Figure 172. Pedestrian Rail Test Installation



Figure 173. System Panels and Anchors, Test No. APR-1



Figure 174 Slots Cut in Baseplates to Aid in Rail Fabrication

# 13 FULL-SCALE CRASH TEST NO. APR-1

#### 13.1 Test No. APR-1

The 2,428-lb (1,101-kg) small car impacted the aluminum pedestrian rail at a speed of 45.2 mph (72.7 km/h) and an angle of 25.1 degrees. A summary of the test results and sequential photographs are shown in Figure 175. Additional sequential photographs are shown in Figures 176 and 177. Documentary photographs of the crash test are shown in Figure 178.

### **13.2 Weather Conditions**

Test no. APR-1 was conducted on October 24, 2014, at approximately 1:30 p.m. The weather conditions, as per the National Oceanic and Atmospheric Administration (station 14939/LNK), were reported and are shown in Table 13.

Temperature	74° F
Humidity	56%
Wind Speed	9 mph
Wind Direction	230° from True North
Sky Conditions	Partly Cloudy
Visibility	10.00 Statute Miles
Pavement Surface	Dry
Previous 3-Day Precipitation	0.66 in.
Previous 7-Day Precipitation	0.66 in.

Table 13. Weather Conditions, Test No. APR-1

#### **13.3 Test Description**

Initial vehicle impact was to occur with the centerline of the vehicle aligned with the centerline of the downstream post of panel no. 12, as shown in Figures 166 and 179. This location was selected in order to evaluate the potential for windshield damage as the result of a panel sliding up the hood and the debris field. The first point of vehicle contact was the second spindle upstream from the downstream post of panel no. 11. A sequential description of the impact events is contained in Table 14. The vehicle came to rest upright and 45.5 ft (13.9 m)

behind the centerline of panel no. 26. The vehicle trajectory and final position are shown in Figures 175 and 180.

TIME	EVENT		
(sec) 0	Right side of front bumper contacted downstream end of panel no. 11		
0.002	Right side of front bumper began to deform		
0.010	Right fender began to deform		
0.036	Vehicle began to yaw toward channelizer		
0.042	Right-front tire overrode downstream end of panel no. 11		
0.052	Panel no. 11 disengaged from post baseplates		
0.056	Panel no. 12 disengaged from post baseplates		
0.074	Front bumper contacted panel no. 13		
0.082	Hood began to deform		
0.102	Left fender began to deform		
0.110	Right rear tire became airborne		
0.114	Panel no. 13 disengaged from post baseplates		
0.128	Panel no. 14 disengaged from post baseplates		
0.138	Vehicle pitched downward		
0.144	Upstream post of panel no. 15 disengaged from upstream post baseplate		
0.176	Vehicle rolled slightly toward barrier		
0.218	Left fender contacted panel no. 15		
0.312	Right-rear tire contacted ground		

Table 14. Sequential Description of Impact Events, Test No. APR-1

# 13.4 System Damage

Damage to the pedestrian rail is shown in Figures 181 through 186. The welds fractured between the posts and baseplates on panel nos. 11 through 15, resulting in the panels disengaging from the baseplates. All anchors remained undamaged. The final locations of the disengaged panels are shown in Table 15.

All components of panel no. 11 remained intact, except the post welds fractured and the posts disengaged from the baseplates. The entire panel twisted slightly. The welds partially fractured at the upstream and downstream ends of the bottom and top horizontal rails, the downstream end of the middle horizontal rail, and around some of the spindles of panel no. 11. The upstream end of the middle rail and downstream end of the top rail partially ruptured. The post baseplates encountered minor deformations. Contact marks were found on the lower portion of the downstream post.

Panel no. 12 remained intact, except the post welds fractured and the posts disengaged from the baseplates and five spindles disengaged. The entire panel twisted slightly. The welds partially fractured on panel no. 12 at the upstream end of the bottom and middle horizontal rails, the downstream end of the middle horizontal rail, and around some of the spindles. The bottom horizontal rail tore at spindle locations. The post baseplates were bent. Contact marks were found on the upstream and front faces of the upstream post.

All components of panel no. 13 remained intact, except the post welds fractured and the posts disengaged from the baseplates. The panel was bent. The welds partially fractured at both the upstream and downstream ends of all three horizontal rails and the downstream post cap of panel no. 13. The downstream end of the bottom rail partially ruptured. The post baseplates were bent, and the downstream slot in the downstream post baseplate sheared. Dents were found on the upstream face of the upstream post and on the back face of the top rail. Contact marks were found on the top of the upstream post and on the upstream face of the upstream post near the bottom and middle horizontal rails.

All components of panel no. 14 remained intact, except the post welds fractured and the posts disengaged from the baseplates. The panel was bent. The welds fractured at both the

upstream and downstream ends of all three horizontal rails of panel no. 14. The post baseplates were bent.

All components of panel no. 15 remained intact, except the post welds fractured and the posts disengaged from the baseplates. The panel was bent. The welds fractured at both the upstream and downstream ends of all three horizontal rails and around some of the spindles of panel no. 15. The downstream end of the top rail tore. Some spindles were bent. The post baseplates were bent, and the downstream slot in the upstream post baseplate sheared.

All components of panel no. 16 remained intact. The panel was bent. The welds partially fractured at both the upstream and downstream ends of all three horizontal rails, except the downstream end of the top rail. Welds between the posts and the baseplates began to fracture; however, the posts remained attached to the baseplates. The downstream post baseplate was bent.

Table 15. Final Locations of Disengaged Panels, Test No. APR-1

Panel No.	Final Location	Reference Location
11	55 ft (16.8 m) behind system	Centerline of panel no. 13 (initial location)
12	70 ft (21.3 m) behind system	Centerline of panel no. 20
13	6 ft (1.8 m) behind system	Centerline of panel no. 23
14	29 ft (8.8 m) in front of system	Joint between panel nos. 13 and 14
15	13.5 ft (4.1 m) in front of system	Centerline of panel no. 15 (initial location)

## **13.5 Vehicle Damage**

The damage to the vehicle was moderate, as shown in Figures 187 and 189. The maximum occupant compartment deformations are listed in Table 16 along with the deformation limits established in MASH for various areas of the occupant compartment. Note that none of the MASH-established deformation limits were violated. Occupant compartment damage is shown in Figure 189. Complete occupant compartment and vehicle deformations and the corresponding locations are provided in Appendix G.

The majority of the damage was concentrated on the front of the vehicle. The front plastic bumper disengaged and was fractured. The hood and front bumper were dented and deformed backward into the radiator. The right-front fender deformed outward, and the right headlight fractured. The right-front rim was dented, and the tire was torn and deflated. The right-front Aarm disengaged from the wheel. The left-front fender was dented and encountered contact marks behind the tire. The left headlight disengaged but remained attached by the cable. All window glass remained undamaged.

LOCATION	MAXIMUM DEFORMATION in. (mm)	MASH ALLOWABLE DEFORMATION in. (mm)
Wheel Well & Toe Pan	<sup>1</sup> ⁄ <sub>4</sub> (6)	≤ 9 (229)
Floorpan & Transmission Tunnel	<sup>1</sup> ⁄ <sub>4</sub> (6)	≤ 12 (305)
Side Front Panel (in Front of A-Pillar)	<sup>1</sup> /2 (13)	≤ 12 (305)
Side Door (Above Seat)	<sup>1</sup> / <sub>2</sub> (13)	≤ 9 (229)
Side Door (Below Seat)	<sup>1</sup> ⁄ <sub>4</sub> (6)	≤ 12 (305)
Roof	0 (0)	$\leq 4 (102)$
Windshield	0 (0)	≤ 3 (76)

Table 16. Maximum Occupant Compartment Deformations by Location

## 13.6 Occupant Risk

The calculated occupant impact velocities (OIVs) and maximum 0.010-sec occupant ridedown accelerations (ORAs) in both the longitudinal and lateral directions are shown in Table 17. Note that the OIVs and ORAs were within the suggested limits provided in MASH. The calculated THIV, PHD, and ASI values are also shown in Table 17. The results of the occupant risk analysis, as determined from the accelerometer data, are summarized in Figure 175. The recorded data from accelerometers and rate transducers are shown graphically in Appendix H.

Note, the DTS unit was designated as the primary unit during this test, as it was mounted closer to the c.g. of the vehicle.

Evaluation Criteria		Trans	MASH	
		DTS (Primary)	SLICE-2	Limits
OIV	Longitudinal	-19.08 (-5.82)	-18.79 (-5.73)	≤40 (12.2)
ft/s (m/s)	Lateral	3.89 (1.19)	2.89 (0.88)	≤40 (12.2)
ORA	Longitudinal	-1.85	-2.11	≤ 20.49
g's	Lateral	-3.33	-3.35	≤ 20.49
MAX.	Roll	10.61	-5.64	≤75
ANGULAR DISPL.	Pitch	7.99	-1.82	≤75
deg.	Yaw	50.72	51.33	not required
THIV ft/s (m/s)		20.26 (6.17)	20.08 (6.12)	not required
PHD g's		3.65	3.79	not required
ASI		0.62	0.60	not required

Table 17. Summary of OIV, ORA, THIV, PHD, and ASI Values, Test No. APR-1

# **13.7 Discussion**

The analysis of the test results for test no. APR-1 showed that the pedestrian rail allowed controlled penetration of the 1100C vehicle through the longitudinal channelizer. Neither detached elements nor fragments showed potential for penetrating the occupant compartment or for presenting undue hazard to other traffic. Note, none of the pedestrian rail panels went over the hood, near the windshield, or underneath the vehicle. Deformations of, or intrusions into, the occupant compartment that could have caused serious injury did not occur. The OIVs and ORAs were within the suggested limits provided in MASH. The test vehicle remained upright during

and after the collision. Vehicle roll, pitch, and yaw angular displacements, as shown in Appendix I, were deemed acceptable, because they did not adversely influence occupant risk safety criteria or cause rollover. After impact, the vehicle penetrated behind the channelizer. Therefore, test no. APR-1 was determined to be acceptable according to the MASH safety performance criteria for longitudinal channelizers, test designation no. 2-90.

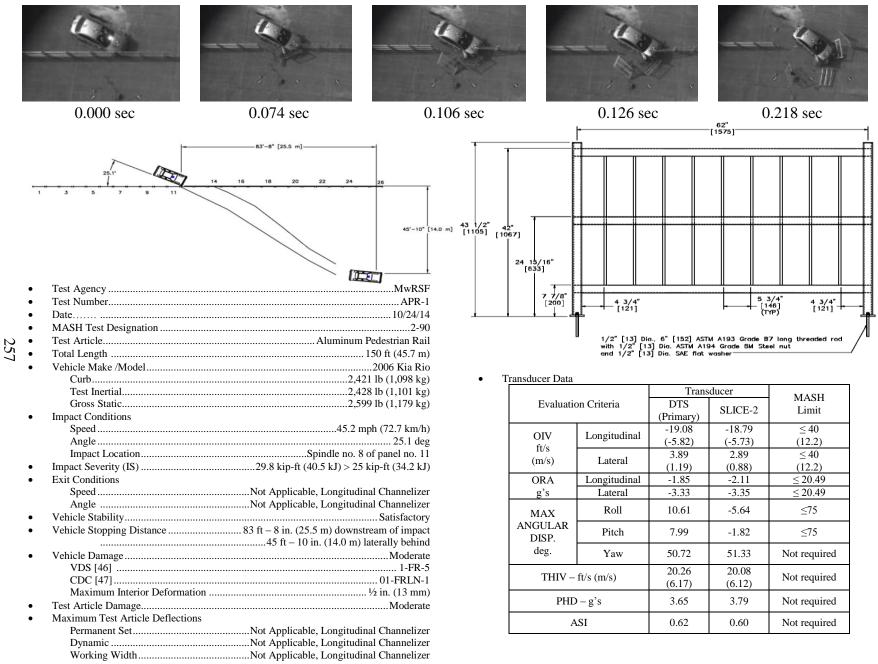


Figure 175. Summary of Test Results and Sequential Photographs, Test No. APR-1

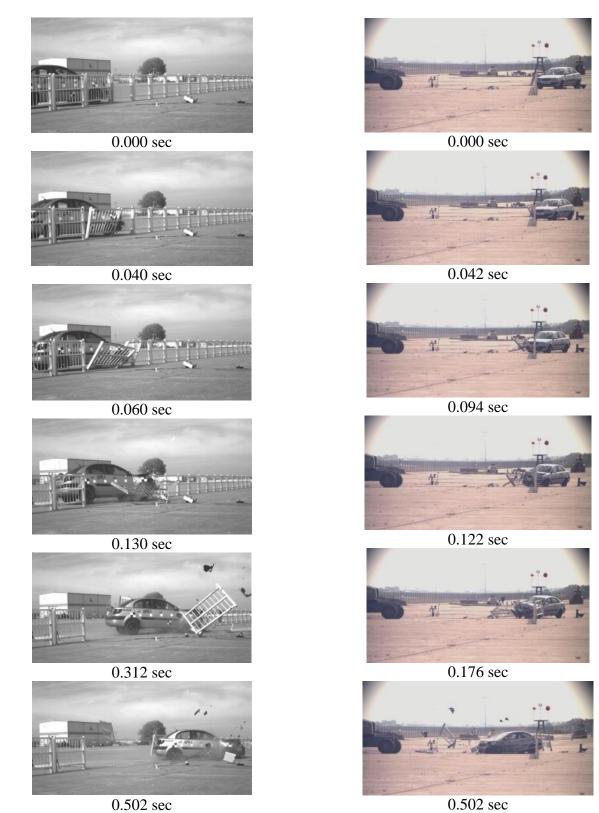


Figure 176. Additional Sequential Photographs, Test No. APR-1

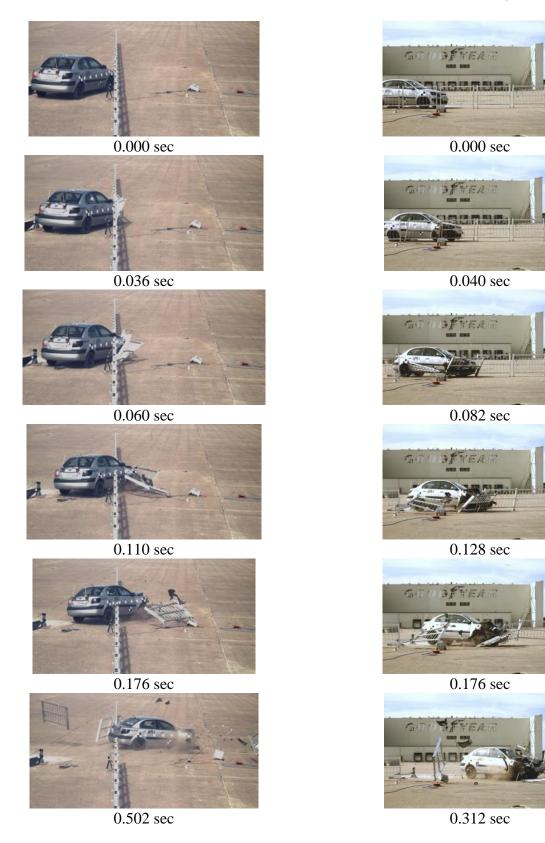


Figure 177. Additional Sequential Photographs, Test No. APR-1

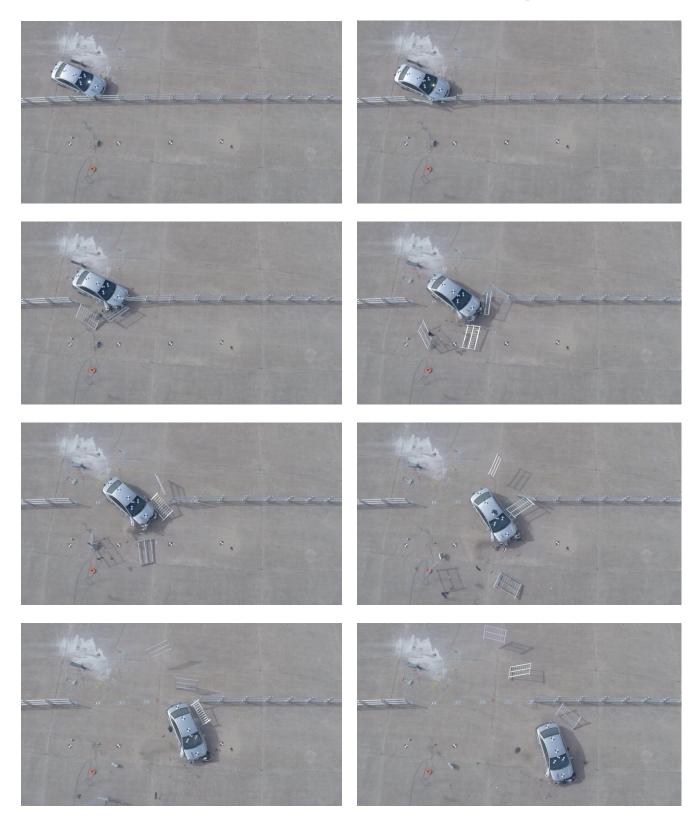


Figure 178. Documentary Photographs, Test No. APR-1

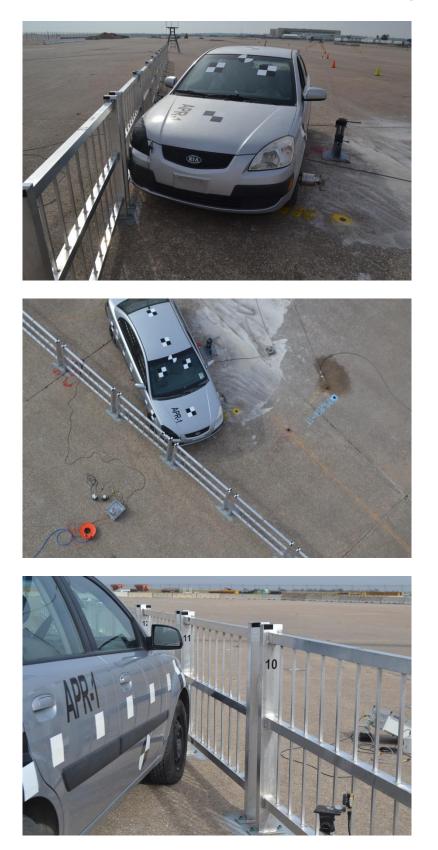


Figure 179. Impact Location, Test No. APR-1



Figure 180. Vehicle Final Position and Trajectory Marks, Test No. APR-1







Figure 181. System Damage, Test No. APR-1



Figure 182. Panel No. 11 Damage, Test No. APR-1



Figure 183. Panel No. 12 Damage, Test No. APR-1



Figure 184. Panel No. 13 Damage, Test No. APR-1

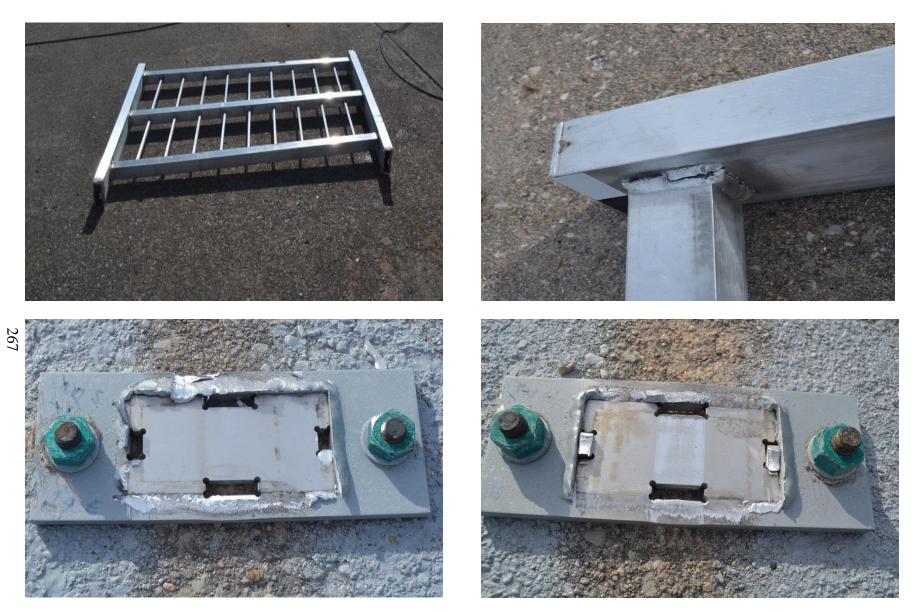
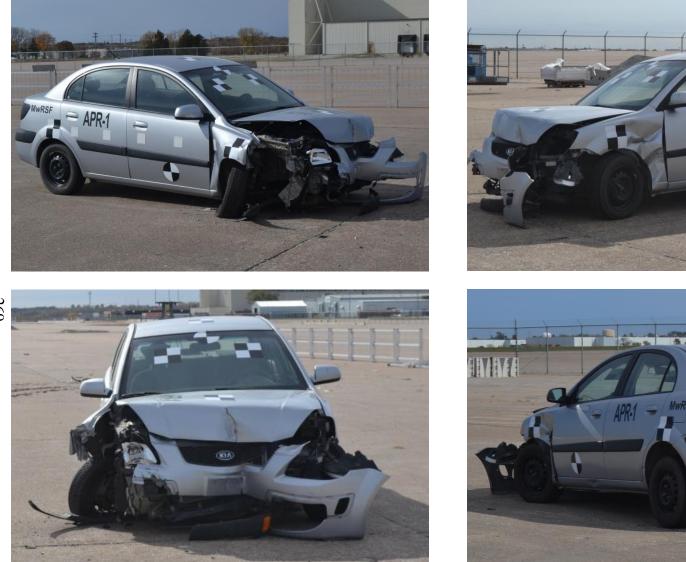


Figure 185. Panel No. 14 Damage, Test No. APR-1



Figure 186. Panel No. 15 Damage, Test No. APR-1

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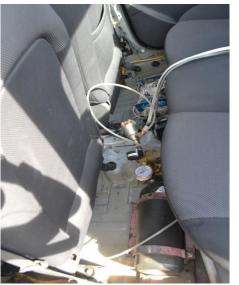
Figure 188. Vehicle Damage, Test No. APR-1

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Figure 189. Occupant Compartment Damage, Test No. APR-1





# 14 FULL-SCALE CRASH TEST NO. APR-2

# 14.1 Test No. APR-2

The 2,437-lb (1,105-kg) small car impacted the aluminum pedestrian rail at a speed of 44.5 mph (71.6 km/h) and an angle of 0 degrees. A summary of the test results and sequential photographs are shown in Figure 190. Additional sequential photographs are shown in Figures 191 and 192. Documentary photographs of the crash test are shown in Figure 193.

# **14.2 Weather Conditions**

Test no. APR-2 was conducted on November 12, 2014, at approximately 2:15 p.m. The weather conditions, as per the National Oceanic and Atmospheric Administration (station 14939/LNK), were reported and are shown in Table 18.

Temperature	23° F
Humidity	48%
Wind Speed	20 mph
Wind Direction	320° from True North
Sky Conditions	Sunny
Visibility	10.00 Statute Miles
Pavement Surface	Dry
Previous 3-Day Precipitation	0.06 in.
Previous 7-Day Precipitation	0.07 in.

Table 18. Weather Conditions, Test No. APR-2

# **14.3 Test Description**

Initial vehicle impact was to occur with the centerline of the vehicle aligned with the centerline of the upstream post of panel no. 1, as shown in Figures 167 and 194. This location was selected in order to evaluate the potential for windshield and roof damage, vehicle instability, occupant risk, and the debris field. The actual point of impact was the centerline of the upstream post of panel no. 1. A sequential description of the impact events is contained in Table 19. The vehicle came to rest 15 ft – 11 in. (4.9 m) upstream from the upstream post of

panel no. 10 and parallel to the centerline of the system. The vehicle trajectory and final position are shown in Figures 190 and 195.

TIME (sec)	EVENT
0	Front bumper contacted upstream post of panel no. 1
0.002	Front bumper deformed
0.006	Hood deformed
0.014	Downstream post of panel no. 1 contacted upstream post of panel no. 2
0.024	Panel no. 1 disengaged from post baseplates
0.030	Downstream post of panel no. 2 contacted upstream post of panel no. 3
0.038	Downstream post of panel no. 3 contacted upstream post of panel no. 4
0.044	Panel no. 2 disengaged from post baseplates
0.050	Downstream post of panel no. 4 contacted upstream post of panel no. 5
0.054	Panel no. 3 disengaged from post baseplates
0.060	Downstream post of panel no. 5 contacted upstream post of panel no. 6
0.068	Panel no. 4 disengaged from post baseplates
0.070	Downstream post of panel no. 6 contacted upstream post of panel no. 7
0.078	Panel no. 5 disengaged from post baseplates
0.088	Downstream post of panel no. 7 contacted upstream post of panel no. 8
0.216	Panel no. 1 overrode hood
0.268	Panel no. 2 overrode hood
0.342	Panel no. 3 overrode hood
0.364	Front bumper contacted upstream post of panel no. 4
0.476	Front bumper contacted upstream post of panel no. 5
0.684	Panel no. 6 disengaged from post baseplates
0.710	Downstream post of panel no. 8 contacted upstream post of panel no. 9
0.872	Panel no. 7 disengaged from post baseplates
0.892	Downstream post of panel no. 9 contacted upstream post of panel no. 10
0.908	Panel no. 8 disengaged from post baseplates
1.580	Front bumper contacted upstream post of panel no. 9
1.592	Panel no. 9 disengaged from post baseplates
1.774	Vehicle stopped and began to roll backward

 Table 19. Sequential Description of Impact Events, Test No. APR-2

#### 14.4 System Damage

Damage to the pedestrian rail is shown in Figures 196 through 206. The welds fractured between the posts and baseplates for panel nos. 1 through 9, resulting in the panels disengaging away from the baseplates. Panel nos. 1 through 9 otherwise remained intact and were bent. All anchors remained undamaged. The final locations of the disengaged panels are shown in Table 20.

Panel no. 1 had fractured welds at the downstream end of the middle and top rails and the bottom of the vertical spindles. The upstream post fractured above the bottom rail. The downstream post also fractured, but this occurred below the middle rail. The downstream end of the bottom rail tore. All spindles bent.

Panel nos. 2, 3, 4, and 5 had fractured welds at the upstream end of the bottom, middle, and top horizontal rails, the downstream end of the middle and top rails, and the bottom of some vertical spindles. The upstream and downstream posts fractured below the middle rail on panel nos. 2 through 5. The downstream end of the bottom rail tore on panel nos. 2 through 5. All spindles on panel nos. 2 through 5 bent. Gouges were found on the bottom rail of panel no. 5.

Panel no. 6 had fractured welds at the upstream end of the bottom, middle, and top rails and the downstream end of the middle and top rails. The upstream post fractured below the middle rail. The downstream post bent at the middle rail. The downstream end of the bottom rail tore. The middle of all three rails bent and tore. The top rail had a <sup>5</sup>/<sub>8</sub>-in. (16-mm) diameter hole near the downstream post. All spindles bent and one spindle fractured.

The downstream post of panel no. 7 fractured and bent below the middle rail. The upstream post bent at the middle rail. The upstream and downstream ends of the bottom, middle, and top rails tore. Gouges were found on the upstream face of the downstream post. All spindles bent.

Panel no. 8 had fractured welds at the upstream end of the bottom, middle, and top rails and the downstream end of the middle and top rails. The downstream end of the bottom rail tore. The upstream and downstream posts bent at the middle rail. All spindles bent.

The horizontal rails of panel no. 9 buckled near their midspans. The bottom and middle rails tore near the middle. Welds fractured at the upstream end of the bottom, middle, and top rails and the downstream end of the middle rail. The downstream post bent at the middle rail. Significant gouging was found on the upstream post. All spindles bent.

Panel no. 10 remained intact, and the panel remained attached to both baseplates. The entire panel was bent. The welds partially fractured at the upstream and downstream ends of the bottom, middle, and top rails and at the upstream and downstream baseplates. The upstream and downstream posts were bent.

Panel	Final Location	Reference Location		
No.		Centerline of upstream post on panel no. 10		
1	39 ft (11.9 m) right of system	35 ft (10.7 m) upstream		
2	4 ft (1.2 m) right of system	30 ft (9.1 m) upstream		
3	5 ft (1.5 m) left of system	30 ft (9.1 m) upstream		
4	21 ft (6.4 m) right of system	16 ft (4.9 m) downstream		
5	6 ft (1.8 m) left of system	6 ft (1.8 m) upstream		
6	8.5 ft (2.6 m) right of system	2 ft (0.6 m) downstream		
7	30 ft (9.1 m) right system	5 ft (1.5 m) downstream		
8	32 ft (9.8 m) right of system	7 ft (2.1 m) upstream		
9	2.5 ft (0.8 m) left of system	2.7 ft (0.8 m) upstream		

Table 20. Final Location of Disengaged Panels, Test No. APR-2

# 14.5 Vehicle Damage

The damage to the vehicle was moderate, as shown in Figures 207 through 210. The maximum occupant compartment deformations are listed in Table 21 along with the deformation limits established in MASH for various areas of the occupant compartment. Note that none of the MASH-established deformation limits were violated. Occupant compartment damage is shown

in Figure 209. Complete occupant compartment and vehicle deformations and the corresponding locations are provided in Appendix G.

The majority of the damage was concentrated on the front of the vehicle. The front plastic bumper disengaged and fractured. The entire front end, including the radiator supports and steel bumper, crushed inward. The hood was bent and deformed upward. Both the right-front and leftfront fenders were bent, deformed, and dented. Both headlights were fractured and disengaged but remained attached by the cables. Contact marks were found on the hood and on the undercarriage of the vehicle. The lower-right corner of the windshield was cracked.

LOCATION	MAXIMUM DEFORMATION in. (mm)	MASH ALLOWABLE DEFORMATION in. (mm)
Wheel Well & Toe Pan	<sup>3</sup> / <sub>8</sub> (9.5)	≤ 9 (229)
Floorpan & Transmission Tunnel	<sup>1</sup> ⁄ <sub>4</sub> (6.4)	≤ 12 (305)
Side Front Panel (in Front of A-Pillar)	<sup>1</sup> ⁄ <sub>4</sub> (6.4)	≤ 12 (305)
Side Door (Above Seat)	0 (0)	≤ 9 (229)
Side Door (Below Seat)	<sup>1</sup> ⁄ <sub>4</sub> (6.4)	≤ 12 (305)
Roof	0 (0)	≤ 4 (102)
Windshield	0 (0)	≤ 3 (76)

Table 21. Maximum Occupant Compartment Deformations by Location

# 14.6 Occupant Risk

The calculated occupant impact velocities (OIVs) and maximum 0.010-sec occupant ridedown accelerations (ORAs) in both the longitudinal and lateral directions are shown in Table 22. The calculated THIV, PHD, and ASI values are also shown in Table 22. The results of the occupant risk analysis, as determined from the accelerometer data, are summarized in Figure 190. The recorded data from the accelerometers and the rate transducers are shown graphically in Appendix I. Note, the DTS unit was designated as the primary unit during this test as it was mounted closer to the c.g. of the vehicle.

		Trans	MASH Limits	
Evaluation Criteria		SLICE-2		
OIV	Longitudinal	-21.95 (-6.69)	-21.69 (-6.61)	≤ 40 (12.2)
ft/s (m/s)	Lateral	-0.23 (-0.07)	-1.19 (-0.36)	≤40 (12.2)
ORA	Longitudinal	-20.91	-19.41	≤ 20.49
g's	Lateral	-6.74	-3.87	≤ 20.49
MAX.	Roll	-2.07	8.63	≤75
ANGULAR DISPL.	Pitch	3.93	17.68	≤75
deg.	Yaw	-1.88	9.57	not required
	THIV ft/s (m/s)         21.95 (6.69)         21.79 (6.64)		not required	
-	PHD g's	20.91	19.5	not required
ASI		0.9	0.87	not required

Table 22. Summary of OIV, ORA, THIV, PHD, and ASI Values, Test No. APR-2

# 14.7 Discussion

The analysis of the test results for test no. APR-2 showed that the pedestrian rail allowed controlled penetration of the 1100C vehicle through the longitudinal channelizer. Neither detached elements nor fragments showed potential for penetrating the occupant compartment or for presenting undue hazard to other traffic. Deformations of, or intrusions into, the occupant compartment that could have caused serious injury did not occur. The test vehicle remained upright during and after the collision and came to rest within the longitudinal line of the channelizer. Vehicle roll, pitch, and yaw angular displacements, as shown in Appendix I, were deemed acceptable, because they did not adversely influence occupant risk safety criteria or cause rollover.

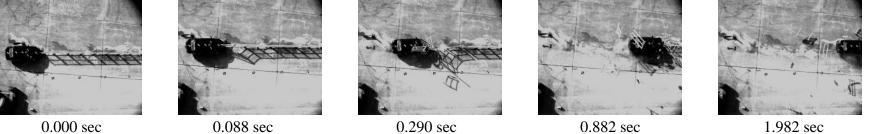
The longitudinal ORA for the backup accelerometer unit was greater than the specified MASH limit. The test results revealed that the pedestrian rail system imparted a high longitudinal ORA to the hypothetical vehicle occupants when struck end-on. The longitudinal ORA from the DTS (the primary accelerometer unit) was 19.41 g's, while the longitudinal ORA from the SLICE-2 (the backup accelerometer unit) was 20.91 g's.

During this testing program, there was no existing guidance or policy within the crash test laboratories regarding comparison of accelerations from different transducer units used during the same test near the c.g. of a test vehicle or which value to choose if the values varied. Consequently, feedback was sought from FHWA in February 2015. Following the discussion with FHWA and quoting them directly,

"We've not seen this situation before (i.e., 2 conflicting accel readings). After review of video, we certainly agree with your detailed assessment of test in regards to acceleration spike. As there is no existing policy for comparing accelerations from different transducer units on same test, we feel it best to recognize the implication of a higher value."

The documentation of the correspondence with FHWA is shown in Appendix J. Based on the FHWA response, test no. APR-2 was initially determined to be unacceptable according to the MASH safety performance criteria for longitudinal channelizers, test designation no. 2-90.

Cases where varying occupant risk results were acquired from different accelerometer systems was addressed with other crash test laboratories during the AASHTO Task Force 13 Subcommittee #7 meeting on April 30, 2015 in Lincoln, Nebraska. During this discussion, the crash test laboratories and FHWA came to a consensus that the results from the primary accelerometer unit would be reported. The primary accelerometer unit is defined as the unit placed closest to the c.g. More detailed information on this discussion can be found in the minutes from the April 2015 AASHTO Task Force 13 Subcommittee #7 meeting shown in Appendix J. Under this guidance, the primary accelerometer unit provided a longitudinal ORA value below the MASH limit. Therefore, test no. APR-2 was subsequently determined to be acceptable according to the MASH safety performance criteria for longitudinal channelizers, test designation no. 2-90.

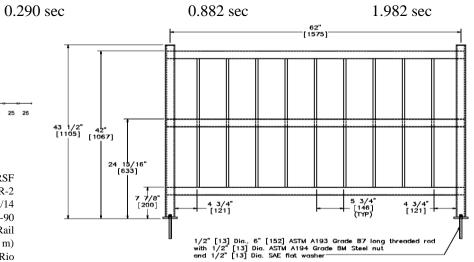


, ", ", **[**]

280

0.088 sec

10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26



		N. DOE
•	6.	MwRSF
•		
•	Date	
٠	MASH Test Designation	
•	Test Article	Aluminum Pedestrian Rail
•	Total Length	
•	Vehicle Make /Model	
	Curb	
	Test Inertial	
	Gross Static	
•	Impact Conditions	
	Speed	
	Angle	
	Impact Location	Upstream post of Panel No. 1
•		
•	Exit Conditions	
	Speed	Not Applicable, Longitudinal Channelizer
	Angle	Not Applicable, Longitudinal Channelizer
•		
•	Vehicle Stopping Distance	
•	Vehicle Damage	
	VDS [46]	
	CDC [47]	
•	Test Article Damage	
•	Maximum Test Article Deflections	
	Permanent Set	Not Applicable, Longitudinal Channelizer
	Dynamic	Not Applicable, Longitudinal Channelizer
		Not Applicable, Longitudinal Channelizer
	-	

Transducer Data ٠

Evaluation Criteria		Transducer		MASH	
		SLICE-2	DTS (primary)	Limit	
OIV ft/s	Longitudinal	-21.95 (-6.69)	-21.69 (-6.61)	$\leq 40$ (12.2)	
m/s (m/s)	Lateral	-0.23 (-0.07)	-1.19 (-0.36)	$\leq 40$ (12.2)	
ORA	Longitudinal	-20.91	-19.41	$\leq 20.49$	
g's	Lateral	-6.74	-3.87	$\leq 20.49$	
MAX ANGULAR DISP.	Roll	-2.07	8.63	≤75	
	Pitch	3.93	17.68	≤75	
deg.	Yaw	-1.88	9.57	not required	
THIV –	THIV – ft/s (m/s)		21.79 (6.64)	not required	
PHD	<b>)</b> – g's	20.91	19.5	not required	
ASI		0.9	0.87	not required	

Figure 190. Summary of Test Results and Sequential Photographs, Test No. APR-2



0.000 sec



0.088 sec



0.426 sec

0.684 sec



0.972 sec



1.172 sec Figure 191. Additional Sequential Photographs, Test No. APR-2



0.000 sec



0.096 sec



0.324 sec



0.724 sec



1.424 sec



1.824 sec



0.000 sec



0.070 sec



0.326 sec



0.892 sec



1.408 sec



1.874 sec



0.000 sec



0.078 sec



0.278 sec



0.626 sec



1.126 sec



1.626 sec

Figure 192. Additional Sequential Photographs, Test No. APR-2

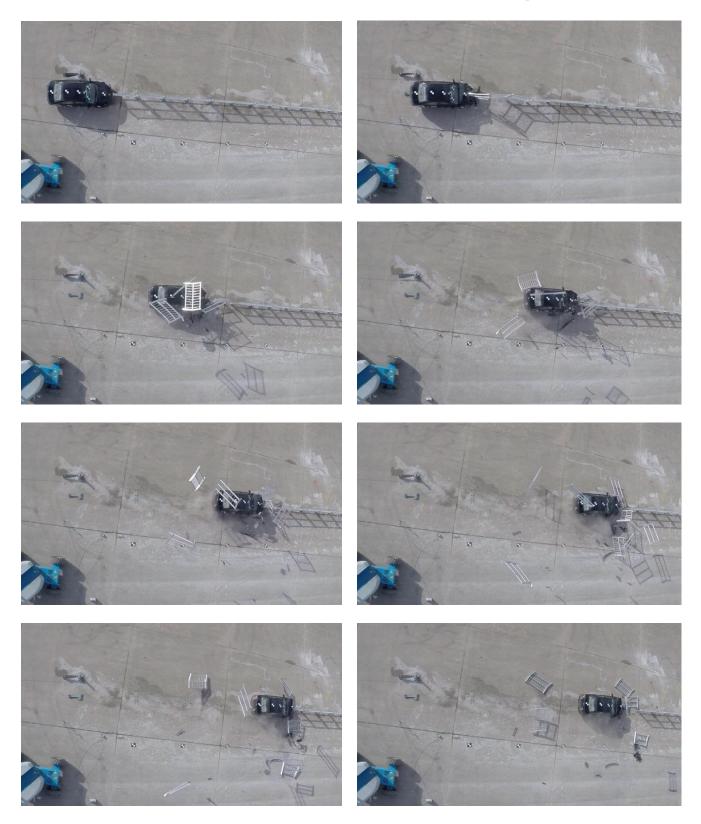


Figure 193. Documentary Photographs, Test No. APR-2







Figure 194. Impact Location, Test No. APR-2



Figure 195. Vehicle Final Position and Trajectory Marks, Test No. APR-2







Figure 196. System Damage, Test No. APR-2



Figure 197. Panel No. 1 Damage, Test No. APR-2

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Figure 198. Panel No. 2 Damage, Test No. APR-2



Figure 199. Panel No. 3 Damage, Test No. APR-2



Figure 200. Panel No. 4 Damage, Test No. APR-2



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Figure 201. Panel No. 5 Damage, Test No. APR-2



Figure 202. Panel No. 6 Damage, Test No. APR-2







Figure 203. Panel No. 7 Damage, Test No. APR-2



Figure 204. Panel No. 8 Damage, Test No. APR-2



Figure 205. Panel No. 9 Damage, Test No. APR-2



Figure 206. Panel No. 10 Damage, Test No. APR-2



Figure 207. Vehicle Damage, Test No. APR-2











Figure 208. Vehicle Damage, Test No. APR-2



Figure 209. Occupant Compartment Damage, Test No. APR-2



Figure 210. Vehicle Undercarriage Damage, Test No. APR-2

# **15 SUMMARY, CONCLUSIONS, AND RECOMMENDATIONS**

The objective of this study was to design a crashworthy pedestrian rail that will protect pedestrians from hazards while not posing undue safety risk to motorists and pedestrians. The new pedestrian rail was to meet the design standards of the ADA, the AASHTO *LRFD Bridge Design Specifications*, and the AASHTO MASH TL-2 safety performance evaluation criteria for longitudinal channelizers.

After a literature review was completed on existing pedestrian rail systems and commercially-available railings, twenty-five pedestrian rail concepts were considered and included various materials, such as steel, PVC, wood, HDPE, and FRP. During this design phase, the geometry and structural capacity of the pedestrian rail concepts were investigated to meet the AASHTO LRFD *Bridge Design Specifications*.

After sponsor review of the original concepts, several prototypes and material options were eliminated. Concepts were refined based on aesthetics, strength, weight, cost, and workability. Eight concepts, including two modular aluminum, two welded aluminum, two PVC, and two wood, were further developed using simplified load cases. After sponsor review, the aluminum concepts were further refined into four concepts and designed. Each component of the systems, including rails, posts, and infill members, post-to-rail, post-to-base, and infill-to-rail connections, and anchorages, were configured.

Seven dynamic bogie tests were conducted on four aluminum pedestrian rail concepts. Each system was configured as a two-panel system, impacted at approximately 45 mph (72.4 km/h) and evaluated in two different impact orientations, except for the fourth concept (test no. WIPR-4). Each system was impacted at a 25-degree angle and within the spindle region of the first panel. Next, each system was impacted using an end-on orientation. For test no. WIPR-4, only the end-on orientation was evaluated. Each system broke away as designed. The concept of a post welded to a baseplate seemed to fracture more cleanly at the base than the concept involving a post inserted into a socket that was welded to the baseplate. Minor permanent deformation was found on all the baseplates. An increased anchor diameter of ½ in. (12.7 mm) eliminated permanent anchor deformations. Shifting the middle rail downward, closer toward the bumper heights of the pickup truck and small car, minimized component damage and allowed the system to behave more like a rigid frame. The factors that helped to improve system behavior included: (1) lowering the middle horizontal rail; (2) extending the spindles from the top to bottom rails and passing the spindles through the middle rail; (3) increasing anchor diameter to ½ in. (12.7 mm); and (4) inserting the rails into cutouts in the posts. Thus, design concept AW2-D (test no. WIPR-4) was recommended to be evaluated through full-scale vehicle crash testing according to the MASH TL-2 safety performance criteria for longitudinal channelizers.

Two full-scale vehicle crash tests were conducted on the pedestrian rail according to the TL-2 safety performance criteria found in MASH for longitudinal channelizers. Test no. APR-1, test designation no. 2-90, consisted of an 1100C small car impacting the pedestrian rail at a speed of 45.2 mph (72.7 km/h) and an angle of 25.1 degrees within the system. The pedestrian rail allowed controlled penetration of the 1100C vehicle through the longitudinal channelizer. Neither detached elements nor fragments showed potential for penetrating the occupant compartment or for presenting undue hazard to other traffic. Note, none of the pedestrian rail panels went over the hood, near the windshield, or underneath the vehicle. Deformations of, or intrusions into, the occupant compartment that could have caused serious injury did not occur. The OIVs and ORAs were within the suggested limits provided in MASH. The test vehicle remained upright during and after the collision. After impact, the vehicle penetrated behind the

channelizer. Therefore, test no. APR-1 was determined to be acceptable according to the MASH safety performance criteria for longitudinal channelizers, test designation no. 2-90.

Test no. APR-2, test designation no. 2-90, consisted of an 1100C small car impacting the pedestrian rail on its upstream end at a speed of 44.5 mph (71.6 km/h) and an angle of 0 degrees. The test vehicle remained upright during and after the collision and came to rest within the longitudinal line of the channelizer. The test results revealed that the pedestrian rail system imparted a high longitudinal ORA to the hypothetical vehicle occupants when struck end-on. The longitudinal ORA from the DTS (the primary accelerometer unit) was 19.41 g's, while the longitudinal ORA from the SLICE-2 (the backup accelerometer unit) was 20.91 g's. Thus, the primary accelerometer produced an acceptable longitudinal ORA according to MASH, while the backup accelerometer was unacceptable.

During this testing program, there was no existing guidance or policy within the crash test laboratories regarding comparison of accelerations from different transducer units used within the same test vehicle or which value to choose if the values varied. Consequently, feedback was sought from FHWA in February 2015. Following the discussion with FHWA and quoting them directly,

"We've not seen this situation before (i.e., 2 conflicting accel readings). After review of video, we certainly agree with your detailed assessment of test in regards to acceleration spike. As there is no existing policy for comparing accelerations from different transducer units on same test, we feel it best to recognize the implication of a higher value."

The documentation of the correspondence with FHWA is shown in Appendix J. Therefore, test no. APR-2 was initially determined to be unacceptable according to the MASH safety performance criteria for longitudinal channelizers, test designation no. 2-90.

Cases where varying occupant risk results were acquired from different accelerometer systems was addressed with other crash test laboratories during the AASHTO Task Force 13 Subcommittee #7 meeting on April 30, 2015 in Lincoln, Nebraska. During the discussion, the crash test laboratories and FHWA came to a consensus that the results from the primary accelerometer unit would be reported. The primary accelerometer unit is defined as the unit placed closest to the c.g. More detailed information on this discussion can be found in the minutes from the April 2015 AASHTO Task Force 13 Subcommittee #7 meeting. Under this guidance, the accelerometer unit that was considered the primary unit provided a longitudinal ORA below the MASH limit. As such, test no. APR-2 was subsequently determined to be acceptable according to the MASH safety performance criteria for longitudinal channelizers. A summary of the safety performance evaluation for each test is provided in Table 23.

The results from both test designation no. 2-90 crash tests were analyzed to determine if any test no. 2-91 crash tests would be required for the pedestrian rail system. In test no. 2-90 with an impact angle of 25 degrees (test no. APR-1), the small car did not have any instability or accelerations that were of concern. Thus, there were no concerns for excessive accelerations with the larger, more massive pickup truck conducted under test designation no. 2-91 at a 25-degree impact angle. Also, the pedestrian rails panels did not get above the hood, near the windshield, or traverse under the car, so occupant compartment deformation or penetration was not a concern given the geometry of the pickup truck.

In test no. 2-90 with an impact angle of 0 degrees (test no. APR-2), the small car did not have any instability concerns, although the accelerations were higher than desired. However, there were no concerns for excessive accelerations with the larger, more massive pickup truck conducted under test designation no. 2-91 at a 0-degree impact angle. In test no. APR-2, some panels shifted downstream, compressed against one another end-on, and buckled upward as a series. As a result, some panels were propelled upward, passing over the top of the small car, while other panels contacted the front of the engine hood. The front-end geometry of the pickup truck is taller than that provided by the small car. Both the engine hood and bumper are higher relative to the ground. Thus, it is not believed that the pedestrian rail panels would become airborne as easily during an end-on, pickup truck crash event as compared to small car end-on impacts. With a higher load height, it is expected that the panels may displace more sideways rather than vertically. Even if some vertical panel displacement occurred, the panels would not likely be propelled at the windshield. Thus, there is no concern for occupant compartment penetration or deformation in tests with the pickup truck. Therefore, it is believed that test designation no. 2-91 is not necessary for this channelizer system.

The current as-tested system did not include ADA-compliant handrails, which may be required for some roadside applications. As such, further design and crash testing may be required to investigate the use of ADA-compliant handrails. In addition, the current system was tested on level terrain. Consequently, no information was ascertained as to the safety performance of this pedestrian rail placed on top of and behind roadside curbs as well as on sloped terrain. If this pedestrian rail is desired for use near curbs or sloped terrain, then it is recommended that further investigation and crash testing be performed.

Initially, it was believed that the pedestrian rail system could be configured with segmented panels with gaps or as a continuous system. The researchers configured and tested a segmented panel system as it was believed to be easier to install. However, the researchers do not currently recommend that this system be installed continuously due to concerns for loading multiple posts simultaneously and the potential for higher longitudinal ORAs when impacted end-on.

Although the pedestrian rail system met the requirements in MASH, it is recommended that the system be modified to improve its safety performance and lower the occupant risk measures. It may be useful to reduce the amount of flying debris that could cause additional risk to pedestrians. However, it should be noted that channelizers do not provide positive protection between pedestrians and errant vehicles. Thus, the errant vehicle itself also poses a risk to pedestrians. In addition, it may be beneficial to consider other design modifications that reduce the tendency for panel segment to contact one another in the form of a longer compressed column. Such modifications may include staggered placement or post sections that allow for improved shedding of the upstream panel section under end-on impact events. Finally, future considerations should be directed toward inclusion of an ADA-compliant handrail that does not pose undue safety risk to motorists and pedestrians. Further crash testing may be required to accommodate these modifications.

Table 23 Summary	of Safety	Performance	<b>Evaluation Results</b>
1 abic 25. Summary	of Safety	1 chlorinance	L'valuation Results

Evaluation Factors		E	Evaluation Criteria			Test No. APR-2	
Structural Adequacy	C.	Acceptable test article performance controlled stopping of the vehicle.	ptable test article performance may be by redirection, controlled penetration, or olled stopping of the vehicle.			S	
	D.	Detached elements, fragments or other debris from the test article should not penetrate or show potential for penetrating the occupant compartment, or present an undue hazard to other traffic, pedestrians, or personnel in a work zone. Deformations of, or intrusions into, the occupant compartment should not exceed limits set forth in Section 5.3 and Appendix E of MASH.			S	S	
	F.	. The vehicle should remain upright during and after collision. The maximum roll and pitch angles are not to exceed 75 degrees.			S	S	
Occupant Risk	H.	procedure) should satisfy the following limits:			S	-	
Ittok		Occupant Impact Velocity Limits				S	
			Component	Preferred	Maximum		
		Longitudinal and Lateral	30 ft/s (9.1 m/s)	40 ft/s (12.2 m/s)			
	I.	The Occupant Ridedown Acceleration (ORA) (see Appendix A, Section A5.3 of MASH for calculation procedure) should satisfy the following limits:					
		calculation procedure) should satisf	y the following mints.				
		1 /	pant Ridedown Acceleration Lin	nits	S	S*	
		1 /		nits Maximum	S	S*	
		Occup	ant Ridedown Acceleration Lin		S	S*	
Vehicle Trajectory	N.	Occup Component	Pant Ridedown Acceleration Lin Preferred 15.0 g's	Maximum	S S	S*	
	N.	Occup           Component           Longitudinal and Lateral           Vehicle trajectory behind the test art	Pant Ridedown Acceleration Lin Preferred 15.0 g's	Maximum			

S – Satisfactory U – Unsatisfactory

NA - Not Applicable

\*The primary accelerometer unit provided a longitudinal ORA below the MASH limit, and the backup accelerometer unit provided a longitudinal ORA above the MASH limit.

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# **17 APPENDICES**

# Appendix A. Pooled Fund Survey for Pedestrian Rail Highest Priority Need

3/28/12 Wisconsin Department of Transportation Hill Farms State Transportation Building 4802 Sheboygan Avenue P.O. Box 7910 Madison, WI 53707-7910

Dear Erik Emerson,

The Midwest Roadside Safety Facility (MwRSF) is working on the initial phase of a pedestrian rail project that will ultimately lead to the development of a crashworthy ADA-compliant pedestrian-only rail. The project has been sponsored by the Wisconsin Department of Transportation, who has requested the input from the member states of the Pooled Fund.

Situations arise where a barrier or rail is required to prevent pedestrians from crossing or dropping into an area which may be acceptable for an errant vehicle. These types of pedestrian rails are located (1) on top of short culverts and retaining walls (less than 8 ft tall); (2) next to high pedestrian and vehicle facilities to prevent jaywalking; and (3) to keep pedestrians off public/private property such as train tracks. Examples are shown in Figures 1 through 3. It is important to keep pedestrians from entering these locations, but it is not feasible to install a crashworthy barrier as shown in Figure 4. These rails would not need to redirect or stop an errant vehicle, and they must not present additional hazards to the motoring public. In addition, pedestrian rails must comply with the Americans with Disabilities Act (ADA). Currently, no permanent pedestrian-only rails have been crash tested and accepted for use on roadsides.



Figure 1. Rail over Culvert



Figure 3. Rail around Private Property



Figure 2. Rail Restricting Jaywalkers



Figure 4. Concrete Traffic/Pedestrian Rail

Since only a small number of rails will actually be developed and tested, identifying the most prominent need is required. To complete the initial phase of the project, the researchers at MwRSF need to establish which pedestrian rails provide the greatest value to the state of Wisconsin and the member states of the Pooled Fund. Thus, MwRSF researchers are seeking input on what rails are the most common, what they are being used for, and what additional/updated standard plans or drawing details have been established for pedestrian-only permanent rails.

The compiled pedestrian rail needs will be organized into a limited number of design categories that will result in the smallest number of full-scale crash tests as possible. It is anticipated these rails will be tested under TL-2 criteria since they are not used on high-speed facilities. Priorities for the project will be assigned based on (1) the importance of the pedestrian rail to the states participating; (2) the number of different system configurations that can be addressed simultaneously; and (3) the potential for the development of a successful design under MASH.

A review of the WisDOT website was done to find current pedestrian-only rail standards. The details of pedestrian-type railings obtained from the website were limited to (1) wire and (2) chain link fences. The acquired drawings found are attached. Drawing standards for a true pedestrian-only barrier were not found. Please respond with the most resent/additional pedestrian rail standards which are used in the state of Wisconsin.

#### Please reply no later than 4/13/2012 with the following information:

- A completed copy of the attached survey
- Any additional/updated details, state standards, or plans concerning any type of pedestrian-only rail installed near a roadway
- Any photographs of these pedestrian rails currently in use

Questions should be directed to Mitch Wiebelhaus, MwRSF graduate research assistant, at <u>mitchw1@huskers.unl.edu</u> or (402) 472-9043 or Karla Lechtenberg at <u>kpolivka2@unl.edu</u> or (402) 472-9070. The completed form and all additional documents may be e-mailed, faxed, or mailed to:

Mitch Wiebelhaus <u>mitchw1@huskers.unl.edu</u> 130 Whittier Building 2200 Vine Street Lincoln, Nebraska 68583-0853 Fax: (402) 472-2022

Once again your response is requested no later than 4/13/2012. Thank you for your efforts and time.

Sincerely, Mitch Wiebelhaus, B.S.C.E., E.I. Graduate Research Assistant Usage Summary for Permanent Pedestrian-Only Rail

- (1) Identify how useful the development of the listed pedestrian rails would be to your state by putting an X in the box.
- (2) Include the approximate percentage of pedestrian rails which are comprised of the rails implemented in your state.
- (3) Rank the pedestrian rail location/circumstance in order of their benefit to your state with 1 being the most beneficial.
- (4) List additional location/situation your state has for pedestrian rails and indicate its usefulness, percentage of rails, and benefit to your state.
- (5) Include pictures, details, and drawings of current pedestrian rails installed near roadways in your state.

Pedestrian Rail	Usefulness (1)						Rank
Locations/Situation	Not Useful		Somewhat Useful		Very Useful	(2)	(3)
On top of culverts							
On top of retaining walls							
To prevent jaywalking							
Around private/public property							
Additional Locations/Situations (4)							

Please direct questions and return the completed form with the additional materials to:

Mitch Wiebelhaus <u>mitchw1@huskers.unl.edu</u> 130 Whittier Building 2200 Vine Street Lincoln, Nebraska 68583-0853 (402) 472-9043 Fax: (402) 472-2022

# Appendix B. Original Design Concepts

Material properties for Concepts 1 through 19 and Designs 1 through 6 are shown in Table B-1 and Table B-2, respectively. These values may vary from nominal to account for temperature or degradation variations.

Material	σ <sub>y</sub> (psi)	E (ksi)	ρ (lb/ft <sup>3</sup> )
Steel	50,000	29,000	503
PVC	7,000	300	90
Wood	12,000	1,600	28

Table B-1. Material Properties, Concepts 1 through 19

Table B-2. Material Properties, Designs 1 through 6

Material	$\sigma_{y}$ (psi)	E (ksi)
PVC	4,500	300
Wood	12,000	1,600
HDPE	2,175	1,600
FRP	24,000	2,320

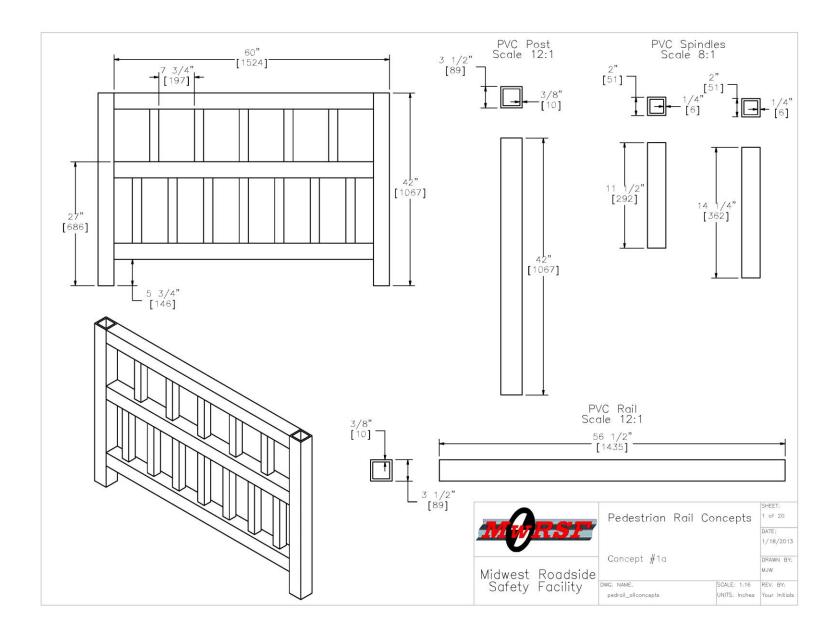


Figure B-1. Concept 1: PVC Posts, Rails, and Spindles

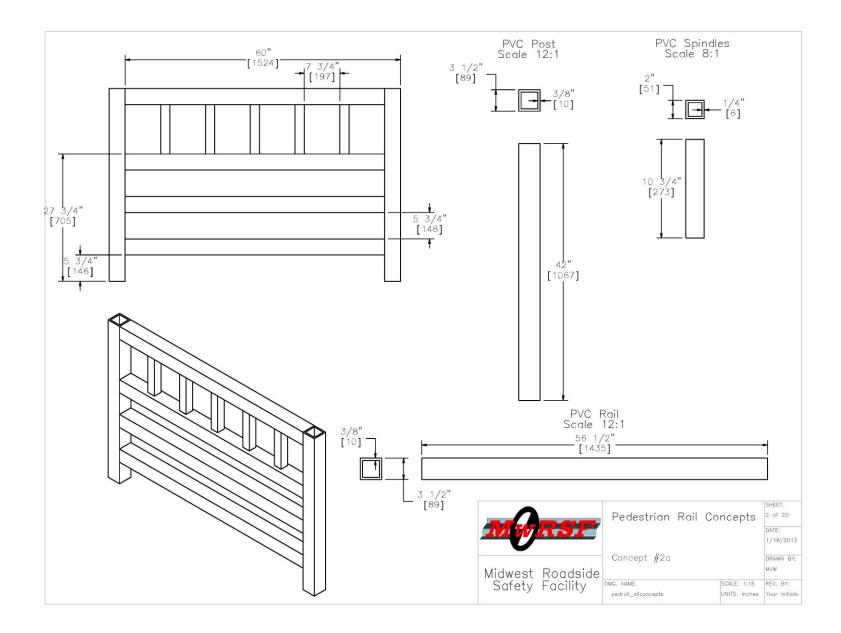


Figure B-2. Concept 2: PVC Posts, Rails, and Spindles

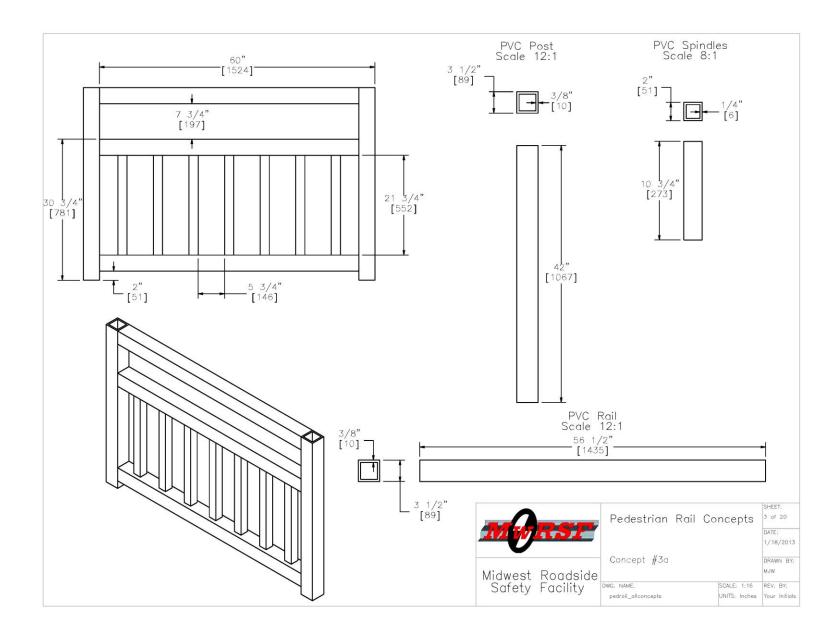


Figure B-3. Concept 3: PVC Posts, Rails, and Spindles

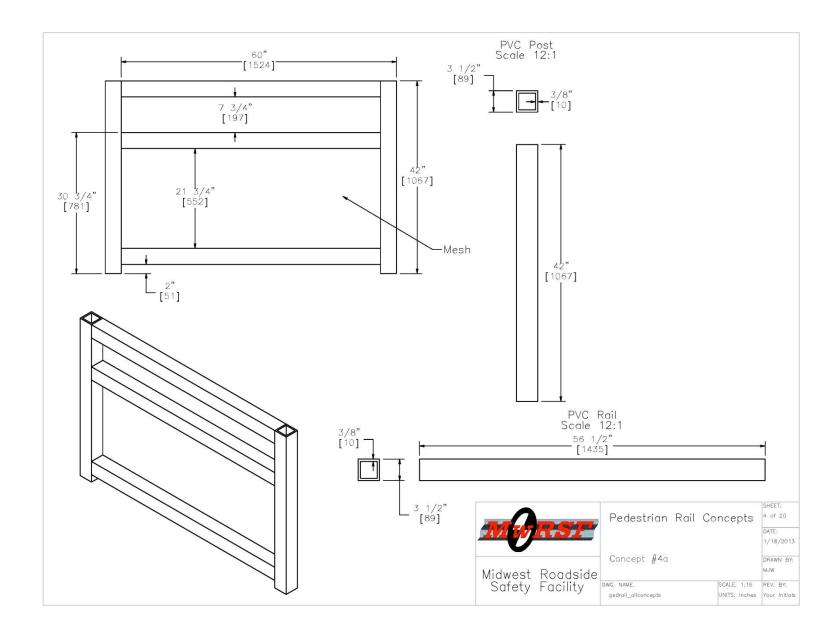


Figure B-4. Concept 4: PVC Posts and Rails with Mesh

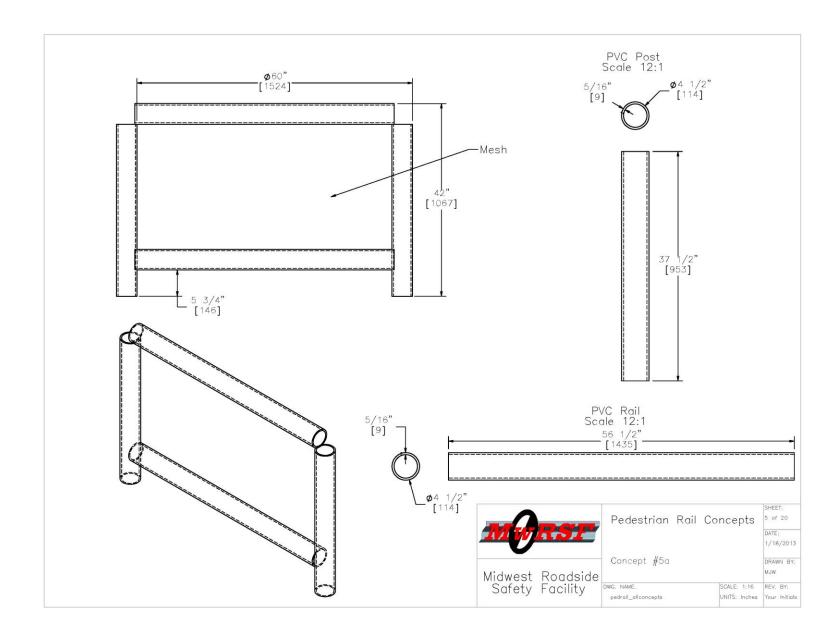


Figure B-5. Concept 5: PVC Posts and Rails with Mesh

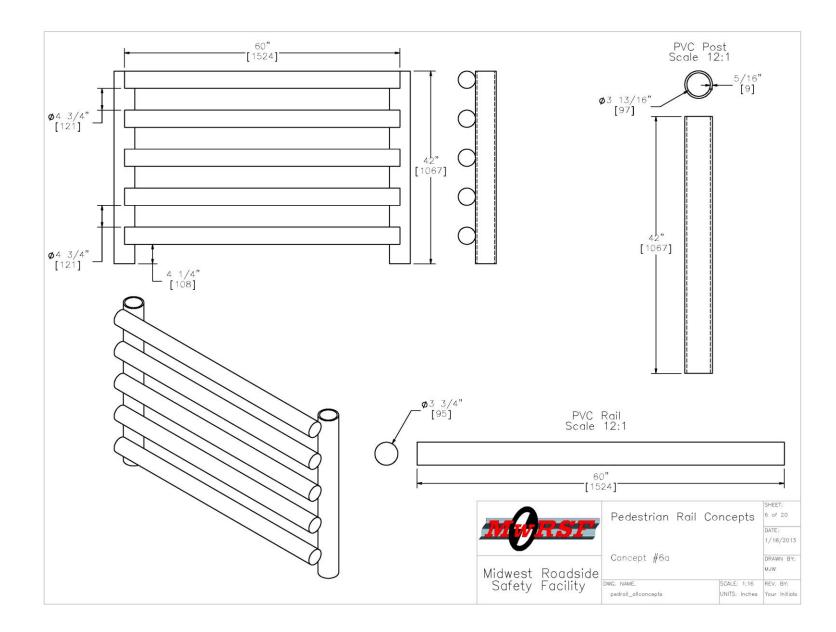


Figure B-6. Concept 6: PVC Posts and Rails

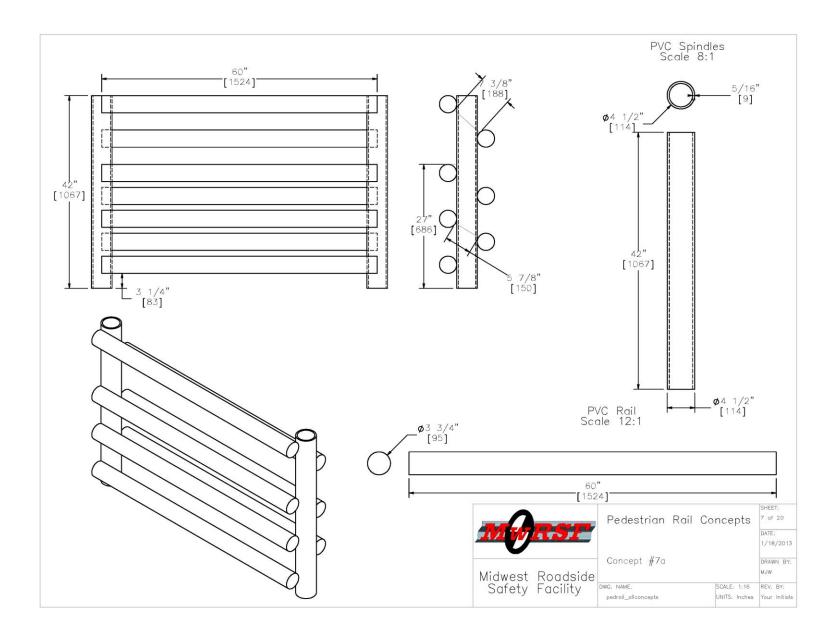


Figure B-7. Concept 7: PVC Posts and Rails

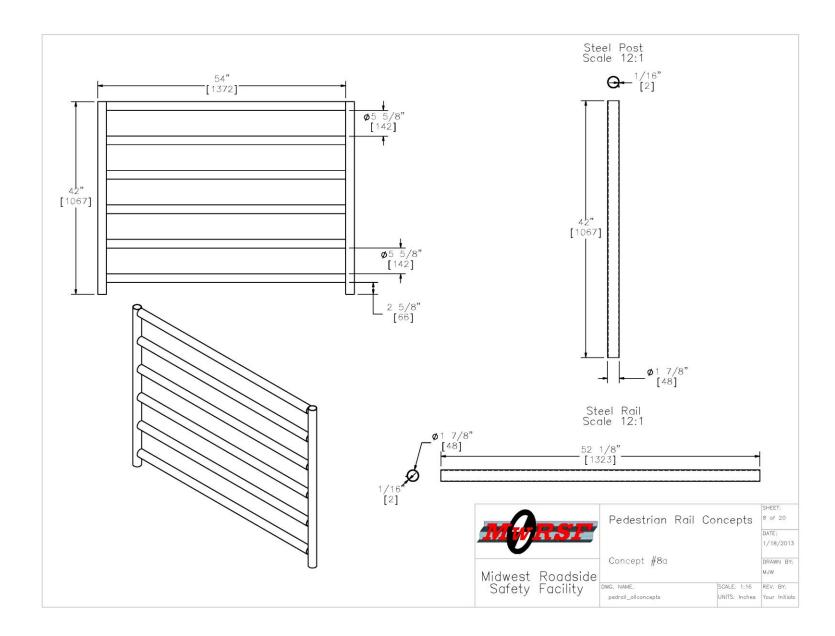


Figure B-8. Concept 8: Steel Posts and Rails

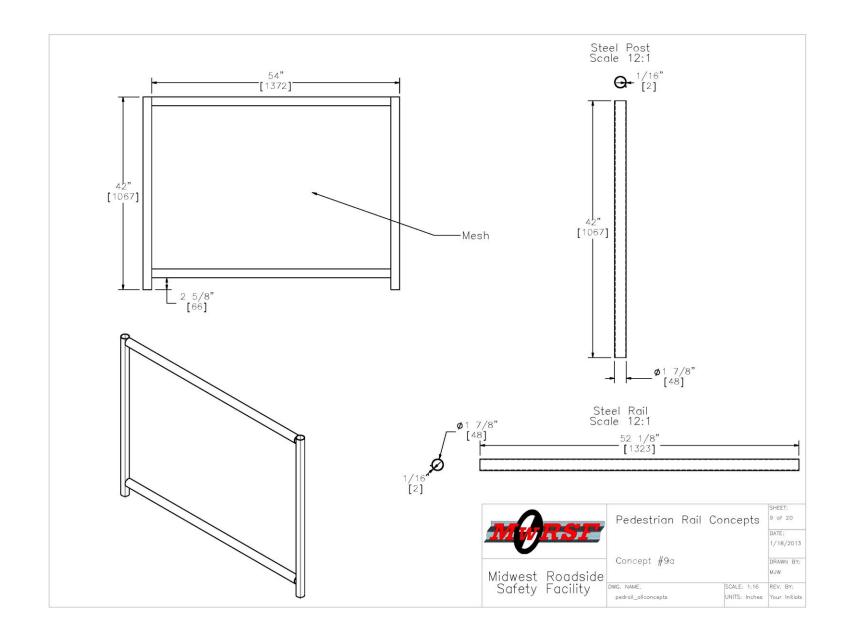


Figure B-9. Concept 9: Steel Posts and Rails with Mesh

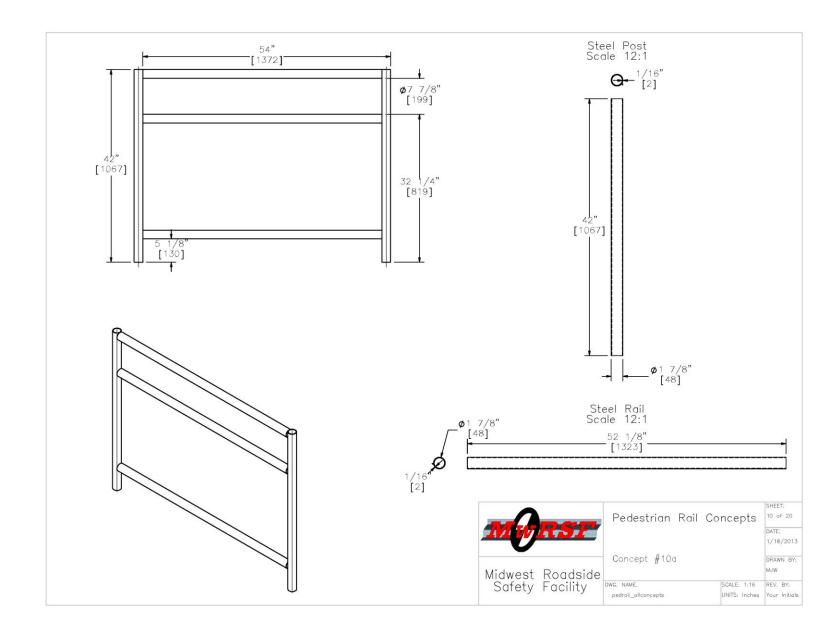


Figure B-10. Concept 10: Steel Posts and Rails with Mesh

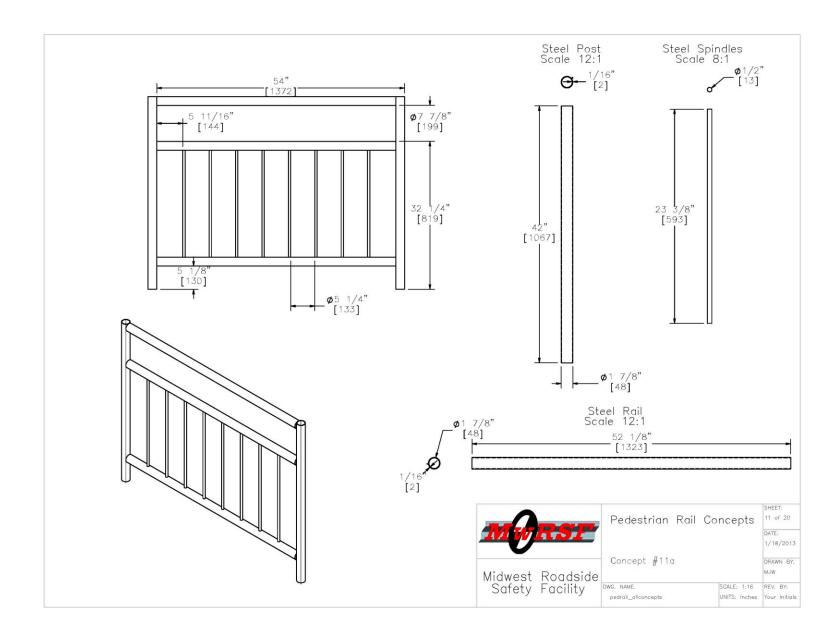


Figure B-11. Concept 11: Steel Posts, Rails, and Spindles

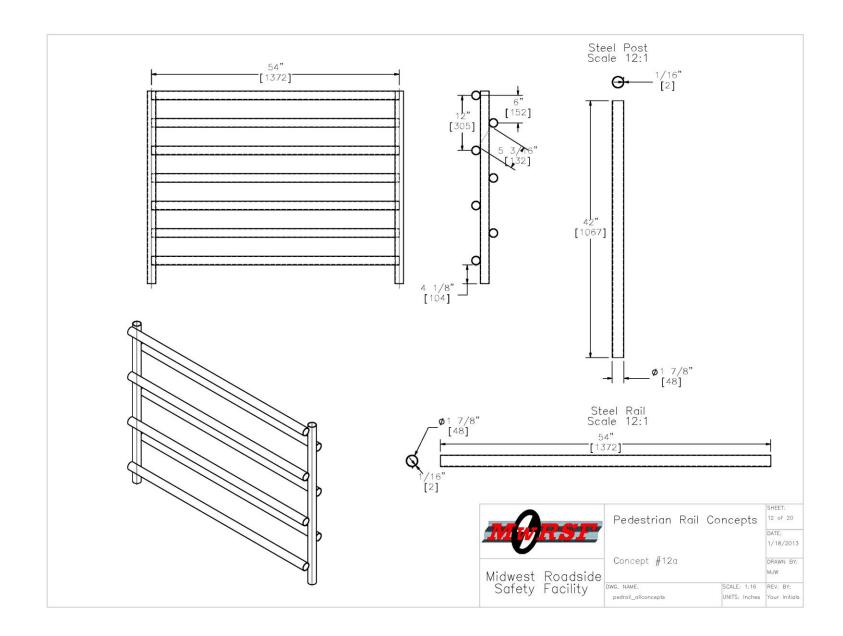


Figure B-12. Concept 12: Steel Posts and Rails

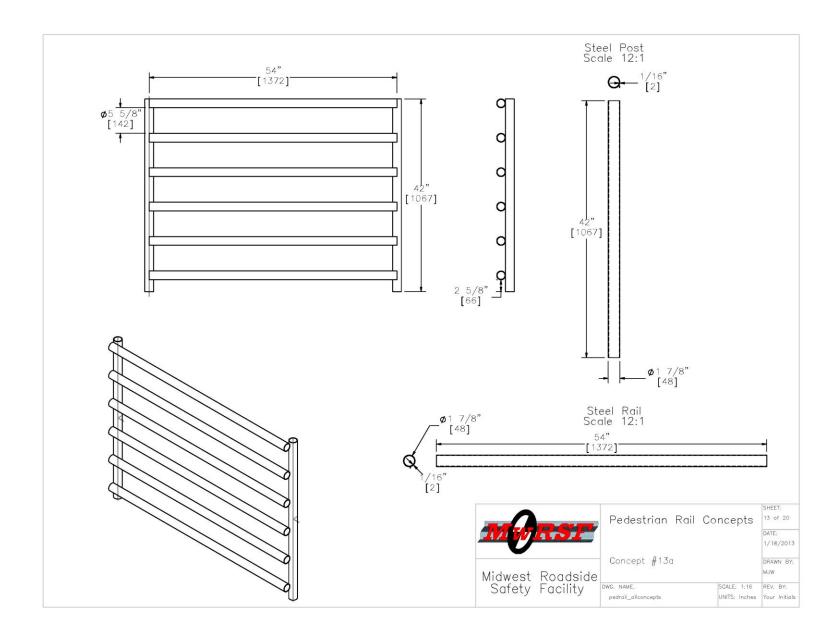


Figure B-13. Concept 13: Steel Posts and Rails

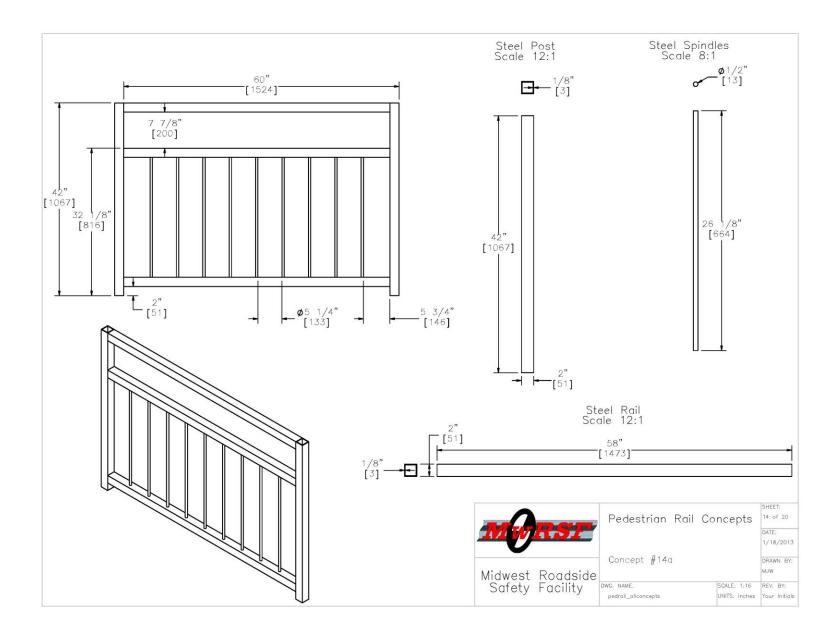


Figure B-14. Concept 14: Steel Posts, Rails, and Spindles

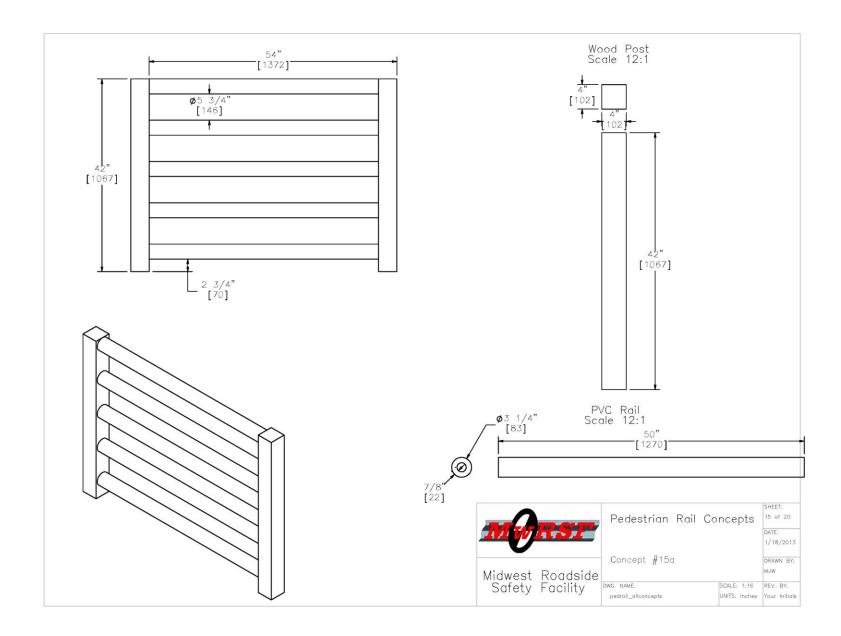


Figure B-15. Concept 15: Wood Posts and PVC Rails

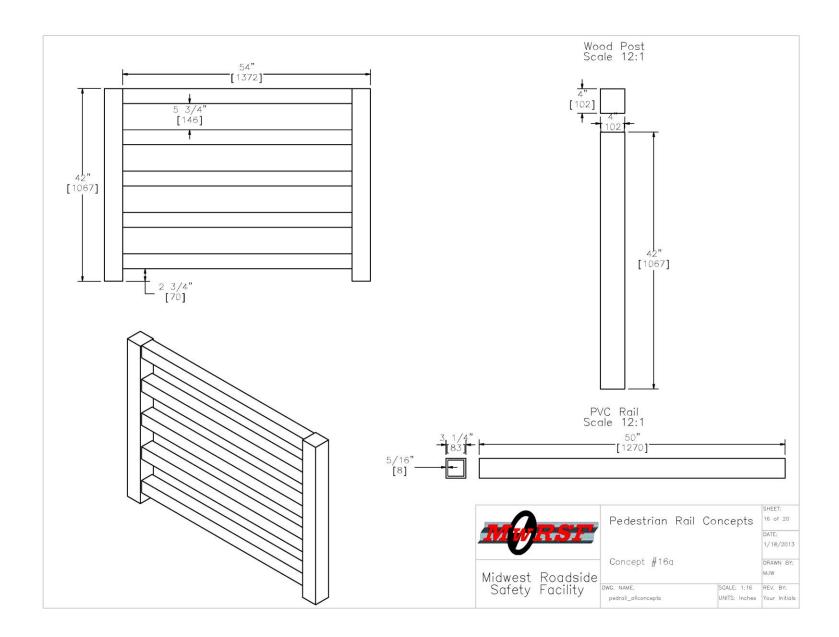
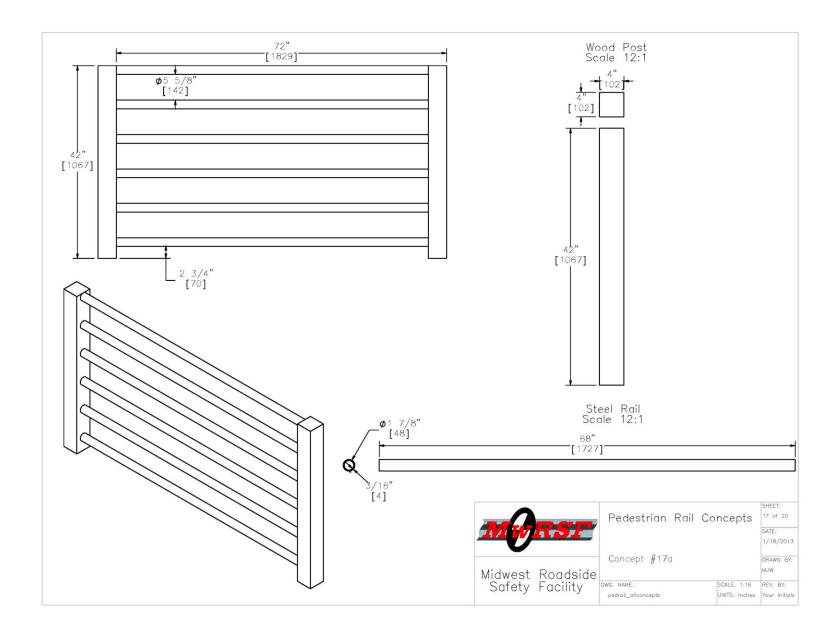


Figure B-16. Concept 16: Wood Posts and PVC Rails



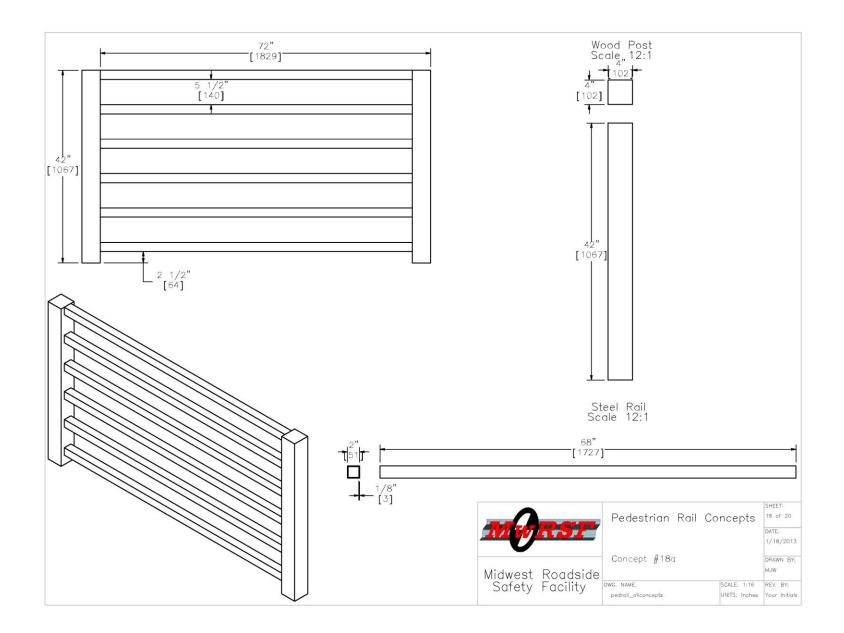


Figure B-18. Concept 18: Wood Posts and Steel Rails

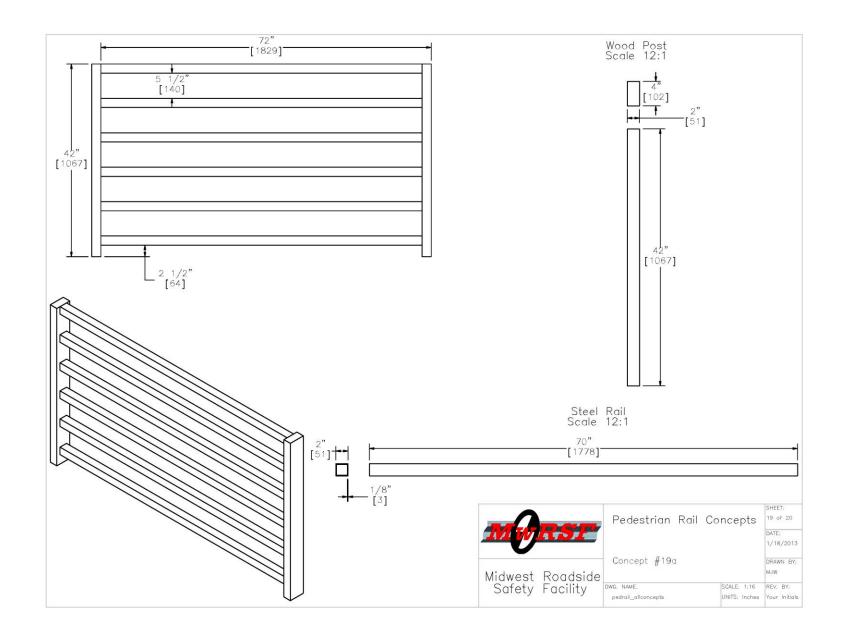


Figure B-19. Concept 19: Wood Posts and Steel Rails

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# Figure B-20. Design 1: PVC Posts, Rails, and Spindles

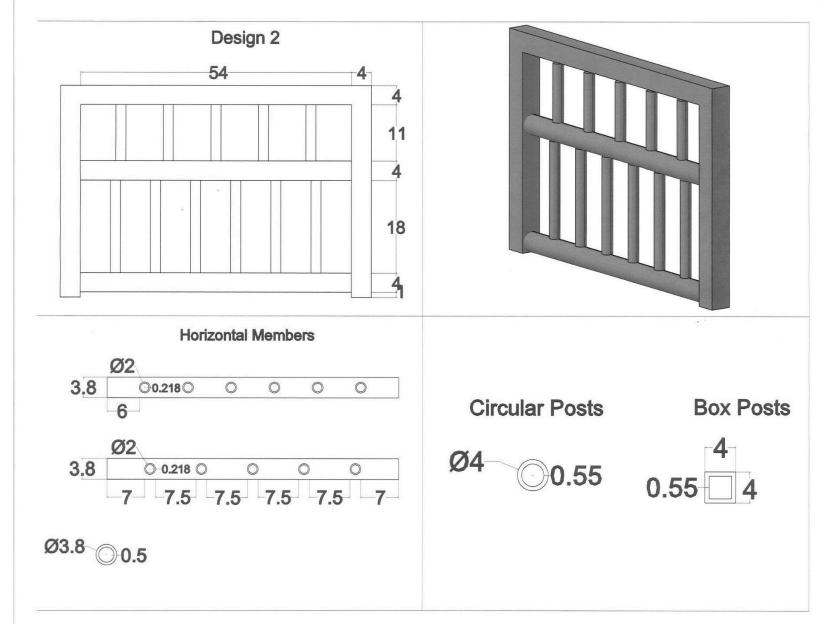


Figure B-21. Design 2: PVC Posts, Rails, and Spindles

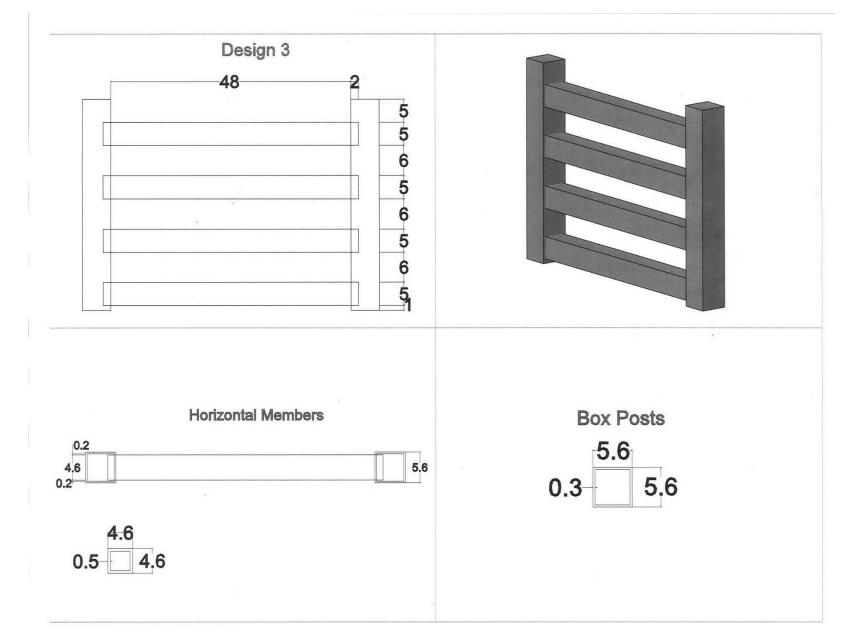


Figure B-22. Design 3: HDPE Posts and Rails

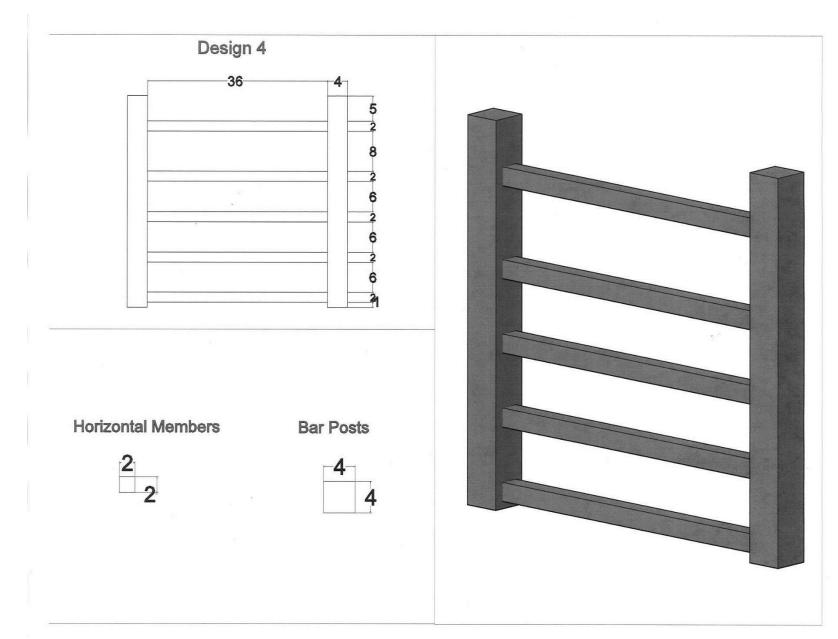


Figure B-23. Design 4: Wood Posts and Rails

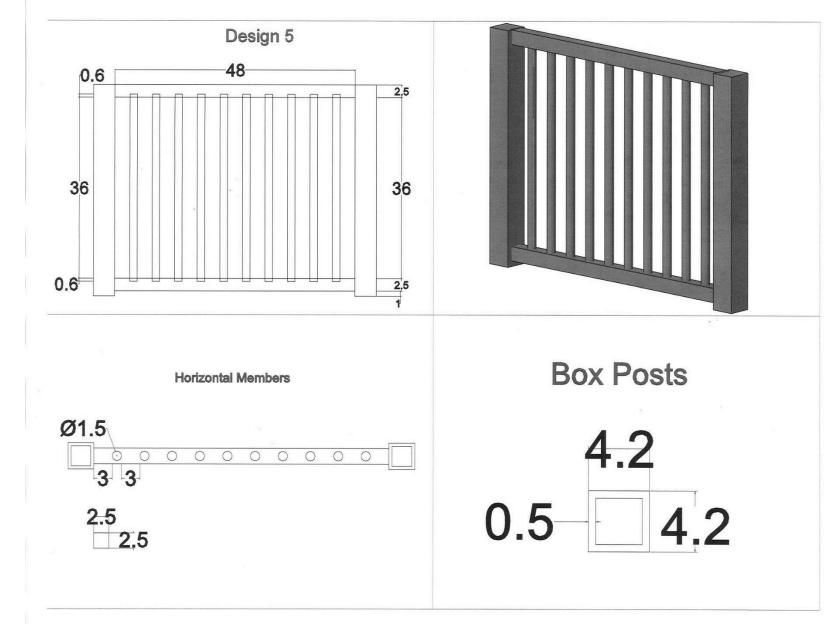


Figure B-24. Design 5: HDPE Posts, Wood Rails, FRP Spindles

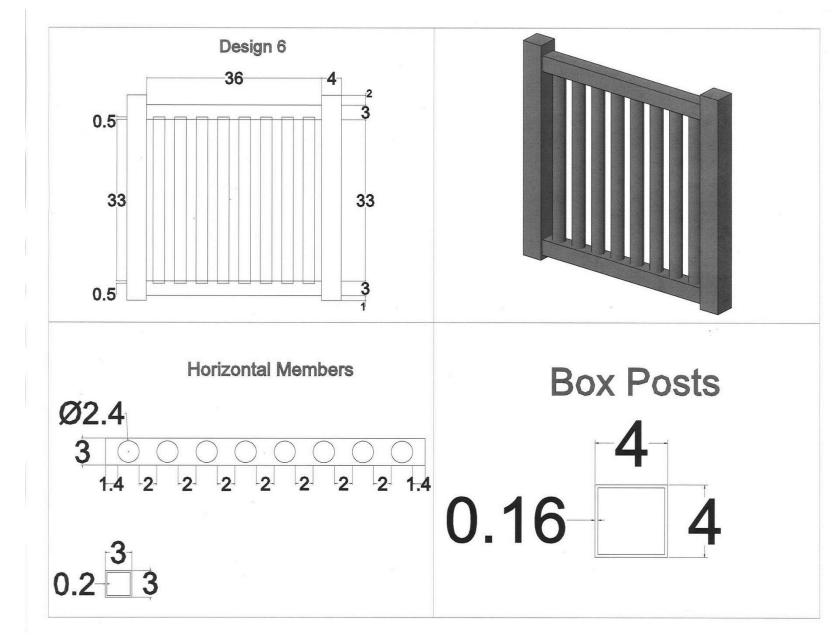


Figure B-25. Design 6: FRP Posts, HDPE Rails, PVC Spindles

January 18, 2016 MwRSF Report No. TRP-03-321-15

### Appendix C. Design Calculations

The material strengths for 6061-T6 aluminum, 5356 aluminum weld filler, and 535 aluminum alloy castings that were used in the example calculations are shown in Tables C-1, C-2, and C-3, respectively.

Table C-1.	Material Strengths for 6061-T6 Aluminum from Tables A.3.4, A.3.5, and A3.1 in
ADM [38]	

Non-Welded Strength Extrusions, All Thicknesses ksi (MPa)		StrengthStrengthExtrusions,Sheet & Plate,Thicknesses $0.010 \le t \le 4.000$ in.		Weld-Affected Strength All Shapes, t ≤ 0.375 in. ksi (MPa)		Weld-Affected Strength All Shapes, t > 0.375 in. ksi (MPa)	
$\mathbf{F}_{\mathrm{tu}}$	38 (260)	$\mathbf{F}_{\mathrm{tu}}$	42 (290)	F <sub>tuw</sub>	24 (165)	F <sub>tuw</sub>	24 (165)
$F_{ty}$	35 (240)	F <sub>ty</sub>	35 (240)	F <sub>tyw</sub>	15 (105)	F <sub>tyw</sub>	11 (80)
F <sub>cy</sub>	35 (240)	$F_{cy}$	35(240)	F <sub>cyw</sub>	15 (105)	F <sub>cyw</sub>	11 (80)
F <sub>su</sub>	24 (165)	$F_{su}$	27 (185)	F <sub>suw</sub>	15 (105)	F <sub>suw</sub>	15 (105)
F <sub>sy</sub>	21(145)	$F_{sy}$	21 (145)	F <sub>syw</sub>	9 (62)	F <sub>syw</sub>	6.6 (46)
Where: $F_{tu}$ = Tensile Ultimate Strength							

 $F_{ty}$  = Tensile Yield Strength

 $F_{cv}$  = Compressive Yield Strength

 $F_{su}$  = Shear Ultimate Strength

F<sub>tuw</sub> = Tensile Ultimate Strength of Weld-Affected Zones

F<sub>tyw</sub> = Tensile Yield Strength of Weld-Affected Zones

F<sub>cvw</sub> = Compressive Yield Strength of Weld-Affected Zones

F<sub>suw</sub> = Shear Ultimate Strength of Weld-Affected Zones

 $F_{sv}$  = Shear Yield Strength

 $F_{syw}$  = Shear Yield Strength of Weld-Affected Zones

Table C-2. Material Strengths for 5356 Aluminum Weld Filler from Table J.2.1 in ADM [38]

Weld Strength ksi (MPa)				
F <sub>tuw</sub>	35 (240)			
F <sub>suw</sub>	17 (115)			

Table C-3. Material Strengths for 535 Aluminum Alloy Castings from Table A.3.6 in ADM [38]

Weld Strength ksi (MPa)				
F <sub>tu</sub>	26.2 (180)			
F <sub>ty</sub>	13.5 (93)			

The capacity of a weld was determined for four different connections: Concepts AW2-A and AW2-D post-to-base; Concept AW2-C sleeve-to-base; Concepts AW2-A, AW2-C, and AW2-D rail-to-post; and Concepts AW2-A, AW2-C, and AW2-D spindle-to-post. An example of determining the moment of inertia of the weld group and all the calculations is shown in Figure C-1 for Concept AW2-A post-to-base weld. The weld calculations are shown in Figures C-2 through C-5 for all concepts.

Example calculations of the baseplate by Method nos. 1 and 2 are shown for Concept AW2-A in Figures C-6 and C-7. The calculations for all baseplate designs are shown in C-8 through C-15.

The calculations for the anchor capacity in both tension and shear for all concepts is shown in Figures C-16 through C-21.

The final design calculations are shown in Figures C-22 through C-25.

#### Weld Capacity

Inputs:  $S_{W} (Weld Size) = \frac{1}{4} \text{ in.}$   $F_{suw(filler)} = 17,000 \text{ psi}$   $F_{suw(base metal)} = 15,000 \text{ psi}$   $F_{tuw(base metal)} = 24,000 \text{ psi}$  b (Flange Width) = 2 in. h (Web Width) = 4 in.  $\varphi = 0.75$   $e (Effective Throat) = S_{W} * \cos 45^{\circ} = 0.25 * \cos 45^{\circ} = 0.1768 \text{ in.}$ From Equation 32,  $\varphi R_{n} = \varphi F_{sw} L_{we}$   $F_{sw} = \text{Shear Strength of Weld [psi], which is the Least of:}$   $F_{suw(filler)} * e = 17,000 \text{ psi} * e = 17,000 * 0.1768 = 3,005 \text{ psi}$   $F_{suw(base metal)} * S_{W} = 15,000 \text{ psi} * S_{W} = 15,000 * 0.25 = 3,750 \text{ psi}$ 

 $F_{tuw(base\ metal)} * S_w = 24,000\ psi * S_w = 24,000 * 0.25 = 6,000\ psi$ Therefore,  $F_{sw} = 3,005\ psi$ .

 $L_{we}$  (Weld Effective Length) = 2 \* 2 in. + 2 \* 4 in. = 12 in.

$$\varphi R_n = \varphi F_{sw} L_{we} = 0.75 * 3,005 \text{ psi} * 12 \text{ in.} = 27,047 \text{ lb}$$

From Equation 33,  $\varphi M_n = \frac{\varphi F_{suw(filler)I}}{c}$ 

$$I \text{ (Moment of Inertia, Weld Group)} = 2(I_{flange} + A_{flange}d_{flange}^{2}) + 2(I_{web} + A_{web}d_{web}^{2})$$

$$I_{flange} = \frac{be^{3}}{12} = \frac{2 * 0.1768^{3}}{12} = 0.00092 \text{ in.}^{4}$$

$$I_{web} = \frac{eh^{3}}{12} = \frac{0.1768 * 4^{3}}{12} = 0.9428 \text{ in.}^{4}$$

$$A_{flange} = be = 2 * 0.1768 = 0.3536 \text{ in.}^{2}$$

$$A_{web} = eh = 0.1768 * 4 = 0.7071 \text{ in.}^{3}$$

$$d_{flange} = \left(\frac{h}{2}\right) + \left(\frac{e}{2}\right)\cos 45^{\circ} = \left(\frac{4}{2}\right) + \left(\frac{0.1768}{2}\right)\cos 45^{\circ} = 2.0625 \text{ in.}$$

$$d_{web} = 0$$

$$I = 2(0.00092 + 0.3536 * 2.0625^{2}) + 2(0.9428 + 0.7071 * 0^{2}) = 4.895 \text{ in.}^{4}$$

 $c(\text{Distance to Neutral Axis}) = \left(\frac{h}{2}\right) + \left(\frac{e}{2}\right)\cos 45^\circ = \left(\frac{4}{2}\right) + \left(\frac{0.1768}{2}\right)\cos 45^\circ$ = 2.0625 in.

$$\varphi M_n = \frac{\varphi F_{suw(filler)}I}{c} = \frac{0.75 * 17,000 \text{ psi} * 4.895 \text{ in.}^4}{2.0625 \text{ in.}} = 30,363 \text{ in.} -\text{lb}$$

Figure C-1. Example Calculation of Weld, Concept AW2-A

Input Name	Input Value	Units
$S_W$ (weld size)	1/4	in.
F <sub>suw(filler)</sub>	17000	psi
F <sub>suw(base metal)</sub>	15000	psi
F <sub>tuw(base metal)</sub>	24000	psi
b flange width	2	in.
h web width	4	in.
Φ	0.75	

Connection: 2" x 4"	x 1/4"	Post-to-Base	Plate, 1/	4" Weld
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ate, 1/4" Weld	Outrout Value	I Inita
Output Name	Output Value	Units
e (effective throat)	0.176776695	in.
F <sub>sw</sub>	3005.20382	lb/in.
$L_{we}$ (effective weld length)	12	in.
$R_n$ (nominal shear capacity)	27046.83438	lb
$\mathbf{M}_{\mathbf{n}}$ (nominal moment capacity)	30262.63013	inlb
$\mathbf{M}_{\mathbf{n}}$ (nominal moment capacity)	2521.885845	ft-lb
C (distance to neutral axis (NA))	2.0625	in.
I (moment of inertia of weld group)	4.895425463	in. <sup>4</sup>
$I_{flange}$ (moment of inertia of flange)	0.000920712	in. <sup>4</sup>
$I_{web}$ (moment of inertia of web)	0.942809042	in. <sup>4</sup>
$A_{\text{flange}}$ (area of one flange)	0.353553391	in. <sup>2</sup>
Aweb (area of one web)	0.707106781	in. <sup>2</sup>
$d_{flange}$ (distance from NA section to NA flange)	2.0625	in.
$d_{web}$ (distance from NA section to NA web)	0	in.

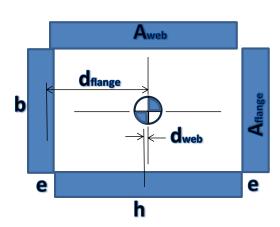


Figure C-2. Post-to-Base Weld, Concepts AW2-A and AW2-D

Input Name	Input Value	Units
$S_W$ (weld size)	3/16	in.
F <sub>suw(filler)</sub>	17000	psi
F <sub>suw(base metal)</sub>	15000	psi
F <sub>tuw(base metal)</sub>	24000	psi
b flange width	2 1/8	in.
h web width	3 1/8	in.
Φ	0.75	

Connection: 2 1/8" x	3 1/8" x 1/4'	' Sleeve-to-Base Pla	te, 3/16" Weld
----------------------	---------------	----------------------	----------------

Output Name	Output Value	Units
e (effective throat)	0.132582521	in.
F <sub>sw</sub>	2253.902865	lb/in.
$L_{we}$ (effective weld length)	10 1/2	in.
$R_n$ (nominal shear capacity)	17749.48506	lb
$\mathbf{M}_{\mathbf{n}}$ (nominal moment capacity)	16911.21663	inlb
$M_n$ (nominal moment capacity)	1409.268053	ft-lb
C (distance to neutral axis)	1.609375	in.
I (moment of inertia of weld group)	2.134626609	in. <sup>4</sup>
$I_{\text{flange}}$ (moment of inertia of flange)	0.000412702	in. <sup>4</sup>
$I_{web}$ (moment of inertia of web)	0.337174788	in. <sup>4</sup>
A <sub>flange</sub> (area of one flange)	0.281737858	
Aweb (area of one web)	0.41432038	in. <sup>2</sup>
$d_{\text{flange}}$ (distance from NA section to NA flange)	1.609375	in.
$d_{web}$ (distance from NA section to NA web)	0	in.

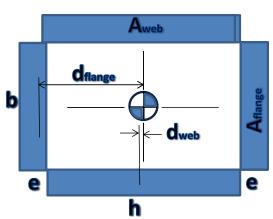


Figure C-3. Post-to-Base Weld, Concept AW2-C

Input Name	Input Value	Units
$S_W$ (weld size)	1/8	in.
F <sub>suw(filler)</sub>	17000	psi
F <sub>suw(base metal)</sub>	15000	psi
F <sub>tuw(base metal)</sub>	24000	psi
b flange width	2	in.
h web width	2	in.
Φ	0.75	

0 Weld		
Output Name	Output Value	Units
e (effective throat)	0.088388348	in.
F <sub>sw</sub>	1502.60191	lb/in.
$L_{we}$ (effective weld length)	8	in.
$R_n$ (nominal shear capacity)	9015.61146	lb
$M_n$ (nominal moment capacity)	6108.589015	inlb
$M_n$ (nominal moment capacity)	509.0490846	ft-lb
C (distance to neutral axis)	1.03125	in.
I (moment of inertia of weld group)	0.494077053	in. <sup>4</sup>
$I_{flange}$ (moment of inertia of flange)	0.000115089	in. <sup>4</sup>
$I_{web}$ (moment of inertia of web)	0.058925565	in. <sup>4</sup>
A <sub>flange</sub> (area of one flange)	0.176776695	in. <sup>2</sup>
Aweb (area of one web)	0.176776695	in. <sup>2</sup>
$d_{flange}$ (distance from NA section to NA flange)	1.03125	
$d_{web}$ (distance from NA section to NA web)	0	in.

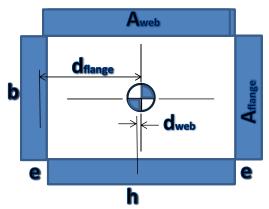


Figure C-4. Rail-to-Post Weld, Concepts AW2-A, AW2-C, and AW2-D

Input Name	Input Value	Units
$S_W$ (weld size)	1/8	in.
F <sub>suw(filler)</sub>	17000	psi
F <sub>suw(base metal)</sub>	15000	psi
F <sub>tuw(base metal)</sub>	24000	psi
b flange width	1/2	in.
h web width	1/2	in.
Φ	0.75	

|--|

Output Name	Output Value	Units
e (effective throat)	0.088388348	in.
F <sub>sw</sub>	1502.60191	lb/in.
$L_{we}$ (effective weld length)	2	in.
$R_n$ (nominal shear capacity)	2253.902865	lb
$M_n$ (nominal moment capacity)	403.0416582	inlb
$M_n$ (nominal moment capacity)	33.58680485	ft-lb
C (distance to neutral axis)	0.28125	in.
I (moment of inertia of weld group)	0.008890625	in. <sup>4</sup>
$I_{flange}$ (moment of inertia of flange)	2.87722E-05	in. <sup>4</sup>
$I_{web}$ (moment of inertia of web)	0.000920712	in. <sup>4</sup>
A <sub>flange</sub> (area of one flange)	0.044194174	in. <sup>2</sup>
Aweb (area of one web)	0.044194174	in. <sup>2</sup>
$d_{flange}$ (distance from NA section to NA flange)	0.28125	
$d_{web}$ (distance from NA section to NA web)	0	in.

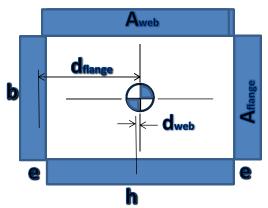


Figure C-5. Spindle-to-Rail Weld, Concepts AW2-A, AW2-C, and AW2-D

### **Baseplate Capacity**

Inputs: B=3 in. (Width of Baseplate) N=7.5 in. (Depth of Baseplate)  $b_f=2$  in. (Width of Flange) d=4 in. (Depth of Web)  $M_u=1,537.5$  lb-ft (Moment on Baseplate)  $F_y$  = yield stress [psi] =  $F_{tyw} = 15,000$  psi  $\phi = 0.90$ 

$$m = \frac{N - 0.95d}{2} = \frac{7.5 - 0.95(4)}{2} = 1.85 \text{ in.}$$
$$n = \frac{B - 0.80b_f}{2} = \frac{3 - 0.80(2)}{2} = 0.7 \text{ in.}$$
$$l = \text{Greater of } m \text{ and } n = 1.85 \text{ in.}$$

$$P_u = \frac{M_u}{d} = \frac{1,537.5 \text{ ft} - \text{lb} (12 \text{ in.}/\text{ft})}{4 \text{ in.}} = 4,613 \text{ lb}$$

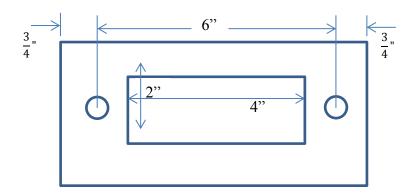
$$t_{min} = l \sqrt{\frac{2P_u}{\varphi F_y BN}} = 1.85 \text{ in.} \sqrt{\frac{2 * 4,613 \text{ lb}}{0.9 * 15,000 \text{ psi} * 3 \text{ in.} * 7.5 \text{ in.}}} = 0.322 \text{ in}.$$

Choose a baseplate thickness of t =  $\frac{3}{8}$  in. The nominal capacity,  $\varphi P_n$ , of a  $\frac{3}{8}$ -in. baseplate is:

$$\varphi P_n = \varphi \frac{F_y BN}{2} \left(\frac{t}{l}\right)^2 = 0.90 \frac{15,000 \text{ psi} * 3 \text{ in.} * 7.5 \text{ in.}}{2} \left(\frac{\frac{3}{8} \text{ in.}}{1.85 \text{ in.}}\right)^2 = 6,240 \text{ lb}$$

 $\varphi P_n = 6,240 \text{ lb} > P_u = 4,613 \text{ lb}$ , so the design is adequate.

Figure C-6. Example Calculation of Baseplate - Method No. 1, Concept AW2-A



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 $\frac{3}{4}$ "-6" 3 " Inputs: 4 B = 3 in. (Width of Baseplate) N = 7.5 in. (Depth of Baseplate) N' = 6.75 in. 4" 2" A' = 3 in. (Distance from Bolt to Center of Post)  $b_f = 2$  in. (Width of Flange) d = 4 in. (Depth of Web)  $M_u = 1,537.5$  lb-ft × 12 in/ft = 18,450 lb-in.  $P_{u} = 575 \ lb$  $\Phi = 0.90$  $A_1 = B * N = 3 * 7.5 = 22.5 in.^2$  $A_1 = b + N = 0 + 1.2$   $A_2 = 4 * A_1 = 90 \text{ in.}^2$   $m = \frac{N - 0.95d}{2} = \frac{7.5 - 0.95(4)}{2} = 1.85 \text{ in.}$ 

$$e = \frac{M_u}{P_u} = \frac{18,450 \ lb - in}{575 \ lb} = 32.1 \ in.$$
  $e_{crit} = \frac{N}{2} = \frac{7.5}{2} = 3.75 \ in.$ 

 $e > e_{crit}$ , so use design with a large eccentricity

$$\begin{split} F_p &= 0.65 \times 0.85 f_c^{\prime} \sqrt{\frac{A_2}{A_1}} \le 0.65 \times 1.7 f_c^{\prime} = 0.65 \times 0.85(2500)(4) \le 0.65 \times 1.7(2500) = 2,762.5 \ psi \\ f^{\prime} &= \frac{F_p B N^{\prime}}{2} = \frac{2762.5 * 3 * 6.75}{2} = 27,970.3 \ lb \\ A &= \frac{f^{\prime} \pm \sqrt{(f^{\prime})^2 - 4\left(\frac{F_p B}{6}\right)(PA^{\prime} + M)}}{\frac{F_p B}{3}} \\ &= \frac{27,970.3 \ lb \pm \sqrt{(27,970.3 \ lb)^2 - 4\left(\frac{2,762.5 \ psi * 3 \ in.}{6}\right)(575 \ lb * 3 \ in. +1,537.5 \ ft - lb * 12 \ in./ft)}{\frac{2,762.5 \ psi * 3 \ in.}{3}} \end{split}$$

$$= 0.749 in$$

$$T = \frac{F_p AB}{2} - P = \frac{2,762.5 \text{ psi} * 0.749 \text{ in.* 3 in.}}{2} - 575 \text{ lb} = 3,104 \text{ lb}$$
$$M_{pl} = \frac{F_p A}{2} \left(m - \frac{1}{3}A\right) = \frac{2,762.5 \text{ psi} * 0.749 \text{ in.}}{2} \left(1.85 - \frac{1}{3} * 0.749\right) = 1,655 \text{ lb} - \text{in./in.}$$
$$t_{min} = \sqrt{\frac{4M_{pl}}{0.9F_y}} = \sqrt{\frac{4 * 1,655 \text{ lb} - \text{in./in.}}{0.9 * 15,000 \text{ psi}}} = 0.7 \text{ in.}$$

Even though this method requires a minimum 0.7 in. thick,  $t_{min}$ , baseplate, a  $t = \frac{3}{8}$ -in. thick baseplate was selected for the design. According to this method, the nominal bending capacity of the plate is:

$$\varphi M_n = \frac{\varphi F_y t^2}{4} = \frac{0.9 * 15,000 \text{ psi} * (.375 \text{ in.})^2}{4} = 474.6 \text{ lb} - \text{in./in.}$$
$$\varphi M_n = 474.6 \text{ lb} - \text{in./in.} < M_{pl} = 1,655 \text{ lb} - \text{in./in.}$$

Figure C-7. Example Calculation of Baseplate - Method No. 2, Concept AW2-A

Concept:	Concept: AW2-A							
Variable	Input	Unit	Description					
b	2	in.	width of post					
d	4	in.	depth of post					
В	3	in.	width of base plate					
N	7.5	in.	depth of base plate					
F <sub>tyw</sub>	15000	psi	tensile yield strength					
M <sub>u</sub>	1537.5	ft-lb	moment at base of post					
ф	0.9							
Variable	Calculation	Unit	Equation	Description				
P <sub>u</sub>	4612.5	lb	=M <sub>u</sub> /d	maximum vertical force on base plate				
m	1.85	in.	=(N-0.95d)/2	location of critical section along N				
n	0.7	in.	=(B-0.80b)/2	location of critical section along B				
	1.85	in.		the greater of m and n				
t <sub>min</sub>	0.322	in.	=l*sqrt(2P <sub>u</sub> /(φF <sub>tyw</sub> BN))	minimum base plate thickness				
t	0.375	in.		actual base plate thickness				
φP <sub>n</sub>	6240.3	lb	=φF <sub>tyw</sub> BN/2*(t/l)^2	nominal base plate capacity				

 $\phi P_n > P_u$  so design is good

Figure C-8. Capacity of Baseplate - Method No. 1, Concept AW2-A

Concept: AW2-C

Variable	Input	Unit	Description	
b	2	in.	width of post	
d	3	in.	depth of post	
В	3.5	in.	width of base plate	
Ν	7.5	in.	depth of base plate	
F <sub>tyw</sub>	15000	psi	tensile yield strength	
M <sub>u</sub>	1537.5	ft-lb	moment at base of post	
φ	0.9			
Variable	Calculation	Unit	Equation	Description
P <sub>u</sub>	6150	lb	=M <sub>u</sub> /d	maximum vertical force on base plate
m	2.325	in.	=(N-0.95d)/2	location of critical section along N
n	0.95	in.	=(B-0.80b)/2	location of critical section along B
I	2.325	in.		the greater of m and n
t <sub>min</sub>	0.433	in.	=I*sqrt(2P <sub>u</sub> /( $\phi$ F <sub>tyw</sub> BN))	minimum base plate thickness
t	0.375	in.		actual base plate thickness
φP <sub>n</sub>	4609.5	lb	=φF <sub>tyw</sub> BN/2*(t/l)^2	nominal base plate capacity

 $\Phi P_{n<}P_u$  so design is not good

Figure C-9. Capacity of Baseplate – Method #No. 1, Concept AW2-C

Concept:	Concept: AM-1							
Variable	Input	Unit	Description					
b	2.375	in.	width of post					
d	2.375	in.	depth of post					
В	5	in.	width of base plate					
N	8.5	in.	depth of base plate					
F <sub>ty</sub>	13500	psi	tensile yield strength					
M <sub>u</sub>	1537.5	ft-lb	moment at base of post					
φ	0.9							
Variable	Calculation	Unit	Equation	Description				
Pu	7768.4	lb	=M <sub>u</sub> /d	maximum vertical force on base plate				
m	3.122	in.	=(N-0.95d)/2	location of critical section along N				
n	1.55	in.	=(B-0.80b)/2	location of critical section along B				
I	3.122	in.		the greater of m and n				
t <sub>min</sub>	0.542	in.	=l*sqrt(2P <sub>u</sub> /(φF <sub>ty</sub> BN))	minimum base plate thickness				
t	0.5625	in.		actual base plate thickness				
φP <sub>n</sub>	8382.0	lh	=φF <sub>tv</sub> BN/2*(t/l)^2	nominal base plate capacity				

 $\Phi P_n > P_u$  so design is good

Figure C-10. Capacity of Baseplate - Method No. 1, Concept AM-1

Concept: AW2-D

Variable	Input	Unit	Description	
b	2	in.	width of post	
d	4	in.	depth of post	
В	3	in.	width of base plate	
N	7.75	in.	depth of base plate	
F <sub>tyw</sub>	15000	psi	tensile yield strength	
M <sub>u</sub>	1537.5	ft-lb	moment at base of post	
ф	0.9			
Variable	Calculation	Unit	Equation	Description
Pu	4612.5	lb	=M <sub>u</sub> /d	maximum vertical force on base plate
m	1.975	in.	=(N-0.95d)/2	location of critical section along N
n	0.7	in.	=(B-0.80b)/2	location of critical section along B
Ι	1.975	in.		the greater of m and n
t <sub>min</sub>	0.339	in.	=l*sqrt(2P <sub>u</sub> /(φF <sub>tyw</sub> BN))	minimum base plate thickness
t	0.375	in.		actual base plate thickness
φP <sub>n</sub>	5657.9	lb	=φF <sub>tyw</sub> BN/2*(t/l)^2	nominal base plate capacity

 $\phi P_n > P_u$  so design is good

Figure C-11. Capacity of Baseplate – Method No. 1, Concept AW2-D

Concept:	AW2-A			_
Variable	Input	Unit	Description	
Mu	18450	in-lb	max moment at base of post	
Pu	575	lb	axial load on base plate	
В	3	in	width of BP	
N	7.5	in	length of BP	
N'	6.75	in	distance from edge of plate to far bolt along N	
Α'	3	in	distance from anchor to centerline of post along N	
b	2	in	flange of post	
d	4	in	web of post	
f <sub>c</sub> '	2500		compression strength of concrete	
A <sub>1</sub>	22.5	in. <sup>2</sup>	area of base plate	
A <sub>2</sub>	90	in. <sup>2</sup>	area of supporting concrete foundation	
F <sub>tyw</sub>	15000	psi	tensile yield strength	
ф	0.9			
φ <sub>c</sub>	0.65			
Variable	Output	Unit	Calculation	Description
e	32.08696	in.	M <sub>u</sub> /P <sub>u</sub>	eccentricity
e <sub>crit</sub>	3.75	in	N/2	critical eccentricity
			e>e <sub>crit</sub>	therefore use large eccentricity base plate design
m	1.85	in.	(N-0.95d)/2	location of critical section along N
F <sub>p</sub>	2762.5	psi	$\phi_c 0.85 f_c$ '*sqrt(A <sub>2</sub> /A <sub>1</sub> )	maximum design bearing stress
	2762.5	psi	≤¢c1.7fc'	maximum design bearing stress
f'	27970.3	lb		
A	0.749	in.	$(f'\pm sqrt((f')^{2-4*}(F_{p}B/6)(P_{u}A'+M)))/(F_{p}B/3)$	length of bearing stress block along N
Т	2528.7	lb	F <sub>p</sub> AB/2-P <sub>u</sub>	tension in bolt
M <sub>pl</sub>	1655.6	lb-in./in.	0.5*F <sub>p</sub> A(m-1/3A)	required bending moment per width
t <sub>min</sub>	0.700	in.	sqrt(4*M <sub>pl</sub> /(φF <sub>tyw</sub> )	minimum base plate thickness
t	0.375			actual base plate thickness
φM <sub>n</sub>	474.6	lb-in./in.	$\phi F_{tyw} t^2 / 4$	nominal moment capacity

 $\Phi M_n < M_{pl}$  so design is not good Figure C-12. Capacity of Baseplate – Method No. 2, Concept AW2-A

Concept:	AW2-C			_
Variable	Input	Unit	Description	
M <sub>u</sub>	18450	in-lb	max moment at base of post	
P <sub>u</sub>	575	lb	axial load on base plate	
В	3.5	in	width of BP	
N	7.5	in	length of BP	
N'	6.75	in	distance from edge of plate to far bolt along N	
Α'	3	in	distance from anchor to centerline of post along N	
b		in	flange of post	
d	3	in	web of post	
f <sub>c</sub> '	2500		compression strength of concrete	
A <sub>1</sub>	26.25	in. <sup>2</sup>	area of base plate	
A <sub>2</sub>		in. <sup>2</sup>	area of supporting concrete foundation	
F <sub>tyw</sub>	15000	psi	tensile yield strength	
ф	0.9			
φ <sub>c</sub>	0.65			
Variable	Output	Unit	Calculation	Description
e	32.08696	in.	M <sub>u</sub> /P <sub>u</sub>	eccentricity
e <sub>crit</sub>	3.75	in	N/2	critical eccentricity
			e>e <sub>crit</sub>	therefore use large eccentricity base plate design
m	2.325	in.	(N-0.95d)/2	location of critical section along N
Fp	2762.5	psi	$\phi_c 0.85 f_c$ '*sqrt(A <sub>2</sub> /A <sub>1</sub> )	maximum design bearing stress
	2762.5	psi	≤¢ <sub>c</sub> 1.7f <sub>c</sub> '	maximum design bearing stress
f'	32632.0	lb		
А	0.638	in.	(f'±sqrt((f')^2-4*(F <sub>p</sub> B/6)(P <sub>u</sub> A'+M)))/(F <sub>p</sub> B/3)	length of bearing stress block along N
Т	2511.2	lb	F <sub>p</sub> AB/2-P <sub>u</sub>	tension in bolt
M <sub>pl</sub>	1862.5	lb-in./in.	0.5*F <sub>p</sub> A(m-1/3A)	required bending moment per width
t <sub>min</sub>	0.743	in.	sqrt(4*M <sub>pl</sub> /(φF <sub>tyw</sub> )	minimum base plate thickness
t	0.375			actual base plate thickness
φM <sub>n</sub>	474.6	lb-in./in.	φF <sub>tyw</sub> t²/4	nominal moment capacity

 $\phi M_n < M_{pl}$  so design is not good

Figure C-13. Capacity of Baseplate – Method No. 2, Concept AW2-C

Concept:	AM-1			_
Variable	Input	Unit	Description	
Mu	18450	in-lb	max moment at base of post	
Pu	575	lb	axial load on base plate	
В	5	in	width of BP	
Ν	8.5	in	length of BP	
N'	6.75	in	distance from edge of plate to far bolt along N	
Α'	2.75	in	distance from anchor to centerline of post along N	
b	2.375	in	flange of post	
d	2.375	in	web of post	
f <sub>c</sub> '	2500		compression strength of concrete	
A <sub>1</sub>		in. <sup>2</sup>	area of base plate	
A <sub>2</sub>	170	in. <sup>2</sup>	area of supporting concrete foundation	
F <sub>ty</sub>	13500	psi	tensile yield strength	
ф	0.9			
φ <sub>c</sub>	0.65			
Variable	Output	Unit	Calculation	Description
e	32.08696	in.	M <sub>u</sub> /P <sub>u</sub>	eccentricity
e <sub>crit</sub>	4.25	in	N/2	critical eccentricity
			e>e <sub>crit</sub>	therefore use large eccentricity base plate design
m	3.121875	in.	(N-0.95d)/2	location of critical section along N
F <sub>p</sub>	2762.5	psi	$\phi_c 0.85 f_c$ '*sqrt(A <sub>2</sub> /A <sub>1</sub> )	maximum design bearing stress
	2762.5	psi	≤¢ <sub>c</sub> 1.7f <sub>c</sub> '	maximum design bearing stress
f'	46617.2	lb		
A	0.439	in.	(f'±sqrt((f')^2-4*(F <sub>p</sub> B/6)(P <sub>u</sub> A'+M)))/(F <sub>p</sub> B/3)	length of bearing stress block along N
Т	2458.4	lb	F <sub>p</sub> AB/2-P <sub>u</sub>	tension in bolt
M <sub>pl</sub>	1805.1	lb-in./in.	0.5*F <sub>p</sub> A(m-1/3A)	required bending moment per width
t <sub>min</sub>	0.731	in.	sqrt(4*M <sub>pl</sub> /(φF <sub>ty</sub> )	minimum base plate thickness
t	0.5625	in.		actual base plate thickness
φM <sub>n</sub>	961.1	lb-in./in.	$\phi F_{ty} t^2 / 4$	nominal moment capacity

 $\phi M_n < M_{pl}$  so design is not good Figure C-14. Capacity of Baseplate – Method No. 2, Concept AM-1

Concept:	AW2-D			_
Variable	Input	Unit	Description	
M <sub>u</sub>	18450	in-lb	max moment at base of post	
Pu	575	lb	axial load on base plate	
В	3	in	width of BP	
N	7.75	in	length of BP	
N'	7	in	distance from edge of plate to far bolt along N	
Α'	3.125	in	distance from anchor to centerline of post along N	
b	-	in	flange of post	
d	4	in	web of post	
f <sub>c</sub> '	2500		compression strength of concrete	
A <sub>1</sub>	23.25		area of base plate	
A <sub>2</sub>	93	in. <sup>2</sup>	area of supporting concrete foundation	
F <sub>tyw</sub>	15000	psi	tensile yield strength	
ф	0.9			
φ <sub>c</sub>	0.65			
Variable	Output	Unit	Calculation	Description
е	32.08696	in.	M <sub>u</sub> /P <sub>u</sub>	eccentricity
e <sub>crit</sub>	3.875	in	N/2	critical eccentricity
			e>e <sub>crit</sub>	therefore use large eccentricity base plate design
m	1.975	in.	(N-0.95d)/2	location of critical section along N
F <sub>p</sub>	2762.5	psi	$\phi_c 0.85 f_c$ * sqrt(A <sub>2</sub> /A <sub>1</sub> )	maximum design bearing stress
	2762.5	psi	≤¢ <sub>c</sub> 1.7f <sub>c</sub> '	maximum design bearing stress
f'	29006.3	lb		
А	0.723	in.	(f'±sqrt((f')^2-4*(F <sub>p</sub> B/6)(P <sub>u</sub> A'+M)))/(F <sub>p</sub> B/3)	length of bearing stress block along N
т	2420.5	lb	F <sub>p</sub> AB/2-P <sub>u</sub>	tension in bolt
M <sub>pl</sub>	1731.4	lb-in./in.	0.5*F <sub>p</sub> A(m-1/3A)	required bending moment per width
t <sub>min</sub>	0.716	in.	sqrt(4*M <sub>pl</sub> /(φF <sub>tyw</sub> )	minimum base plate thickness
t	0.375			actual base plate thickness
φM <sub>n</sub>	474.6	lb-in./in.	$\phi F_{tyw} t^2 / 4$	nominal moment capacity

 $\phi M_n < M_{pl}$  so design is not good Figure C-15. Capacity of Baseplate – Method No. 2, Concept AW2-D

# **Anchor Capacity**

		<b>TENSION AN</b>	<b>ICHORS</b>	(FRON	T FACE	)		
	Embe	dment Depth, h <sub>ef</sub> :		in.				
	Steel	Bar Diameter, d <sub>a</sub> :	0.375					
		Area of Steel, A.:	0.078	in. <sup>2</sup>		Tensi	on Stren	oths
	Front (Tension) 4	Anchor Spacing, s:		in.		Terisi	Shoren	
Fror	nt (Tension) Anchor to			in.		Failu	re Mode	Load (kips)
1101			10			C+,	eel Fracture:	(KIDS) 7313
		trength, $\tau_{uncr} = \tau_{cr}$ : Steel Stength, $f_{uta}$ :	125000			-	te Breakout:	8648
		crete Strength, f' <sub>c</sub> :	2500	-		-	ond Failure:	5552
		teinforced? (y/n):	2500 n	121		B		
	roundation	λ <sub>a</sub>	1					
			1					
			Tension	Shear				
A	CI Steel Strength Red	uction Factor, φ.:	0.75		Ì			
	oncrete Strength Red		0.65	0.75	ĺ			
	dhesive Strength Redu		0.65	NA	ĺ			
		- · · · · · · · · · · · · · · · · · · ·	2.00					
	Required Constitut	-1507 5*10/-	2075	lh				
	Required Capacity	=1537.5*12/s	3075	D				
	TENSION CAPA							
		Steel Fracture:						
		φN <sub>sa</sub> =	7312.50	Ib				
	-	n anata Des altas t	TN .	/A -				
	Co	ncrete Breakout:				ν <sub>cp,N</sub> <sup>≁</sup> N <sub>b</sub>		
			N <sub>b</sub> =	$k_c * \lambda_a h_{ef}^{1.5}$				
				k <sub>c</sub> :		-	n place, 17 for pos	
				Ψ <sub>c,N</sub> :		(1.25 for cas	t in anchors, 1.4 fo	or post installed)
			N <sub>b</sub> =	9503.29	lb	_		
			C <sub>ac</sub> :	10				
			Ψ <sub>cp,N</sub> :					
			Ψ <sub>ed,N</sub> :	1				
					2			
		A	$h_{co} = 9 h_{ef}^{2}$ :		in. <sup>2</sup>			
			A <sub>Nc</sub> :		in. <sup>2</sup>			
			A <sub>Nc</sub> /A <sub>Nco</sub> :	1				
		φN <sub>cb</sub> =	8647.99	lb				
		/·		/* *		* NI		
	Adhesiv	e / Bond Failure:				a" IN <sub>ba</sub>		
				$\tau_{cr} \pi d_a h_{ef}$				
			N <sub>ba</sub> =	8541.21	di D			
				(2*2.)2				
			A <sub>Nao</sub> =	(2*C <sub>Na</sub> ) <sup>2</sup>				
					10*d <sub>a</sub> *√(1			
				C <sub>Na</sub> =		L in.		
			A <sub>Nao</sub> =					
			A <sub>Na</sub> =	74.15	in. <sup>2</sup>			
			A <sub>Na</sub> /A <sub>Nao</sub> :	1				
			ψ <sub>cp, Na</sub> :	1	(should b	e the same	e as ψ <sub>cp, N</sub> )	
			ψ <sub>cp,Na</sub> : Ψ <sub>ed,Na</sub> :			e the same	e as ψ <sub>cp,N</sub> )	

Figure C-16. Capacity of Anchors-Tension, Concepts AW2-A and AW2-C

		SHEAR	ANCHC	)RS (BA	CK FAC	E)						
Er	nbedment D	epth, h <sub>ef</sub> :	5	in.		Shear	Streng	ths				
	teel Bar Dian		0.375	in.				Load	1			
		Steel, A <sub>s</sub> :	0.078	in. <sup>2</sup>		Failure	e Mode	(kips)				
	Anchor S			in.		Stee	I Fracture:	3803				
Anchor to De	eck Edge Dist	ance, c <sub>a1</sub> :	10	in.		Concrete	Breakout:	11790				
	Steel Ste	ngth, f <sub>uta</sub> :	125000	psi		Concre	te Pryout:	12812				
	Concrete Stre	ength, f' <sub>c</sub> :	2500	•								
	ndation Thic			in.								
Foundati	on Reinforce											
	Bond Stre	ength, τ <sub>cr</sub> : λ,	1450 1	psı								
		٨.,	1									
	Required	Capacity	225	lb								
				CAPAC	Т							
					φV <sub>sa</sub> =0.6	*A.f.						
			51681		φv <sub>sa</sub> =0.0 3802.50							
				₽ sa−	3802.30	1.5						
			Concrete	Breakout:	φV <sub>cbg</sub> = A	Vc/Aven *u	ν ec, v Ψed. v	Ψ <sub>c,V</sub> Ψ <sub>h,V</sub> *	V <sub>b</sub>			
					V <sub>b1</sub> =	7 * (I <sub>e</sub> /d <sub>a</sub> ) <sup>0</sup>	<sup>0.2</sup> *Vd <sub>a</sub> * V	'c * C <sub>a1</sub> <sup>1.5</sup>				
							3.00					
					V <sub>b1</sub> =	10273.10	lb					
					V <sub>b2</sub> =	$9*\lambda_a c_{a1}^{1.5}$	'√f' <sub>c</sub>					
						14230.25						
				$V_b = \min(C)$	$V_{b1}, V_{b2}) =$							
					ψ <sub>ec,V</sub> :							
					Ψ <sub>ed,V</sub> :							
					ψ <sub>c,V</sub> :			icked, 1.2 for c	racked reinfo	rced, 1.0 for cr	acked unrein	forced
						1.463850						
					ψ <sub>ec,V</sub> :	1						
				A=	4.5*(c <sub>a1</sub> ) <sup>2</sup> =	450	in. <sup>2</sup>					
				· ·vco -	A <sub>vc</sub> =							
						0.746667						
				$\phi V_{cb} =$	11790.01	lb						
		Concre	ete Pryout	Strength:	$\phi V_{cpg} = k$	<sub>ep</sub> N <sub>epg</sub>						
					k <sub>cp</sub> =							
					N <sub>cpg</sub> =	Min (N <sub>cbg</sub> ,	N <sub>ag</sub> )					
		NT A	/ 4		 			NI	/ <b>A</b> •		* **	
					$\psi_{cp,N} * N_b$					<sub>Na</sub> Ψed,Na Ψcp	<sub>Na</sub> ~ N <sub>ba</sub>	
		N <sub>b</sub> =	$k_c * h_{ef}^{1.5} $						$\tau_{cr} \pi d_a h_{ef}$			
			k <sub>c</sub> :					N <sub>ba</sub> =	8541.21	lb		
		<b>۲</b>	Ψ <sub>c,N</sub> :	1.4					(2*2 .)			
		N <sub>b</sub> =	9503.29	۵ı				A <sub>Nao</sub> =	(2*C <sub>Na</sub> ) <sup>2</sup>	10*4 *-"	(1100)	
		<u>.</u>	10							10*d <sub>a</sub> *V(τ <sub>c</sub>	(0011 /	
		C <sub>ac</sub> :	10					•	C <sub>Na</sub> = 74.15			
		Ψ <sub>cp,N</sub> :	1					A <sub>Nao</sub> =	74.15			
		Ψ <sub>ed,N</sub> :	1									
		Ψec,N:	1	in. <sup>2</sup>				A <sub>Na</sub> /A <sub>Nao</sub> :				
	A <sub>N</sub>	$h_{ef} = 9 h_{ef}^2$ : A <sub>Nc</sub> :		in. <sup>2</sup>				Ψ <sub>ec, Na</sub> : Ψ:		(should be	the come	- as **
		A <sub>Nc</sub> /A <sub>Nco</sub> :						Ψ <sub>cp,Na</sub> :			are same	α συ ψι
		Nc' ANCO	1					Ψ <sub>ed, Na</sub> :	1			
		N <sub>cb</sub> =	13304.60	lb				N <sub>a</sub> =	8541.21	lb		
									_			

Figure C-17. Capacity of Anchors-Shear, Concepts AW2-A and AW2-C

		TENSION A			NT FAC	E)		
	Embedr	ment Depth, h <sub>ef</sub> :	5	in.				
	Steel B	ar Diameter, d <sub>a</sub> :	0.5	in.				
	A	rea of Steel, A <sub>s</sub> :	0.142	in. <sup>2</sup>		Tensio	on Streng	gths
	Front (Tension) Ar	-						Load
Front	(Tension) Anchor to d			in.		Failur	e Mode	(kips)
		rength, $\tau_{uncr} = \tau_{cr}$ :				Ste	el Fracture:	13313
		eel Stength, futa:		-			e Breakout:	8648
		ete Strength, f' <sub>c</sub> :		-			ond Failure:	7402
		inforced? (y/n):						
		λ <sub>a</sub>	1					
			Tension	Shear				
	I Steel Strength Reduc							
	ncrete Strength Reduc			0.75				
CI Adł	hesive Strength Reduc	tion Factor, φ <sub>a</sub> :	0.65	NA				
	Required Capacity	=1537.5*12/s	2952	lb				
	TENSION CAPA							
		Steel Fracture:						
		φN <sub>sa</sub> =	13312.50	lb				
		aroto Presi	4N - 4	/▲ ∸		***		
	Con	crete Breakout:				<sub>cp,N</sub> *N <sub>b</sub>		
			N <sub>b</sub> =	$k_c * \lambda_a h_{ef}^{1.5}$				
				k <sub>c</sub> :			place, 17 for pos	
			NT	Ψ <sub>c,N</sub> :		(1.25 for cast	in anchors, 1.4 fo	r post installed)
			N <sub>b</sub> =	9503.29	מו			
			C <sub>ac</sub> :	10				
			-ac. Ψ <sub>cp,N</sub> :	10				
			Ψ <sub>ed,N</sub> :	1				
			i eu, N					
		A	$h_{\rm Nco} = 9 h_{\rm ef}^{2}$ :	225	in.²			
			A <sub>Nc</sub> :		in. <sup>2</sup>			
			A <sub>Nc</sub> /A <sub>Nco</sub> :					
		$\phi N_{cb} =$	8647.99	lb				
	Adhesive	/ Bond Failure:				* N <sub>ba</sub>		
				$\tau_{cr} \pi d_a h_{ef}$				
			N <sub>ba</sub> =	11388.27	Ib			
				(2*C ) <sup>2</sup>				
			A <sub>Nao</sub> =	(2*C <sub>Na</sub> ) <sup>2</sup>	10*-1 * "	(4400)		
					10*d <sub>a</sub> *V(τ <sub>0</sub>			
				C <sub>Na</sub> = 131.82		ın.		
			A <sub>Nao</sub> =					
			A <sub>Na</sub> =	131.82				
			A <sub>Na</sub> /A <sub>Nao</sub> :	1				
			Ψ <sub>cp,Na</sub> :		(should be	e the same	as ψ <sub>cp,N</sub> )	
			Ψ <sub>ed, Na</sub> :	1				

Figure C-18. Capacity of Anchors-Tension, Concept AW2-D

		SHEAR	ANCHO	DRS (BA	CK FAC	E)						
	nbedment			in.		Shear	Streng	ths		_	_	
S	teel Bar Dia		0.5			Failure	Mode	Load				
		f Steel, A <sub>s</sub> :	0.142					(kips)				
		Spacing, s:	6.25				I Fracture:	6923				
nchor to De	eck Edge Dis		-	in.			Breakout:	11016				
_		ength, f <sub>uta</sub> :	125000			Concre	te Pryout:	17082		_		
	Concrete St		2500									
	ndation Thi on Reinford			in.								
Foundati		rength, $\tau_{cr}$ :	1450	nci								
	Dona St	λ.	1450	p31								
	Require	d Capacity	225	lb								
			SHEAR	CAPAC	ITY							
			Stee	Fracture:	φV <sub>sa</sub> =0.6	*A. Nfuta						
					6922.50							
				r · sa	0522.30							
			Concrete	Breakout:	φV <sub>cbg</sub> = A	Avc/Avco *4	ν <sub>ec,V</sub> Ψ <sub>ed,V</sub> ν	Ψc,v Ψh,v *	V <sub>b</sub>			
					V <sub>b1</sub> =	7 * (I <sub>e</sub> /d <sub>a</sub> )	<sup>0.2</sup> *Vd <sub>a</sub> * Vf	f'c * C <sub>a1</sub> <sup>1.5</sup>				
							4.00					
					V <sub>b1</sub> =	11862.36	lb					
					V <sub>b2</sub> =	9*c <sub>a1</sub> <sup>1.5</sup> *vt	r c					
						14230.25						
				$V_b = min$ (	$V_{b1}, V_{b2}) =$	11862.36	lb					
_												
					Ψ <sub>ed,V</sub> :	1	(only reduced	d for anchor a	djacent to de	ck discontinui	ty)	
					ψ <sub>c,V</sub> :	1.4	(1.4 for uncra	cked deck, 1.	2 for cracked	reinforced, 1.	) for cracked u	nreinforced d
					$\psi_{h,V}$ :							
					ψ <sub>ec,V</sub> :	1						
_							in 2					
_				A <sub>vco</sub> =	4.5*(c <sub>a1</sub> ) <sup>2</sup> =	450	in			-	-	
_						271.875				-	-	
					A <sub>Vco</sub> /A <sub>Vc</sub> =	0.604167						
_				άV –	11015.74	Ib						
				Ψ v cb -	11015.74	ID						
		Concr	ete Prvout	Strength	$\phi V_{cpg} = k$	n Nana						
1		Cond			i cpg – k	ch .ch8						
					k <sub>cp</sub> =	2						
1						– Min (N <sub>cbg</sub> ,						
1					· · cpg	(cog/	ag,					
1		Ncho= AN	/A <sub>Nco</sub> *ψ <sub>ec,1</sub>	vVed N Ward	Ψ <sub>cn N</sub> * N⊦			Nag= AN	/A <sub>Nao</sub> * ψ	, <sub>Na</sub> Ψed,Na Ψ	n Na * Nbr	
			$k_c * h_{ef}^{1.5} \gamma$		p.,0				$\tau_{cr} \pi d_a h_{er}$		-r	
-		. • <sub>b</sub> –	k <sub>c</sub> <sup>∞</sup> n <sub>ef</sub> k <sub>c</sub> :						11388.27		-	
				1.4				IN ba=	11300.27		-	
		N. –	ψ <sub>c,N</sub> : 9503.29					Δ. –	(2*C <sub>Na</sub> ) <sup>2</sup>			
			,505.29					ANao -		= 10*d <sub>a</sub> *V(1	/1100)	
		C <sub>ac</sub> :	10						C <sub>Na</sub> = C <sub>Na</sub> =		1	
								A <sub>Nao</sub> =				
		Ψ <sub>cp,N</sub> :							131.8182		-	
		Ψ <sub>ed,N</sub> :						A <sub>Na</sub> = A <sub>Na</sub> /A <sub>Nao</sub> :				
		- 0*h <sup>2</sup>		in. <sup>2</sup>					-		-	
	A	$h_{co} = 9 h_{ef}^{2}$ : $A_{Nc}$ :		in. <sup>2</sup>				Ψ <sub>ec,Na</sub> :			e the come	
		A <sub>Nc</sub> : A <sub>Nc</sub> /A <sub>Nco</sub> :						Ψ <sub>cp,Na</sub> :			e the same	as y <sub>cp,N</sub> )
		ANC/ANCO	1					Ψ <sub>ed,Na</sub> :	1			
		N <sub>cb</sub> =	13304.60	lb				N <sub>o</sub> =	11388.27	7 Ib		
	1			1	N=	11388.27	lb	- ·a		1		

Figure C-19. Capacity of Anchors-Shear, Concept AW2-D

		TENSION			INT FAC	,C)	-	
		ent Depth, h <sub>ef</sub> :		in.				
	Steel Ba	r Diameter, d <sub>a</sub> :		in.				
	Ar	ea of Steel, A <sub>s</sub> :	0.142	in. <sup>2</sup>		Tensio	on Streng	gths
_	Front (Tension) And			in.		Failur	re Mode	Load
Front	(Tension) Anchor to de			in.				(kips)
		ength, $\tau_{uncr} = \tau_{cr}$ :		-			eel Fracture:	13313
		el Stength, f <sub>uta</sub> :		·			te Breakout:	5065
		te Strength, f' <sub>c</sub> :		· · · · · ·		В	ond Failure:	5182
	Foundation Rei							
		λ <sub>a</sub>	1					
			Tension	Shear				
AC	I Steel Strength Reduct	ion Factor, $\phi_s$ :	0.75	0.65				
CI Co	ncrete Strength Reduct	ion Factor, $\phi_c$ :	0.65	0.75				
CI Ad	hesive Strength Reduct	ion Factor, $\phi_a$ :	0.65	NA				
	Required Capacity	=1537.5*12/s	3355	lb				
	<b>TENSION CAPA</b>	СІТҮ						
		Steel Fracture:	φN <sub>sa</sub> =A <sub>se</sub>	,Nf <sub>uta</sub>				
			13312.50					
	Conc	rete Breakout:				v <sub>cp,N</sub> *N <sub>b</sub>		
			N <sub>b</sub> =	$k_c \ * \lambda_a h_{ef}^{-1.5}$	√f' <sub>c</sub>			
				k <sub>c</sub> :	17	(24 for cast i	n place, 17 for pos	t installed)
				Ψc, N	1.4	(1.25 for cas	t in anchors, 1.4 fo	r post installed)
			N <sub>b</sub> =	5565.72	lb			
			C <sub>ac</sub> :	7				
			ψ <sub>cp,N</sub> :					
			Ψ <sub>ed,N</sub> :	1				
			r eu, N					
		A,	$h_{\rm Nco} = 9 h_{\rm ef}^{2}$ :	110.25	in. <sup>2</sup>			
			A <sub>Nc</sub> :					
			A <sub>Nc</sub> /A <sub>Nco</sub> :					
		φN <sub>cb</sub> =	5064.80	lb				
				/ <b>A</b> •		* 11		
	Adhesive	Bond Failure:				• N <sub>ba</sub>		
				$\tau_{cr} \pi d_a h_{ef}$				
			וא <sub>ba</sub> =	7971.79	10			
			Δ	(2*C <sub>Na</sub> ) <sup>2</sup>				
			ANao -		10*d <sub>a</sub> *ν(τ	/1100)		
				C <sub>Na</sub> =				
			A <sub>Nao</sub> =					
			A <sub>Nao</sub> –	131.82				
			A <sub>Na</sub> –					
			A Na' A Nao	1				
				1	(should be	e the same	as w)	
			Ψ <sub>cp,Na</sub> :	1	13110010 0			
			$\psi_{ed,Na}$ :	1				

Figure C-20. Capacity of Anchors-Tension, Concept AM-1

		SHEAR	ANCHO	DRS (BA	CK FAC	E)						
_												
						Ch.	C+.					
	bedmentl		3.5			Shear	Streng	ths				
St	eel Bar Dia		0.5			Failure	Mode	Load				
		f Steel, A <sub>s</sub> :	0.142					(kips)				
		Spacing, s:	5.5				Fracture:	6923				
nchor to De				in.		Concrete		10504				
_		ength, f <sub>uta</sub> :	125000			Concre	te Pryout:	11688				
	oncrete St		2500									
	dation Thio		7	in.								
Foundatio	n Reinford	rength, τ <sub>cr</sub> :	n 1450	nci								
_	bonu su	λ,	1450	psi								
	Require	d Capacity	225	lb								
				САРАС	Т							
					φV <sub>sa</sub> =0.6	*A.sf.						
			JIEE		φ v <sub>sa</sub> =0.0							
				Ψ <sup>v</sup> sa	0922.50	10						
										-	-	
			Concrete	Breakout.	ΦV <sub>cha</sub> = A	Vc/A <sub>Vc0</sub> *4	lerv Water I	ΨενΨκυ*	V <sub>b</sub>			
					V <sub>b1</sub> =	7 * (l <sub>e</sub> /d <sub>a</sub> ) <sup>(</sup>	<sup>1.2</sup> *Vd <sub>3</sub> * Vf	"c * C <sub>31</sub> <sup>1.5</sup>				
					01	، (۱۵٫۵۵٫) ۱٫:		- 01		1	1	
					V <sub>b1</sub> =	11549.75				1	1	
					01	11545.75	10					
					V.a =	9*c <sub>a1</sub> <sup>1.5</sup> *Vf	,					
					. 02	14230.25						
						11200.20						
				$V_b = min$ (	$V_{b1}, V_{b2}) =$	11549.75	lb					
						[						
					$\psi_{ed,V}$ :	1	(only reduce	d for anchor a	djacent to deo	ck discontinui	ty)	
					ψ <sub>c,V</sub> :	1.4	(1.4 for uncra	cked deck, 1.2	2 for cracked r	reinforced, 1.0	) for cracked u	nreinforced de
					$\psi_{h,V}$ :	1.46385						
_					ψ <sub>ec,V</sub> :	1						
_							. 2					
				A <sub>vco</sub> =	4.5*(c <sub>a1</sub> ) <sup>2</sup> =	450	in. <sup>2</sup>					
						266.25	in.~					
_					A <sub>Vco</sub> /A <sub>Vc</sub> =	0.591667						
				φV. –	10503.54	lb						
				Ψ <sup>+</sup> cb <sup>-</sup>	10505.54	IJ						
		Concr	ete Pryout	Strength:	$\phi V_{cpg} = k$	cp N <sub>cno</sub>						
			,	<b>.</b>		17 8						
					k <sub>cp</sub> =	2						
						Min (N <sub>cbg</sub> ,	N <sub>ag</sub> )					
		Ncbg= ANd	A <sub>Nco</sub> * $\psi_{ec,1}$	vΨed,N Ψc,N	$\psi_{cp,N} * N_b$			N <sub>ag</sub> = A <sub>Na</sub>	$A_{Nao} * \psi_{ec}$	,Na Ψed,Na Ψe	<sub>p,Na</sub> * N <sub>ba</sub>	
			$k_{c} * h_{ef}^{1.5} $						$\tau_{cr} \pi d_a h_{ef}$			
			k <sub>c</sub> :						7971.79			
			ψ <sub>c,N</sub> :	1.4								
		N <sub>b</sub> =	5565.72					A <sub>Nao</sub> =	$(2*C_{Na})^2$			
										= 10*d <sub>a</sub> *v(1	/1100)	
		C <sub>ac</sub> :	7						C <sub>Na</sub> =		1	
		ψ <sub>cp,N</sub> :						A <sub>Nao</sub> =				
		Ψ <sub>ed,N</sub> :							131.8182			
		ψ <sub>ec,N</sub> :						A <sub>Na</sub> /A <sub>Nao</sub> :				
	A	$h_{rec} = 9 h_{ef}^{2}$ :						ψ <sub>ec,Na</sub> :				
	,	$A_{\text{Nc}} = \gamma H_{\text{ef}}$ :						ψ <sub>cp,Na</sub> :			e the same	as ψ <sub>cn №</sub> )
		A <sub>Nc</sub> /A <sub>Nco</sub> :						Ψcp,Na <sup>+</sup> Ψed,Na <sup>+</sup>				· rcp,N/
								, cu,ivd				
		N <sub>cb</sub> =	7792.00	lb				N <sub>a</sub> =	7971.79	lb		
					N <sub>cpg</sub> =	7792.00	lb					

Figure C-21. Capacity of Anchors-Shear, Concept AM-1

Variables		Condition	Load Case	Equation	Calculation	Nomin	nal Capaci	ty	Requi Design	
ail - 2" x 2" x ¼" 6061 A	luminum Tube					L			Design	2500
$A_w = 0.4375 \text{ in}^2$			Shear	$\phi V_n = \phi F_s A_w$	= 0.9*21000*0.4375	= 82	68.8 lb	>	348.2 1	lb
b = 1.75 in.		No Welding	Flexural Yielding	$\phi M_n = \phi 1.3 F_{ty} S_t$	= 0.9*1.3*35000*0.2759/12	= 9	41.5 ft-lb	>	562.5 1	ft-lb
t = 0.25 in.			Flexural Rupture	$\phi M_n = \phi 1.42 F_{tu} S_t$	= 0.75*1.42*38000*0.2759/12	= 9	30.5 ft-lb	>	562.5 1	ft-lb
$b/t = 7 < S_1$			Shear	$\phi V_n = \phi F_s A_w$	= 0.9*9000*0.4375	= 35	43.8 <b>l</b> b	>	348.2	íb
$F_s = F_{sy}$ or $F_{syw}$		Fully Welded	Flexural Yielding	$\phi M_n = \phi 1.3 F_{tyw} S_t$	= 0.9*1.3*15000*0.2759/12	= 4	03.5 ft-lb	>	356.5 1	ît-lb
$S_t = 0.2759 \text{ in}^3$			Flexural Rupture	$\phi M_n = \phi 1.42 F_{tuw} S_t$	= 0.75*1.42*24000*0.2759/12	= 5	87.7 ft-lb	>	356.5	ít-lb
$A_{wzt} = 0.25 \text{ in}^2$		One-side welded	Flexural Yielding			-	54.6 ft-lb	>	562.5 1	ít-lb
Agt =0.4688 in <sup>2</sup>		one sale wealed	Flexural Rupture	$\phi Mn = \phi 1.42 [F_{tu}(1 - \frac{Awzt}{Ag})/k_t + F_{tuw}(\frac{Awzt}{Ag})]S_t$	= 0.75*1.42*(38000*(125/.4688)/1.0+24000*0.25/0.4688)*0.2759/12	= 7	47.7 ft-lb	>	562.5	ít-lb
ost - 2" x 4" x 1/4" 6061	Aluminum Tube			T						
$A_w = 2 in^2$			Shear	$\phi V_n = \phi F_s A_w$			7800 lb	>	450 1	
b = 3.5 in.		No Welding	Flexural Yielding	$\phi M_n = \phi 1.3 F_{ty} S_t$			27.7 ft-lb	>	1537.5	
t = 0.5 in.	-		Flexural Rupture	$\phi M_n = \phi 1.42 F_{tu} S_t$			74.6 ft-lb	>	1537.5	
$b/t = 7 < S_1$			Shear	$\phi V_n = \phi F_s A_w$			5200 lb	>	450 1	-
$F_s = F_{sy}$ or $F_{syw}$		Fully Welded	Flexural Yielding	$\varphi M_n = \varphi 1.3 F_{tyw} S_t$			40.4 ft-lb	>	1537.5	
$S_t = 1.3268 \text{ in}^3$			Flexural Rupture	$\phi M_n = \phi 1.42 F_{tuw} S_t$	= 0.75*1.42*24000*1.3268/12	= 28	26.1 ft-lb	>	1537.5	ıt−lb
bindles - 1/2" x 1/2" solid	l square		Chann	$aV = a\Gamma$ A	0.0*21000*0.25		1725 1	<u> </u>	7.0.1	n.
$A_g = 0.25 \text{ in}^2$		No Welding	Shear Flexural Yielding	$\varphi V_n = \varphi F_{sy} A_g$			4725 lb	>	7.9	
$S_t = 0.0104 \text{ in}^3$		NO Welding	Flexural Rupture	$\varphi M_n = \varphi 1.3 F_{ty} S_t$			35.5 ft-lb	>	4.0 1	
	-			$\varphi M_n = \varphi 1.42 F_{tu} S_t$			35.1 ft-lb 2025 lb	>	4.0 1	
		Fully Welded	Shear Flexural Yielding	$\varphi V_n = \varphi F_{syw} A_g$ $\varphi M_n = \varphi 1.3 F_{tyw} S_t$			15.2 ft-lb	>	2.7 1	
		Fully welded	Flexural Rupture	$\phi M_n - \phi 1.5 \Gamma_{tyw} S_t$ $\phi M_n = \phi 1.42 \Gamma_{tyw} S_t$			22.2 ft-lb	>	2.7 1	
ase Plate - 7 1/2" x 3" x 3	3/" 6061 Aluminun	n Plate	T lexiliar reapraire	$\psi_{1}\psi_{1} - \psi_{1} + 2\Gamma_{tuw}S_{t}$	- 0.75*1.42*24000*0.0104/12	-	22.2 It-ID	<u> </u>	2.7	.t-10
B = 3  in.	$t = \frac{3}{8}$ in.		Method #1 - Vertical Force	$\varphi P_n = \varphi F_{tyw} BN/2 * (t/I)^2$	= 0.9*15000*3*7.5/2*(.375/1.85)^2	= 62	40.3 lb	>	4612.5	lb
N = 7.5 in.	l = 1.85 in.	Fully Welded	Method #2 - Moment	$\phi M_n = \phi F_{typ} t^2/4$		= 4	74.6 inlb/	in. <	1656 i	inI
ost to Base Plate Weld - 1				The Trywess						
e = 0.1768 in.	$I = 4.8954 \text{ in.}^4$	XX7 11	Shear	$\phi R_n = \phi F_{sw} L_{we}$	= 0.75*17000*0.1768*12	= 270	46.8 lb	>	450 1	lb
$L_{we} = 12$ in.	c = 2.0625 in.	Weld	Moment	$\phi M_n = \phi F_{su} * I/c$	= 0.75*17000*4.8954/2.0625/12	= 25	21.9 ft-lb	>	1537.5	ft-lb
ail to Post Weld - ½" 535	6 Filler									
e = 0.0884 in.	$I = 0.4941 \text{ in.}^4$	Weld	Shear	$\phi R_n = \phi F_{sw} L_{we}$	= 0.75*17000*0.0884*8	= 90	15.6 lb	>	348.2	lb
$L_{we} = 8$ in.	c = 1.03125 in.	WCRI	Moment	$\phi M_n = \phi F_{su} * I/c$	= 0.75*17000*0.4941/1.03125/12	= 5	09.0 ft-lb	>	356.5 1	ft-lb
oindles to Rail Weld - 1/8"			1	1						
e = 0.0884 in.	$I = 0.0089 \text{ in.}^4$	Weld	Shear	$\phi R_n = \phi F_{sw} L_{we}$	= 0.00 17000 0.0001 2		53.9 lb	>	7.9 1	
$L_{we} = 2$ in.	c = 0.28125 in.		Moment	$\phi M_n = \phi F_{su} * I/c$	= 0.75*17000*0.0089/0.28125/12	=	33.6 ft-lb	>	2.7 1	ft-lb
nchor Bolts - 3/8" Diamete		led Rod, Embedded 5				701			2075	
$A_{se,N} = A_{se,V} = 0.078 \text{ in.}^2$	N <sub>ba</sub> = 8541 lb		Tension (Steel)	$\phi N_{sa} = \phi A_{se,N} f_{uta}$		= 731				1
f <sub>uta</sub> = 125,000 psi	$A_{Vc}/A_{Vco} = 0.7467$		Tension (Concrete Breakout)	$\phi N_{cb} = \phi A_{Nc} / A_{Nco} \psi_{ed,N} \psi_{c,N} \psi_{cp,N} N_b$	= 0.65*1*1.4*1*9503	= 864				]
$A_{Nc}/A_{Nco} = 1$	$\psi_{ec,V} = 1$	Anchor	Tension (Adhesive Bond)	$\phi N_a = \phi A_{Nc} / A_{Nco} \Psi_{cd,Na} \Psi_{cp,Na} N_{ba}$	= 0.65*1*1*1*8541	= 555	2 lb	>	3075	I
$\psi_{ed,N} = 1$	$\psi_{ed,V}\!=1$	1 1101101	Shear (Steel)	$\phi N_{sa} = \phi 0.6 A_{se, V} f_{uta}$	= 0.65*0.6*0.078*125000	= 380	3 lb	>	225	I
$\psi_{\rm c,N}{=}1.4$	$\Psi_{c,V} = 1.4$		Shear (Concrete Breakout)	$\varphi V_{cbg} = A_{Vc} / A_{Vco} * \psi_{ec, V} \psi_{ed, V} \psi_{c, V} \psi_{h, V} * V_b$	= 0.75*0.7467*1*1*1.4*1.46*10273	= 1179	90 lb	>	225	I
$\Psi_{cp,N} = 1$	$\psi_{h,V} = 1.46$		Shear (Concrete Pryout)	$\Phi V_{cpg} = k_{cp} N_{cpg}$	= 0.75*2*8541	= 128	12 lb	>	225	]
$N_{b} = 9503 \text{ lb}$	$V_{b} = 10273 \text{ lb}$									
$\Psi_{ed,Na} = 1$	k <sub>cp</sub> = 2									
	ep <b>2</b>									
$\Psi_{cp,Na} = 1$	N <sub>cpg</sub> = 8541 lb									

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Variab	les	Condition	Load Case	Equation	Calculation	Nominal Capa	city	Required Design Load
Rail - 2" x 2" x 1/8" 60	61 Aluminum Tube		l	l				Louid
$A_w = 0.4375 \text{ in}^2$			Shear	$\phi V_n = \phi F_s A_w$	= 0.9*21000*0.4375	= 8268.8 lb	>	348.2 lb
b = 1.75 in.		No Welding	Flexural Yielding	$\phi M_n = \phi 1.3 F_{tv} S_t$	= 0.9*1.3*35000*0.2759/12	= 941.5 ft-lt	>	562.5 ft-lb
t = 0.25 in.			Flexural Rupture	$\varphi M_n = \varphi 1.42 F_{tu} S_t$	= 0.75*1.42*38000*0.2759/12	= 930.5 ft-lt	>	562.5 ft-lb
$b/t = 7 < S_1$			Shear	$\phi V_n = \phi F_s A_w$	= 0.9*9000*0.4375	= 3543.8 lb	>	348.2 lb
$F_s = F_{sy} \text{ or } F_{syw}$		Fully Welded	Flexural Yielding	$\phi M_n = \phi 1.3 F_{tyw} S_t$	= 0.9*1.3*15000*0.2759/12	= 403.5 ft-lt	>	356.5 ft-lb
$S_t = 0.2759 \text{ in}^3$			Flexural Rupture	$\phi M_n = \phi 1.42 F_{tuw} S_t$	= 0.75*1.42*24000*0.2759/12	= 587.7 ft-lt	>	356.5 ft-lb
$A_{wzt} = 0.25 \text{ in}^2$			Flexural Yielding	$\varphi Mn = \varphi 1.30 [F_{tv} (1 - \frac{Awzt}{Ag}) + F_{tvw} (\frac{Awzt}{Ag})]S_t$	= 0.9*1.3*(35000*(125/.4688)+15000*0.25/0.4688)*0.2759/12	= 654.6 ft-lt	>	562.5 ft-lb
Agt =0.4688 in <sup>2</sup>		One-side welded	Flexural Rupture	$\varphi Mn = \varphi 1.42 [F_{tu}(1 - \frac{Awzt}{Ag})/k_t + F_{tuw}(\frac{Awzt}{Ag})]S_t$	= 0.75*1.42*(38000*(125/.4688)/1.0+24000*0.25/0.4688)*0.2759/12	= 747.7 ft-lt	>	562.5 ft-lb
Post - 2" x 3" x 1/8" 60	61 Aluminum Tube		· · · · ·	· · · · · · · ·				
$A_{\rm w} = 0.75  {\rm in}^2$	$b/t = 11 < S_1$		Shear	$\phi V_n = \phi F_s A_w$	= 0.9*21000*0.75	= 14175 lb	>	450 lb
b = 2.75 in.	$F_s = F_{sy} \text{ or } F_{syw}$	No Welding	Flexural Yielding	$\phi M_n = \phi 1.3 F_{ty} S_t$	= 0.9*1.3*35000*0.4890/12	= 1668.7 ft-lt	>	1537.5 ft-lb
t = 0.25 in.	$S_t = 0.4890 \text{ in}^3$		Flexural Rupture	$\phi M_n = \phi 1.42 F_{tu} S_t$	= 0.75*1.42*38000*0.4890/12	= 1649.2 ft-lt	>	1537.5 ft-lb
Spindles - 1/2" x 1/2" :	solid square			-				
$A = 0.25 \text{ in}^2$			Shear	$\phi V_n = \phi F_{sy} A_g$	= 0.9*21000*0.25	= 4725 lb	>	7.9 lb
$S_t = 0.0104 \text{ in}^3$		No Welding	Flexural Yielding	$\phi M_n = \phi 1.3 F_{ty} S_t$	= 0.9*1.3*35000*0.0104/12	= 35.5 ft-lt	>	4.0 ft-lb
			Flexural Rupture	$\phi M_n = \phi 1.42 F_{tu} S_t$	= 0.75*1.42*38000*0.0104/12	= 35.1 ft-lt	>	4.0 ft-lb
			Shear	$\phi V_n = \phi F_{svw} A_g$	= 0.9*9000*0.25	= 2025 lb	>	7.9 lb
		Fully Welded	Flexural Yielding	$\phi M_n = \phi 1.3 F_{tyw} S_t$	= 0.9*1.3*15000*0.0104/12	= 15.2 ft-lt	>	2.7 ft-lb
			Flexural Rupture	$\phi M_n = \phi 1.42 F_{tuw} S_t$	= 0.75*1.42*24000*0.0104/12	= 22.2 ft-lt	>	2.7 ft-lb
Base Plate - 7 1/2" x 3	1/2" x 3/8" 6061 Al	uminum Plate	•	· · ·				
B = 3.5 in.	$t = \frac{3}{8}$ in.	E-lle W-lle l	Method #1 - Vertical Force	$\phi P_n = \phi F_{tyw} BN/2 * (t/l)^2$	= 0.9*15000*3.5*7.5/2*(.375/2.325)^2	= 4609.5 lb	<	6150 lb
N = 7.5 in.	1 = 2.325 in.	Fully Welded	Method #2 - Moment	$\varphi M_n = \varphi F_{tyw} t^2 / 4$	= 0.9*15000*(.375)^2/4	= 474.6 in1	o∕in. <	1862.5 inlb/in.
Sleeve - 3 5/8" x 2 5/8	" x 1/4" 6061 Alum	inum Plate						
$A_w = 1.5625 \text{ in}^2$	$b/t = 6.25 < S_1$		Shear	$\phi V_n = \phi F_s A_w$	= 0.9*9000*1.5625	= 12656.3 lb	>	450 lb
b = 3.125 in.	$F_s = F_{sy}$ or $F_{syw}$	Fully Welded	Flexural Yielding	$\phi M_n = \phi 1.3 F_{tyw} S_t$	= 0.9*1.3*15000*1.3837/12	= 2023.7 ft-lt	>	1537.5 ft-lb
t = 0.5 in.	$S_t = 1.3837 \text{ in}^3$		Flexural Rupture	$\phi M_n = \phi 1.42 F_{tuw} S_t$	= 0.75*1.42*24000*1.3837/12	= 2947.3 ft-lt	>	1537.5 ft-lb
Sleeve to Base Plate V	Weld - 3/16'' 5356 F	Filler						
e = 0.1326 in.	$I = 2.1346 \text{ in.}^4$	Weld	Shear	$\phi R_n = \phi F_{sw} L_{we}$	= 0.75*17000*0.1326*10.5	= 17749.5 lb	>	450 lb
$L_{we} = 10.5$ in.	c = 1.6094 in.	wen	Moment	$\phi M_n = \phi F_{su} * I/c$	= 0.75*17000*2.1346/1.6094/12	= 1409.2 ft-lt	<	1537.5 ft-lb
Rail to Post Weld - 1/8"	5356 Filler		1	1				
e = 0.0884 in.	$I = 0.4938 \text{ in.}^4$	Weld	Shear	$\phi R_n = \phi F_{sw} L_{we}$	= 0.75*17000*0.0884*8	= 9015.6 lb	>	348.2 lb
L <sub>we</sub> =8 in.	c = 1.03125 in.	W CRI	Moment	$\phi M_n = \phi F_{su} * I/c$	= 0.75*17000*0.4941/1.03125/12	= 509.0 ft-lt	>	356.5 ft-lb
Spindles to Rail Weld								
e = 0.0884 in.	$I = 0.0089 \text{ in.}^4$	Weld	Shear	$\phi R_n = \phi F_{sw} L_{we}$		= 2253.9 lb	>	7.9 lb
$L_{we} = 2$ in.	c = 0.28125 in.		Moment	$\phi M_n = \phi F_{su} * I/c$	= 0.75*17000*0.0089/0.28125/12	= 33.6 ft-lt	>	2.7 ft-lb
Anchor Bolts - 3/8" Dia		readed Rod, Embedde		T				
$A_{se,V} = A_{se,V} = 0.078 \text{ in.}^2$	N <sub>ba</sub> = 8541 lb		Tension (Steel)	$\phi N_{sa} = \phi A_{sc,N} f_{uta}$	= 0.75*0.078*125000	= 7313 lb	>	
$f_{uta} = 125,000 \text{ psi}$	$A_{Vc}\!/A_{Vco}{=}0.7467$		Tension (Concrete Breakout)	$\phi N_{cb} = \phi A_{Nc} / A_{Nco} \psi_{ed,N} \psi_{c,N} \psi_{cp,N} N_b$	= 0.65*1*1.4*1*9503	= 8648 lb	>	3075 lb
$A_{Nc}/A_{Nco} = 1$	$\psi_{ec,V} = 1$		Tension (Adhesive Bond)	$\phi N_a = \phi A_{Nc} / A_{Nco} \psi_{cd,Na} \psi_{cp,Na} N_{ba}$	= 0.65*1*1*1*8541	= 5552 lb	>	3075 lb
$\Psi_{ed,N} = 1$	$\Psi_{ed,V} = 1$	Anchor	Shear (Steel)	$\phi N_{sa} = \phi 0.6 A_{sc.V} f_{uta}$	= 0.65*0.6*0.078*125000	= 3803 lb	>	225 lb
$\psi_{c,N} = 1.4$	$\psi_{c,V} = 1.4$		Shear (Concrete Breakout)	$ \phi V_{cbg} = A_{Vc}/A_{Vco} * \psi_{ec,V} \psi_{ed,V} \psi_{c,V} \psi_{h,V} * V_b $	= 0.75*0.7467*1*1*1.4*1.46*10273	= 11790 lb	>	225 lb
$\Psi_{cp,N} = 1$	$\Psi_{h,V} = 1.46$		Shear (Concrete Pryout)	$\phi V_{cpg} = k_{cp} N_{cpg}$	= 0.75*2*8541	= 12812 lb	>	225 lb
$N_{b} = 9503 \text{ lb}$	$V_{b} = 10273 \text{ lb}$							
	5							
W 1	k - 2							1
$\psi_{ed,Na} = 1$ $\psi_{cp,Na} = 1$	$k_{cp} = 2$ $N_{cpg} = 8541$ lb							

Figure C-23. Final Design Calculations, Concept AW2-C

Variabl	es	Condition	Load Case	Equation	Calculation	Nominal	Capacit	7	Required Desig Load
Rail - 2" x 2" x 1/8"	6061 Aluminum	Tube							Louid
$A_w = 0.4375 \text{ in}^2$			Shear	$\phi V_n = \phi F_s A_w$	= 0.9*21000*0.4375	= 8268.	8 lb	>	348.2 lb
b = 1.75 in.		No Welding	Flexural Yielding	$\varphi M_n = \varphi 1.3 F_{ty} S_t$	= 0.9*1.3*35000*0.2759/12	= 941.	5 ft-lb	>	562.5 ft-lb
t = 0.25 in.			Flexural Rupture	$\phi M_n = \phi 1.42 F_{tu} S_t$	= 0.75*1.42*38000*0.2759/12	= 930.	5 ft-lb	>	562.5 ft-lb
$b/t = 7 < S_1$			Shear	$\phi V_n = \phi F_s A_w$	= 0.9*9000*0.4375	= 3543.	8 lb	>	348.2 lb
$F_s = F_{sy}$ or $F_{syw}$		Fully Welded	Flexural Yielding	$\phi M_n = \phi 1.3 F_{tyw} S_t$	= 0.9*1.3*15000*0.2759/12	= 403.	5 ft-lb	>	356.5 ft-lb
$S_t = 0.2759 \text{ in}^3$			Flexural Rupture	$\phi M_n = \phi 1.42 F_{tuw} S_t$	= 0.75*1.42*24000*0.2759/12	= 587.	7 ft-lb	>	356.5 ft-lb
$A_{wzt} = 0.25 \text{ in}^2$		0 11 11 1	Flexural Yielding	$\varphi Mn = \varphi 1.30 [F_{ty}(1 - \frac{Awzt}{Ag}) + F_{tyw}(\frac{Awzt}{Ag})]S_t$	= 0.9*1.3*(35000*(125/.4688)+15000*0.25/0.4688)*0.2759/12	= 654.	6 ft-lb	>	562.5 ft-lb
Agt =0.4688 in <sup>2</sup>		One-side welded	Flexural Rupture	$\phi Mn = \phi 1.42 [F_{tu}(1-^{Awzt}/_{Ag})/k_t + F_{tuw}(^{Awzt}/_{Ag})]S_t$	$t_1 = 0.75*1.42*(38000*(125/.4688)/1.0+24000*0.25/0.4688)*0.2759/12$	= 747.	7 ft-lb	>	562.5 ft-lb
ost - 2" x 4" x 1/4"	" 6061 Aluminur	n Tube							
$A_w = 2 in^2$			Shear	$\phi V_n = \phi F_s A_w$	= 0.9*21000*2	= 3780	0 lb	>	450 lb
b = 3.5 in.		No Welding	Flexural Yielding	$\varphi M_n = \varphi 1.3 F_{ty} S_t$	= 0.9*1.3*35000*1.3268/12	= 4527.	7 ft-lb	>	1537.5 ft-lb
t = 0.5 in.			Flexural Rupture	$\varphi M_n = \varphi 1.42 F_{tu} S_t$	= 0.75*1.42*38000*1.3268/12	= 4474.	6 ft-lb	>	1537.5 ft-lb
$b/t = 7 < S_1$			Shear	$\phi V_n = \phi F_s A_w$	= 0.9*9000*2	= 1620	0 lb	>	450 lb
$F_s = F_{sy}$ or $F_{syw}$		Fully Welded	Flexural Yielding	$\varphi M_n = \varphi 1.3 F_{tyw} S_t$	= 0.9*1.3*15000*1.3268/12	= 1940.	4 ft-lb	>	1537.5 ft-lb
$S_t = 1.3268 \text{ in}^3$			Flexural Rupture	$\phi M_n = \phi 1.42 F_{tuw} S_t$	= 0.75*1.42*24000*1.3268/12	= 2826.	1 ft-lb	>	1537.5 ft-lb
pindles - 1/2'' x 1/2	2'' solid square			-					
$A = 0.25 \text{ in}^2$			Shear	$\phi V_n = \phi F_{sy} A_g$	= 0.9*21000*0.25	= 472	5 lb	>	10.5 ⊯
$S_t = 0.0104 \text{ in}^3$		No Welding	Flexural Yielding	$\phi M_n = \phi 1.3 F_{ty} S_t$	= 0.9*1.3*35000*0.0104/12	= 35.	5 ft-lb	>	7.0 ft-lb
			Flexural Rupture	$\phi M_n = \phi 1.42 F_{tu} S_t$	= 0.75*1.42*38000*0.0104/12	= 35.	1 ft-lb	>	7.0 ft-lb
			Shear	$\phi V_n = \phi F_{syw} A_g$	= 0.9*9000*0.25	= 202	5 lb	>	10.5 lb
		Fully Welded	Flexural Yielding	$\phi M_n = \phi 1.3 F_{tyw} S_t$	= 0.9*1.3*15000*0.0104/12	= 15.	2 ft-lb	>	4.7 ft-lb
			Flexural Rupture	$\phi M_n = \phi 1.42 F_{tuw} S_t$	= 0.75*1.42*24000*0.0104/12	= 22.	2 ft-lb	>	4.7 ft-lb
Base Plate - 7 3/4"	x 3" x ¾" 6061	Aluminum Plate							
$\mathbf{B} = 3$ in.	$t = \frac{3}{8}$ in.	Fully Welded	Method #1 - Vertical Force	$\varphi P_n = \varphi F_{tyw} BN/2^* (t/l)^2$	= 0.9*15000*3*7.75/2*(.375/1.975)^2	= 5657.	9 lb	>	4312.5 lb
N = 7.75 in.	1 = 1.975 in.	ruiy weided	Method #2 - Moment	$\phi M_n = \phi F_{tyw} t^2 / 4$	= 0.9*15000*(.375)^2/4	= 474.	6 inlb/ir	L <	1731.4 inlb/in
Post to Base Plate V	Weld - 1/4" 5356	ó Filler							
e = 0.1768 in.	$I = 4.8961 \text{ in.}^4$	Weld	Shear	$\phi R_n = \phi F_{sw} L_{we}$	= 0.75*17000*0.0884*8	= 9015.	6 lb	>	450 lb
$L_{we} = 12$ in.	c = 2.0625 in.	wen	Moment	$\phi M_n = \phi F_{su} * I/c$	= 0.75*17000*0.4941/1.03125/12	= 509.	0 ft-lb	>	1537.5 ft-lb
ail to Post Weld -	/8" 5356 Filler								
e = 0.0884 in.	$I = 0.4938 \text{ in.}^4$	Weld	Shear	$\phi R_n = \phi F_{sw} L_{we}$	= 0.75*17000*0.0884*8	= 9016.	8 lb	>	348.2 lb
Lwe =8 in.	c = 1.03125 in.	Weki	Moment	$\phi M_n = \phi F_{su} * I/c$	= 0.75*17000*0.4938/1.03125/12	= 508.	8 ft-lb	>	356.5 ft-lb
pindles to Rail We	1	ler	T						
e = 0.0884 in.	$I = 0.0089 \text{ in.}^4$	Weld	Shear	$\phi R_n = \phi F_{sw} L_{we}$	= 0.75*17000*0.0884*2	= 2253.	9 lb	>	7.9 lb
	c = 0.28125 in.		Moment	$\phi M_n = \phi F_{su} * I/c$	= 0.75*17000*0.0089/0.28125/12	= 33.	6 ft-lb	>	2.7 ft-lb
		B7 Threaded Rod, Er	nbedded 5'', at 6 1/4'' spacing						
$A_{se,V} = 0.142 \text{ in.}^2$	$N_{ba}\!=11388~lb$		Tension (Steel)	$\phi N_{sa} = \phi A_{se,N} f_{uta}$	= 0.75*0.142*125000	= 1331	3 lb	>	3075 lb
f <sub>uta</sub> = 125,000 psi\ <sub>v</sub>	$V_{c}/A_{Vco} = 0.6042$		Tension (Concrete Breakout)	$\phi N_{cb} = \phi A_{Nc} / A_{Nco} \psi_{ed,N} \psi_{c,N} \psi_{cp,N} N_b$	= 0.65*1*1.4*1*9503	= 864	8 lb	>	3075 lb
$A_{Nc}/A_{Nco} = 1$	$\Psi_{ec,V} = 1$		Tension (Adhesive Bond)	$\phi N_a = \phi A_{Nc} / A_{Nco} \psi_{ed,Na} \psi_{cp,Na} N_{ba}$	$= 0.65^{*}1^{*}1^{*}1^{*}11388$	= 740	2 lb	>	3075 lb
$\Psi_{ed,N} = 1$	$\Psi_{ed,V} = 1$	Anchor	Shear (Steel)	$\phi N_{sa} = \phi 0.6 A_{se} \sqrt{f_{uta}}$	= 0.65*0.6*0.142*125000	= 692	3 lb	>	225 lb
$\Psi_{c,N} = 1.4$	$\Psi_{c,V} = 1.4$		Shear (Concrete Breakout)	$\phi V_{cbg} = A_{Vc}/A_{Vco} * \psi_{ec,V} \psi_{ed,V} \psi_{c,V} \psi_{h,V} * V_b$	= 0.75*0.6042*1*1*1.4*1.46*1186?	= 1101	6 lb	>	225 lb
$\psi_{c,N} = 1.4$ $\psi_{cp,N} = 1$	$\psi_{c,V} = 1.4$ $\psi_{h,V} = 1.46$		Shear (Concrete Pryout)	$ \phi V_{cpg} = k_{cp} N_{cpg} $	= 0.75 0.0042 1 1 1.4 1.40 11802	= 1708		>	225 lb
$\psi_{cp,N} = 1$ N <sub>b</sub> = 9503 lb			shear (Concrete Fryolit)	Y cpg cp cpg	- 0.15 2 11300	_ 1708	- 10		<i>223</i> IO
-	-								
$\psi_{ed,Na}=1$	$k_{cp} = 2$								
$\Psi_{cp,Na} = 1$	$N_{cpg} = 11388 \text{ lb}$								

Figure C-24. Final Design Calculations, Concept AW2-D

Variables	s	Condition	Load Case	Equation	Calculation	Nominal Capacity	, 1	Required Design Load
Rail - 2" Dia. Schedule 4	40 6061 Aluminun	n Pipe						
$A_g = 1.0745 \text{ in}^2$	$\mathbf{F}_{s} = \mathbf{F}_{sy}$		Shear	$\phi V_n = \phi F_s A_g/2$	= 0.9*21000*1.0745/2 =	10154.0 lb	>	348.2 lb
$R_b = 2.22$ in.	$S = 0.5606 \text{ in}^3$		Flexural Tensile Yielding	$\phi M_n = \phi 1.17 F_{ty} S$	= 0.9*1.17*35000*0.5606/12 =	1721.7 ft-lb	>	562.5 ft-lb
t = 0.31 in.	$R_b/t=7.16\leq S_1$	No Welding	Flexural Tensile Rupture	$\varphi M_n = \varphi 1.24 F_{tu} S/k_t$	= 0.75*1.24*38000*0.5606/12 =	1651.0 ft-lb	>	562.5 ft-lb
$L_v = 60$ in.	F <sub>b</sub> = 52.87 ksi		Flexural Compressive Yielding	$\phi M_n = \phi 1.17 F_{cy} S$	= 0.9*1.17*35000*0.5606/12 =	1721.7 ft-lb	>	562.5 ft-lb
$\lambda_t=22.63\leq S_1$			Flexural Local Buckling	$\phi M_n = \phi F_b S$	= 0.9*(52.87*1000)*0.5606/12 =	2222.9 ft-lb	>	562.5 ft-lb
Post - 2" Dia. Schedule 8		n Pipe		l .				
$A_g = 1.4773 \text{ in}^2$	$\mathbf{F}_{s} = \mathbf{F}_{sy}$		Shear	$\varphi V_n = \varphi F_s A_g/2$	= 0.9*21000*1.4773/2 =	13960.5 lb	>	450 lb
$R_b = 2.16$ in.	$S = 0.7309 \text{ in}^3$		Flexural Tensile Yielding	$\phi M_n = \phi 1.17 F_{ty} S$	= 0.9*1.17*35000*0.7309/12 =	2244.8 ft-lb	>	1537.5 ft-lb
t = 0.44 in.	$R_b/t = 4.91 \leq S_1$	No Welding	Flexural Tensile Rupture	$\phi M_n = \phi 1.24 F_{tu} S/k_t$	= 0.75*1.24*38000*0.7309/12 =	2152.5 ft-lb	>	1537.5 ft-lb
$L_v = 41$ in.	F <sub>b</sub> = 54.92 ksi		Flexural Compressive Yielding	$\phi M_n = \phi 1.17 F_{cy} S$	= 0.9*1.17*35000*0.7309/12 =	2244.8 ft-lb	>	1537.5 ft-lb
$\lambda_t = 18.64 \ \leq S_1$			Flexural Local Buckling	$\phi M_n = \phi F_b S$	= 0.9*(54.92*1000)*0.7309/12 =	3010.6 ft-lb	>	1537.5 ft-lb
Spindles - 3/4" Dia. Sche		minum Pipe		l .				
$A_g = 0.2577 \text{ in}^2$	$\mathbf{F}_{s} = \mathbf{F}_{sy}$		Shear	$\phi V_n = \phi F_s A_g/2$	= 0.9*21000*0.2577/2 =	2435.3 lb	>	7.9 lb
$R_b = 0.97$ in.	$S_t = 0.0566 \text{ in}^3$		Flexural Tensile Yielding	$\phi M_n = \phi 1.17 F_{ty} S$	= 0.9*1.17*35000*0.0566/12 =	98.3 ft-lb	>	4.0 ft-lb
t = 0.083 in.	$\mathrm{R_{b}/t} = 11.69 \leq \mathrm{S_{1}}$	No Welding	Flexural Tensile Rupture	$\varphi M_n = \varphi 1.24 F_{tu} S/k_t$	= 0.75*1.24*38000*0.0566/12 =	97.1 ft-lb	>	4.0 ft-lb
$L_v = 12.125$ in.	F <sub>b</sub> = 49.56 ksi		Flexural Compressive Yielding	$\phi M_n = \phi 1.17 F_{cy} S$	= 0.9*1.17*35000*0.0566/12 =	98.3 ft-lb	>	4.0 ft-lb
$\lambda_t = 13.99 \ \leq S_1$			Flexural Local Buckling	$\phi M_n = \phi F_b S$	= 0.9*(49.56*1000)*0.0566/12 =	210.4 ft-lb	>	4.0 ft-lb
Base Plate - 7 1/2" x 3"	x 3/8" 535 Alumin	um Alloy Casting						
B = 5 in.	t = 9/16 in.	Fully Welded	Method #1 - Vertical Force	$\phi P_n = \phi F_{ty} BN/2^* (t/l)^2$	= 0.9*13500*5*8.5/2*(0.5625/3.122)^2 =	8381.4 lb	>	7768.4 lb
N = 8.5 in.	l = 3.122 in.	-		$\phi M_n = \phi F_{ty} t^2 / 4$	= 0.9*13500*(0.5625)^2/4 =	961.1 inIb/in	. <	1805.1 inIb/in.
Anchor Bolts - 1/2" Dian	meter A193 B7 T	hreaded Rod, Embedd	led 3 1/2", at 5 1/2" spacing	1				
$A_{se,N} = A_{se,V} = 0.142 \text{ in.}^2$	$N_{ba} = 7972 \text{ lb}$		Tension (Steel)	$\phi N_{sa} = \phi A_{se,N} f_{uta}$	= 0.75*0.142*125000 =	13313 lb	>	3075 lb
f <sub>uta</sub> = 125,000 psi A	$A_{Vc}/A_{Vco} = 0.5917$		Tension (Concrete Breakout)	$\phi N_{cb} = \phi A_{Nc} / A_{Nco} \psi_{ed,N} \psi_{c,N} \psi_{cp,N} N_b$	= 0.65*1*1.4*1*5566 =	5065 lb	>	3075 Ib
$A_{Nc}/A_{Nco} = 1$	$\psi_{ec,V} = 1$		Tension (Adhesive Bond)	$\phi N_a = \phi A_{Nc} / A_{Nco} \psi_{ed,Na} \psi_{cp,Na} N_{ba}$	= 0.65*1*1*1*7972 =	5182 lb	>	3075 lb
$\Psi_{ed,N} = 1$	$\Psi_{ed,V} = 1$	Anchor	Shear (Steel)	$\phi N_{sa} = \phi 0.6 A_{sc,V} f_{uta}$	= 0.65*0.6*0.142*125000 =	6923 lb	>	225 lb
$\psi_{\mathrm{c},\mathrm{N}}{=}1.4$	$\Psi_{c,V} = 1.4$		Shear (Concrete Breakout)	$\phi V_{cbg} = A_{Vc} A_{Vco} * \psi_{ec,V} \psi_{ed,V} \psi_{c,V} \psi_{h,V} * V_b$	= 0.75*0.5917*1*1.4*1.46*11550 =	10504 lb	>	225 lb
$\Psi_{cp,N} = 1$	$\psi_{h,V} = 1.46$		Shear (Concrete Pryout)	$\oint \mathbf{V}_{cpg} = \mathbf{k}_{cp} \mathbf{N}_{cpg}$	= 0.75*2*7792 =	11688 lb	>	225 lb
N <sub>b</sub> =5566 lb	V <sub>b</sub> = 11550 lb							
$\psi_{ed,Na}=1$	$k_{cp} = 2$							

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 $\begin{array}{c|c} & & & & \\ \hline \psi_{\text{cp,Na}} = 1 & & & \\ \hline W_{\text{cp,Na}} = 1 & & & \\ \hline \end{array} \\ \hline Figure C-25. Final Design Calculations, Concept AM-1 \end{array}$ 

# Appendix D. Material Specifications

Pedestrian Rail Design AW2-A and AW2-D (WIPR-1 and WIPR-4)											
ltem No.	Description	Material Spec	Reference								
a1	2"x4"x1/4" [51x102x6] Aluminum Post, 43" [1092] long	6061-T6	H# 21311648								
a2	Aluminum Post Cap - 1/8" [3] Plate	6061-T6	R# 14-0473 L# 21635829								
a3	Aluminum Post Base	6061-T6	R# 14-0473 L# 212073 & 539961								
d1	2"x2"x1/8" [51x51x3] Aluminum Rail - 60" [1524] long	6061-T6	H# 201405597								
d2	1/2"x1/2" [13x13] Square Aluminum Spindle - 24 1/4" [616] long	6061-T6	H# 201405836								
d3	3/8" [10] Dia. Threaded Rod	ASTM A193 Grade B7 Galv.	Grainger COC R# 14-0433 - 4FHG3								
d4	3/8" [10] Dia. Nut	ASTM A194 Grade 8M Galv	Grainger COC R# 14-0433 - 1XA24								
d5	3/8" [10] Dia. SAE Flat Washer	ASTM F436 Type 1 Galv.	Grainger COC R# 14-0433 - 6PE80								
d6	Ероху	Minimum bond strength = 1,450 psi [10.0 MPa]	June 2014 C300								

	Pedestrian Rail Design AW2-C (WIPR-2)												
Item	n No.	Description	Material Spec	Reference									
С	:1	2"x3"x1/8" [51x76x3] Aluminum Post, 43" [1092] long	6061-T6	H# 21393458									
С	:2	Aluminum Post Cap - 1/8" [3] Plate	6061-T6	R# 14-0473 L# 21635829									
С	:3	Aluminum Post Base	6061-T6	R# 14-0473 L# 212073 & 539961									
N C	:4	1/4" [6] Dia., 3" [76] Long Bolt and Nut Bolt	ASTM A193 Grade B8M Class 2, Nut ASTM A194 Grade 8M	Grainger COC - IVZA6									
	11	2"x2"x1/8" [51x51x3] Aluminum Rail - 60" [1524] long	6061-T6	H# 201405597									
d	12	1/2"x1/2" [13x13] Square Aluminum Spindle - 24 1/4" [616] long	6061-T6	H# 201405836									
d	13	3/8" [10] Dia. Threaded Rod	ASTM A193 Grade B7 Galv.	Grainger COC R# 14-0433 - 4FHG3									
d	14	3/8" [10] Dia. Nut	ASTM A194 Grade 8M Galv.	Grainger COC R# 14-0433 - 1XA24									
d	<u>15</u>	3/8" [10] Dia. SAE Flat Washer	ASTM F436 Type 1 Galv.	Grainger COC R# 14-0433 - 6PE80									
d	16	Ероху	Minimum bond strength = 1,450 psi [10.0 MPa]	June 2014 C300									

Figure D-1. Bill of Materials and Material Reference, Test Nos. WIPR-1, WIPR-2, and WIPR-4

	Pedestrian Rail Design AM-1 (WIPR-3)								
Item No.	Description	Material Spec	Reference						
b1	2" [51] Dia. Schedule 80 post, 39" [991] long	6061-T6 Aluminum	Item# G00369485 L# 21684972 H# S14033401						
b2	2" [51] Dia. Schedule 40 rail, 56 1/2" [1435] long	6061-T6 Aluminum	Item# G03369473 L# 21633667 H# S14010202						
b3	3/4" [19] Dia. Schedule 10 picket, 22" [559] long	6063-T6 Aluminum	Cast# 34391						
b4	No. 3 Elbow (2" [51])	6061-T6 Aluminum	See Alex						
b5	No. 5 Tee (2" [51])	6061-T6 Aluminum	See Alex						
b6	No. 7 Cross (2" [51])	6061-T6 Aluminum	See Alex						
b7	No. 48 Heavy-Duty Base Flange (2" [51], 2-hole)	6061-T6 Aluminum	See Alex						
b8	1/2" [13] Dia. Threaded Rod ASTM	A193 Grade B7 Galv.	Grainger COC R# 14-0433 - 4FHF3						
b9	1/2" [13] Dia. Nut	ASTM A194 Grade 8M Galv.	Ken						
b10	1/2" [13] Dia. SAE Flat Washer	ASTM F436 Type 1 Galv.	Ken						
b11	Ероху	Minimum bond strength = 1,450 psi [10.0 MPa]	June 2014 C300						

		Pedes	trian Rail Design (APR-1 and APR-2)	
	ltem No.	Description	Material Spec	Reference
	a1	2"x4"x1/4" [51x102x6] Aluminum Post, 43" [1092] long	6061-T6	R#15-0098 H# 21550443
З	a2	Aluminum Post Cap - 1/8" [3] Plate	6061-T6	R#15-0098 No Definite Heat #
372	a3	Aluminum Post Base	6061-T6	R#15-0098 L# 2307073D0
10	d1	2"x2"x1/8" [51x51x3] Aluminum Rail - 63 1/2" [1613] long	6061-T6	R#15-0098 H# 21836702
	d2	2"x2"x1/8" [51x51x3] Aluminum Rail - 63 1/2" [1613] long with holes	6061-T6	R#15-0098 H# 21836702
	d3	1/2"x1/2" [13x13] Square Aluminum Spindle - 32 1/8" [816] long	6061-T6	R#15-0098 H# 201408541
	d4	1/2" [13] Dia. UNC, 6" [152] long Threaded Rod	ASTM A193 Grade B7 Galv.	R# 15-0188 H# E21306214 L# 1401071935C
	d5	1/2" [13] Dia. Steel Nut	ASTM A194 Grade 8M Galv	R# 15-0188 H# NF12104365 L# 325254B
	d6	1/2" [13] Dia. SAE Steel Flat Washer	ASTM F436 Type 1 Galv.	R# 15-0188 H# 342288 L# C7313D
	d7	Ероху	Minimum bond strength = 1,450 psi [10.0 MPa]	TECHNICAL DATA AVAILABLE ONLINE

Figure D-2. Bill of Materials and Material Reference, Test Nos. WIPR-3, APR-1, and APR-2

	CATE O	F TES	T					Certi	Page 01 c ificatior 10-APR-20	n Date
39584	ORDER NUN 4 PART NUMB	ER	6661	EARLE M. 1800 N U KANSAS (	JNIVERS	SAL AVEN	UE		oice Numk 5107282	ber
SOLD TO:	RIVERS N	IETAL	PRODUCT	S SH	IP TO:	RIV	ERS META	AL PRODUC	CTS	
	3100 N 3 LINCOLN		68504				0 NORTH COLN NI			
2 X 4 X	ion: 60 .250 W X 1311648	20'		DED PORTH		Li	STM B22: ne Tota	1 Q 1: 39.270	03 FT	
Specifica ASTM B223 EN 10204			QQ-	-A-200/8			ams q	Q A 200/8	8 97	
				NINIUM CI						
DESCRIPT: MIN	ION: SI 0.4 0.8	FE	CU 0.15	MN	MG 0.8	CR 0.04	ZN	TI		
OTHERS	: EACH	TOT	AL 5	אד ז		DED				*v
VENDOR:	911497 SAPA PROI	FILES	NORTH 2	AMERICA		UNTRY OF		: USA		
VENDOR:	911497	FILES	NORTH 2	AMERICA	C0.	UNTRY OF		: USA		
VENDOR:	911497 SAPA PROI	FILES YLD KS 44.1 45.5	NORTH J STR I	MERICA MECHAI ULT TEL KSI 47.0 48.9	CO NICAL N % IN	UNTRY OF PROPERTI ELONG 02 IN 13.5 14.0	ES %RED IN ARE.	HARDNE. A		
VENDOR:	911497 SAPA PROI	FILES YLD KS 44.1 45.5	NORTH J STR I	MERICA MECHAI ULT TEL KSI 47.0 48.9	CO NICAL N % IN	UNTRY OF PROPERTI ELONG 02 IN 13.5 14.0	ES %RED IN ARE.	HARDNE		
	911497 SAPA PROI	FILES YLD KS 44.1 45.5	NORTH J STR I	MERICA MECHAI ULT TEL KSI 47.0 48.9	CO NICAL N % IN	UNTRY OF PROPERTI ELONG 02 IN 13.5 14.0	ES %RED IN ARE.	HARDNE. A		
VENDOR :	911497 SAPA PROI	FILES YLD KS 44.1 45.5 cribed from rification req cet to examin material cor g any specifi ulse, fictitiou	NORTH 2 STR I SI the manufactur uirements of th nation. vered by this re cation forming s, or fraudulen	AMERICA MECHAI ULT TEL KSI 47.0 48.9 er's Certificate of e information on port will meet the a part of the desc	CO NICAL N % IN IN	UNTRY OF PROPERTI ELONG 02 IN 13.5 14.0 	ES %RED IN ARE	HARDNE; A	with mercury whi BUSICK BUSICK	le in

Figure D-3. 2"x4"x<sup>1</sup>/<sub>4</sub>" Aluminum Post Material Certificate, Test Nos. WIPR-1 and WIPR-4

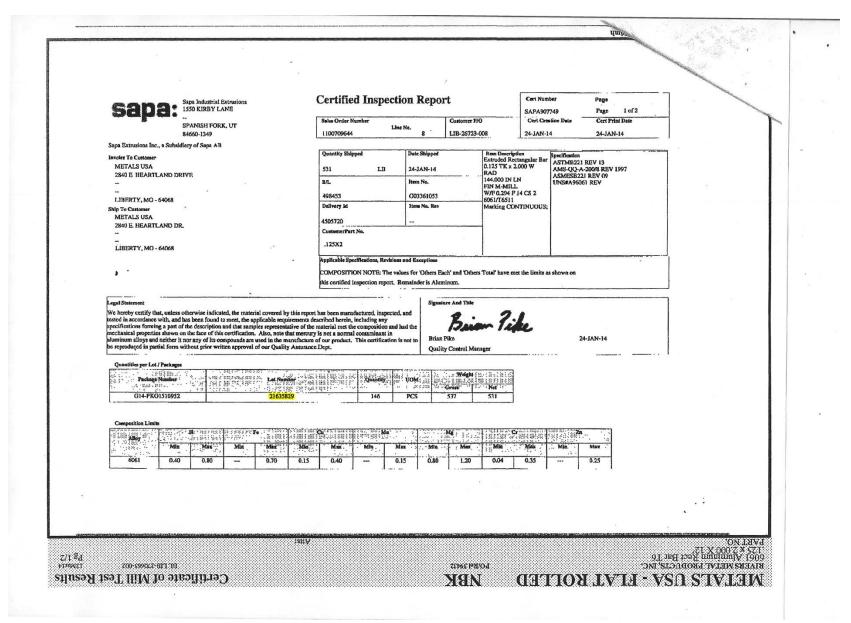
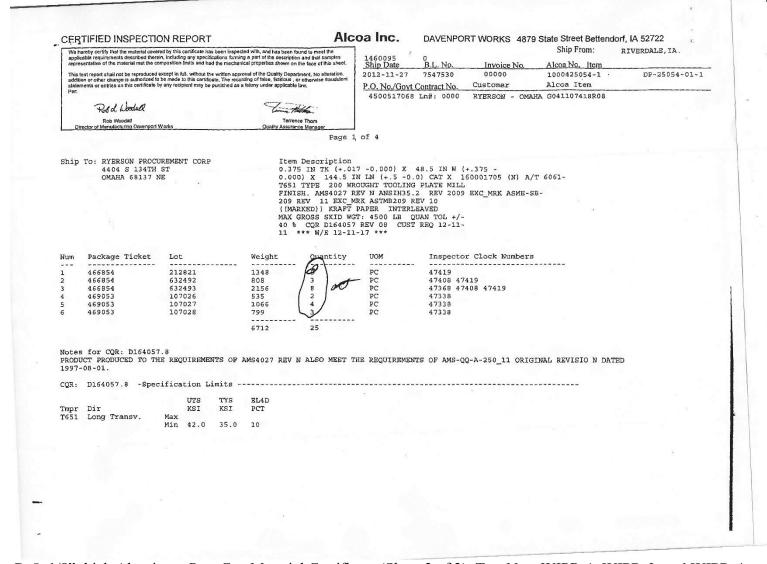
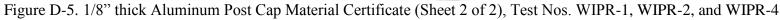


Figure D-4. 1/8" thick Aluminum Post Cap Material Certificate (Sheet 1 of 2), Test Nos. WIPR-1, WIPR-2, and WIPR-4





£ . . Alcoa Inc. CERTIFIED INSPECTION REPORT DAVENPORT WORKS 4879 State Street Bettendorf, IA 52722 Ship From: RIVERDALE, IA. We hereby certify that the material covered by this certificate has been inspected with, and has been found to meet the . . 1436681 Ship Date 0 B.L. No. applicable requirements described therein, including any specifications forming a part of the description and that samples representative of the material met the composition limits and had the mechanical properties shown on the face of this sheet. Alcoa No. Item Invoice No. This test report shall not be reproduced except in full, without the written approval of the Quality Department. No alteration, addition or other change is authorized to be made to this certificate. The recording of false, fictitous, or otherwise fraudulent statements or entities on this certificate by any recipient may be purished as a felory under applicable law. 2012-09-30 7388421 00000 1000404652-1 DP-04652-1 Customer Alcoa Item P.O. No./Govt Contract No. 4500504158 Ln#: 0000 RYERSON - COON RAP G041107416R08 Rold Woodald line table Terrence Thom Rob Woodall Director of Manufacturing Davenport Work Quality Assurance Manage Page 2 of 4 CQR: D164055.8 -Specification Limits -----UTS TYS EL4D Tmpr Dir KSI KSI PCT T651 Long Transv. Max Min 42.0 35.0 10 Other Other Chemical Composition SI FE CU MN MG CR ZN TI Each Total Aluminum Max 0.8 0.7 0.40 0.15 1.2 0.35 0.25 0.15 0.05 0.15 Alloy 6061 Min 0.40 0.15 0.8 0.04 REMAIN Lot: 212073 - Mechanical, Physical, Metallography, Quantometer Results -----UTS TYS EL4D No-> Dir Tmpr Test KSI KSI PCT T651 Long Transv. 48.1 43.7 2 14.1 48.2 43.8 14.5 Cast Number Chemical - OES SI FE CU MN MG CR ZN TI 12L02301 Actuals 0.66 0.5 0.24 0.01 1.0 0.16 0.01 0.05 - Mechanical, Physical, Metallography, Quantometer Results -Lot: 539961 No-> UTS TYS EL4D Tmpr Dir Test KSI KSI PCT T651 48.1 Long Transv. 2 43.6 14.5 48 43.5 14.6 Cast Number Chemical - OES SI FE CU MN MG ZN CR TT H2122011 Actuals 0.64 0.4 0.25 0.05 0.9 0.15 0.03 0.03

Figure D-6. Aluminum Post Base Material Certificate (Sheet 1 of 2), Test Nos. WIPR-1, WIPR-2, and WIPR-4

$ \begin{array}{c ccccccccccccccccccccccccccccccccccc$	appli repre This additi	licable requirements described there esentative of the material met the co- test report shall not be reproduced ( tion or other change is authorized to ements or entries on this certificate b	ed by this certificate has been inspec in, including any specifications formit imposition limits and had the mechar except in full, without the written appu- be made to this certificate. The reco- ry any recipient may be punished as	ng a part of the description nical properties shown on the roval of the Quality Department roting of false, fictitious, or	ind to meet the and that samples he face of this sheet. ment. No alteration, otherwise fraudulent	oa Inc. 1436681 Ship Date 2012-09-30 P.O. No./Govt ( 4500504158		Invoice No. 00000 Customer RYERSON - COOM	Ship From: Alcoa No. Item 1000404652-1 Alcoa Item N RAP G041107416R0	RIVERDALE, IA. DP-04652-1 8	
Dender of Mandataring Damport Works         Quarky Assumed Marger           Page 1 of 4           Ship To: RYERSON PROCUREMENT CORP (COON RAPIDS 55433 MN)         Item Description 0.25 TNK (+.014 -0.000) X 48.5 IN W (+.375 - 0.000) X 144.5 IN LN (+.5 0.0) CAX X 160001700 (N) A/T 6061- T651 TYPE 200 WROUGH TOOLING PLATE MILL PINTSH. AMS4027 REV N ANSITIA5.2 REV 2009 EXC_MRK ASME-SB- 209 REV 10 ((MARKED)) KRAFT PAPER I INTERLEAVED MAX GROSS SKID WGT: 4500 LB QUAN YOL +/- 30 % CQR D164055 REV 08 CUST REQ 12-09- 30 *** W/E 12-10-06 ***           Num         Package Ticket         Lot         Weight         Quantity         UOM         Inspector Clock Numbers           1         441631         212073         1068         6         PC         47200           2         441631         572941         1068         6         PC         47200           441631         572947         1068         6         PC         47200           441631         572941         1068         6         PC         47200           441634         572947         1068         6         PC         47200           441634         572941         1068         6         PC         47200           441634         572946         1068         6         PC         47200           441753         539966         899         5	1	Rold Woodall		lame the	the						
Ship To: RYERSON PROCUREMENT CORP 455 85TH AVE NW COON RAFIDS 55433 MN       Item Description 0.25 IN TK (+.014 -0.000) X 48.5 IN W (+.375 - 0.000) X 144.5 IN LN (+.5 -0.0) CAT X 160001700 (N) A/T 6061- 7651 TYPE 200 RWCUENT TOOLING PLATE MILL FINISH. AMS4027 REV N ANDUCHT TOOLING PLATE MILL FINISH. AMS4027 REV N B CUST REQ 12-09- 30 % CQR D164055 REV 08 CUST REQ 12-09- 30 % *** W/E 12-10-06 ***         Num       Package Tickst       Lot       Weight       Quantity       UOM       Inspector Clock Numbers         1       441631       212073       1068       6       PC       47200         2       441631       572941       1068       6       PC       47200         441631       572941       1068       6       PC       47200         441634       572941       1068       6       PC       47200         441634       572948       1068       6       PC       47200         441753       539966       556       3       PC       47200         441753       539966       1068       6       PC       47200         10       441753       539966       89       5       PC       47200     <	Di	Rob Woodall Director of Manufacturing Davenport V	Works	Terrence Quality Assurance	e Thom e Manager						
455       857H AVE NW       0.25 IN TK (+.014 -0.00) X 48.5 IN W (+.375 -         COON RAPIDS 55433 MN       0.000 X 144.5 IN IN (+.5 -0.0) CAT X 160001700 (N) A/T 6061-         TG51       TYPE 200 WROUGHT TOOLING PLATE MILL         FINIS       FINIS         V0       D0 W 104 (MARKED)         V0       COON RAPIDS 55433 MN         V0       D0 X 144.5 IN IN (+.5 -0.0) CAT X 160001700 (N) A/T 6061-         TG51       TYPE 200 WROUGHT TOOLING PLATE MILL         FINIS       AGROSS SKID WGT: 4500 LBU         V0       (MARKED)       KROSS SKID WGT: 4500 LBU         V0       (MARKED)       KROSS SKID WGT: 4500 LBU         V0       MARGONS SKID WGT: 4500 LBU       MARGONS SKID WGT: 4500 LBU         V1       Weight       Quantity       UOM         V1       41631       572941       1068         5       FC       47200         5       441631       572947       1068         6       PC       47200         5       441634       572947       1068         6       PC       47200         5       441634       572947       1068         6       PC       47200         7       441634       572947       <		2 - 2- 3			Page 1	of 4					
1       441631       212073       1068       6       PC       47200         2       441631       572941       1068       6       PC       47200         3       441631       572947       1068       6       PC       47200         4       441631       572948       1068       6       PC       47200         5       441634       572947       1068       6       PC       47200         5       441634       572941       1068       6       PC       47200         6       441634       572947       1068       6       PC       47200         7       441634       572947       1068       6       PC       47200         8       441634       572947       1068       6       PC       47200         8       441753       539961       536       3       PC       47200         10       441753       539966       889       5       PC       47200         11       441753       539968       1067       6       PC       47200         12       441753       539968       1067       6       PC       47200      <				0.000 T651 FINIS 209 R ((MAR MAX G 30 %	) X 144.5 I TYPE 200 WR H. AMS4027 R EV 11 EXC_M KED)) KRAFT ROSS SKID WG CQR D164055	N LN (+.5 -0.1 OUGHT TOOLING EV N ANSIH35. RK ASTMB209 RI PAPER INTERLI T: 4500 LB QI REV 08 CUST	D) CAT X 1 PLATE MILL 2 REV 2009 EV 10 EAVED JAN TOL +/-	L60001700 (N) A/			
1       441631       212073       1068       6       PC       47200         2       441631       572941       1068       6       PC       47200         3       441631       572947       1068       6       PC       47200         4       441631       572948       1068       6       PC       47200         5       441634       212073       1068       6       PC       47200         5       441634       572941       1068       6       PC       47200         6       441634       572947       1068       6       PC       47200         7       441634       572947       1068       6       PC       47200         8       441634       572948       1068       6       PC       47200         9       441753       539961       536       3       PC       47200         10       441753       539967       711       4       PC       47200         12       441753       539968       1067       6       PC       47200         13       441753       539968       1067       6       PC       47200      <	Num				Quantity			or Clock Numbers			
5 441634 212073 1068 6 PC 47200 6 441634 572941 1068 6 PC 47200 7 441634 572947 1068 6 PC 47200 8 441634 572948 1068 6 PC 47200 9 441753 539961 536 3 PC 47200 10 441753 539966 889 5 PC 47200 11 441753 539967 711 4 PC 47200 12 441753 539968 1067 6 PC 47200 13 441753 539968 1067 6 PC 47200 13 441753 572941 1067 6 PC 47200 13 441753 572941 1067 7 9 PC 47200 14 41753 539968 1067 7 9 PC 47200 13 441753 539968 1067 7 9 PC 47200 14 41753 539968 1067 7 9 PC 47200 15 441753 539968 1067 7 9 PC 47200 16 7 7 9 PC 47200 17 41753 539968 1067 7 9 PC 47200 18 441753 539968 1067 7 9 PC 47200 19 441753 539968 1067 7 9 PC 47200 10 47200 10 PC 47200 10 41753 1067 9 PC 47200 10 41753 1	1 2 3	441631 441631	212073 572941	1068 1068	6	PC PC	47200 47200				
7 441634 572947 1068 6 PC 47200 8 441634 572948 1068 6 PC 47200 9 441753 539961 536 3 PC 47200 10 441753 539966 889 5 PC 47200 11 441753 539967 711 4 PC 47200 12 441753 539968 1067 6 PC 47200 13 441753 539968 1067 6 PC 47200 14 441753 539968 1067 7 6 PC 47200 13 441753 572941 1067 7 6 PC 47200 14 441753 539968 1067 7 7 PC 47200 13 441753 539968 1067 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7	4 5	441634	212073	1068	6	PC	47200				
8 441634 572948 1068 6 PC 47200 9 441753 539961 536 3 PC 47200 10 441753 539966 889 5 PC 47200 11 441753 539967 711 4 PC 47200 12 441753 539968 1067 6 PC 47200 13 441753 572941 1067 6 PC 47200 13 441753 572941 067 7 6 PC 47200 12814 72 Notes for CQR: D164055.8 PRODUCT PRODUCED TO THE REQUIREMENTS OF AMS4027 REV N ALSO MEET THE REQUIREMENTS OF AMS-QQ-A-250 11 ORIGINAL REVISIO N DATED	6 7										
10 441753 539966 989 5 PC 47200 11 441753 539967 711 4 PC 47200 12 441753 539968 1067 6 PC 47200 13 441753 572941 1067 6 PC 47200 13 441753 572941 1067 6 PC 47200 12814 72 Notes for CQR: D164055.8 ROBUCT PRODUCED TO THE REQUIREMENTS OF AMS4027 REV N ALSO MEET THE REQUIREMENTS OF AMS-QQ-A-250 11 ORIGINAL REVISIO N DATED	8		572948			PC	47200				
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PRODUCT PRODUCED TO THE REQUIREMENTS OF AMS4027 REV N ALSO MEET THE REQUIREMENTS OF AMS-QQ-A-250_11 ORIGINAL REVISIO N DATED											
1997-08-01.	PRODU	UCT PRODUCED TO TH		AMS4027 REV N	ALSO MEET T	HE REQUIREMEN	rs of Ams-Q	Q-A-250_11 ORIG	INAL REVISIO N DAT	ED	
	1997.	-08-01.									

Figure D-7. Aluminum Post Base Material Certificate (Sheet 2 of 2), Test Nos. WIPR-1, WIPR-2, and WIPR-4

CERTIFICATE O	F TEST		)		Page 01 of 01	_
					Certification Dat 10-APR-2014	e
CUSTOMER ORDER NU 39584 CUSTOMER PART NUM 0001	•	EARLE M. JORG 1800 N UNIVER KANSAS CITY	SAL AVENU	JE	Invoice Number S107283	
SOLD TO: RIVERS	METAL PRODUC	CTS SHIP TO:	RIVE	ERS METAL	PRODUCTS	
3100 N LINCOLN	38TH I NE 68504			) NORTH 3 COLN NE		
Description: 6 2 X 2 X .125 W X HEAT: 201405597	20'	UDED PORTHOLE T ITEM: 107209	Liı		Q 139.6207 FT	
Specifications: ASTM B221 13	Q	Q-A-200/8		AMS QQ	A 200/8 97	
		UMINIUM CHEMICA	L ANALYS			
DESCRIPTION:	FE CU	MN MG 5 0.8 0.15 1.2	CR	ZN	ΨT	
OTHERS : EACH 0.05		AL REMAIN	IDER		-4	
RCPT: R328024 VENDOR: SERVICE	CENTER META	LS CC	UNTRY OF	ORIGIN	: USA	
		MECHANICAL	PROPERTI	ES		
DESCRIPTION	YLD STR KSI 40.3 42.7	KSI IN 42.6 44.9	ELONG 1 02 IN 11.5 14.85	%RED IN AREA	HARDNESS	
	ecification requirements of	turer's Certificate of Test after v f the information on the certificat		Material did not o our possession.	come in contact with mercury while in LARRY BUSICK	
		report will meet the applicable ring a part of the description.	requirements	Har	A Bunch	

Figure D-8. 2"x2"x1/8" Aluminum Rail Material Certificate, Test Nos. WIPR-1, WIPR-2, and WIPR-4

SUSTOMER ORDER NUMBER 39584 1800 N UNIVERSAL AVENUE SUSTOMER PART NUMBER 01 513315 SOLDTO: RIVERS METAL PRODUCTS SHIPTO: RIVERS METAL PRODUCTS 3100 N 307H LINCOLN NE 68504 LINCOLN NE 68504 Description: 6061-T6511 EXTRUDED BAR AMS QQ-A-200/8 LINCOLN NE 68504 Description: 6061-T6511 EXTRUDED BAR AMS QQ-A-200/8 LINCOLN NE 68504 Description: 71.0000 LB Same of the second state of the second stat	CERTIF	ICATE OF TEST		U.S.		Page 01 of 01 Certification Date NOT VALID
3954       1800 N UNIVERSAL AVENUE KANSAS CITY MO 64120       S107281         SUSTOMER PART NUMBER       0       513315         OLD TO:       RIVERS METAL PRODUCTS 3100 N 38TH LINCOLN NE 68504       SHIP TO:       RIVERS METAL PRODUCTS 3100 NORTH 38TH LINCOLN NE 68504         Description:       6061-T6511 EXTRUDED BAR AMS QQ-A-200/8 Line Total 71.0000 LB       TIEM: 513315 CST 2.09LB 71.00LB         Specifications:       *** NO VALUE TEST REPORT FOR ORDER *** MESSAGE - DOES NOT EXIST       State of the manufacture's Catification for the manufactu	USTOMER	R ORDER NUMBER				NOT VHILD
SOLD TO:       RIVERS METAL PRODUCTS 3100 N 387H LINCOLN NE 68504       SHIP TO:       RIVERS METAL PRODUCTS 3100 NORTH 387H LINCOLN NE 68504         Description:       6061-T6511 EXTRUDED BAR AMS QQ-A-200/8 Line Total 71.0000 LB       Jine Total 71.0000 LB         Description:       6061-T6511 EXTRUDED BAR AMS QQ-A-200/8 Line Total 71.00LB       71.0000 LB         Specifications:       **** NOVALID TEST REPORT FOR ORDER **** MESSAGE - DOES NOT EXIST       71.00LB         The above data were transcribed from the manufacture's Cettificate of Test after vertification for completeness and specification forming a part of the description. The willing subject to examination.       Material did not come in contact with mercury while in or possesion.	USTOMER	R PART NUMBER				
L/2 SG X 121       Line Total 71.0000 LB         HEAT: 201405836       ITEM: 513315 CST 2.09LB 71.00LB         Specifications:       **** NO VALID TEST REPORT FOR ORDER         **** NO VALID TEST REPORT FOR ORDER       **** MESSAGE - DOES NOT EXIST		RIVERS METAL PROD 3100 N 38TH		SHIP TO:	3100 NORTH 3	8TH
*** NO VALID TEST REPORT FOR ORDER **** MESSAGE - DOES NOT EXIST The above data were transcribed from the manufacturer's Certificate of Test after verification for completeness and specification requirements of the information on the certificate. All test results results results results not be subject to examination. We hereby certify that the material covered by this report will meet the applicable requirements described herein, including any specification forming a part of the description. The will the results of the function forming a part of the description.	L/2 SQ X	K 12'			Line Total	
The above data were transcribed from the manufacturer's Certificate of Test after verification for completeness and specification requirements of the information on the certificate. All test results remain on file subject to examination. We hereby certify that the material covered by this report will meet the applicable requirements described herein, including any specification forming a part of the description. The willful recording of false, fictitious, or fraudulent statements in connection with test results	*** NO	VALID TEST REPORT		R		
The above data were transcribed from the manufacturer's Certificate of Test after verification for completeness and specification requirements of the information on the certificate. All test results remain on file subject to examination. We hereby certify that the material covered by this report will meet the applicable requirements described herein, including any specification forming a part of the description. The willful recording of false, fictitious, or fraudulent statements in connection with test results						
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for completeness and specification requirements of the information on the certificate. All test results remain on file subject to examination. We hereby certify that the material covered by this report will meet the applicable requirements described herein, including any specification forming a part of the description. The willful recording of false, fictitious, or fraudulent statements in connection with test results	m. 1		Fraturaria Cratificat	a of Test of		come in contact with mercury while in
described herein, including any specification forming a part of the description. The willful recording of false, fictitious, or fraudulent statements in connection with test results	for con	mpleteness and specification requirements				
	descril The w	bed herein, including any specification for rillful recording of false, fictitious, or frau	ming a part of the d fulent statements in	lescription.		

Figure D-9. <sup>1</sup>/<sub>2</sub>"x<sup>1</sup>/<sub>2</sub>" Aluminum Spindle Material Certificate, Test Nos. WIPR-1, WIPR-2, and WIPR-4



April 15 2014

Attn:

Fax #

W.W. Grainger, Inc. 100 Grainger Parkway Lake Forest, IL. 60045-5201

KENNETH L KRENK KENNETH L KRENK 29 WSEC LINCOLN, NE, 68588-0000

Pedestrian Hardware R# 14-0433

Grainger Sales Order #: 1206167220 E000137265 Customer PO #:

Dear KENNETH L KRENK As you requested, we are providing you with the following information. We certify that, to the best of Grainger's actual knowledge, the products described below conform to the respective manufacturer's specifications as described and approved by the manufacturer.

Item #	Description	Vendor Part #	Catalog Page #	<b>Order Quantity</b>
4FHF3	Threaded Rod, B7, Yellow Zinc, 1/2-13x3 ft	U22182.050.3600	2929	1.000
4FHG3	Threaded Rod,B7,Yellow Zinc,3/8-16x6 ft	U22182.037.7200	2929	1.000
4FHF1	Threaded Rod, B7, Yellow Zinc, 3/8-16x3 ft	U22182.037.3600	2929	1.000
1XA24	Hex Nut,Grade 2H,3/8-16,PK50	SHY97	0000	1.000
6PE80	Flat Washer, Ylw Zinc, Fits 3/8 In, Pk 50	HU-0375USSHZYBAGGR	2825	1.000

Lilips

Tim Phillips Process Management Analyst Compliance Team Grainger Industrial Supply

Figure D-10. Certificate of Conformance – 3/8" and 1/2" Threaded Rods, 3/8" Nut, 3/8" Washer, Test Nos. WIPR-1 through WIPR-4

CENTIFI	CATE O	F TEST						Certific	01 of 01 ation Date PR-2014
CUSTOMER 39584 CUSTOMER 0001	4		J P	EARLE M. 1800 N U KANSAS C	NIVERSA	AL AVEN		Invoice T780	
SOLD TO:	RIVERS I 3100 N 3 LINCOLN	38TH		<sup>3</sup> S⊦	IIP TO:	310	ERS META 0 NORTH COLN NI		
Descript 2 X 3 X HEAT: 2	<mark>.125</mark> W X 1393458			ED PORTH				1 Q l: 40.5175 F	Т
Specific ASTM B22	1 12A			A-200/8				Q A 200/8 97	
				INIUM CH					
DESCRIPT MIN MAX	ION: SI 0.4 0.8	FE 0.7	CU 0.15 0.4	MN 0.15	MG 0.8 1.2	CR 0.04 0.35	ZN 0.25	TI 0.15	
OTHERS	: EACH 0.05	TOTA 0.15	L	AL I	REMAIND	ER			
RCPT: R VENDOR:	982552			MECHAI	COU	NTRY OF		: USA	
DESCRIPT	ION	YLD S KSI 40.8 42.6		ULT TEN KSI 45.5 46.5	N %E IN 1	LONG	%RED IN ARE		
	j.								
for com results r	ove data were trans pleteness and spec remain on file subj sby certify that the	cification requir ject to examinat	ements of the ion.	information on t	he certificate.	All test	Material did n our possession	ot come in contact with me LARRY BUS	
describe The wil	ed herein, includin lful recording of f punishable as a fe	g any specifica alse, fictitious,	tion forming a	part of the desc	ription.		0 ja	Innager, Quality Assur	ince

Figure D-11. 2"x3"x<sup>1</sup>/<sub>8</sub>" Aluminum Post Material Certificate, Test No. WIPR-2



April 25 2014

W.W. Grainger, Inc. 100 Grainger Parkway Lake Forest, IL. 60045-5201

Pedestrian Rail R# 14-0472 April 2014 SMT

Fax #

Attn:

Grainger Sales Order #: 1206971310 E000139790 Customer PO #:

KENNETH L KRENK KENNETH L KRENK 29 WSEC LINCOLN, NE, 68588-0000

Dear KENNETH L KRENK As you requested, we are providing you with the following information. We certify that, to the best of Grainger's actual knowledge, the products described below conform to the respective manufacturer's specifications as described and approved by the manufacturer.

Item #	Description	Vendor Part #	Catalog Page #	Order Quantity
1VZA6	Hex Cap Screw, B7, 1/4-20x3, PK10	HXCS.001609.50	2766	1.000

Vias

Tim Phillips Process Management Analyst Compliance Team Grainger Industrial Supply

Figure D-12. Certificate of Conformance – 1/4" Dia. x 3" Bolt and 1/4" Nut, Test No. WIPR-2

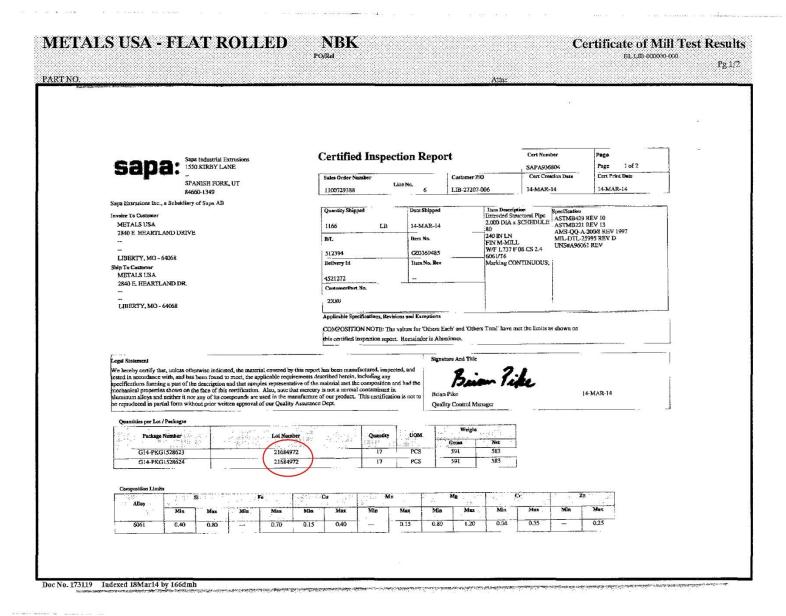


Figure D-13. 2" Dia. Schedule 80 Aluminum Post Material Certificate (Sheet 1 of 2), Test No. WIPR-3

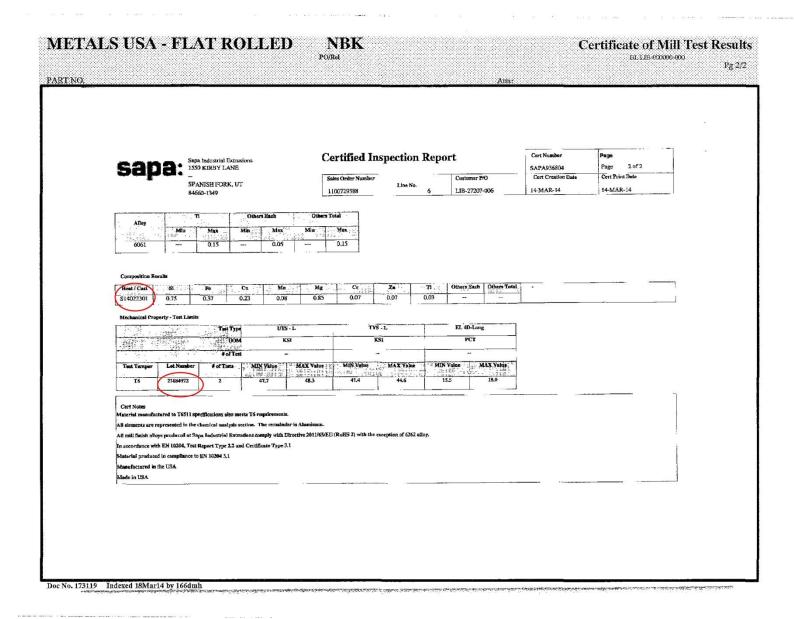


Figure D-14. 2" Dia. Schedule 80 Aluminum Post Material Certificate (Sheet 2 of 2), Test No. WIPR-3

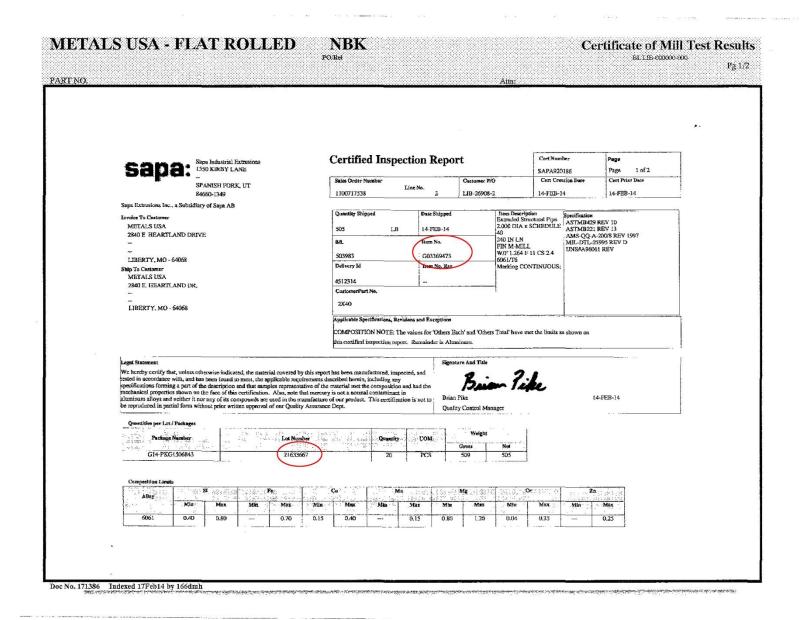


Figure D-15. 2" Dia. Schedule 40 Aluminum Post Material Certificate (Sheet 1 of 2), Test No. WIPR-3

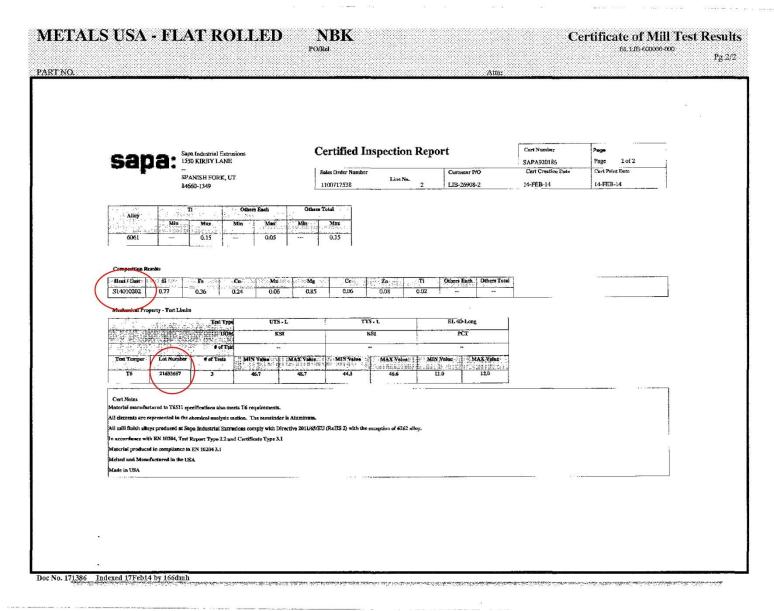


Figure D-16. 2" Dia. Schedule 40 Aluminum Post Material Certificate (Sheet 2 of 2), Test No. WIPR-3

		10 Hemp 713 / 869	stead Ro - 9551 /	ad, Hou FAX 71	iston T2 3 / 869 -	X 77008 - 4254		Inc.			140
SOLD TO	<u> </u>	ERTI			FAN	ALYSIS	2. 		COCEE1		
	SOLD TO SHIPPED TO METALS USA/ SPEC FLAT ROLLED- SAME								606551 LIB-263	78	
MISSOURI 2840 E. HEARTLA LIBERTY, MO 640							Frt. 0		58067 BEST WAY	2	
NF Die Number	Quantity	Description Aliov					Shir	ped	Dat	e Shipped	
NF2493	500 LBS	1X.118 R ASTM B-2 AMS-QQ/ ROHS CC	OUND TI 221 A 200/8	UBE		6061/T-6			12/31/13		
CHEMIC	AL ANALYSIS				[						T
Lot Number	Cast Number	Alloy	sı	FE	CU	MN	MG	CR	ZN	TI	AL
	34391	6061	0.64	0.24	0.2	0.04	0.82	0.05	0.02	0.012	97.9
		85								2000 - 100 	100
											100
Note:	L	<u> </u>	<u> </u>	L			. <u> </u>	I	<u> </u>		
Non-Ferrous Extrusions h	ereby certifies that metal nd in conformance with th		specifications								

Doc No. 169304 Indexed 7.Jan14 by 166dmh

387

Figure D-17. <sup>3</sup>/<sub>4</sub>" Dia. Schedule 10 Aluminum Picket Material Certificate, Test No. WIPR-3

1	
٢Ē	ЛĴ
7'	19

**CERTIFICATE OF TEST** 

August/October 2014 SMT

Pedestrian Rail R#15-0098 TMCO

Page 01 of 01

Certification Date 22-SEP-2014

CUSTOMER ORDER NUMBER 35316 CUSTOMER PART NUMBER 0001 894-020	1800 N UNIVERSAL KANSAS CITY MO	N COMPANY Invoice Number AVENUE S153306 64120						
SOLD TO: TMCO INC ATTENTION ACCOUNTS 535 J STREET LINCOLN NE 68508		TMCO INC 701 S 6TH STREET LINCOLN NE 68508						
Description: 6061 T6 EXTF 2 X 4 X .250 W X 20' HEAT: 21550443 Specifications: ASTM B221 13	ITEM: 116661	Line Total: 257.27 FT						
EN 10204 3.1 AI DESCRIPTION:	EN 10204 3.1 ALUMINIUM CHEMICAL ANALYSIS							
MIN 0.4 0.1 MAX 0.8 0.7 0.4 OTHERS : EACH TOTAL		CR ZN TI .04 .35 0.25 0.15						
0.05 0.15 RCPT: R197775 VENDOR: SAPA PROFILES NORTH		Y OF ORIGIN : USA						
DESCRIPTION KSI 44.3	KSI IN 02 46.7 14.0	G %RED HARDNESS						

The above data were transcribed from the manufacturer's Certificate of Test after verification for completeness and specification requirements of the information on the certificate. All test results remain on file subject to examination. Harry A Busick

Manager, Quality Assurance

Material did not come in contact with mercury while in

We hereby certify that the material covered by this report will meet the applicable requirements described herein, including any specification forming a part of the description.

The willful recording of false, fictitious, or fraudulent statements in connection with test results may be punishable as a felony under federal statutes.

Figure D-18. 2"x4"x1/4" Aluminum Post Material Certificate, Test Nos. APR-1 and APR-2

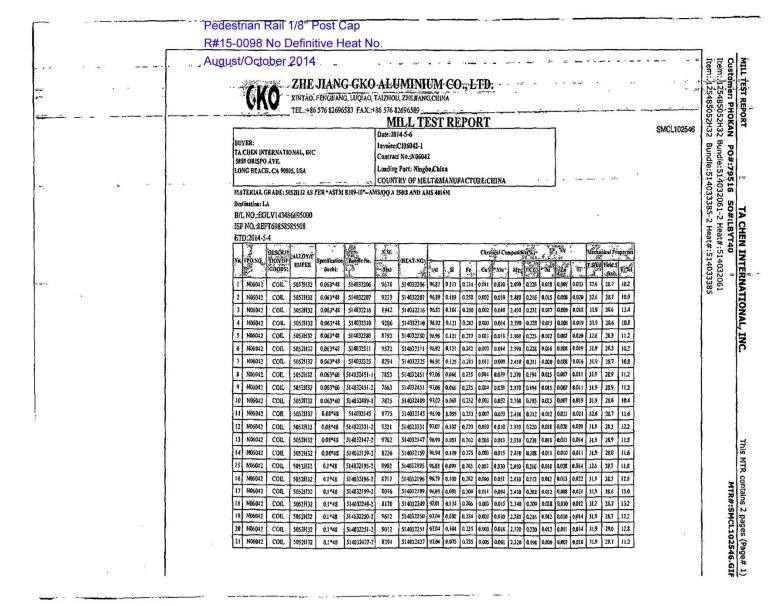


Figure D-19. 1/8" thick Aluminum Post Cap Material Certificate (Sheet 1 of 2), Test Nos. APR-1 and APR-2

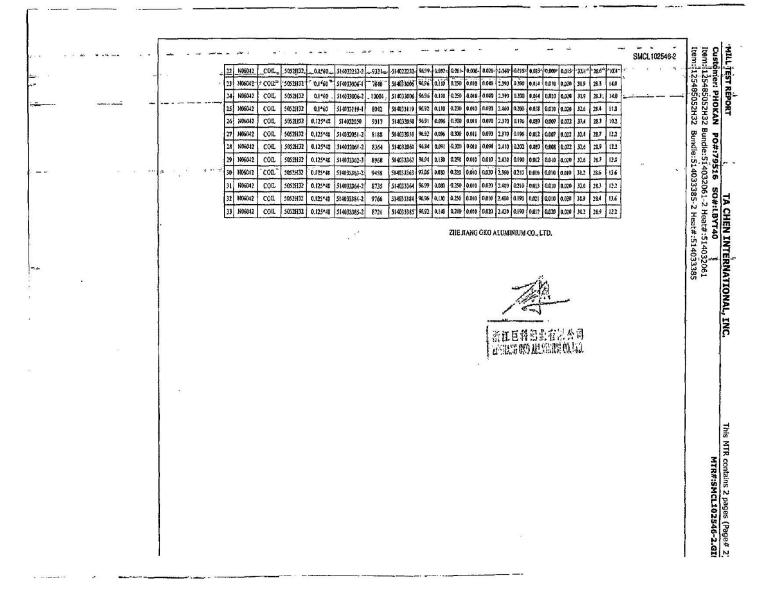


Figure D-20. 1/8" thick Aluminum Post Cap Material Certificate (Sheet 2 of 2), Test Nos. APR-1 and APR-2

## **TEST CERTIFICATE**

# HULAMIN

### Certificate No: 1307LB8422

#### Hulamin Limited Reg. No. 1940/013924/06 VAT Reg. No. 4080149604 HEAD OFFICE: Moses Mabhida Rd, Pietermaritzburg 3/201, P.O. Box 74, Pietermaritzburg 3/200, South Africa Telephone: +27 33 395 6911 Telefax: +27 33 394 6335

	Item Part :	1/1	Combined P/List No	• : R138732	
	HULAMIN Order N	o:247713E	Cust Ref/Part No:		
	Cust Order No :	6043-P.R142815	Certificate No :	1307LB8422	
1 PARKER PLAZA FORT LEE	P/List No : Release No :	2/1274813 RE112504	Dimension : Alloy - Temper :	0.375" X 48.5" X 96.5" 6061 - T651	
BUYER: EMPIRE RESOURCES INC 10 th FLOOR	Shipping FileNo: Lot No :	UR46949 23/07/073D0	Product : 375" X 48.5" X 96.5" PLATE 6061, T651		

Case No: PGW881,PGW882

### MECHANICAL TEST RESULTS

		Metal Id	Alloy Sp		Mechanical Properties							
Lot No.	Cast No.			Spec No	Yield Strength (Ksi)	UTS (Ksi)	Elongation A50 (%)	Earing (%)	TestDate	Gauge Length (Inches)	Bend Test	Actual Gauge (Inches)
Spec				Min Max	35.1	42.0	10			50		0.375 0.392
23/07/073D0	VNKK	59693083	6061	1 2	41.8 41.8	46.3 46.3	16 16		31/07/13 31/07/13	2 2		0.386 0.386

CHEMICAL COMPOSITION

	Cast No.	Alloy	Si (%)	Fe (%)	Cu (%)	Mn (%)	Mg(%)	Cr(%)	Zn (%)	Ti (%)	Each(%)	Total(%)	AI(%)
Min		0.000	0.40		0.15		0.8	0.04					
Max			0.8	0.7	0.40	0.15	1.2	0.35	0.25	0.15	0.05	0.15	
	VNKK	6061	0.72	0.42	0.30	0.10	1.00	0.18	0.02	0.008			97.21

#### CONFORMS TO: ASME SB-209 ASTM B209/10 AMS 4027N AMS-QQA-250/11, 08.1997

For purposes of determining conformance with these specifications, an observed value or a calculated value shall be rounded "to the nearest unit" in the last right-hand digit used in expressing the specification limit, in accordance with the rounding method of ASTM Practice E29, for Using Significant Digits in Test Data to Determine Conformance with Specifications.

WE HEREBY CERTIFY, THAT THE MATERIAL DESCRIBED ABOVE HAS BEEN TESTED AND COMPLIES WITH THE TERMS OF THE ORDER CONTRACT. THE INSPECTION RESULTS INDICATED IN THE CHEMICAL COMPOSITION HAVE BEEN OBTAINED FROM CAST ANALYSIS.

Dr. A. Pitchford(HEAD OF CHEMICAL TESTING)

Ver 2.6 2

1 Rain -aug

V. Maniram(HEAD OF PHYSICAL TESTING)

Printed Date ' 31 Aug 2013

Melted, cast, reflect and processed in Scultr Africa - meets Requirements of RoHS and REACH

1 of 1

		Ϋ́Ξ	1.)		
<b>CERTIFICATE O</b>	F TEST	Ľ			Page 01 of 01
Pedestrian Rail R#15-	0098 TMCO				Certification Date
August/October 2014	SMT				22-SEP-2014
CUSTOMER ORDER NUM	IBER		ORGENSEN CO		Invoice Number
35316		1800 N UNI KANSAS CIT	VERSAL AVEN Y MO 6412		S153305
CUSTOMER PART NUME	884-02000	)			
SOLD TO: TMCO INC	2	SHIP	TMC: TMC	O INC	
ATTENTIC 535 J S LINCOLN		PAYABLE	/01	S 6TH SI COLN NE	CREET 68508
	<u></u>				
Description: 60 2 X 2 X .125 W X HEAT: 21836702	20'	DED PORTHOL ITEM: 107	Li		Q 609.22 FT
Specifications: ASTM B221 13	QQ	-A-200/8		AMS QQ	A 200/8
	ALUI	MINIUM CHEM	ICAL ANALYS	IS	
DESCRIPTION:	EE CU	MNT	MC CD		тт.
SI MIN 0.4 MAX 0.8	FE CU 0.15 0.7 0.4	0	MG CR .8 0.04 .2 0.35	ZN 0.25	TI 0.15
OTHERS : EACH 0.05	TOTAL 0.15	AL REM	AINDER		
RCPT: R431612 VENDOR: SAPA PROI	FILES NORTH A	AMERICA	COUNTRY OF		
			AL PROPERTI		
DESCRIPTION	YLD STR KSI 36.6 40.6	ULT TEN KSI 44.1 48.0	%ELONG IN 02 IN 11.5 13.0	%RED IN AREA	HARDNESS
The above data were trans for completeness and speci results remain on file subje We hereby certify that the described herein, including	fication requirements of th ect to examination. material covered by this re	er's Certificate of Test a e information on the cer port will meet the applic	ifter verification tificate. All test able requirements		ome in contact with mercury while in LARRY BUSICK W A Busick

The willful recording of false, fictitious, or fraudulent statements in connection with test results may be punishable as a felony under federal statutes.

Figure D-22. 2"x2"x1/8" Aluminum Rail Material Certificate, Test Nos. APR-1 and APR-2

Manager, Quality Assurance

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**CERTIFICATE OF TEST** 

Pedestrian Rail R#15-0098 TMCO August/October 2014 SMT Page 01 of 02

Certification Date 24-SEP-2014

CUSTOMER ORDER NUMB 35349 CUSTOMER PART NUMBER 0001	R	EARLE M. JORG L800 N UNIVER KANSAS CITY	ENSEN COM SAL AVENU MO 64120	IPANY JE	Invoice Number S153835
			TIMOC	INC	
SOLD TO: TMCO INC ATTENTION 535 J STR LINCOLN	EET	AYABLE SHIP TO:	701	S 6TH STH COLN NE 6	
Description: 606	1-T6511 EXTE	RUDED BAR AM	IS QQ-A-20	0/8	- Man-Malaka - Sandaka - Sa
1/2 SQ X 12' HEAT: 201408541		ITEM: 513315	Lir	e Total:	256 LB
Specifications: QQ A 200/8 AMS QQA 200/8 97	ASTN MEET	4 B221 13* FS T6 TEMPER			ICIL
		INIUM CHEMICA	AL ANALYSI	S	
DESCRIPTION:					
SI MIN 0.4 MAX 0.8 0	FE CU 0.15	MN MG	CR 0.04	ZN	TI
MAX 0.8 0	.7 0.4	0.15 1.2	0.35	0.25 0	).15
OTHERS : EACH 0.05	TOTAL 0.15	AL REMAIN	IDER		
RCPT: R364734 VENDOR: SERVICE CE	NTER METALS	cc	UNTRY OF	ORIGIN :	USA
		MECHANICAL	PROPERTIE	IS	
DESCRIPTION	YLD STR KSI 46.8 47.5	ULT TEN % KSI IN 49.1 49.8	1 02 IN 11.1 18.1	IN AREA	

The above data were transcribed from the manufacturer's Certificate of Test after verification for completeness and specification requirements of the information on the certificate. All test results remain on file subject to examination. Material did not come in contact with mercury while in our possession. LARRY BUSICK

Manager, Quality Assurance

We hereby certify that the material covered by this report will meet the applicable requirements described herein, including any specification forming a part of the description.

The willful recording of false, fictitious, or fraudulent statements in connection with test results may be punishable as a felony under federal statutes.

Figure D-23.  $\frac{1}{2}x^{1/2}$  Aluminum Spindle Material Certificate (Sheet 1 of 2), Test Nos. APR-1 and APR-2



**CERTIFICATE OF TEST** 

Page 02 of 02

Certification Date 24-SEP-2014

3534	R ORDER NUMBER	EARLE M. JORGEN 1800 N UNIVERSA KANSAS CITY MO		Invoice Number S153835
0001	854-0050	0		
SOLD TO:	TMCO INC ATTENTION ACCOUNTS 535 J STREET LINCOLN NE 68508	SHIP TO: PAYABLE	TMCO INC 701 S 6TH S LINCOLN NE	STREET 5 68508
Descript	ion: 6061-T6511 EX	TRUDED BAR AMS (	Q-A-200/8	

Description: SUB1-18311 EXTROLED BAR ANS QC-R-20078 Line Total: 256 LB HEAT: 201408541 ITEM: 513315 COMMENTS melt source usa chemistry: cast number 04191405 & g02061402 si 0.71/0.77 fe 0.35/0.36 cu 0.32/0.33 mn 0.10/0.11 mg 0.84/0.89 cr 0.09/0.09 zn 0.07/0.02 ti 0.02/0.02 al 97.50/97.41 others each 0.03/0.03 total 0.10/0.10

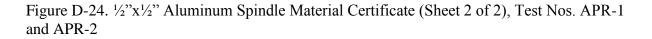
The above data were transcribed from the manufacturer's Certificate of Test after verification for completeness and specification requirements of the information on the certificate. All test results remain on file subject to examination.

Material did not come in contact with mercury while in our possession. LARRY BUSICK

Manager, Quality Assurance

We hereby certify that the material covered by this report will meet the applicable requirements described herein, including any specification forming a part of the description.

The willful recording of false, fictitious, or fraudulent statements in connection with test results may be punishable as a felony under federal statutes.



## Pedestrian Rail Threaded Rods October 2014 R# 15-0188 SMT

## **CERTIFIED MATERIAL TEST REPORT**

FOR ASTM A193-11 B7 STUDS

INSPECTION CERTIFICATE PER BS EN10204:2004 3.1

									0 1	204:20	010.1			
											DA	TE: JA	N.17.2014	
			EITER M			E CO.,L' NA	TD			MFR LOT NO.: 1401071935C				
CUSTOMER: B	RIGH	TON-B	EST IN	TERNA	TION/	L(TAP	WAN)II	NC.						
QTY :	1350	PCS								PO	NUMMB	ER: U	15971	
SAMPLING PLA	N PE	R ASTI	M A 193	-11 GR	-B7									
SIZE & DESCRI	PTION	₹:	1/2-132	X6-1/4"	PI.						PART	NO: 77	5042	
IEAD MARKS:			NDF-I	37										
STEEL PROPER	TIES:													
STEEL GRADE:	SAE	4140				STEEL	SIZE:0	0.472 "			HEAT	NO: E2	21306214	
CHEMISTRY CO	OMPO	SITION	N:											
CHEMIST	Г	C %	Mn %	Si %	P %	S %	Cr %	Mo %	Ni %	Cu %				OTHERS
SPEC.		0.37	0.65	0.15	ΜΛΧ	MAX	0.75	0.15						
		0.49	1.10	0.35	0.035	0.040	1.20	0.25				_		
RESULTS		0.40	0.79	0.23	0.016	0.004	0.94	0.16				a la		
MECHANICAL	PROP	ERTIES	S:						SPECI	FICATI	ON: AST	M A193	3-11 B7	
ITEM			SILE	10000	ELD			1000000	Reduction Temp		pering (	Quenchi	ing Hardness	MACRC
SPEC.		STRENGTH STRENGTH				1000.00	Area				(HRC)	ETCH		
OT LYD (D)			(psi)	MIN		MIN(%)		MIN		6) MIN(°C 593		(°C)	Max	TESTING
STANDARI	) Min				105,000 117,589		16 50 20 58			25	3	820-88	28	4
RESULTS	Max		.228		256	2		6		62	20	860	32	PASSED
TIME (Minutes		134	,220	112,	,200		1	0	0	10	00	80	52	1
TIME (ALIIG COS	4								-	10		00		
MACRO ETCH									SPEC	OF TE	ST METH	HOD: A	ASTM E381-01	(2006)
DIVISION		S	URFAC		DITIO	N		RAND	OM CO.	NDITIC	DN	CEN	NTER SEGREGATION	
SPEC RESULTS		_		S2 S2					R2 R2			C3 C2		
RESUL15	6			32					K2				02	
DIMENSION:									I	FICATI	1		ASME B1.1-2003	
		Shan	k Dia	MAJO		G	0	NO GO		ngth			FRAIGHTNESS	ADD
ITEM				0.4	98"	2.	A	2A		0.0625"			MAX	
STANDARD	Max					12243			6 00" 6	0.0625"		0.58		
STANDARD	Min	2		0.4	88" K	G		NO OK		K	6.25"-0.06 OK	525	0.038" OK	ОК

ALSO MEET THE REQUIREMENTS OF ASME SA-95 SECTION 2. WE CERTIFY THAT THIS DAIA IS A TRUE REPRESENTATION OF INFORMATION PROVIDED BY

THE MATERIAL SUPPLIER AND OUR TESTING LABORATORY.

All parts meet the repuirements of FQA and records of compliance are on file. Marker's ISO#ISO9001:2008\_SGS\_\_\_\_\_\_\_CN11/20818

PLACE OF ORIGIN:CHINA

SIGNATURE TEST LAB MGR. ZHEJIANG NEW ORIENTAL FASTENER CO., LTD

Figure D-25. 1/2" Threaded Rod Material Certificate, Test Nos. APR-1 and APR-2

LOT NO. 325254B Post Office Box 6100 NUCOR Saint Joe, Indiana 46785 Telephone 260/337-1600 FASTENER DIVISION CUSTORER NO/NAME 8061 STRUCTURAL BOLT CO LLC TEST REPORT SERIAL<sup>®</sup> FB410424 TEST REPORT ISSUE DATE 7/24/13 DATE SHIPPED 9/13/13 NUCOR ORDER # CUST PART # 839871 CUSTOMER P.O. # 14790 NAME OF LAB SAMPLER: --CHEMISTRY MATERIAL GRADE -1026L MATERIAL HEAT \*\*CHEMISTRY COMPOSITION (WTZ HEAT ANALYSIS) BY MATERIAL SUPPLIER NUMBER NUMBER C NN P S SI NUCOR STEEL - NEBRASKA RM028016 NF12104365 .23 .75 .011 .021 .25 MIN .20 .60 MAX .55 .040 .050 -- MECHANICAL PROPERTIES IN ACCORDANCE WITH ASTM A563-07a TENSILE STRENGTH SURFACE CORE PROOF LOAD (LBS) N/A N/ HARDNESS HARDNESS 21300 LBS DEG-WEDGE N/A N/A STRESS (PSI) N/A (RC) 28.4 PASS N/A N/A N/A 28.5 31.0 PASS PASS N/A N/A N/A 31.6 PASS N/A N/A PASS N/A N/A AVERAGE VALUES FROM TESTS 29.5 PRODUCTION LOT SIZE 98500 PCS ROTATIONAL CAPACITY TESTED IN ACCORDANCE WITH A325-10, A563-07a SAMPLE #1 PASSED SAMPLE #2 PASSED --VISUAL INSPECTION IN ACCORDANCE WITH ASTM A563-07a 80 PCS. SAMPLED LOT PASSED --COATING - HOT DIP GALVANIZED TO ASTM F2329-13 - GALVANIZING PERFORMED IN THE U.S.A. 1. 0.00283 2. 0.00916 3. 0.00335 4. 0.00213 5. 0.00217 6. 0.00295 7. 0.00455 8. 0.00635 9. 0.00243 10. 0.00343 11. 0.00384 12. 0.00337 13. 0.00251 14. 0.00235 15. 0.00249 15. 0.00249 AVERAGE THICKNESS FROM 15 TESTS .00359 HEAT TREATMENT - AUSTENITIZED, OIL QUENCHED & TEMPERED (MIN 800 DEG F) 0.9790 0.990 0.4750 0.990 --DIMENSIONS PER ASME B18.2.6-2012 CHARACTERISTIC #SAMPLES TESTED Width Across Corners 8 Thickness 32 0.9900 ALL TESTS ARE IN ACCORDANCE WITH THE LATEST REVISIONS OF THE METHODS PRESCRIBED IN THE APPLICABLE SAE AND ASTM SPECIFICATIONS. THE SAMPLES TESTED COMFORM TO THE SPECIFICATIONS AS DESCRIBED/LISTED ABOVE AND WERE MANUFACTURED FREE OF MERCURY CONTAMINATION. NO INTENTIONAL ADDITIONS OF BISMUTH, SELENIUM, TELLURIUM, OR LEAD WERE USED IN THE STEEL USED TO PRODUCE THIS PRODUCT. THE STEEL USED TO PRODUCE THIS PRODUCT. THE STEEL WAS MELTED AND MANUFACTURED IN THE U.S.A. AND THE PRODUCT WAS MANUFACTURED AND TESTED IN THE U.S.A. PRODUCT COMPLIES WITH DFARS 252.225-7014. WE CERTIFY THAT THIS DATA IS A TRUE REPRESENTATION OF INFORMATION PROVIDED BY THE MATERIAL SUPPLIER AND OUNT TESTING LABORATORY. THIS CERTIFIED MATERIAL SET REPORT RELATES ONLY TO THE ITEMS LISTED ON THIS DOCUMENT AND MAY NOT BE REPRODUCED EXCEPT IN FULL.

Pedestrian Rail Nuts October 2014 R# 15-0188 SMT



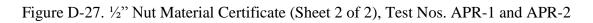
MECHANICAL FASTENER CERTIFICATE NO. A2LA 0139.01 EXPIRATION DATE 12/31/13

NUCOR FASTENER A DIVISION OF NUCOR CORPORATION com w. flynen JOHN W. FERGUSON QUALITY ASSURANCE SUPERVISOR

Page 1 of 1

Figure D-26. <sup>1</sup>/<sub>2</sub>" Nut Material Certificate (Sheet 1 of 2), Test Nos. APR-1 and APR-2

	r Stee	91	2/9	/2013	9:27:41	AM PA	GE	2/002	Fax	Server	
											23010
NUC	=0	2			Mill Cert	ification				2911 E NORF	
NUCOR					2/9/201	3					(402) 644-020 (402) 644-032
		NEBRASKA								1 401	(100) 011 002
Sold To:	NUCOR F/	ASTENER INDIA	NA		S	hip To: NUC	OR FAST	ENER INDIAN	A		
	PO BOX 6 6730 COU ST JOE, IN (260) 337- Fax: (435)	ASTENER INDIA 100 NTY RD 60 146785-0000 1600 734-4581				ST.	JNTY RD 6 IOE, IN 46	ENER INDIAN 60 785-0000			
Custo	omer P.O.	135757						Sales	Order	126701.14	
Produ	uct Group	Special Bar Qu	ality					Part N	umber	31B00875000W68	0
	Grade	1026L					10.11		Lot #	NF1210436511	
	Size	.8750-7/8 Rour	nd Coil						Heat #	NF12104365	
	Product	.8750-7/8 Rour	nd Coil 1026L					B.L. N	umber	N1-246876	
De	escription	1026L						Load I	lumber	N1-193067	
	ner Spec	- <u>*a 10 10 10 10 10</u>						Custome		CH5008	
hereby certify the	hat the materie	al described herein has	been manufactu	red in accorden	e with the specific	allons and standa	rds listed abov	e and that it satisf	es those re	quirements.	
Roll Date: 2	/8/2013	Melt Date: 12/5/	2012 Qty 5	Shipped LB	5: 160,995	Qty Shipped	Pcs: 32			<u>5</u>	
0			~	0		0	~				0
C 0.23%	Mn 0.75%	V 0.003%	Si 0.25%	S 0.021%	P 0.011%	Cu 0.08%	Cr 0.08%	NI 0.04%		10 Al 11% 0.001%	Cb 0.002%
Pb	Sn	Ca	B	Ti	0101110	010010	0,00,10	010 170	- Circ		didd2.0
0.000%	0.005%		0.0002%	0.001%							
		ead,Bismuth or B		9		· · · · ·					
in the United 2. All produc 3. Mercury, i	States. Its produce n any form	d are weld free. , has not been u: M A29-12, ASTA t Nucor Steel Ne reditation cert. av	sed in the provide	duction or t	esting of this r	naterial.	and the second states and the second				
				the state of the second of							
		Chemistry									
Part	<u>C.</u>	Chemistry		tion Ch		_				ai.	
	Receiv	#5008		tion Cha 28	icks	-				×	
	Receiv	<u>H5008</u> c	Verifica ) RM# .	tion Cha 28	oks 01.6	- 3 +++20					



## PRODUCT CERTIFICATION Prestige 23513 Groesbeok Highway Warren, Michigan 48089 (586)773-2700 \* Far (586)773-2298 www.PrestigeStamping.com CERTIFICATION NUMBER Stamping, 119614 Inc. THIS IS TO CERTIFY THE PRODUCT STATED BELOW WAS FABRICATED AND PROCESSED TO THE ORDER AS INDICATED AND CONFORMS TO THE APPLICABLE SPECIFICATIONS AND STANDARDS Customer: THE STRUCTURAL BOLT CO 2140 CORNHUSKER HWY LINCOLN, NE 68521 Customer Part: 1/2"F436 H/DIP Prestige Part: P1088HP300 Part Name: 1/2"F436 H/DIP Purchase Order: 15432-1 Shipment BOL: B173265 Steel Supplier: HORIZON STEEL CO. Grade: CF436 GRADE STEEL Grade: CF436 GRADE STEEL Lot: C7313D Heat: 342288 Carbon: .248 (.21 - .93) Manganese: 1.059 (.43 - 1.6) Phosphorous: .012 (.03 Max.) Sulfur: .0016 (.05 Max.) Silicon: .206 Shipment ID: A0184180 Quantity: 6400 Manufacturers Marking: "P" TEST RESULTS SPECIFICATIONS HARDNESS: HARDNESS: TEST METHOD: ASTM E18 HRC 41 - 43 HRC 38 - 45 CHECKED TO ASTM F606 PLATING: TEST METHOD: ASTM B499 0.0017" Min. HOT DIP GALV TO ASTM F-2329 PLATING: 0.0020" - 0.0030" : : Chemistry is as reported from raw material certification and does not fall under Prestige Stamping's accreditation. This product was produced under an ISO/TS 16949 Quality Assurance System. ISO/TS 16949 Certification No: 0062933. Material was melted and manufactured in the U.S.A. This product was manufactured in Warren, Michigan U.S.A. ACH This product conforms to all requirements for washers as produced according to A.S.T.M. F-436-10. FRAME SCHUBERT Sampling Plan per P.S.I W.I. # 5.4.18.015. The test results only apply to the items tested. Quality Assurance Manager This test report must not be reproduced except in full without prior written approval. Materials used to manufacture these products are mercury, asbestos and radio activity free. No weld repairs made to material. 03/24/14 10:02 KGUZ PAGE 1 of 1 Econ Information System

Pedestrian Rail Washers October 2014 R#15-0188 SMT

Figure D-28. 1/2" Washer Material Certificate, Test Nos. APR-1 and APR-2

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# Appendix E. Vehicle Center of Gravity Determination

est: APR-1	Vehicle: Rio					
2	Vehicle C	G Determination Weight				
VEHICLE	Equipment	(lb)				
+	Unbalasted Car (curb)	2421				
+	Brake receivers/wires	6				
+	Brake Frame	7				
+	Brake Cylinder	28				
+	Strobe Battery	6				
+	Hub	20				
+	CG Plate (SLICEs)	10				
+	DTS	19				
-	Battery	-34				
-	Oil	-10				
-	Interior	-38				
23	Fuel	0				
	Coolant	-8				
-	Washer fluid	-8				
BALLAST	Water					
	Misc.					
	Misc.					
	Estimated Total Weight	2419 lb				
wheel base	98.75 in.					
MASH targets		Test Inertial	Difference			
Test Inertial Wt (lb)	2420 (+/-)55	2428	8.0			
Long CG (in.)	39 (+/-)4	36.44	-2.55848			

-0.63525 Lateral CG (in.) Note: Long. CG is measured from front axle of test vehicle

N/A

Note: Lateral CG measured from centerline - positive to vehicle right (passenger) side

CURB WEIGH	IT (Ib)		
	Left	Rig	ht
Front		800	780
Rear		429	412
FRONT		1580 lb	
REAR		841 lb	
TOTAL		2421 lb	

Dummy =	= 166lbs	5.		
TEST IN	ERTIAL	WEIC	GHT (I	b)
(from scales)				
	Left		Right	
Front		788		744
Rear		453		443
FRONT		1532	lb	
REAR		896	lb	
TOTAL		2428	lb	

NA

Figure E-1. Vehicle Mass Distribution, Test No. APR-1

Test: APR-2

Vehicle: Rio

	Vehicle C	G Determination
		Weight
VEHICLE	Equipment	(lb)
+	Unbalasted Car (curb)	2424
+	Brake receivers/wires	6
+	Brake Frame	8
+	Brake Cylinder	28
+	Strobe Battery	6
+	Hub	20
+	CG Plate (SLICEs)	10
+	DTS	19
	Battery	-36
2-2-2	Oil	-6
-	Interior	-42
:=)	Fuel	0
1 <b>2</b> 3	Coolant	-7
-	Washer fluid	0
BALLAST	Water	
	Misc.	
	Misc.	

Estimated Total Weight

2430 lb

wheel base	98.5	in.		
MASH targets			Test Inertial	Difference
Test Inertial Wt (lb)		2420 (+/-)55	2437	17.0
Long CG (in.)		39 (+/-)4	37.67	-1.32991
Lateral CG (in.)		N/A	-0.48474	NA

Note: Long. CG is measured from front axle of test vehicle

Note: Lateral CG measured from centerline - positive to vehicle right (passenger) side

CURB WEIGH	IT (lb)			
	Left	Rig	ght	
Front		766	761	
Rear		461	436	
FRONT		1527 lb		
REAR		897 lb		
TOTAL		2424 lb		

Dummy =			
TEST IN	ERTIAL	WEIGH	T (lb)
(from scales)			
	Left	R	ight
Front		769	736
Rear		470	462
FRONT		1505 lb	
REAR		932 lb	
TOTAL	2	2437 lb	

Figure E-2. Vehicle Mass Distribution, Test No. APR-2

Appendix F. Fabrication Drawings for Test Nos. APR-1 and APR-2

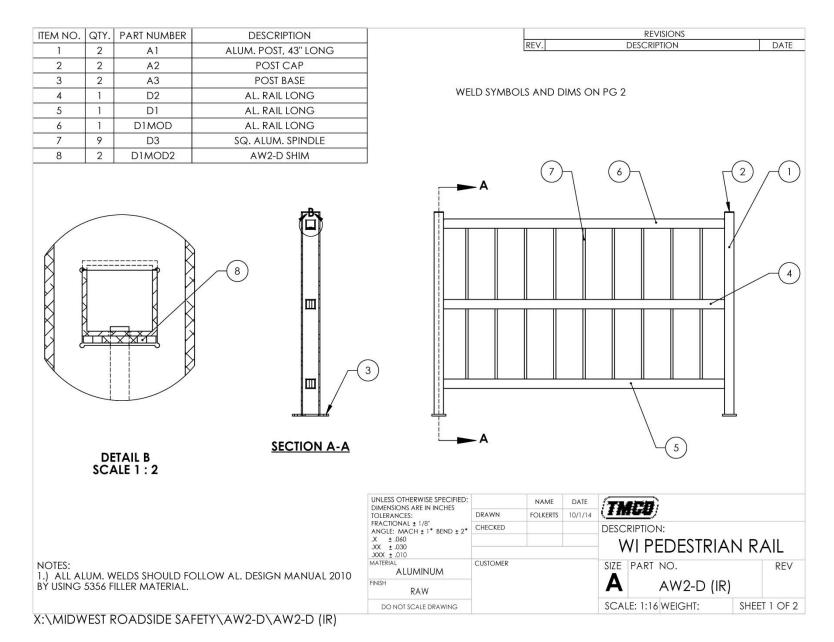


Figure F-1. Fabrication Drawings, Test Nos. APR-1 and APR-2

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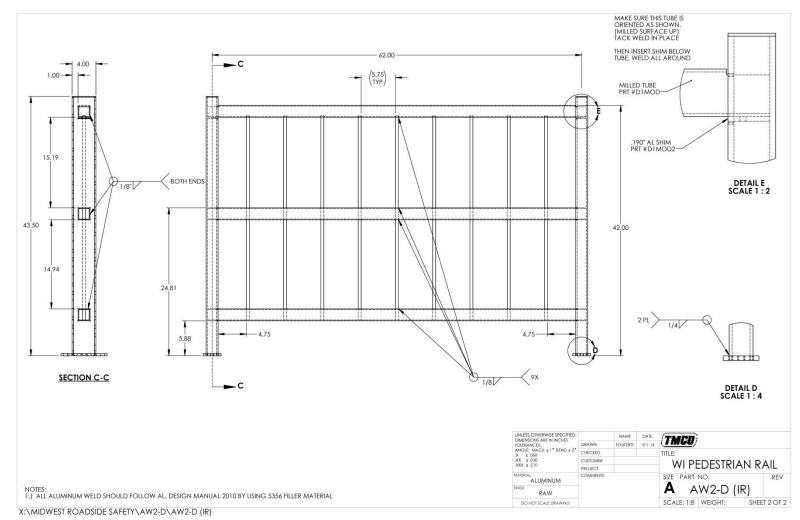


Figure F-2. Fabrication Drawings, Test Nos. APR-1 and APR-2

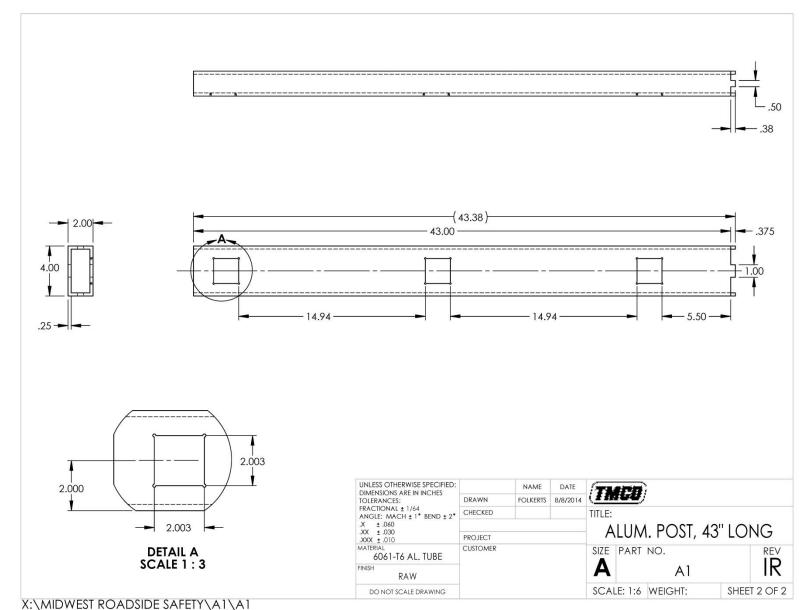


Figure F-3. Fabrication Drawings, Test Nos. APR-1 and APR-2

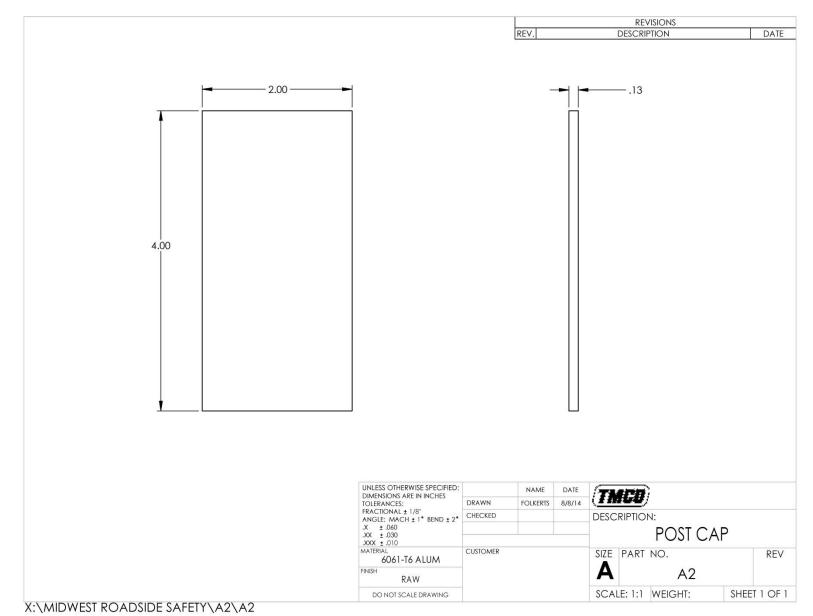


Figure F-4. Fabrication Drawings, Test Nos. APR-1 and APR-2

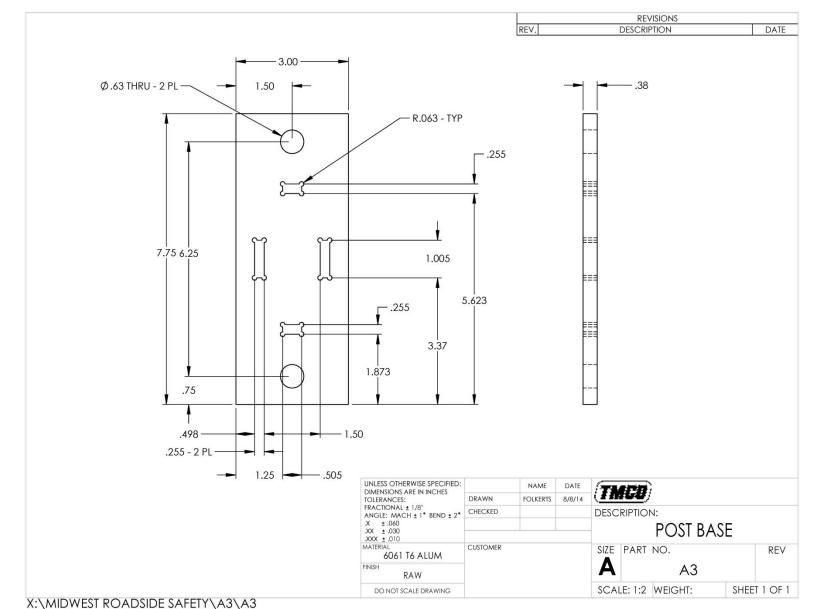


Figure F-5. Fabrication Drawings, Test Nos. APR-1 and APR-2

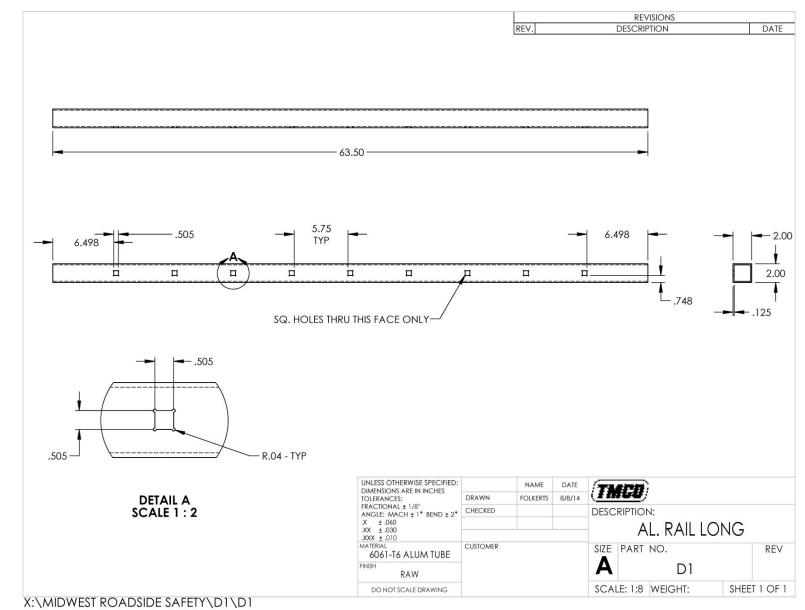


Figure F-6. Fabrication Drawings, Test Nos. APR-1 and APR-2

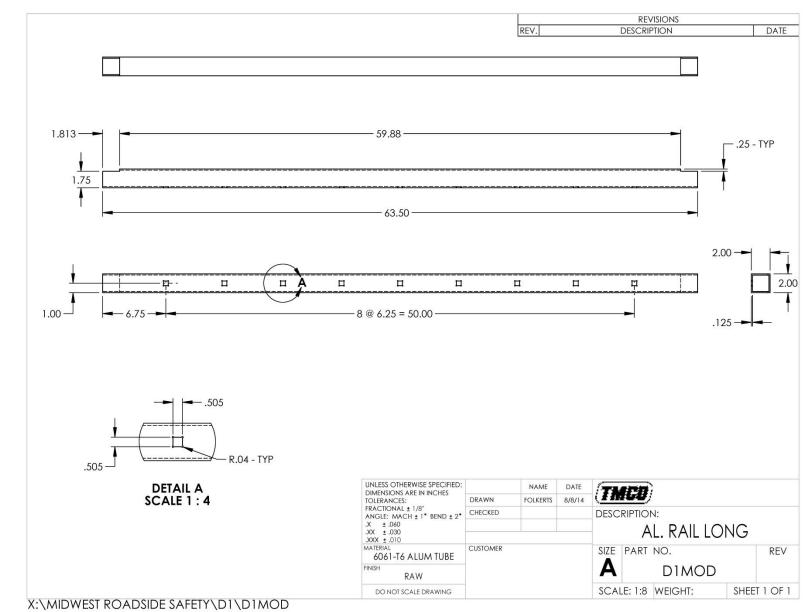


Figure F-7. Fabrication Drawings, Test Nos. APR-1 and APR-2

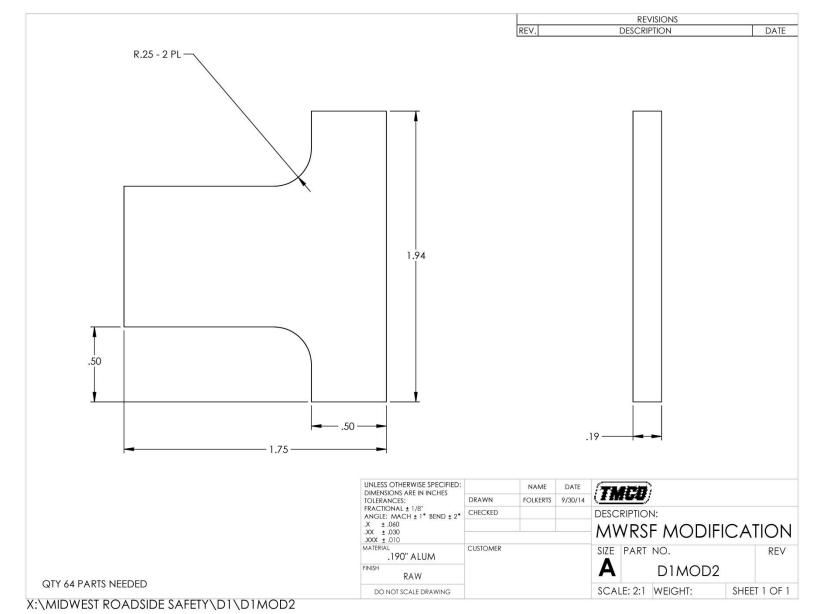


Figure F-8. Fabrication Drawings, Test Nos. APR-1 and APR-2

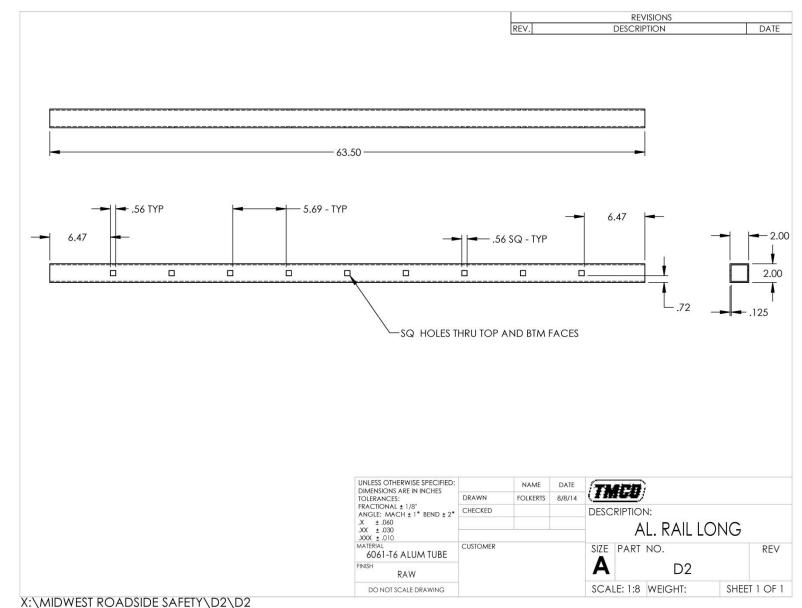


Figure F-9. Fabrication Drawings, Test Nos. APR-1 and APR-2

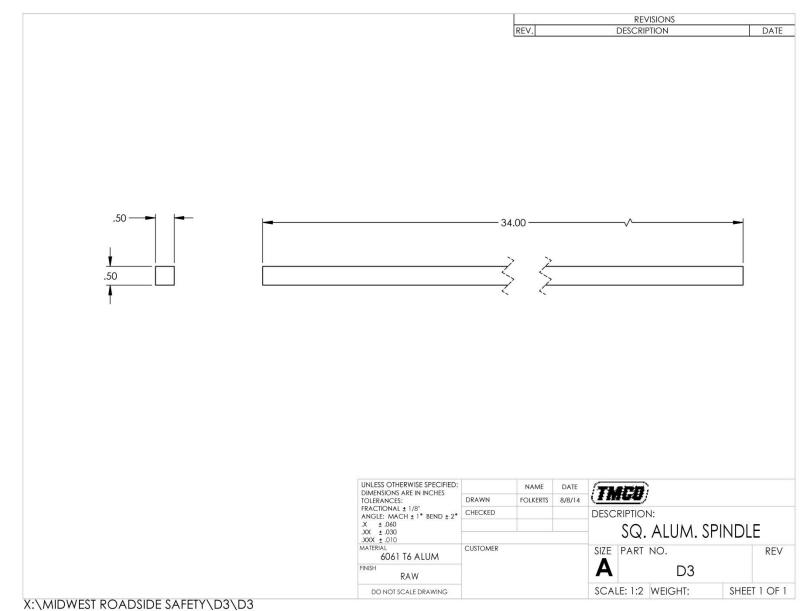


Figure F-10. Fabrication Drawings, Test Nos. APR-1 and APR-2

# Appendix G. Vehicle Deformation Record

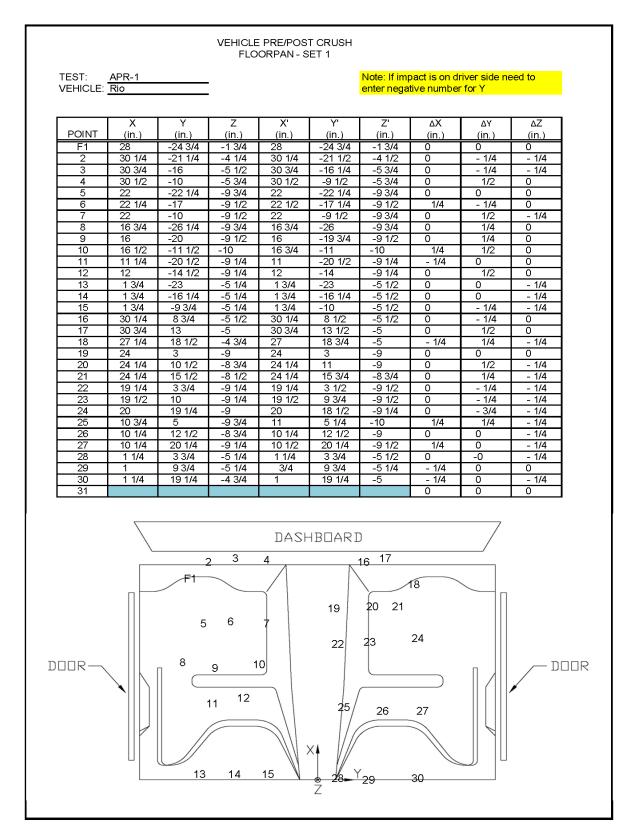


Figure G-1. Floorpan Deformation Data - Set 1, Test No. APR-1

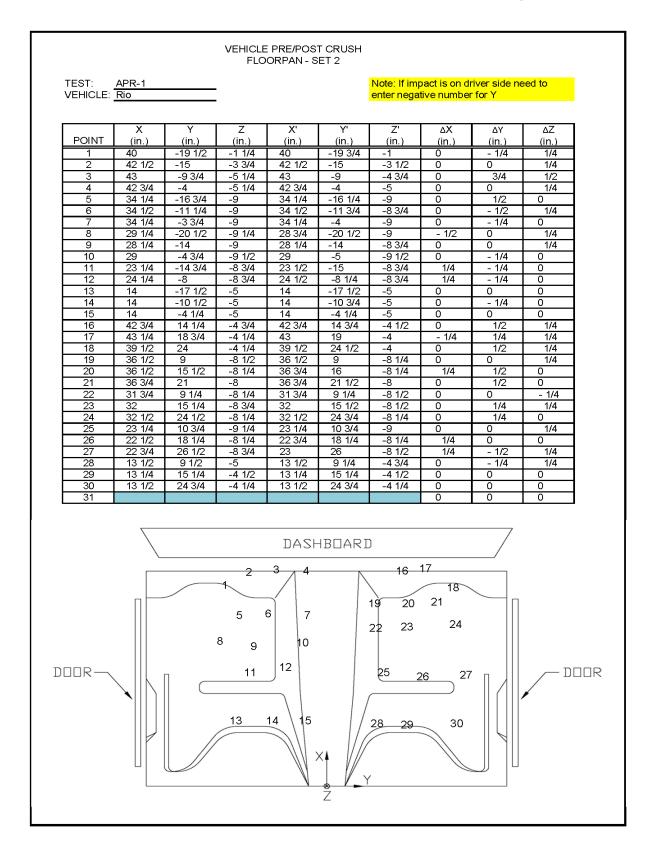


Figure G-2. Floorpan Deformation Data - Set 2, Test No. APR-1

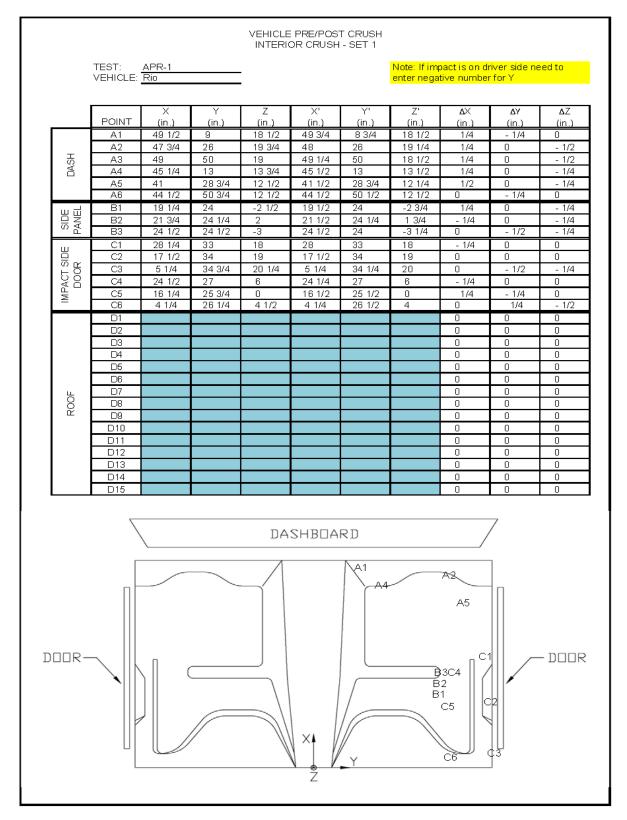


Figure G-3. Occupant Compartment Deformation Data - Set 1, Test No. APR-1

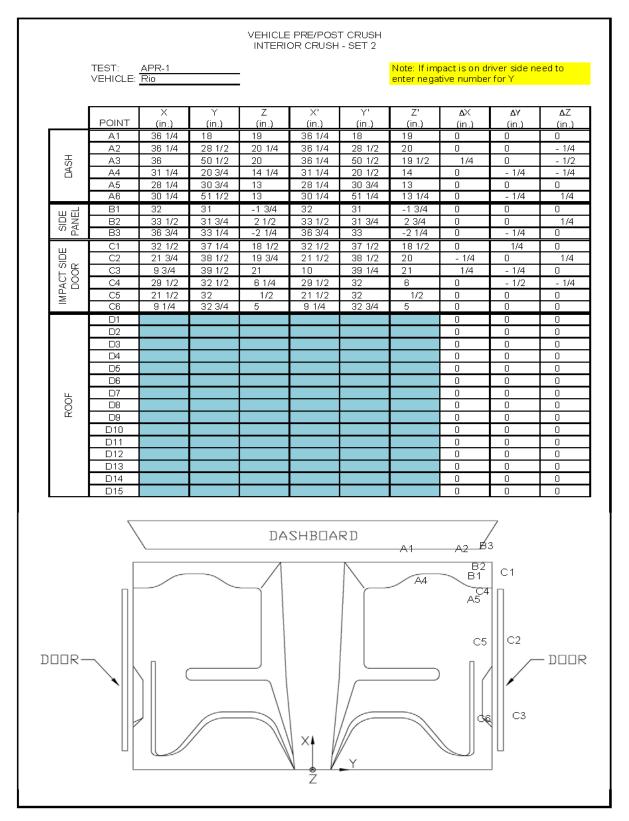


Figure G-4. Occupant Compartment Deformation Data - Set 2, Test No. APR-1

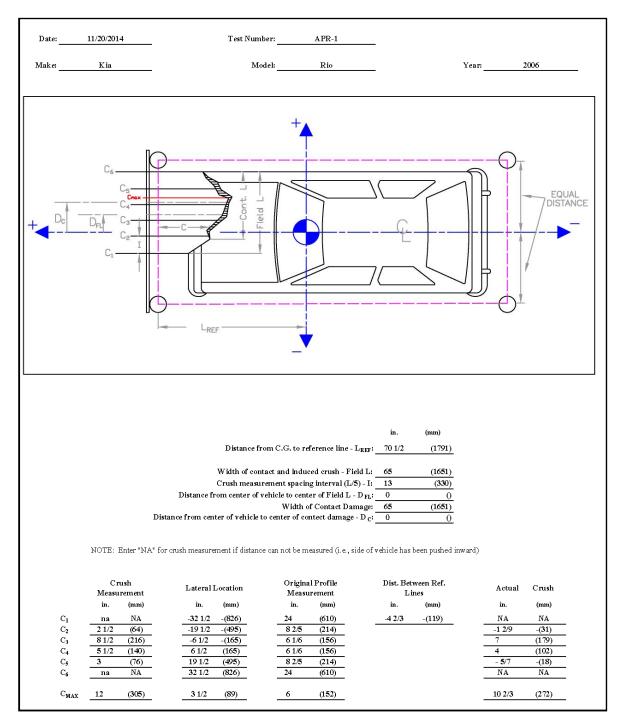


Figure G-5. Exterior Vehicle Crush (NASS) - Front, Test No. APR-1

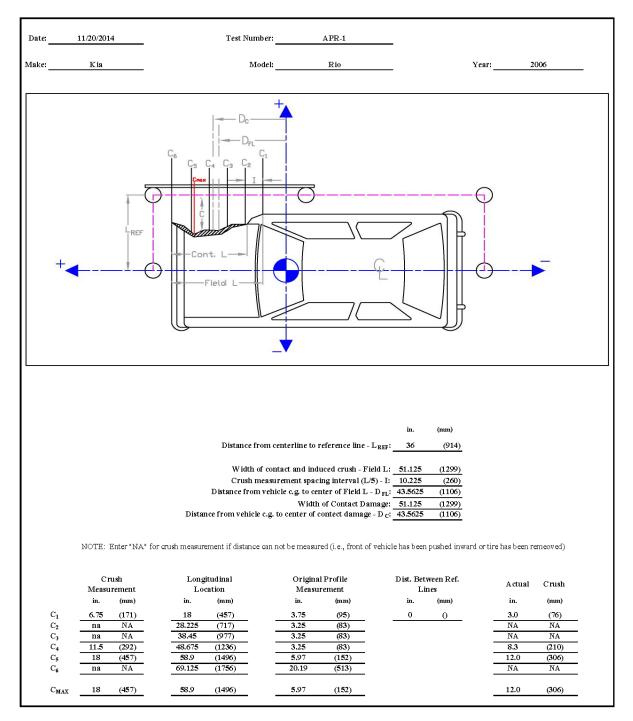


Figure G-6. Exterior Vehicle Crush (NASS) - Side, Test No. APR-1

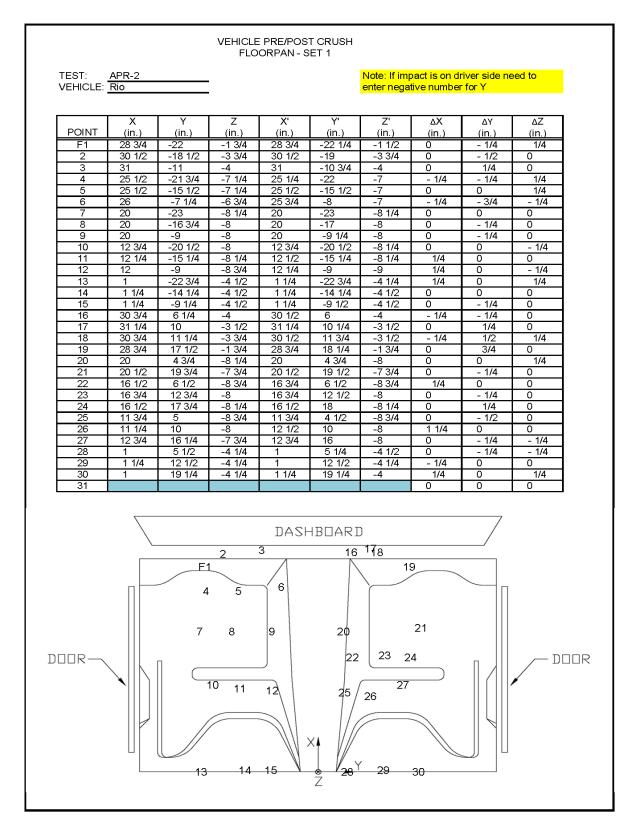


Figure G-7. Floorpan Deformation Data – Set 1, Test No. APR-2

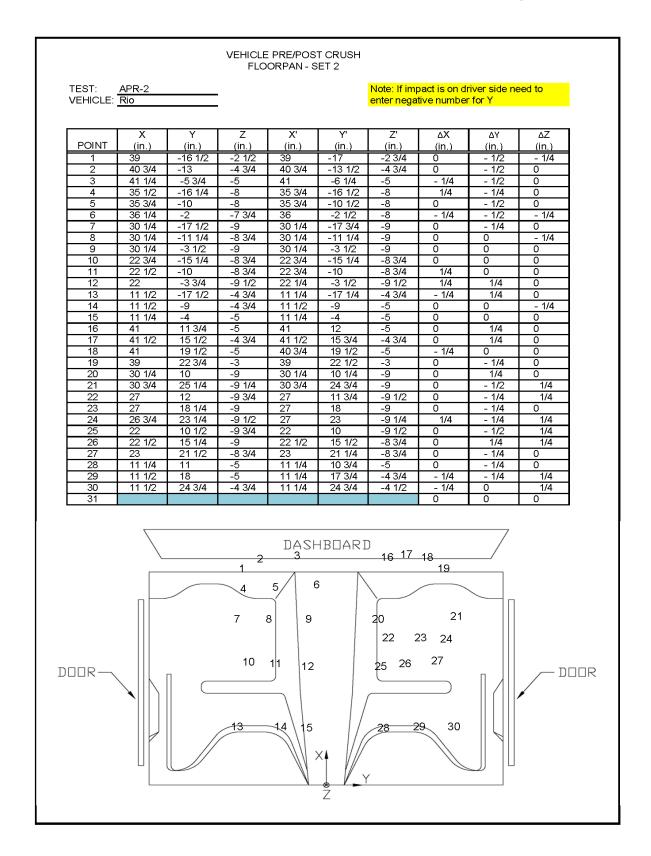


Figure G-8. Floorpan Deformation Data – Set 2, Test No. APR-2

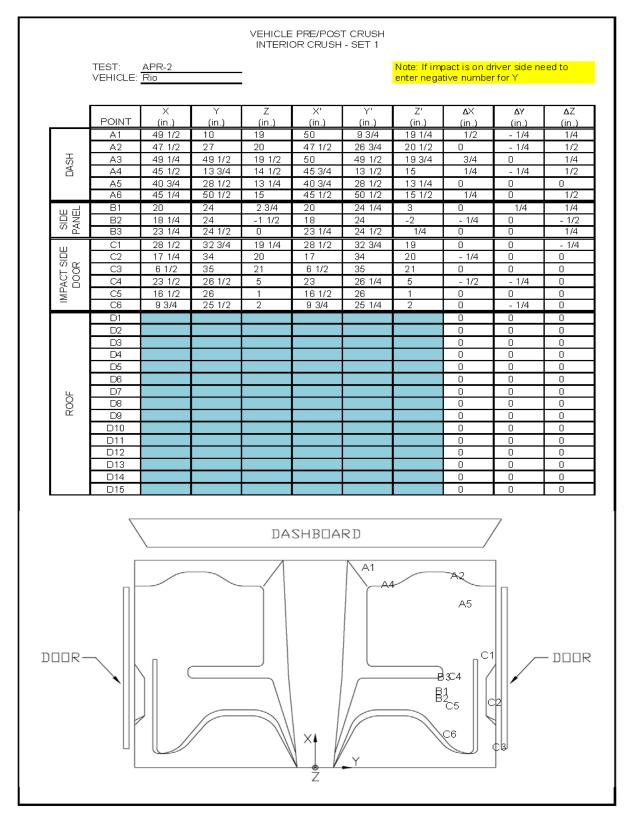


Figure G-9. Occupant Compartment Deformation Data - Set 1, Test No. APR-2

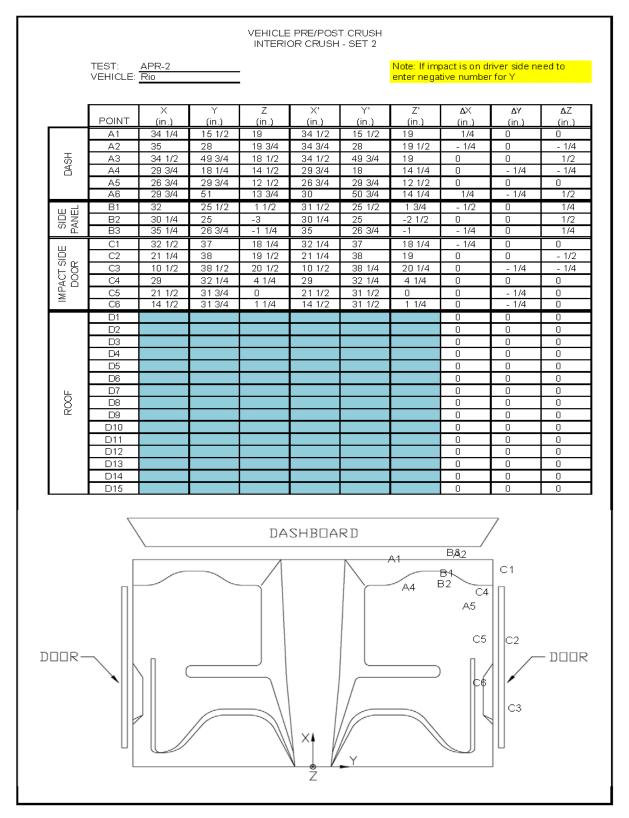


Figure G-10. Occupant Compartment Deformation Data - Set 2, Test No. APR-2

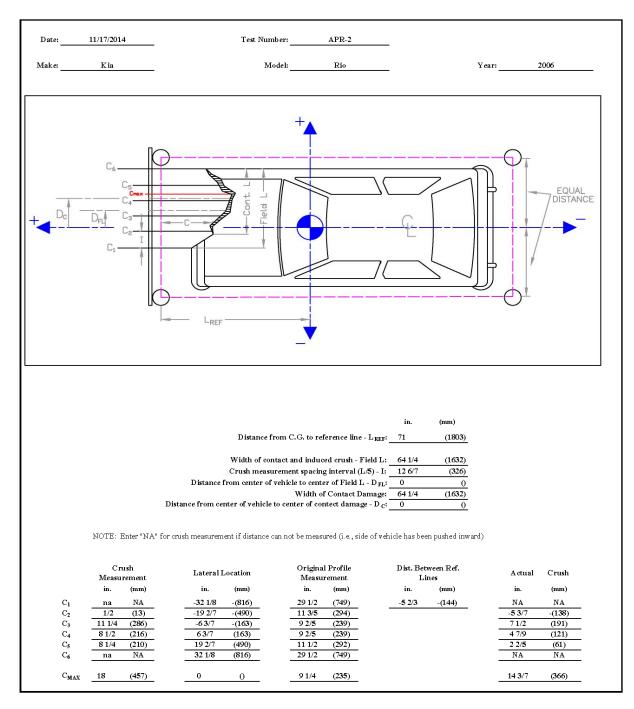


Figure G-11. Exterior Vehicle Crush (NASS) - Front, Test No. APR-2

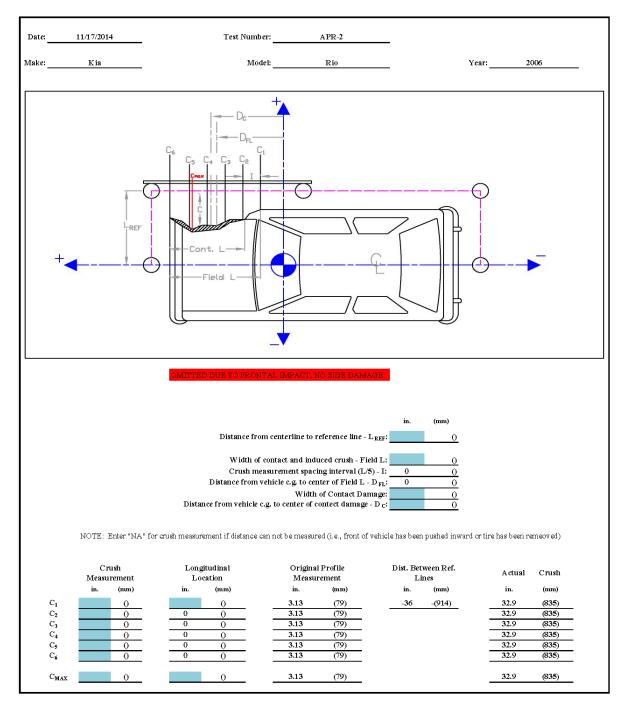


Figure G-12. Exterior Vehicle Crush (NASS) - Side, Test No. APR-2

Appendix H. Accelerometer and Rate Transducer Data Plots, Test No. APR-1

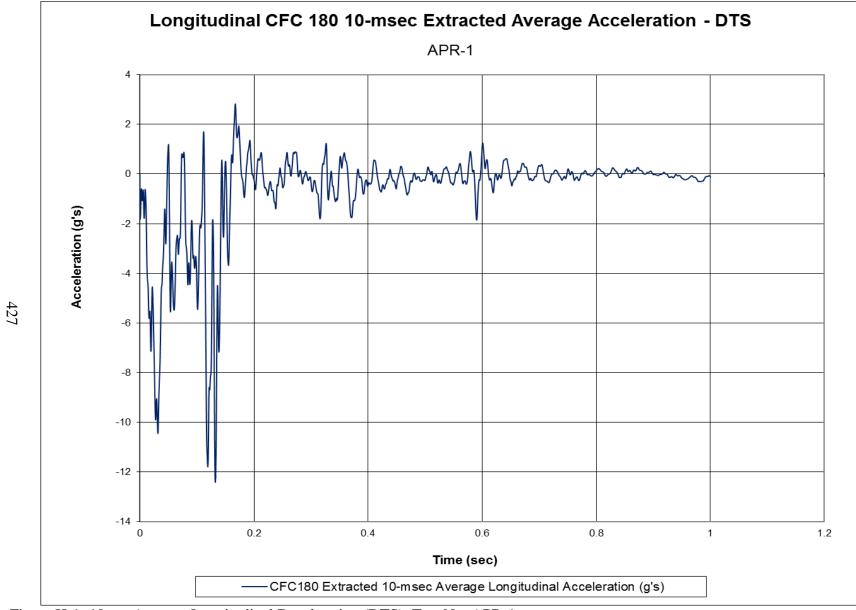


Figure H-1. 10-ms Average Longitudinal Deceleration (DTS), Test No. APR-1

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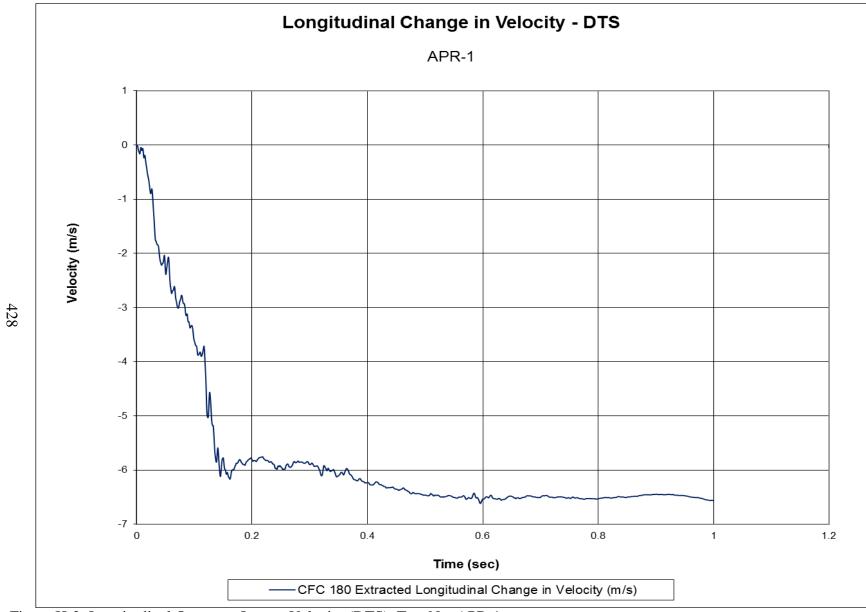


Figure H-2. Longitudinal Occupant Impact Velocity (DTS), Test No. APR-1

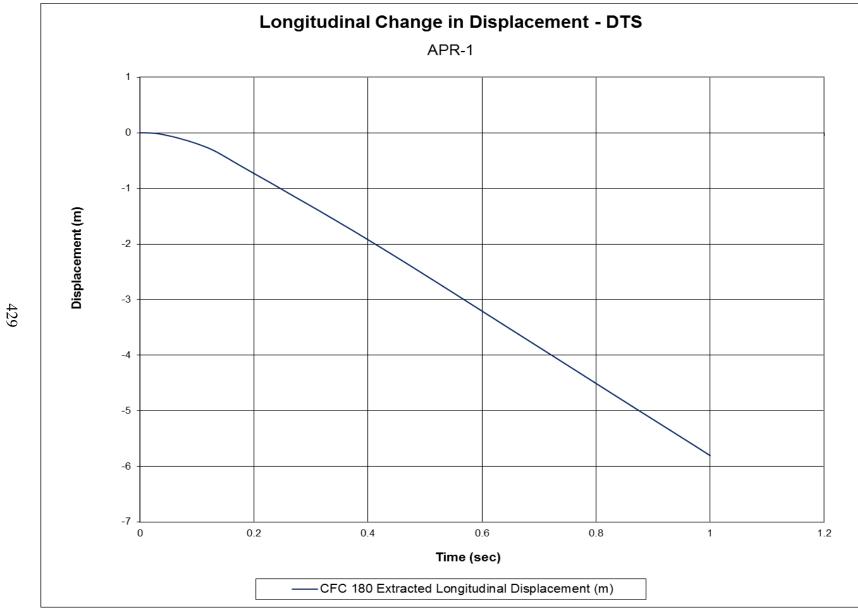


Figure H-3. Longitudinal Occupant Displacement (DTS), Test No. APR-1

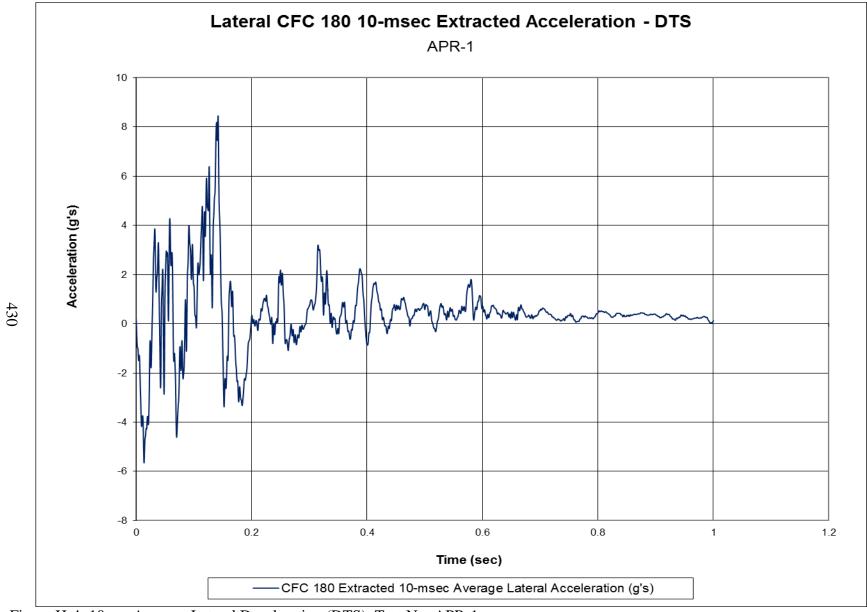


Figure H-4. 10-ms Average Lateral Deceleration (DTS), Test No. APR-1

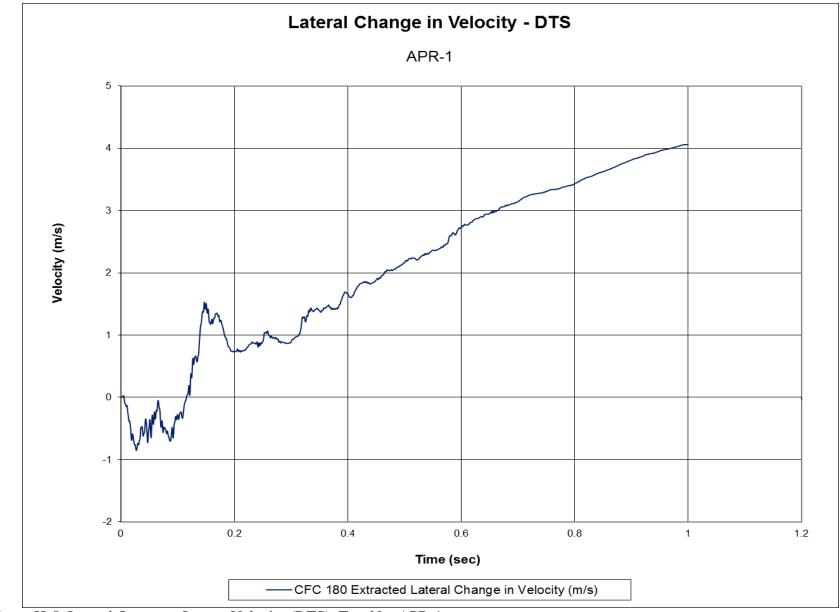


Figure H-5. Lateral Occupant Impact Velocity (DTS), Test No. APR-1

January 18, 2016 MwRSF Report No. TRP-03-321-15

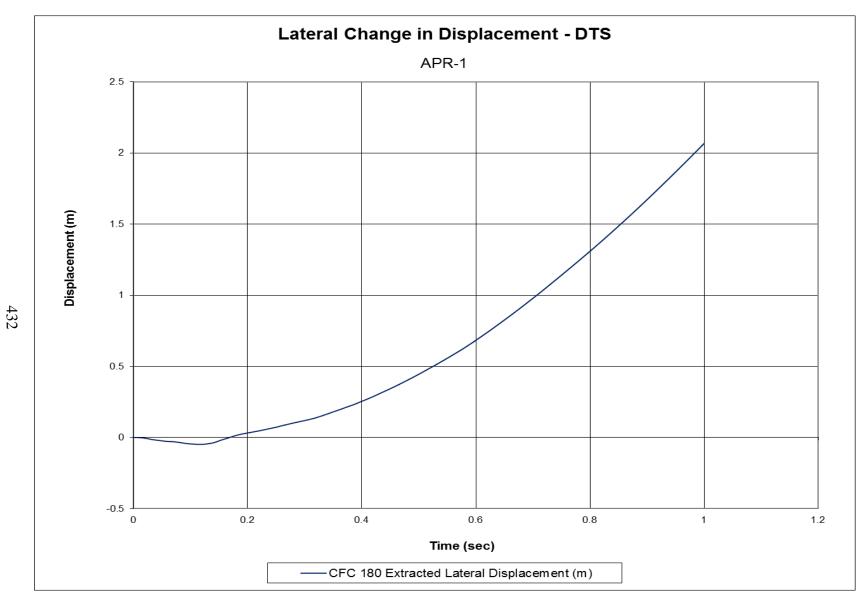


Figure H-6. Lateral Occupant Displacement (DTS), Test No. APR-1



Figure H-7. Vehicle Angular Displacements (DTS), Test No. APR-1

January 18, 2016 MwRSF Renort No. TRP-03-321-15

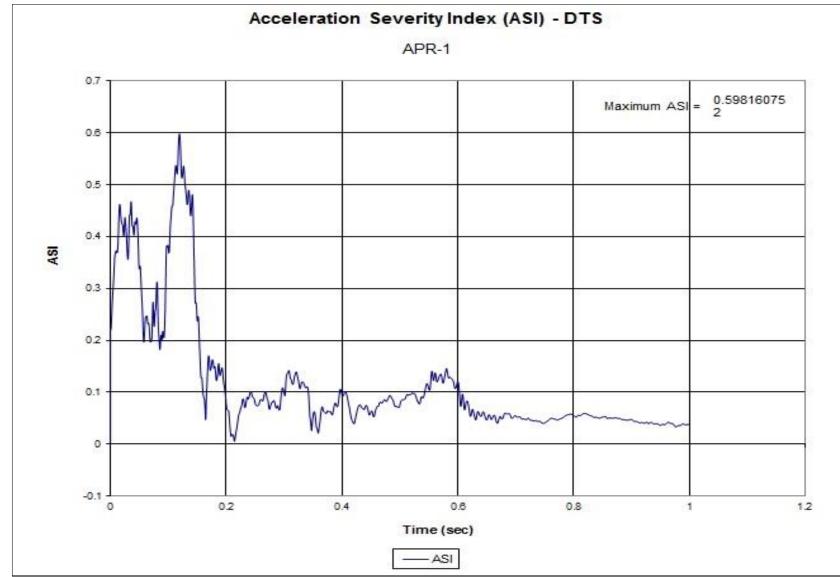
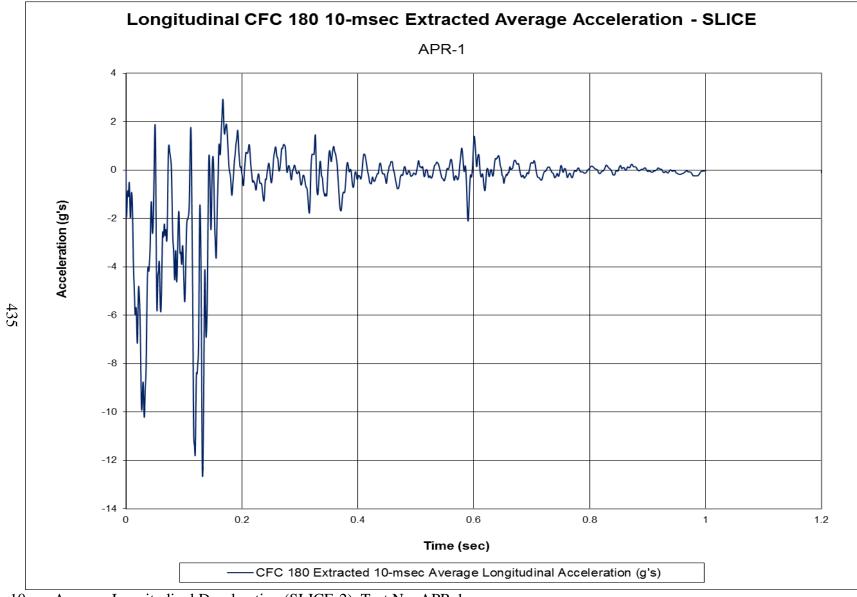


Figure H-8. Acceleration Severity Index (DTS), Test No. APR-1



<sup>10-</sup>ms Average Longitudinal Deceleration (SLICE-2), Test No. APR-1

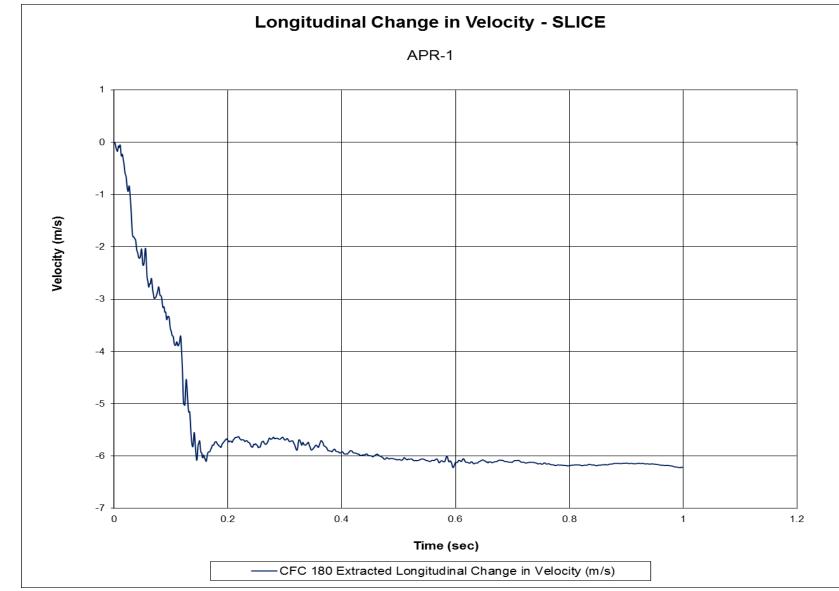


Figure H-9. Longitudinal Occupant Impact Velocity (SLICE-2), Test No. APR-1

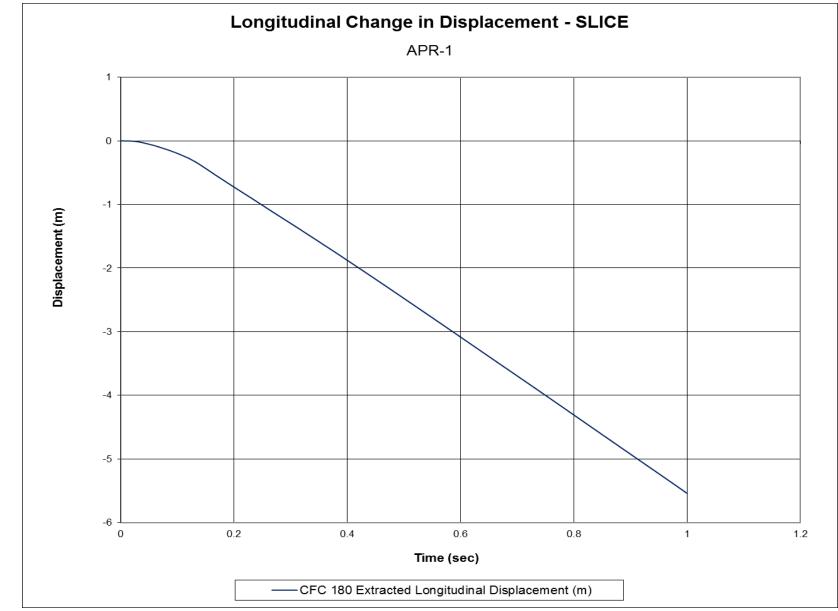


Figure H-10. Longitudinal Occupant Displacement (SLICE-2), Test No. APR-1

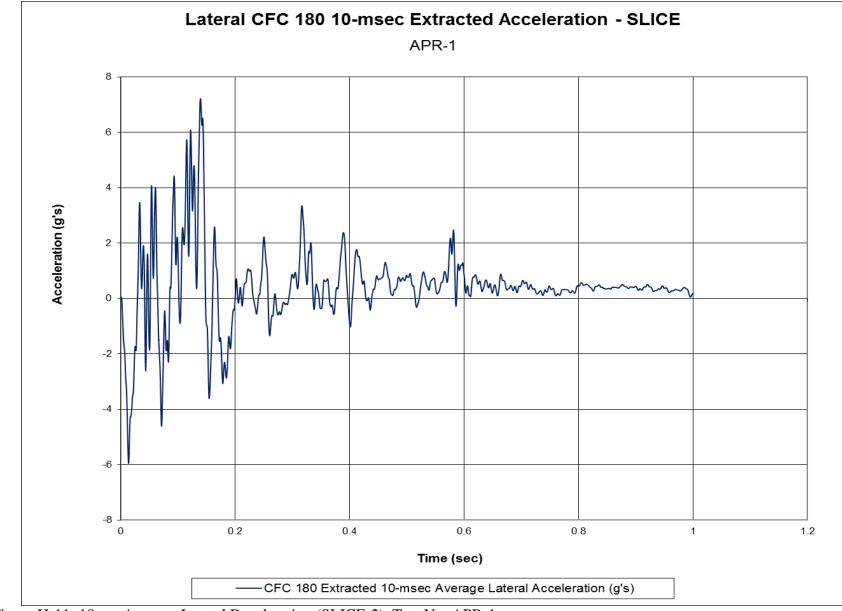


Figure H-11. 10-ms Average Lateral Deceleration (SLICE-2), Test No. APR-1

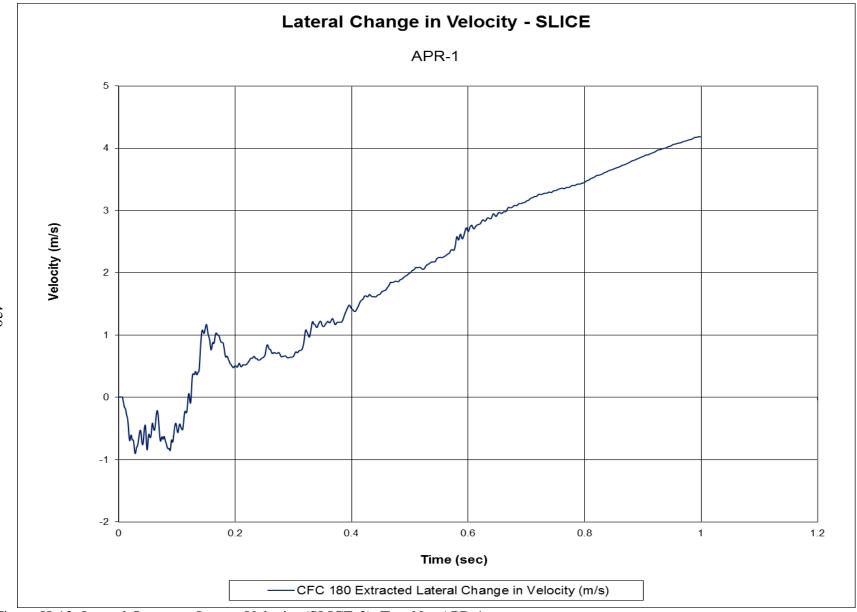


Figure H-12. Lateral Occupant Impact Velocity (SLICE-2), Test No. APR-1

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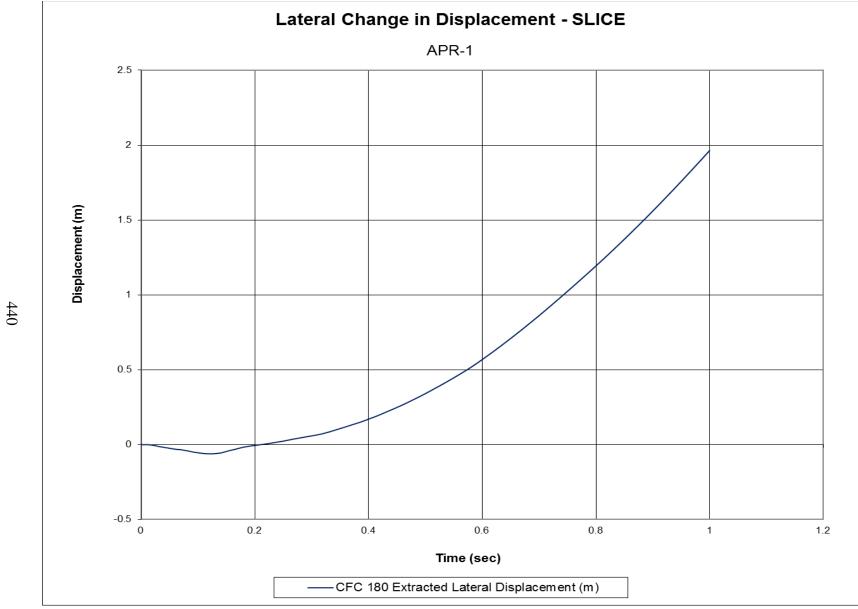


Figure H-13. Lateral Occupant Displacement (SLICE-2), Test No. APR-1

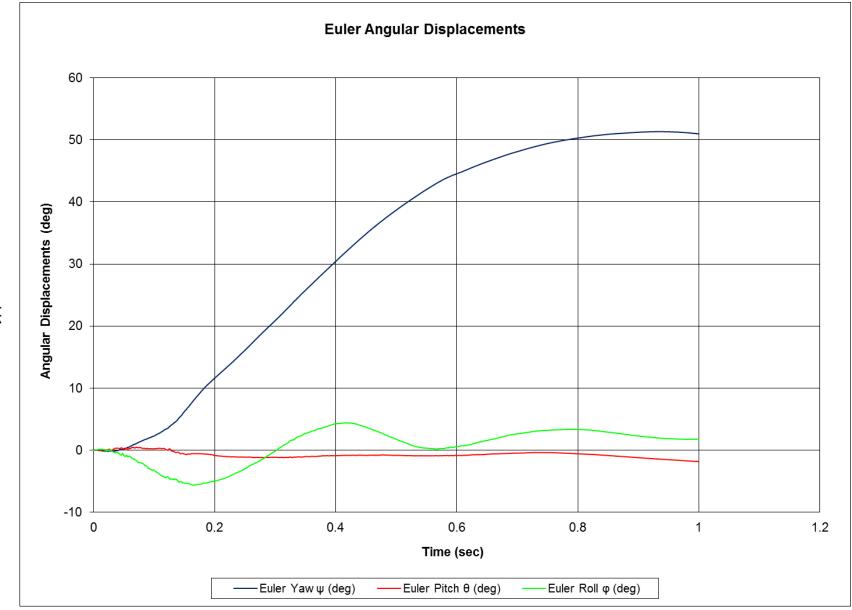


Figure H-14. Vehicle Angular Displacements (SLICE-2), Test No. APR-1

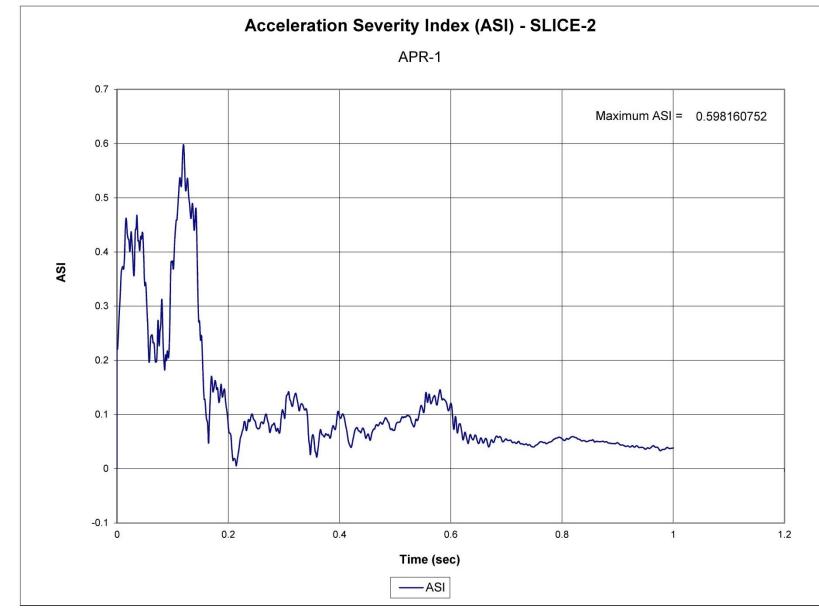


Figure H-15. Acceleration Severity Index (SLICE-2), Test No. APR-1

## Appendix I. Accelerometer and Rate Transducer Data Plots, Test No. APR-2

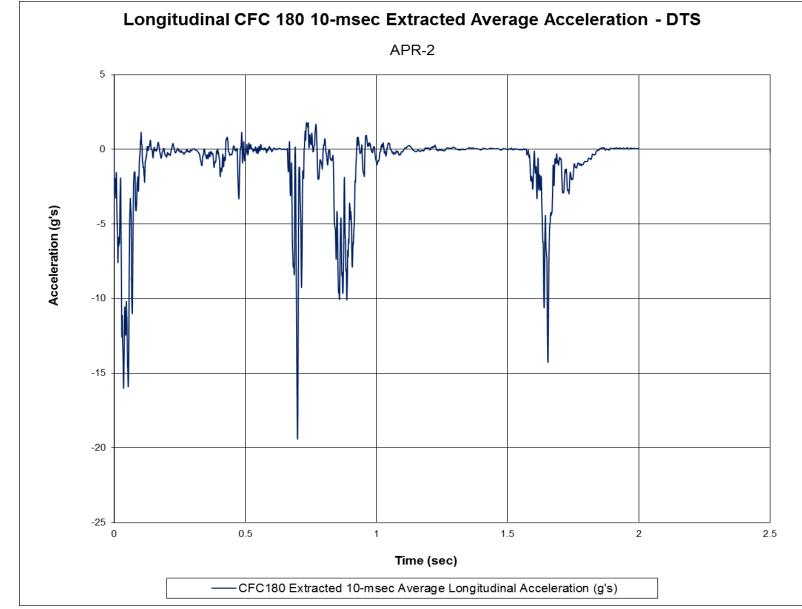


Figure I-1. 10-ms Average Longitudinal Deceleration (DTS), Test No. APR-2

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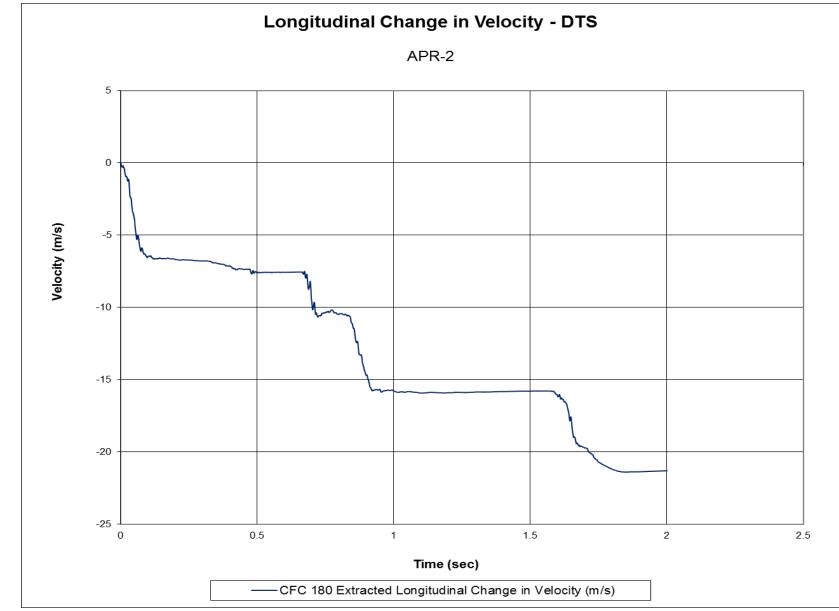


Figure I-2. Longitudinal Occupant Impact Velocity (DTS), Test No. APR-2

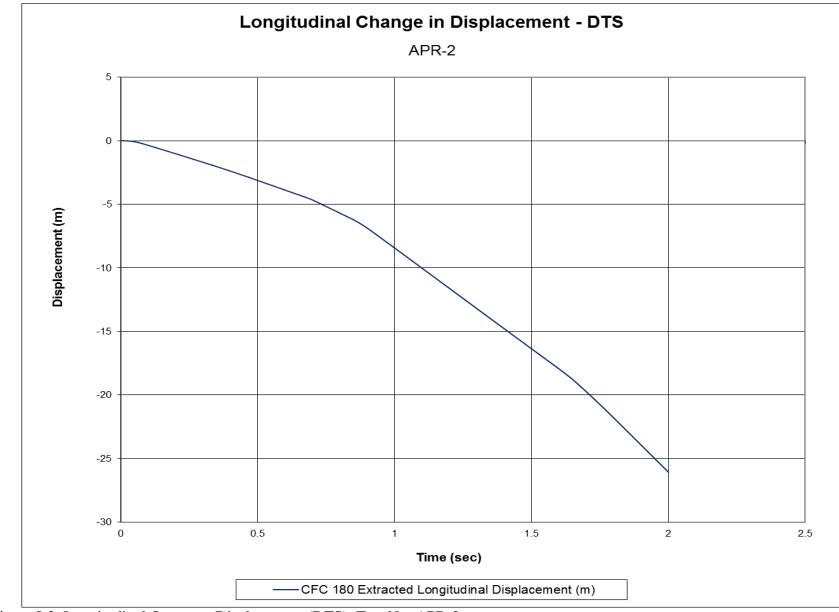


Figure I-3. Longitudinal Occupant Displacement (DTS), Test No. APR-2

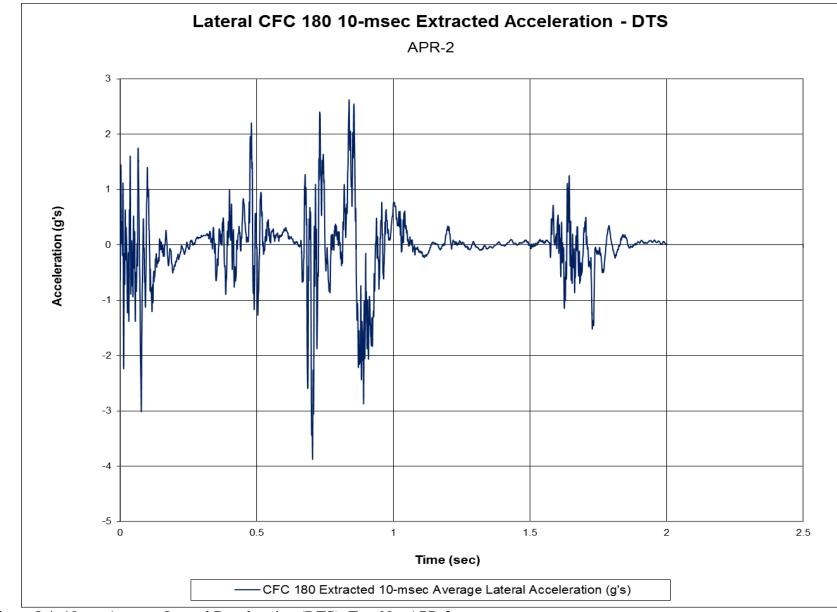


Figure I-4. 10-ms Average Lateral Deceleration (DTS), Test No. APR-2

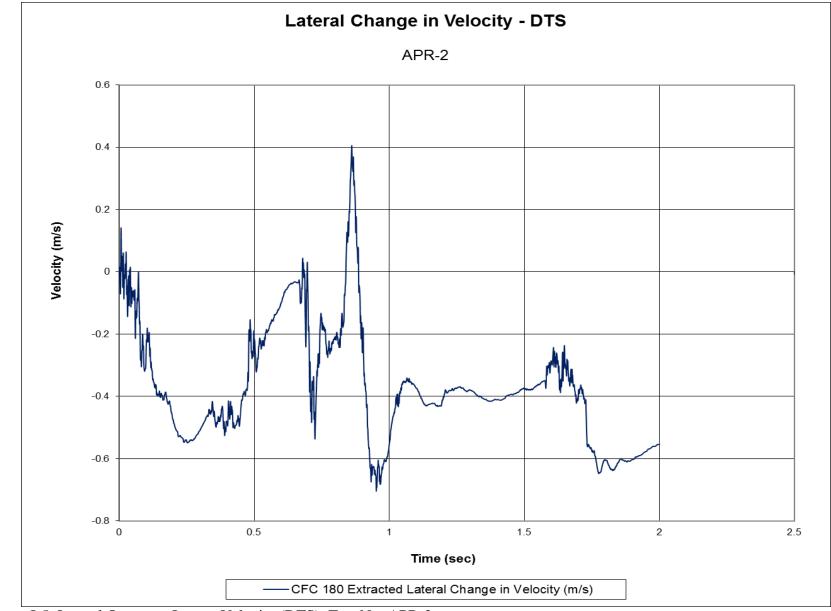


Figure I-5. Lateral Occupant Impact Velocity (DTS), Test No. APR-2

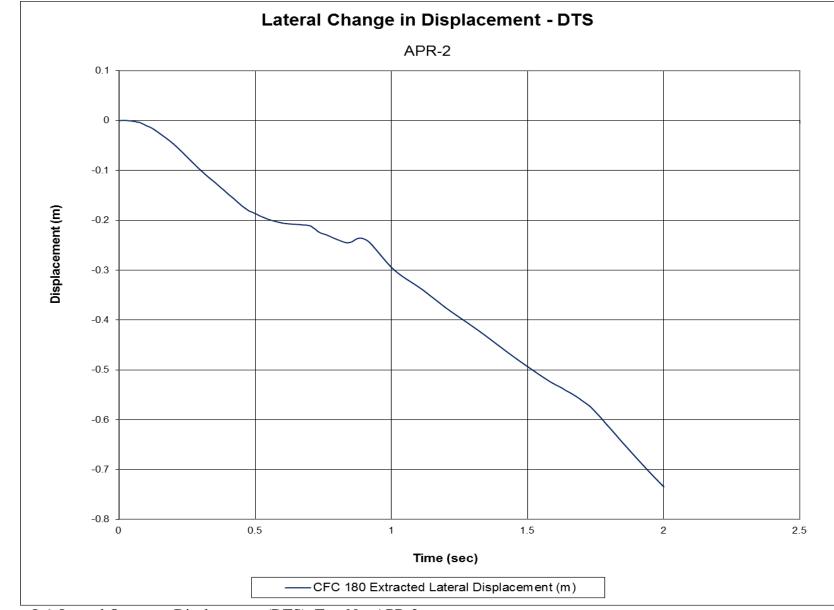
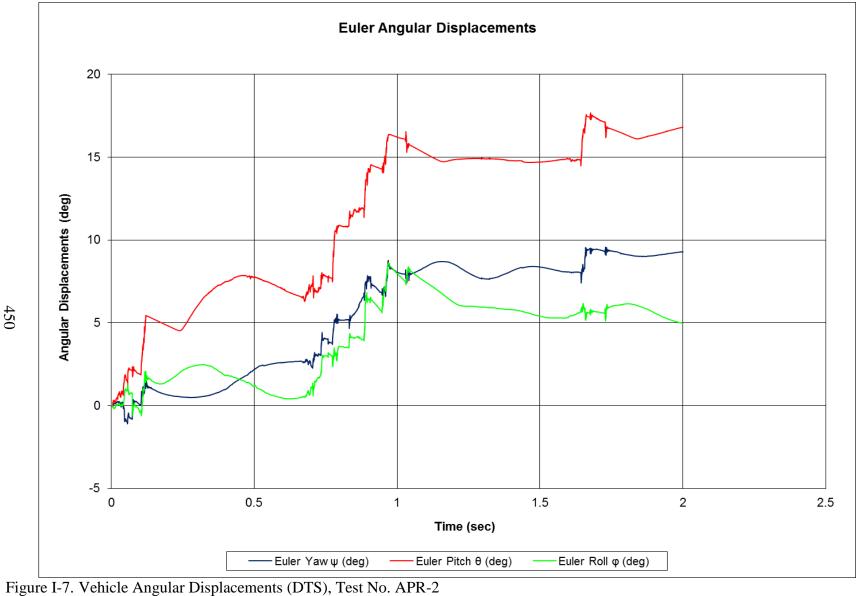


Figure I-6. Lateral Occupant Displacement (DTS), Test No. APR-2



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Acceleration Severity Index (ASI) - DTS

APR-2

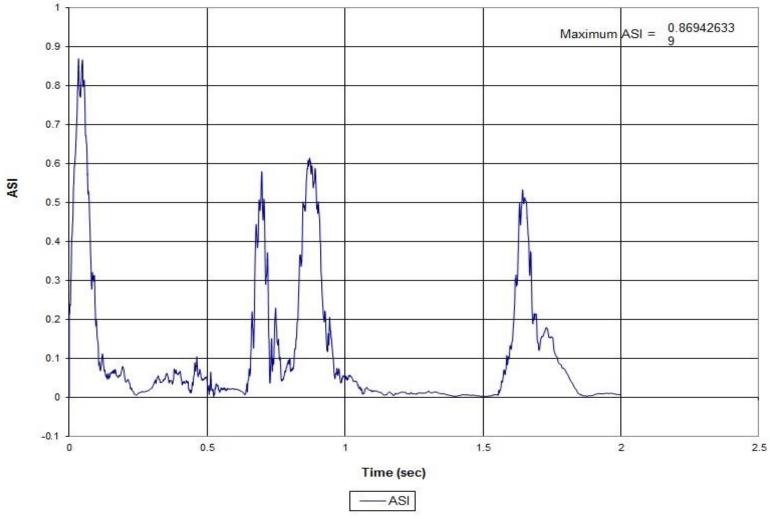


Figure I-8. Acceleration Severity Index (DTS), Test No. APR-2

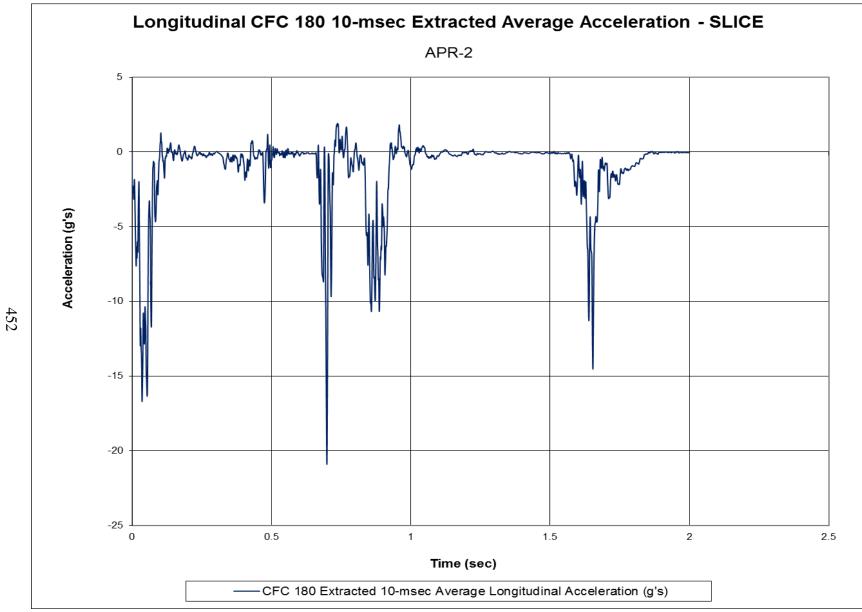


Figure I-9. 10-ms Average Longitudinal Deceleration (SLICE-2), Test No. APR-2

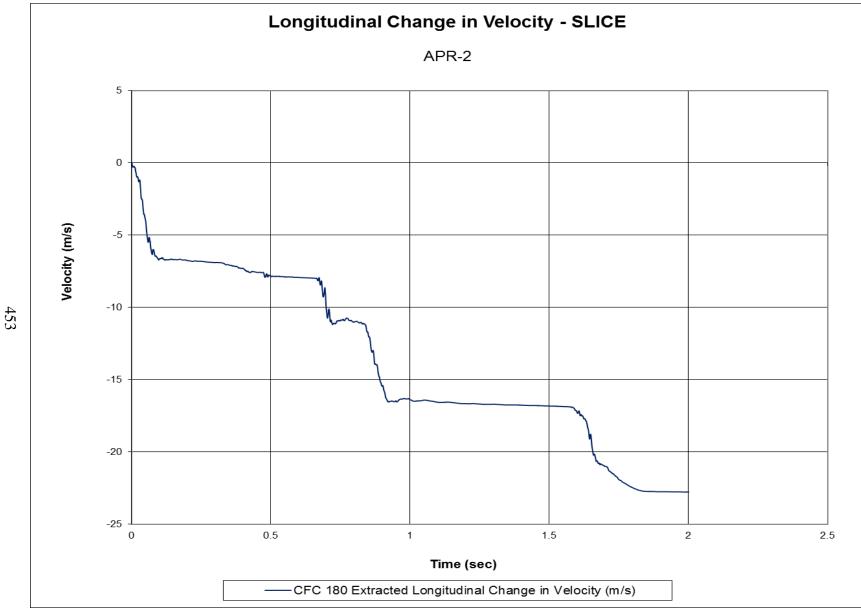


Figure I-10. Longitudinal Occupant Impact Velocity (SLICE-2), Test No. APR-2

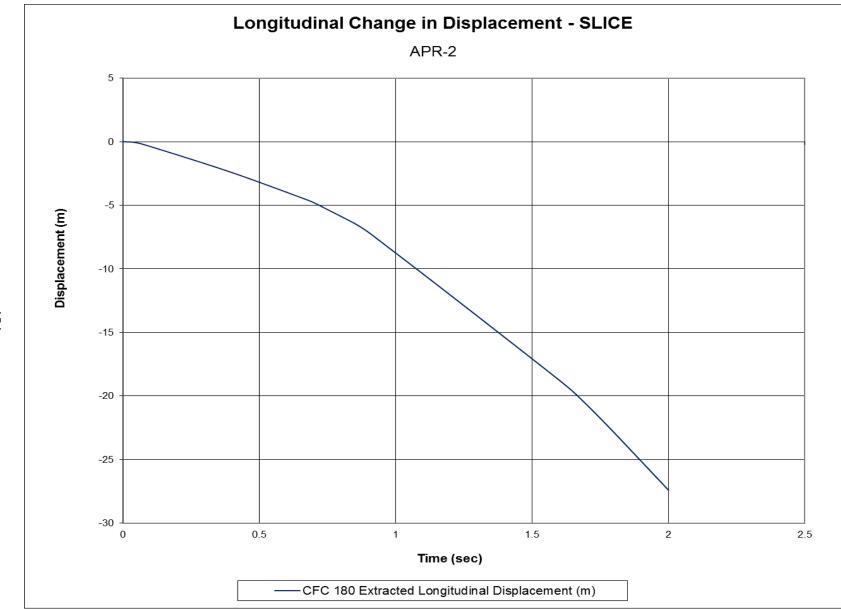


Figure I-11. Longitudinal Occupant Displacement (SLICE-2), Test No. APR-2

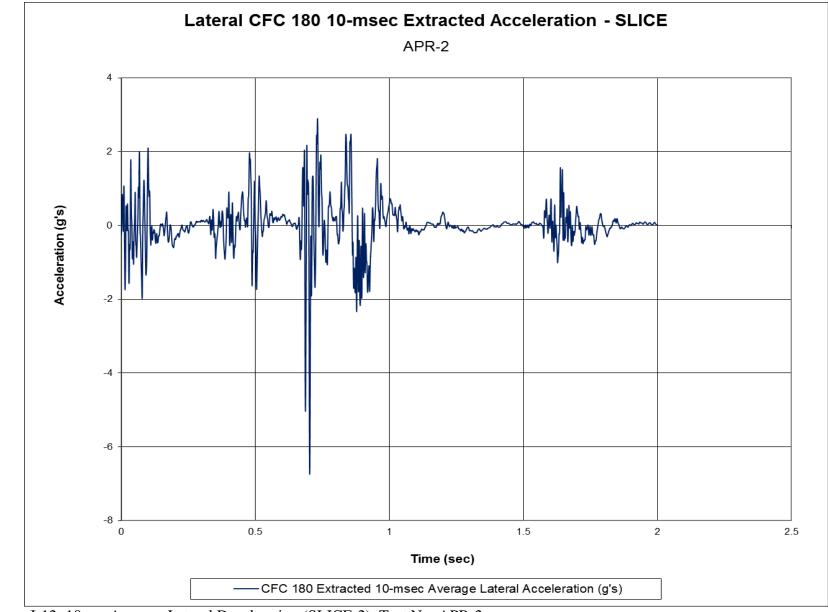


Figure I-12. 10-ms Average Lateral Deceleration (SLICE-2), Test No. APR-2

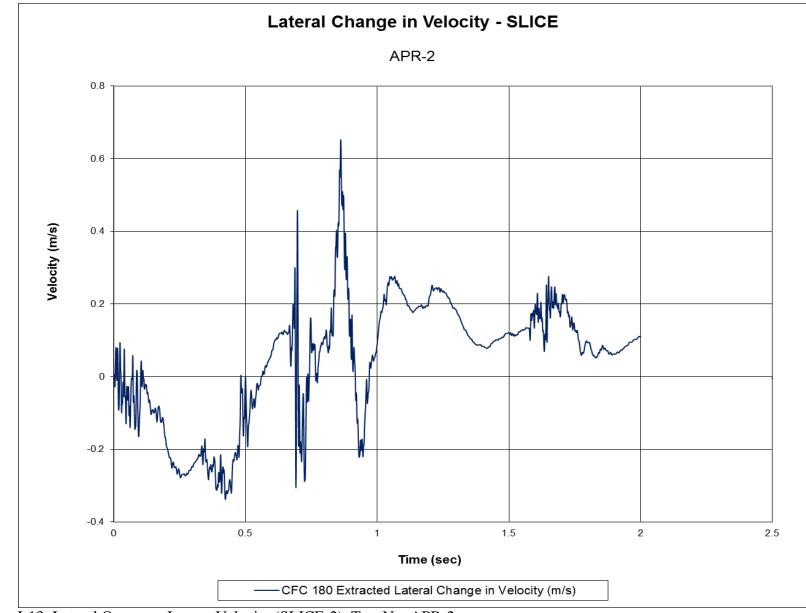


Figure I-13. Lateral Occupant Impact Velocity (SLICE-2), Test No. APR-2

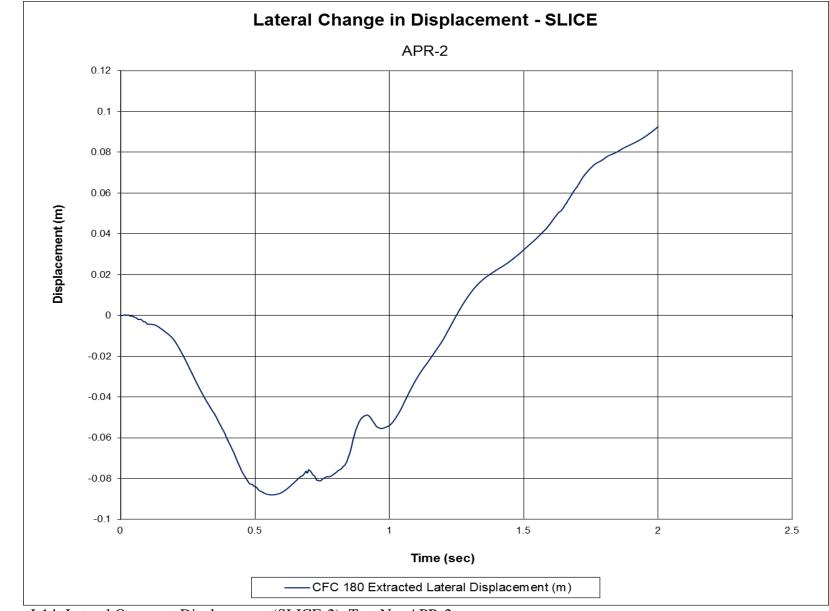


Figure I-14. Lateral Occupant Displacement (SLICE-2), Test No. APR-2



Figure I-15. Vehicle Angular Displacements (SLICE-2), Test No. APR-2

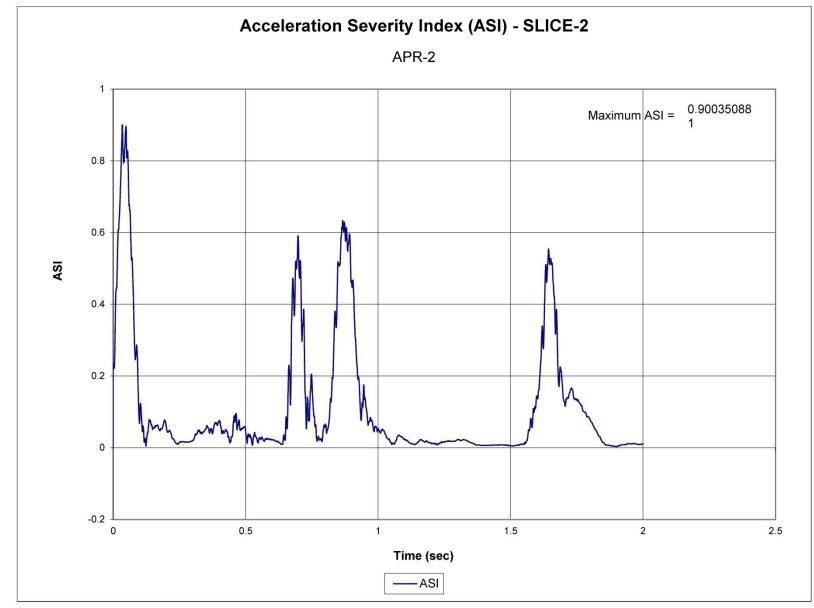


Figure I-16. Acceleration Severity Index (SLICE-2), Test No. APR-2

## Appendix J. Test No. APR-2 Accelerometer Discussion

#### Email Correspondence with FHWA Regarding Test No. APR-2

From: will.longstreet@dot.gov [mailto:will.longstreet@dot.gov] Sent: Thursday, February 12, 2015 9:44 AM To: rbielenberg2@unl.edu Cc: rfaller1@unl.edu; Nick.Artimovich@dot.gov Subject: RE: Pedestrian Rail Test Results

Hi Bob:

Nick & I looked this one over & offer the following.

We've not seen this situation before (i.e., 2 conflicting accel readings). After review of video, we certainly agree w/your detailed assessment of test in regards to acceleration spike. As there is no existing policy for comparing accelerations from different transducer units on same test, we feel it best to recognize the implication of a higher value.

In addition as sponsoring state plans to further develop the design to improve its performance and lower the occupant risk values, was there any thoughts re. flying debris due to impact as well?

Please call me if you wish to further discuss & thanks for your patience.

Best,

### Thursday, April 30, 2015 Task Force 13 Meeting Minutes

#### **Subcommittee #7 Certification of Crash Test Facilities**

Joined online by: John Jewell – CalTrans, Mike Dunlap – KARCO, Steven Matsusaka – KARCO

Sign in sheet sent around – make sure your name is on the list if you wanted to receive email correspondence in regards to ILCs. If not present at the meeting, email either Karla Lechtenberg (kpolivka2@unl.edu) or Lance Bullard (l-bullard@tamu.edu) if you want to receive ILC correspondence and have not been.

ILC discussion

• Accrediting body (A2LA) asking for a "plan" of at least 4 years of ILCs

• Each lab to email Karla Lechtenberg (kpolivka2@unl.edu) and idea for an ILC to be added to the "plan" by July 1, 2015.

• Need to add more "teeth" to the ILCs that are being conducted. Currently only sending out the results, but not discussion on who is correct and why the other labs are not. A report is needed that presents a description of the ILC, the results of the ILC, the differences between labs, the issues of why all are not matching/coming up with the same answer, and resolutions to the differences/issues.

Multiple accelerometer systems in a test vehicle discussion

- All labs use redundant accelerometer systems
- All labs do not analyze nor report all the accelerometer systems used in the tests
- o MwRSF analyzes and reports all data/systems

o Holmes Solution – only uses primary system, compares to other physical results. Only looks at and/or reports the secondary unit data if near the required limits for occupant risk

- o TTI only report primary
- o TRC only report primary

o CalTran – only look at primary unit, only analyze secondary unit if primary has issues. If within uncertainties, just use primary.

- o TRC analyzes and reports all data/units
- Most labs only mount the accelerometer systems at the x,y location of the c.g.

• Some labs stated that if primary fails or does not work then look at the secondary unit. If the occupant risk numbers are close to the threshold then the lab may have to rerun the test since it is unknown what effect not being mounted at the c.g. has on the occupant risk values.

- Consensus of laboratory representatives and FHWA present
- o must report primary accelerometer unit (one at or within 2" of c.g.)
- o may report all accelerometer units used, but must denote which is primary and secondary
- o Placement of all accelerometer units must be noted within the reports.

<sup>1</sup>/<sub>4</sub>-pt offset vs. centerline impact discussion (terminal/crash cushion)

• Currently impact is <sup>1</sup>/<sub>4</sub>-point offset which is critical for vehicle instability

• Not currently required, but centerline impact might be more critical for vehicle decelerations and occupant risk

• Staged devices – concerns for ORA vales

- Non-staged devices OIV/ORA occurs before the vehicle yaws out
- Consensus of laboratory representatives present
- o Conduct the estimation procedure similar to the 1500A vehicle but with an 1100C vehicle

could determine if that might be a critical impact.

o Should be done for staged devices due to possible effects on ORA values

Debris "present undue hazard" discussion

- Began discussion
- MASH subjective on this topic. Not very clear.
- EN1317 uses 2 kg mas as the maximum debris
- Need to develop a consistency among the testing labs
- This topic needs more discussion.

January 18, 2016 MwRSF Report No. TRP-03-321-15

# **END OF DOCUMENT**